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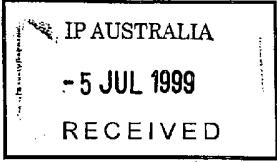
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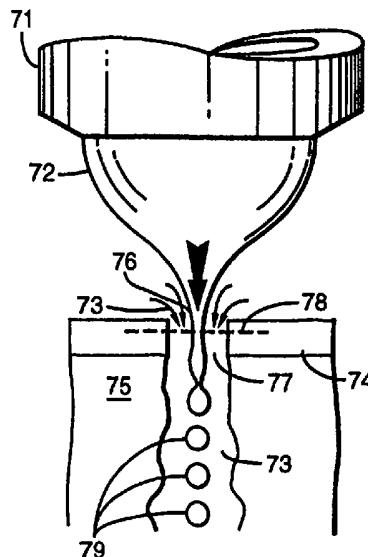
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(54) Title: DEVICE AND METHOD FOR AERATION OF FLUIDS

(57) Abstract

The present invention provides aeration methods using spherical gas bubbles having a size on the order of 0.1 to 100 microns in size. A device of the invention for producing a monodispersion of bubbles includes a source of a stream of gas which is forced through a liquid held under pressure in a pressure chamber with an exit opening therein. The stream of gas surrounded by the liquid in the pressure chamber flows out of an exit orifice of the chamber into a liquid thereby creating a monodispersion of bubbles with substantially uniform diameter. The bubbles are small in size and produced with a relatively small amount of energy relative to comparable systems. Applications of the aeration technology range from oxygenating sewage with monodispersions of bubbles to oxygenation of water for fish maintenance.



DEVICE AND METHOD FOR AERATION OF FLUIDS

FIELD OF THE INVENTION

The invention relates generally to the field of small particle formation and more specifically to fields where it is important to create gas bubbles which are very small and uniform in size.

BACKGROUND OF THE INVENTION

Monodispersed sprays of droplets of micrometric size have attracted the interest of scientist and engineers because of their potential applications in many fields of science and technology. Classifying a polydispersed aerosol (for example, by using a differential mobility analyzer, B. Y. Liu et al. (1974), "A Submicron Standard and the Primary Absolute Calibration of the Condensation Nuclei Counter," *J. Colloid Interface Sci.* 47:155-171 or breakup process of Rayleigh's type of a capillary microjet Lord Rayleigh (1879), "On the instability of Jets," *Proc. London Math. Soc.* 10:4-13, are the current methods to produce the monodispersed aerosols of micrometric droplets needed for such applications. The substantial loss of the aerosol sample during the classification process can severely limit the use of this technique for some applications. On the other hand, although in the capillary break up the size distribution of the droplets can be very narrow, the diameter of the droplets is determined by the jet diameter (approximately twice the jet diameter). Therefore, the generation and control of capillary microjets are essential to the production of sprays of micrometric droplets with very narrow size distribution.

Capillary microjets with diameters ranging from tens of nanometers to hundred of micrometers are successfully generated by employing high electrical fields (several kV) to form the well-known cone-jet electrospray. Theoretical and experimental results and numerical calculations on electrosprays can be obtained from M. Cloupean et al. (1989), "Electrostatic Spraying of Liquids in Cone Jet Mode," *J. Electrostat* 22:135-159, Fernández de la Mora et al. (1994), "The Current Transmitted through an Electrified Conical Meniscus," *J. Fluid Mech.* 260:155-184 and Loscertales (1994), A.M. Gañán-Calvo et al. (1997), "Current and Droplet Size in the Electrospraying of Liquids: Scaling Laws," *J. Aerosol Sci.* 28:249-275, Hartman et al. (1997), "Electrohydrodynamic Atomization in the Cone-Jet Mode," Paper presented at the ESF Workshop on

Electrospray, Sevilla, 28 Feb. - 1 Mar. 1997 among others [see also the papers contained in the Special Issue for Electrosprays (1994)]. In the electrospray technique the fluid to be atomized is slowly injected through a capillary electrified needle. For a certain range of values of the applied voltage and
 5 flow rate an almost conical meniscus is formed at the needle's exit from whose vertex a very thin, charge jet is issued. The jet breaks up into a fine aerosol of high charged droplets characterized by a very narrow droplet size distribution. Alternatively, the use of purely mechanical means to produce capillary microjets is limited in most of applications for several reasons: the
 10 high-pressure values required to inject a fluid through a very narrow tube (typical diameters of the order of few micrometers) and the easy clogging of such narrow tubes due to impurities in the liquid.

The present invention provides a new technique for producing uniform sized monodispersion of gas bubbles based on a mechanical means which
 15 does not present the above inconveniences and can compete advantageously with electrospray atomizers. The jet diameters produced with this technique can be easily controlled and range from below one micrometer to several tens of micrometers.

Any discussion of documents, acts, materials, devices, articles or the
 20 like which has been included in the present specification is solely for the purpose of providing a context for the present invention. It is not to be taken as an admission that any or all of these matters form part of the prior art base or were common general knowledge in the field relevant to the present invention as it existed in Australia before the priority date of each claim of
 25 this application.

SUMMARY OF THE INVENTION

In a first aspect, the present invention is a monodispersion of bubbles
 30 for diffusing gas into a fluid, comprising gas bubbles in a liquid wherein the bubbles are characterized by having approximately the same diameter with a standard deviation in diameter from one bubble to another in a range of from $\pm 0.01\%$ to $\pm 30\%$, the monodispersion being produced by:

forcing a gas from a source opening into a first liquid in a manner so as
 35 to create a flow stream of the gas through the first liquid, wherein the gas is comprised of molecules to be diffused into a second liquid;



moving the first liquid in a pressure chamber surrounding the source opening, out of an exit orifice in the pressure chamber wherein the flow stream of the gas flows out the exit orifice into the second liquid wherein the flow stream breaks up forming bubbles of the gas in the second liquid; and
 5 allowing molecules in the gas bubbles to diffuse into the second liquid.

The present invention provides aeration methods using spherical gas bubbles having a size on the order of 0.1 to 100 microns in size. A device of the invention for producing a monodispersion of bubbles includes a source of a stream of gas which is forced through a liquid held under pressure in a
 10 pressure chamber with an exit opening therein. The stream of gas surrounded by the liquid in the pressure chamber flows out of an exit orifice of the chamber into a liquid thereby creating a monodispersion of bubbles with substantially uniform diameter. The bubbles are small in size and produced with a relatively small amount of energy relative to comparable
 15 systems. Applications of aeration technology range from oxygenating sewerage with monodispersions of bubbles to oxygenation of water for fish maintenance.

Throughout this specification the word "comprise", or variations such as "comprises" or "comprising", will be understood to imply the inclusion of a
 20 stated element, integer or step, or group of elements, integers or steps, but not the exclusion of any other element, integer or step, or group of elements, integers or steps.

BRIEF DESCRIPTION OF THE DRAWINGS

25 Figure 1 is a schematic view showing the basic components of one embodiment of the invention with a cylindrical feeding needle as a source of formulation.



Figure 2 is a schematic view of another embodiment of the invention with two concentric tubes as a source of formulation.

Figure 3 is a schematic view of yet another embodiment showing a wedge-shaped planar source of formulation. Figure 3a illustrates a cross-sectional side view of the planar feeding source and the interaction of the fluids. Figure 3b show a frontal view of the openings in the pressure chamber, with the multiple openings through which the atomizate exits the device. Figure 3c illustrates the channels that are optionally formed within the planar feeding member. The channels are aligned with the openings in the pressure chamber.

Figure 4 is a schematic view of a stable capillary microjet being formed and flowing through an exit opening to thereafter form a monodisperse aerosol.

Figure 5 is a graph of data where 350 measured values of d_j/d_o versus Q/Q_o are plotted.

Figure 6 is a micrograph showing the even dispersement and uniform size of air bubbles created using the method of the invention after expulsion into air.

Figure 7 is a schematic view of the critical area of a device of the type shown in Figure 1 showing gas surrounded by liquid expelled into a liquid to form bubbles.

Figure 8 is a schematic view as in Figure 7 but with the bubbles flowing into a gas.

Figure 9 is a schematic view as in Figure 7 but with two immiscible liquids flowing into a gas.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

Before the present aeration device and method are described, it is to be understood that this invention is not limited to the particular components and steps described, as such may, of course, vary. It is also to be understood that the terminology used herein is for the purpose of describing particular embodiments only, and is not intended to be limiting, since the scope of the present invention will be limited only by the appended claims.

It must be noted that as used herein and in the appended claims, the singular forms "a", "and," and "the" include plural referents unless the context clearly dictates otherwise. Thus, for example, reference to "a bubble" includes a plurality of bubbles and reference to "a gas" includes reference to a mixture of gases, and equivalents thereof known to those skilled in the art, and so forth.

Unless defined otherwise, all technical and scientific terms used herein have the same meaning as commonly understood by one of ordinary skill in the art to which this invention belongs. Although any methods and materials similar or equivalent to those described herein can be used in the practice or testing of the present invention, the preferred methods and materials are now described. All publications mentioned herein are incorporated herein by reference to disclose and describe the methods and/or materials in connection with which the publications are cited.

The publications discussed herein are provided solely for their disclosure prior to the filing date of the present application. Nothing herein is to be construed as an admission that the present invention is not entitled to antedate such publication by virtue of prior invention. Further, the dates of publication provided may be different from the actual publication dates which may need to be independently confirmed.

DEFINITIONS

The terms "bubble", "dispersion of bubbles" and "monodispersion of bubbles" are used interchangeably herein and shall mean small uniformly sized particles of a gas or gaseous formulation that has been dispersed using the device and method of the invention. The particles are generally spherical, and may be comprised of one or more gases or layers of gases.

The terms "air", "particle free air" and the like, are used interchangeably herein to describe a volume of air which is substantially free of other material and, in particular, free of particles intentionally added such as particles of formulation. Air is a mixture of various gas components that may, of course vary, but usually the air will contain approximately 21% oxygen by volume. Air may also contain gases or other air-borne particles. For use in the invention, air may be filtered or treated to remove all unwanted particulate or gaseous matter, or the air may be used in an unfiltered state. Air is the preferred gas for use of the invention in oxygenation of aqueous fluids, e.g. water.

The terms "gas" and "gas formulation" as used herein refer to any gas or gaseous mixture which is desired to be dispersed using the method of the invention. For example, the formulation may be comprised of air, either filtered or unfiltered. Gases such as air may be spiked with a particular gas, such as the spiking of air with additional O₂ gas for use in oxygenation. A gaseous formulation may also contain suspended particulate matter

dispersed within the gas. The gas can be CO₂ to carry out the carbonation of beverages (e.g. water, colas) or a gas containing an unwanted contaminant, e.g. radioactivity or an environmental toxin.

5 The term "aeration" as used herein refers to the dispersion of a gaseous material into a flowable fluid, for example to provide a diffusion surface to introduce a molecule or compound from the gas into the flowable surface. The term is not limited to the dispersion of air per se, although the use of air is preferred, but rather refers to the introduction of any gas to a flowable fluid, e.g. O₂, CO₂, hydrogen, nitrogen, and the like and mixtures thereof. The aeration of a fluid is preferably to allow molecules and/or
10 compounds to diffuse to the fluid through the fluid-bubble interface following expulsion of the bubbles from the device of the invention into the surrounding fluid. A fluid may, however, also be aerated for aesthetic purposes, such as the addition of CO₂ to a beverage to provide carbonation.

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DEVICE IN GENERAL

Different embodiments are shown and described herein (see Figures 1, 2 and 3) which could be used in producing the stable capillary microjet and/or a dispersion of particles which are substantially uniform in size. Although various embodiments are part of the invention, they are merely provided as exemplary devices which can be used to
20 convey the essence of the invention, which is the formation of a stable capillary microjet and/or uniform dispersion of particles.

A basic device comprises (1) a means for supplying a first fluid, preferably a gas, and (2) a pressure chamber supplied with a second fluid which flows out of an exit opening in the pressure chamber, preferably a liquid. The exit opening of the pressure
25 chamber is aligned with the flow path of the means for supplying the first fluid. The embodiments of Figures 1, 2 and 3 clearly show that there can be a variety of different means for supplying the first fluid. Other means for supplying a first fluid flow stream will occur to those skilled in the art upon reading this disclosure.

Further, other configurations for forming the pressure chamber around the means
30 for supplying the first fluid will occur to those skilled in the art upon reading this disclosure. Such other embodiments are intended to be encompassed by the present invention provided the basic conceptual results disclosed here are obtained, i.e. a stable

capillary microjet is formed and/or a dispersion of particle highly uniform in size is formed. To simplify the description of the invention, the means for supplying a first fluid is often referred to as a cylindrical tube (see Figure 1) and the first fluid is generally referred to as a gas. The gas can be any gas depending on the desired use of the device, although it is preferably air. For example, the gas could be air used to create small bubbles for aeration of a liquid to provide a gaseous medium through which components may diffuse into a liquid. Further, for purposes of simplicity, the second fluid is generally described herein as being a liquid, e.g. water. The invention is also generally described with a gas formulation being expelled from the supply means and forming a stable microjet due to interaction with surrounding water flow, which focuses the gas microjet to flow out of an exit of the pressure chamber.

Formation of the microjet and its acceleration and ultimate particle formation are based on the abrupt pressure drop associated with the steep acceleration experienced by the gas on passing through an exit orifice of the pressure chamber which holds the second fluid (i.e. the liquid). On leaving the chamber the flow undergoes a certain pressure difference between the liquid and the gas, which in turn produces a highly curved zone on the liquid surface near the exit port of the pressure chamber and in the formation of a cuspidal point from which a steady microjet flows, provided the amount of the gas drawn through the exit port of the pressure chamber is replenished. Thus, in the same way that a glass lens or a lens of the eye focuses light to a given point, the flow of the liquid surrounds and focuses the gas into a stable microjet. The focusing effect of the surrounding flow of liquid creates a stream of gas which is substantially smaller in diameter than the diameter of the exit orifice of the pressure chamber. This allows the gas to flow out of the pressure chamber orifice without touching the orifice, providing advantages including the feature that the diameter of the stream and the resulting particles are smaller than the diameter of the exit orifice of the chamber. This is particularly desirable because it is difficult to precisely engineer holes which are very small in diameter. Further, in the absence of the focusing effect (and formation of a stable interface cusp) flow of gas out of an opening will result in particles which have a diameter greater than the diameter of the exit opening.

The description provided here generally indicates that the gas leaves the pressure chamber through an exit orifice surrounded by the liquid and thereafter enters into a liquid surrounding environment which may be either a hydrophobic or hydrophilic liquid. This

configuration is particularly useful when it is necessary to create very small highly uniform bubbles which are moved into a liquid surrounding exit opening of the pressure chamber. The need for the formation of very small highly uniform bubbles into a gas occurs in a variety of different industrial applications. For example, water needs to be oxygenated in a variety of situations including small fish tanks for home use and large volume fisheries for industrial use. The additional oxygen can aid the rate of growth of the fish and thereby improve production for the fishery. In another embodiment, oxygen or air bubbles can be forced into liquid sewage in order to aid in treatment. In yet another application of the invention, contaminated gases such as a gas contaminated with a radioactive material can be formed into small uniformed bubbles and blown into a liquid, where the contamination in the gas will diffuse into the liquid, thereby cleaning the gas. The liquid will, of course, occupy substantially less volume and therefore be substantially easier to dispose of than contaminated toxic gas.

Those skilled in the art will recognize that variations on the different embodiments disclosed below will be useful in obtaining particularly preferred results. Specific embodiments of devices are now described.

EMBODIMENT OF FIGURE 1

A first embodiment of the invention where the supply means is a cylindrical feeding needle supplying gas into a pressurized chamber of liquid is described below with reference to Figure 1.

The components of the embodiment of Figure 1 are as follows:

1. Feeding needle - also referred to generally as a fluid source and a tube.
2. End of the feeding needle used to insert the gas to be dispersed.
- 25 3. Pressure chamber.
4. Orifice used as liquid inlet.
5. End of the feeding needle used to evacuate the liquid to be atomized.
6. Orifice through which withdrawal takes place.
7. Atomizate (spray) - also referred to as aerosol.

30 D_f = diameter of the feeding needle; D_o = diameter of the orifice through which the microjet is passed; e = axial length of the orifice through which withdrawal takes place; H = distance

from the feeding needle to the microjet outlet; P_o = pressure inside the chamber; P_a = atmospheric pressure.

Although the device can be configured in a variety of designs, the different designs will all include the essential components shown in Figure 1 or components which perform an equivalent function and obtain the desired results. Specifically, a device of the invention will be comprised of at least one source of a first fluid (e.g., a feeding needle with an opening 2) into which a first fluid such as a gas formulation can be fed and an exit opening 5 from which the gas can be expelled. The feeding needle 1, or at least its exit opening 5, is encompassed by a pressure chamber 3. The chamber 3 has inlet opening 4 which is used to feed a second fluid (e.g. a liquid) into the chamber 3 and an exit opening 6 through which liquid from the pressure chamber and gas from the feeding needle 3 are expelled. When the first fluid is a gas it is preferably expelled into a liquid to create bubbles.

In Figure 1, the feeding needle and pressure chamber are configured to obtain a desired result of producing bubbles wherein the particles are small and uniform in size. The bubbles have a size which is in a range of 0.1 to 100 microns. The particles of any given bubbles will all have about the same diameter with a relative standard deviation of $\pm 0.01\%$ to $\pm 30\%$ or more preferably $\pm 0.01\%$ to $\pm 10\%$. Stating that bubbles will have a diameter in a range of 1 to 5 microns does not mean that different bubbles will have different diameters and that some will have a diameter of 1 micron while others of 5 microns. The bubbles in a given dispersion will all (preferably about 90% or more) have the same diameter $\pm 0.01\%$ to $\pm 30\%$. For example, the bubbles of a given dispersion will have a diameter of 2 microns $\pm 0.01\%$ to $\pm 10\%$.

Such a uniform bubble monodispersion is created using the components and configuration as described above. However, other components and configurations will occur to those skilled in the art. The object of each design will be to supply fluid so that it creates a stable capillary microjet which is accelerated and stabilized by pressure stress exerted by the second fluid on the first fluid surface. The stable microjet created by the second fluid leaves the pressurized area (e.g., leaves the pressure chamber and exits the pressure chamber orifice) and splits into particles or bubbles which have the desired size and uniformity.

The parameter window used (*i.e.* the set of special values for the properties of the liquid used, flow-rate used, feeding needle diameter, orifice diameter, pressure ratio, *etc.*) should be large enough to be compatible with virtually any liquid (dynamic viscosities in the range from 10^{-5} to $1 \text{ kg m}^{-1}\text{s}^{-1}$); in this way, the capillary microjet that emerges from the end of the feeding needle is absolutely stable and perturbations produced by breakage of the jet cannot travel upstream. Downstream, the microjet splits into evenly shaped bubbles simply by effect of capillary instability (see, for example, Rayleigh, "On the instability of jets", Proc. London Math. Soc., 4-13, 1878), similar in a manner to a laminar capillary jet falling from a half-open tap.

When the stationary, steady interface is created, the capillary jet that emerges from the end of the bubble attached at the outlet of the feeding point is concentrically withdrawn into the nozzle. After the gas jet emerges from the attached bubble, the gas is accelerated by pressure forces exerted by the liquid stream, which gradually decreases the jet cross-section. Stated differently the liquid flow acts as a lens and focuses and stabilizes the gas microjet as it moves toward and into the exit orifice of the pressure chamber.

When the first fluid of the invention is a gas, and the second fluid is a liquid, the inertia of the first fluid is low, and the gas abruptly decelerates very soon after it issues from the cusp of the attached bubble. In such an instance, the microjet is so short that it is almost indistinguishable from the stable cusp of the gas-liquid interface.

When the first fluid of the invention is a gas and the second fluid is a liquid, and the two fluid stream is expelled into a gaseous atmosphere, a liquid jet with a regularly spaced gaseous formation of bubbles is formed. The regularity of the bubbles is such that the liquid jet is deformed in a very regular manner, resulting in a highly monodisperse stream of hollow droplets. The gas inside these hollow droplets may be manipulated by appropriate chemical, thermal or mechanical means to expand further upon expulsion from the device, causing the hollow bubbles to break into even finer droplets. Alternatively, if the liquid used is curable, the hollow droplets may be cured to a hollow, solid form.

The forces exerted by the second fluid flow on the first fluid surface should be steady enough to prevent irregular surface oscillations. Therefore, any turbulence in the gas and liquid motion should be avoided; even if the gas velocity is high, the characteristic size of the orifice should ensure that the fluid motion is laminar (similar to the boundary layers formed on the jet and on the inner surface of the nozzle or hole).

STABLE CAPILLARY MICROJET

Figure 4 illustrates the interaction of a gas and a liquid to form bubbles using the method of the invention. The feeding needle 60 has a circular exit opening 61 with an internal radius R_1 which feeds a gas 62 out of the end, forming a drop with a radius in the range of R_1 to R_1 plus the thickness of the wall of the needle. Thereafter the drop narrows in circumference to a much smaller circumference as is shown in the expanded view of the tube (i.e. feeding needle) 5 as shown in Figures 1 and 4. The exiting gas flow comprises an infinite amount of streamlines 63 that after interaction of the gas with the surrounding liquid narrows to form a stable cusp at the interface 64 of the two fluids. The surrounding liquid also forms an infinite number of liquid streamlines 65, which interact with the solid surfaces and the exiting gas to create the effect of a virtual focusing funnel 66. The exiting gas is focused by the focusing funnel 66 resulting in a stable capillary microjet 67, which remains stable until it exits the opening 68 of the pressure chamber 69. After exiting the pressure chamber, the microjet begins to break-up, forming monodispersed particles 70.

The liquid flow, which affects the gas withdrawal and its subsequent deceleration after the jet is formed, should be very rapid but also uniform in order to avoid perturbing the fragile capillary interface (the surface of the drop that emerges from the jet).

As illustrated in Figure 4, the exit opening 61 of the capillary tube 60 is positioned close to an exit opening 68 in a planar surface of a pressure chamber 69. The exit opening 68 has a minimum diameter D_0 and is in a planar member with a thickness e . The diameter D_0 is referred to as a minimum diameter because the opening may have a conical configuration with the narrower end of the cone positioned closer to the source of liquid flow. Thus, the exit opening may be a funnel-shaped nozzle although other opening configurations are also possible, e.g. an hour glass configuration. Liquid in the pressure chamber continuously flows out of the exit opening. The flow of the liquid causes the gas drop expelled from the tube to decrease in circumference as the gas moves away from the end of the tube in a direction toward the exit opening of the pressure chamber.

In actual use, it can be understood that the opening shape which provokes maximum liquid acceleration (and consequently the most stable cusp and microjet with a given set of parameters) is a conically shaped opening in the pressure chamber. The conical opening is positioned with its narrower end toward the source of gas flow.

The distance between the end 61 of the tube 60 and the beginning of the exit opening 68 is H. At this point it is noted that R_1 , D_0 , H and e are all preferably on the order of hundreds of microns. For example, $R_1 = 400\mu\text{m}$, $D_0 = 150\mu\text{m}$, $H = 1\text{mm}$, $e = 300\mu\text{m}$. However, each could be 1/100 to 10x these sizes.

5 The end of the gas stream develops a cusp-like shape at a critical distance from the exit opening 68 in the pressure chamber 69 when the applied pressure drop ΔP_1 across the exit opening 68 overcomes the liquid-gas surface tension stresses γ/R^* appearing at the point of maximum curvature — e.g. $1/R^*$ from the exit opening.

10 A steady state is then established if the gas flow rate Q ejected from the drop cusp is steadily supplied from the capillary tube. This is the stable capillary cusp which is an essential characteristic of the invention needed to form the stable microjet. More particularly, a steady, thin gas jet with a typical diameter d_j is smoothly emitted from the stable cusp-like drop shape and this thin gaseous jet extends over a distance in the range of microns to millimeters. The length of the stable microjet will vary from very short (e.g. 1
15 micron) to very long (e.g. 50 mm) with the length depending on the (1) flow-rate of the gas (2) the Reynolds number of the gas and liquid streams flowing out of the exit opening of the pressure chamber and (3) the Weber number of the gas jet. The gas jet is the stable capillary microjet obtained when supercritical flow is reached. As mentioned, in the case of a gas jet the microjet may be so small as to be almost indistinguishable from the stable cusp. This jet
20 demonstrates a robust behavior provided that the pressure drop ΔP_1 applied to the liquid is sufficiently large compared to the maximum surface tension stress (on the order of γ/d_j) that act at the liquid-gas interface. The stable microjet is formed without the need for other forces, i.e. without adding force such as electrical forces on a charged fluid. However, for some applications it is preferable to add charge to particles, e.g. to cause the particles to
25 adhere to a given surface. The shaping of gas exiting the capillary tube by the liquid flow forming a focusing funnel creates a cusp-like meniscus resulting in the stable microjet. This is a fundamental characteristic of the invention.

 The microjet eventually destabilizes due to the effect of surface tension forces. Destabilization results from small natural perturbations moving downstream, with the fastest
30 growing perturbations being those which govern the break up of the microjet, eventually creating a uniform sized monodispersion of bubbles 70 as shown in Figure 4. The microjet,

even as it initially destabilizes, passes out of the exit orifice of the pressure chamber without touching the peripheral surface of the exit opening.

MATHEMATICS OF A STABLE MICROJET

5 Cylindrical coordinates (r, z) are chosen for analyzing the shape of a stable microjet, i.e. a liquid jet undergoing "supercritical flow." The cusp-like meniscus formed by the fluid coming out of the tube is pulled toward the exit of the pressure chamber by a pressure gradient created by the flow of a second, immiscible fluid.

10 The cusp-like meniscus formed at the tube's mouth is pulled towards the hole by the pressure gradient created by the liquid stream. From the cusp of this meniscus, a steady gas thread with the shape of radius $r = \xi$ is withdrawn through the hole by the action of both the suction effect due to ΔP_b and the tangential viscous stresses τ_s exerted by the liquid on the jet's surface in the axial direction. The averaged momentum equation for this configuration may be written (assuming $Dp_l \ll P_g R_g T$, where R_g is the gas constant and T its
15 temperature):

$$\frac{d}{dz} \left[P_g + \frac{\rho_g Q^2}{2\pi^2 \xi^4} \right] = \frac{2\tau_s}{\xi}, \quad (1)$$

where Q is the gas flow rate upon exiting the feeding tube, P_g is the gas pressure, and ρ_g is the gas density, assuming that the viscous extensional term is negligible compared to the kinetic energy term, as will be subsequently justified. The gas pressure P_g is given by the capillary equation.

$$P_g = P_l + \gamma/\xi. \quad (2)$$

20 where γ is the liquid-gas surface tension. As shown in the Examples, the pressure drop ΔP_l is sufficiently large as compared to the surface tension stress γ/ξ to justify neglecting the latter in the analysis. This scenario holds for the whole range of flow rates in which the microjet is absolutely stable. In fact, it will be shown that, for a given pressure drop ΔP_b , the minimum gas flow rate that can be sprayed in steady jet conditions is achieved when the
25 surface tension stress γ/ξ is of the order of the kinetic energy of the liquid (of the order of

Δp_l), since the surface tension acts like a "resistance" to the motion (it appears as a *negative* term in the right-hand side term of Eq. (1)). Thus,

$$Q_{min} \sim \left(\frac{\gamma d_j^3}{\rho_g} \right)^{1/2} \quad (3)$$

For sufficiently large flow rates Q compared to Q_{min} , the simplified averaged momentum equation in the axial direction can be expressed as

5

$$\frac{d}{dz} \left(\frac{\rho_g Q^2}{2\Pi^2 \xi^4} \right) = \frac{dP_l}{dz} + \frac{2\tau_s}{\xi}, \quad (4)$$

where one can identify the two driving forces for the gas flow on the right-hand side. In general, the pressure gradient in the liquid is on the average much larger than the viscous shear term $2\tau_s/\xi$ owing to the surface stress. On the other hand, the axial viscous forces in the gas are many orders of magnitude smaller than the pressure forces. The neglect of all viscous terms in Eq. (4) is then justified. Notice that in this limit on the gas flow is *quasi-isentropic* in the average (the liquid almost follows Euler-Bernoulli equations) as opposed to most micrometric extensional flows. Thus, integrating (4) from the stagnation regions of both fluids up to the exit, one obtains a *simple* and *universal* expression for the jet diameter at the hole exit:

10

$$d_j \approx \left(\frac{8\rho_g}{\Pi^2 \Delta P_l} \right)^{1/4} Q^{1/2}, \quad (5)$$

15 which for a given pressure drop ΔP_l is *independent* of geometrical parameters (hole and tube diameters, tube-hole distance, etc.), liquid and gas viscosities, and liquid-gas surface tension.

The proposed system obviously requires delivery of the gas to be atomized and the liquid to be used in the resulting drop production. Both should be fed at a rate ensuring that the system lies within the stable parameter window. Multiplexing is effective when the flow-
20 rates needed exceed those on an individual cell. More specifically, a plurality of

feeding sources or feeding needles may be used to increase the rate at which aerosols are created. The flow-rates used should also ensure the mass ratio between the flows is compatible with the specifications of each application.

5 The gas and liquid can be dispensed by any type of continuous delivery system (e.g. a compressor or a pressurized tank the former and a volumetric pump or a pressurized bottle the latter). If multiplexing is needed, the liquid flow-rate should be as uniform as possible among cells; this may entail propulsion through several capillary needles, porous media or any other medium capable of distributing a uniform flow among different feeding points.

10 Each individual device should consist of a feeding point (a capillary needle, a point with an open microchannel, a microprotuberance on a continuous edge, etc.) 0.002-2 mm (but, preferentially 0.01-0.4 mm) in diameter, where the drop emerging from the microjet can be anchored, and a small orifice 0.002-2 mm (preferentially 0.01-0.25 mm) in diameter facing the drop and separated 0.01-2 mm (preferentially 0.2-0.5 mm) from the feeding point. The orifice communicates the withdrawal liquid around the drop, at an increased pressure, with the zone where the atomizate is produced, at a decreased pressure. The device can be made from a variety of materials (metal, polymers, ceramics, glass).

Figure 1 depicts a tested prototype where the gas to be atomized is inserted through one end of the system 2 and the liquid is introduced via the special inlet 4 in the pressure chamber 3. The prototype was tested at gas feeding rates from 10 to 2000 mBar above the atmospheric pressure P_a at which the atomized gas was discharged. The whole enclosure around the feeding needle 1 was at a pressure $P_o > P_a$. The gas feeding pressure, P_g , should always be slightly higher than the liquid propelling pressure, P_o . Depending on the pressure drop in the needle and the gas feeding system, the pressure difference ($P_g - P_o >$ 0) and the flow-rate of the gas to be atomized, Q , are linearly related provided the flow is laminar - which is indeed the case with this prototype. The critical dimensions are the distance from the needle to the plate (H), the needle diameter (D_o), the diameter of the orifice through which the microjet 6 is discharged (d_o) and the axial length, e , of the orifice (i.e. the thickness of the plate where the orifice is made). In this prototype, H was varied from 0.3 to 0.7 mm on constancy of the distances ($D_o = 0.45$ mm, $d_o = 0.2$ mm) and $e = 0.5$ mm. The quality of the resulting spray 7 did not vary appreciably with changes in H provided the operating regime (i.e. stationary drop and microjet) was maintained. However,

the system stability suffered at the longer H distances (about 0.7 mm). The other atomizer dimensions had no effect on the spray or the prototype functioning provided the zone around the needle (its diameter) was large enough relative to the feeding needle.

5

WEBER NUMBER

Adjusting parameters to obtain a stable capillary microjet and control its breakup into monodisperse particle is governed by the Weber number and the liquid-to-gas velocity ratio or α which equal V_l/V_g . The Weber number or "We" is defined by the following equation:

$$We = \frac{\rho_l V_l^2 d}{\gamma}$$

10 wherein ρ_l is the density of the gas, d is the diameter of the stable microjet, γ is the liquid-gas surface tension, and V_l^2 is the velocity of the liquid squared.

When carrying out the invention the parameters should be adjusted so that the Weber number is greater than 1 in order to produce a stable capillary microjet. However, to obtain a particle dispersion which is monodisperse (i.e. each particle has the same size ± 0.01 15 to $\pm 30\%$) the parameters should be adjusted so that the Weber number is less than about 40. The monodisperse aerosol is obtained with a Weber number in a range of about 1 to about 40 ($1 \leq We \leq 40$).

OHNESORGE NUMBER

20

A measure of the relative importance of viscosity on the jet breakup can be estimated from the Ohnesorge number defined as the ratio between two characteristic times: the viscous time t_v and the breaking time t_b . The breaking time t_b is given by [see Rayleigh (1878)]

$$t_b \sim \left(\frac{\rho_l d^2}{\gamma} \right)^{1/2} \quad (2)$$

Perturbations on the jet surface are propagated inside by viscous diffusion in times t_v of the order of

$$t_v \sim \rho_l d^2 / \mu_l \quad (3)$$

where μ_l is the viscosity of the liquid. Then, the Ohnesorge number, Oh, results

$$Oh = \frac{\mu_l}{(\rho_l \gamma d)^{1/2}} \quad (4)$$

If this ratio is much smaller than unity viscosity plays no essential role in the phenomenon under consideration. Since the maximum value of the Ohnesorge number in actual experiments conducted is as low as 3.7×10^{-2} , viscosity plays no essential role during the process of jet breakup.

EMBODIMENT OF FIGURE 2

A variety of configurations of components and types of fluids will become apparent to those skilled in the art upon reading this disclosure. These configurations and fluids are encompassed by the present invention provided they can produce a stable capillary microjet of a first fluid from a source to an exit port of a pressure chamber containing a second fluid. The stable microjet is formed by the first fluid flowing from the feeding source to the exit port of the pressure chamber being accelerated and stabilized by pressure stress exerted by the second fluid in the pressure chamber on the surface of the first fluid forming the microjet. The second fluid forms a focusing funnel when a variety of parameters are correctly tuned or adjusted. For example, the speed, pressure, viscosity and miscibility of the first and second fluids are chosen to obtain the desired results of a stable microjet of the first fluid focused into the center of a funnel formed with the second fluid. These results are also obtained by adjusting or tuning physical parameters of the device, including the size of the opening from which the first fluid flows, the size of the opening from which both fluids exit, and the distance between these two openings.

The embodiment of Figure 1 can, itself, be arranged in a variety of configurations. Further, as indicated above, the embodiment may include a plurality of feeding needles. A

plurality of feeding needles may be configured concentrically in a single construct, as shown in Figure 2.

The components of the embodiment of Figure 2 are as follows:

21. Feeding needle - tube or source of fluid.
- 5 22. End of the feeding needle used to insert the liquids to be atomized.
23. Pressure chamber.
24. Orifice used as liquid inlet.
25. End of the feeding needle used to evacuate the gas to be atomized.
26. Orifice through which withdrawal takes place.
- 10 27. Atomizate (spray) or aerosol.
28. First gas to be atomized (inner core of particle).
29. Second fluid to be atomized (outer coating of particle).
30. Liquid for creation of microjet.
31. Internal tube of feeding needle.
- 15 32. External tube of feeding needle.

D = diameter of the feeding needle; d = diameter of the orifice through which the microjet is passed; e = axial length of the orifice through which withdrawal takes place; H = distance from the feeding needle to the microjet outlet; γ = surface tension; P_o = pressure inside the chamber; P_a = atmospheric pressure.

The embodiment of Figure 2 is preferably used when attempting to form a spherical particle of one substance surrounded by another substance. The device of Figure 2 is comprised of the same basic component as per the device of Figure 1 and further includes
 25 a second feeding source 32 which is positioned concentrically around the first cylindrical feeding source 31. The second feeding source may be surrounded by one or more additional feeding sources with each concentrically positioned around the preceding source.

The process is based on the microsuction which the liquid-gas or liquid-liquid interphase undergoes (if both are immiscible), when said interphase approaches a point
 30 beginning from which one of the fluids is suctioned off while the combined suction of the two fluids is produced. The interaction causes the fluid physically surrounded by the other to form a capillary microjet which finally breaks into spherical drops. If instead of two

fluids (gas-liquid), three or more are used that flow in a concentric manner by injection using concentric tubes, a capillary jet composed of two or more layers of different fluids is formed which, when it breaks, gives rise to the formation of spheres composed of several approximately concentric spherical layers of different fluids. The size of the outer sphere
5 (its thickness) and the size of the inner sphere (its volume) can be precisely adjusted. This can allow the manufacture of layered bubbles for a variety of end uses.

The method is based on the breaking of a capillary microjet composed of a nucleus of a gas and surrounded by other liquids and gases which are in a concentric manner injected by a special injection head, in such a way that they form a stable capillary microjet
10 and that they do not mix by diffusion during the time between when the microjet is formed and when it is broken. When the capillary microjet is broken into spherical drops under the proper operating conditions, which will be described in detail below, these drops exhibit a spherical nucleus, the size and eccentricity of which can be controlled.

In the case of spheres containing two materials, the injection head 25 consists of
15 two concentric tubes with an external diameter on the order of one millimeter. Through the internal tube 31 is injected the material that will constitute the nucleus of the microsphere, while between the internal tube 31 and the external tube 32 the coating is injected. The fluid of the external tube 32 joins with the fluid of tube 31 as the fluids exit the feeding needle, and the fluids thus injected are accelerated by a stream of gas or liquid that passes through a
20 small orifice 24 facing the end of the injection tubes. When the drop in pressure across the orifice 24 is sufficient, the fluids form a completely stationary capillary microjet, if the quantities of liquids that are injected are stationary. This microjet does not touch the walls of the orifice, but passes through it wrapped in the stream of gas or funnel formed by gas from the tube 32. Because the funnel of fluid focuses the exiting fluid, the size of the exit
25 orifice 26 does not dictate the size of the particles formed.

When the parameters are correctly adjusted, the movement of the fluid is uniform at the exit of the orifice 26 and the viscosity forces are sufficiently small so as not to alter either the flow or the properties of the liquids; for example, if there are biochemical
30 molecular specimens having a certain complexity and fragility, the viscous forces that would appear in association with the flow through a micro-orifice might degrade these substances.

Figure 2 shows a simplified diagram of the feeding needle 21, which is comprised of the concentric tubes 30, 31 through the internal and external flows of the fluids 28, 29

that are going to compose the microspheres comprised of two immiscible fluids. The difference in pressures $P_0 - P_a$ ($P_0 > P_a$) through the orifice 26 establishes a flow of liquid present in the chamber 23 and which is going to surround the microjet at its exit. The same pressure gradient that moves the liquid is the one that moves the microjet in an axial direction through the hole 26, provided that the difference in pressures $P_0 - P_a$ is sufficiently great in comparison with the forces of surface tension, which create an adverse gradient in the direction of the movement.

There are two limitations for the minimum sizes of the inside and outside jets that are dependent (a) on the surface tensions γ_1 of the outside fluid 29 with the liquid 30 and γ_2 of the outside fluid 29 with the inside fluid (e.g. gas) 28, and (b) on the difference in pressures $\Delta P = P_0 - P_a$ through the orifice 26. In the first place, the jump in pressures ΔP must be sufficiently great so that the adverse effects of the surface tension are minimized. This, however, is attained for very modest pressure increases: for example, for a 10 micron jet of a gas having a surface tension of 0.05 N/m (tap water), the necessary minimum jump in pressure is in the order of $0.05 \text{ (N/m)} / 0.00001 \text{ m} = \Delta P = 50 \text{ mBar}$. But, in addition, the breakage of the microjet must be regular and axilsymmetric, so that the drops will have a uniform size, while the extra pressure ΔP cannot be greater than a certain value that is dependent on the surface tension of the outside gas with the gas γ_1 and on the outside diameter of the microjet. It has been experimentally shown that this difference in pressures cannot be greater than 20 times the surface tension γ_1 divided by the outside radius of the microjet.

Therefore, given some inside and outside diameters of the microjet, there is a range of operating pressures between a minimum and a maximum; nonetheless, experimentally the best results are obtained for pressures in the order of two to three times the minimum.

The viscosity values of the gases must be such that the gases with the greater viscosity μ_{\max} verifies, for a diameter d of the jet predicted for this gas and a difference through the orifice ΔP , the inequality:

$$\mu_{\max} \leq \frac{\Delta P d^2 D}{Q}$$

With this, the pressure gradients can overcome the extensional forces of viscous resistance exerted by the gas when it is suctioned toward the orifice.

Moreover, the gases must have very similar densities in order to achieve the concentricity of the nucleus of the microsphere, since the relation of velocities between the gases moves according to the square root of the densities $v_1/v_2 = (\rho_2/\rho_1)^{1/2}$ and both jets, the inside jet and the outside jet, must assume the most symmetrical configuration possible, which does not occur if the fluids have different velocities (Figure 2). Nonetheless, it has been experimentally demonstrated that, on account of the surface tension γ_2 between the two fluids, the nucleus tends to migrate toward the center of the microsphere, within prescribed parameters.

The distance between the plane of the internal tube 31 (the one that will normally project more) and the plane of the orifice may vary between zero and three outside diameters of the external tube 32, depending on the surface tensions between the fluids and with the liquid, and on their viscosity values. Typically, the optimal distance is found experimentally for each particular configuration and each set of liquids used.

The proposed dispersion system obviously requires fluids that are going to be used in the resulting bubbles to have certain flow parameters. Accordingly, flows for this use must be:

- Flows that are suitable so that the system falls within the parametric window of stability. Multiplexing (i.e. several sets of concentric tubes) may be used, if the flows required are greater than those of an individual cell.

- Flows that are suitable so that the mass relation of the fluids falls within the specifications of each application. Of course, a greater flow of liquid may be supplied externally by any means in specific applications, since this does not interfere with the functioning of the atomizer.

Therefore, any means for continuous supply of gas (compressors, pressure deposits, etc.) and of liquid (volumetric pumps, pressure bottles, etc.) may be used. If multiplexing is desired, the flow of gas must be as homogeneous as possible between the various cells, which may require impulse through multiple capillary needles, porous media, or any other medium capable of distributing a homogeneous flow among different feeding points.

Each dispersion device will consist of concentric tubes 31, 32 with a diameter ranging between 0.05 and 2 mm, preferably between 0.1 and 0.4 mm, on which the drop from which the microjet emanates can be anchored, and a small orifice (between 0.001 and 2 mm in diameter, preferably between 0.1 and 0.25 mm), facing the drop and separated from the point of feeding by a distance between 0.001 and 2 mm, preferably between 0.2 and 0.5 mm. The orifice puts the liquid that surrounds the drop, at higher pressure, in touch with the area in which the dispersion is to be attained, at lower pressure.

EMBODIMENT OF FIGURE 3

The embodiments of Figures 1 and 2 are similar in a number of ways. Both have a feeding piece which is preferably in the form of a feeding needle with a circular exit opening. Further, both have an exit port in the pressure chamber which is positioned directly in front of the flow path of fluid out of the feeding source. Precisely maintaining the alignment of the flow path of the feeding source with the exit port of the pressure chamber can present an engineering challenge particularly when the device includes a number of feeding needles. The embodiment of Figure 3 is designed to simplify the manner in which components are aligned. The embodiment of Figure 3 uses a planar feeding piece, which by virtue of the withdrawal effect produced by the pressure difference across a small opening through which fluid is passed permits multiple microjets to be expelled through multiple exit ports of a pressure chamber thereby obtaining multiple aerosol streams. Although a single planar feeding member is shown in Figure 3 it, of course, is possible to produce a device with a plurality of planar feeding members where each planar feeding member feeds fluid to a linear array of outlet orifices in the surrounding pressure chamber. In addition, the feeding member need not be strictly planar, and may be a curved feeding device comprised of two surfaces that maintain approximately the same spatial distance between the two pieces of the feeding source. Such curved devices may have any level of curvature, e.g. circular, semicircular, elliptical, hemi-elliptical, etc.

The components of the embodiment of Figure 3 are as follows:

41. Feeding piece.
42. End of the feeding piece used to insert the gas to be dispersed.
43. Pressure chamber.
44. Orifice used as liquid inlet.

- 45. End of the feeding needle used to evacuate the gas to be dispersed.
- 46. Orifices through which withdrawal takes place.
- 47. Dispersion bubbles.
- 48. First fluid containing material to be dispersed.
- 5 49. Second fluid for creation of microjet.
- 50. Wall of the propulsion chamber facing the edge of the feeding piece.
- 51. Channels for guidance of fluid through feeding piece.

d_j = diameter of the microjet formed; ρ_A = density of first fluid (48); ρ_B = density of second fluid (49); v_A = velocity of the first fluid (48); v_B = velocity of the second fluid (49); e =
 10 axial length of the orifice through which withdrawal takes place; H = distance from the feeding needle to the microjet outlet; P_0 = pressure inside the chamber; Δp_g = change in pressure of the gas; P_a = atmospheric pressure; Q = volumetric flow rate

The proposed dispersion device consists of a feeding piece 41 which creates a planar feeding channel through which a where a first fluid 48 flows. The flow is
 15 preferably directed through one or more channels of uniform bores that are constructed on the planar surface of the feeding piece 41. A pressure chamber 43 that holds the propelling flow of a second liquid 49, houses the feeding piece 41 and is under a pressure above maintained outside the chamber wall 50. One or more orifices, openings or slots (outlets) 46 made in the wall 52 of the propulsion chamber face the edge of the feeding
 20 piece. Preferably, each bore or channel of the feeding piece 41 has its flow path substantially aligned with an outlet 46.

When the second fluid 49 is a liquid and the first fluid 48 is a gas, the facts that the liquid is much more viscous and that the gas is much less dense virtually equalize the fluid and gas velocities. The gas microthread formed is much shorter; however, because
 25 its rupture zone is almost invariably located in a laminar flowing stream, dispersion in the size of the microbubbles formed is almost always small. At a volumetric gas flow-rate Q_g and a liquid overpressure ΔP_l , the diameter of the gas microjet is given by

$$d_j \cong \left(\frac{8\rho_g}{\pi^2 \Delta P_l} \right)^{1/4} Q_g^{1/2}$$

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- 23 -

BUBBLES INTO LIQUID OR GAS

Figures 7 and 8 are useful in showing how bubbles may be formed in either a liquid (Figure 7) or a gas (Figure 8). In Figure 7 a tubular feeding source 71 is continually supplied with a flow of gas which forms a stable cusp 72 which is surrounded by the flow of liquid 73 in the pressure chamber 74 which is continually supplied with a flow of liquid 73. The liquid 73 flows out of the chamber 74 into a liquid 75 which may be the same as or different from the liquid 73.

The cusp 72 of gas narrows to a capillary supercritical flow 76 and then enter the exit opening 77 of the chamber 74. At a point 78 in the exit opening 77 the supercritical flow 76 begins to destabilize but remains as a critical capillary flow until leaving the exit opening 77. Upon leaving the exit opening 77 the gas stream breaks apart and forms bubbles 79 each of which are substantially identical to the others in shape and size. The uniformity of bubbles is such that one bubble differs from another (in terms of measured physical diameter) in an amount in a range of standard deviation of $\pm 0.01\%$ to $\pm 30\%$ with a preferred deviation being less than 1%. Thus, the uniformity in size of the bubbles is greater than the uniformity of the particles formed as described above in connection with Figure 1 when liquid particles are formed.

Gas in the bubbles 79 will diffuse into the liquid 75. Smaller bubbles provide for greater surface area contact with the liquid 75. Smaller bubbles provide for greater surface area contact with the liquid 75 thereby allowing for a faster rate of diffusion than would occur if the same volume of gas were present in a smaller number of bubbles. For example, ten bubbles each containing 1 cubic mm of gas would diffuse gas into the liquid much more rapidly than one bubble containing 10 cubic mm of gas. Further, smaller bubbles rise to the liquid surface more slowly than larger bubbles. A slower rate of ascent in the liquid means that the gas bubbles are in contact with the liquid for a longer period of time thereby increasing the amount of diffusion of gas into the liquid. Thus, smaller bubbles could allow a greater amount of oxygen to diffuse into water (e.g., to sewage or where fish are raised) or allow a greater amount of a toxic gas (e.g., a radioactive gas) to diffuse into a liquid thereby concentrating the toxin for disposal. Because the bubbles are so uniform in size the amount of gas diffusing into the liquid can be uniformly calculated which is important in certain applications such as when diffusing CO_2 into carbonated drinks.

Figure 8 shows the same components as shown in Figure 7 except that the liquid 75 is replaced with a gas 80. When the stream of bubbles 79 disassociate the liquid 73 forms an outer spherical cover thereby providing hollow droplets 81 which will float in the gas 80. The hollow droplets 81 have a large physical or actual diameter relative to their aerodynamic diameter. Hollow droplets fall in air at a much slower rate compared to liquid droplets of the same diameter. Because the hollow droplets 81 do not settle or fall quickly in air they can evaporate and diffuse the evaporated liquid into surrounding air. Eventually the hollow droplets 81 will burst and form many smaller particles which diffuse these particles into the surrounding air. Thus, it is understood that the aerodynamic diameter of the hollow droplets is very small compared to their actual physical diameter. The creation of hollow droplets 81 which burst and form very small particles is applicable in a wide range of different applications including the creation of mists of water for cooling systems.

EMULSIONS

Figure 9 is similar to Figures 7 and 8. However, rather than a gas 72 as in Figure 7 the feeding source 71 provides a stream of liquid 82 which may be miscible but is preferably immiscible in the liquid 73. Further the liquid 73 may be the same as or different from the liquid 75 but is preferably immiscible in the liquid 75. The creation of emulsions using such a configuration of liquids has applicability in a variety of fields particularly because the liquid particles formed can have a size in the range of from about 1 to about 200 microns with a standard deviation in size of one particle to another being as little as 0.01%. The size deviation of one particle to another can vary up to about 30% and is preferably less than $\pm 5\%$ and more preferably less than $\pm 1\%$.

The system operates to expel the liquid 82 out of the exit orifice 77 to form spheres 83 of liquid 82. Each sphere 83 has an actual physical diameter which deviates from other spheres 83 by a standard deviation of $\pm 0.01\%$ to $\pm 30\%$, preferably 10% or less and more preferably 1% or less. The size of the spheres 83 and flow rate of liquid 82 is controlled so that each sphere 83 contain a single particle (e.g. a single cell) to be examined. The stream of spheres 83 is caused to flow past a sensor and/or energy source of any desired type thereby allowing for sphere-by-sphere analysis of the sample of liquid 82.

EMULSIONS

In Figure 9 the liquid 75 can be a gas (e.g., air). The liquid 82 could be water which is surrounded by a second liquid 73 which is a compound for creating a desired odor e.g., a perfume. The system then forms particles 83 which have a water center and an outer coating of fuel. Such water/perfume particles will rapidly disperse the perfume at a low cost.

While the present invention has been described with reference to the specific embodiments thereof, it should be understood by those skilled in the art that various changes may be made and equivalents may be substituted without departing from the true spirit and scope of the invention. In addition, many modifications may be made to adapt a particular situation, material, composition of matter, process, process step or steps, to the objective, spirit and scope of the present invention. All such modifications are intended to be within the scope of the claims appended hereto.

15

THE CLAIMS DEFINING THE INVENTION ARE AS FOLLOWS:-

1. A monodispersion of bubbles for diffusing gas into a fluid, comprising gas bubbles in a liquid wherein the bubbles are characterized by having approximately the same diameter with a standard deviation in diameter from one bubble to another in a range of from $\pm 0.01\%$ to $\pm 30\%$, the monodispersion being produced by:
 - forcing a gas from a source opening into a first liquid in a manner so as to create a flow stream of the gas through the first liquid, wherein the gas is comprised of molecules to be diffused into a second liquid;
 - moving the first liquid in a pressure chamber surrounding the source opening, out of an exit orifice in the pressure chamber wherein the flow stream of the gas flows out the exit orifice into the second liquid wherein the flow stream breaks up forming bubbles of the gas in the second liquid; and
 - allowing molecules in the gas bubbles to diffuse into the second liquid.
2. The monodispersion of bubbles of claim 1, wherein the monodispersion comprises more than one thousand bubbles and the bubbles have a diameter in a range of from 0.1 micron to 100 microns.
3. The monodispersion of claim 1 or claim 2, wherein the gas is contaminated with a compound soluble in the fluid.
4. The monodispersion of any one of the preceding claims, wherein the gas is selected from the group consisting of air, oxygen, and carbon dioxide, and wherein the liquid is an aqueous liquid is selected from the group consisting of water, sweetened water, and sewage.
5. The monodispersion of any one of claims 2 to 4, wherein the bubbles are each substantially identical to each other in physical diameter with a standard deviation of $\pm 5\%$ or less.
6. The monodispersion of bubbles of claim 5, wherein the standard deviation is 1% or less.
7. The monodispersion of claim 1, wherein the bubbles have a size in a range of from 0.1 micron to 100 microns;



wherein the bubbles are emitted at regularly spaced intervals from the exit orifice of the pressure chamber; and

wherein the bubbles have a diameter in a range of from 1 micron to 20 microns and are comprised of a gas selected from the group consisting of air
5 and oxygen.

8. A monodispersion of bubbles for diffusing gas into a liquid, substantially as described with reference to the accompanying figures and examples.

10

Dated this sixth day of February 2002

Universidad De Sevilla
Patent Attorneys for the Applicant:

F B RICE & CO



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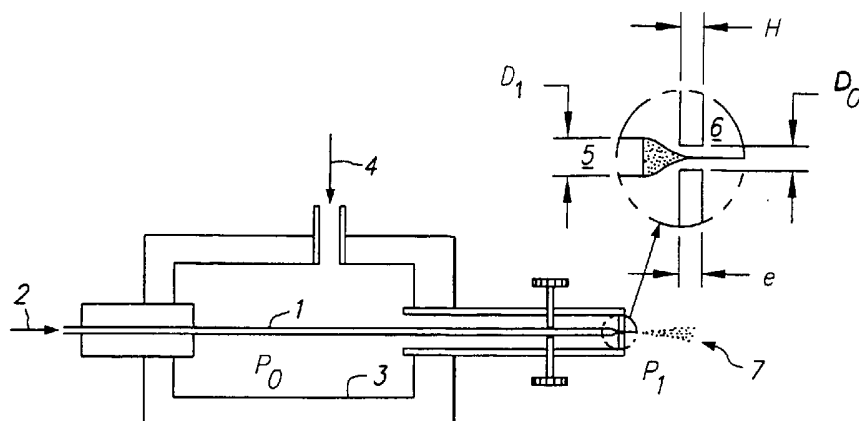


FIG. 1

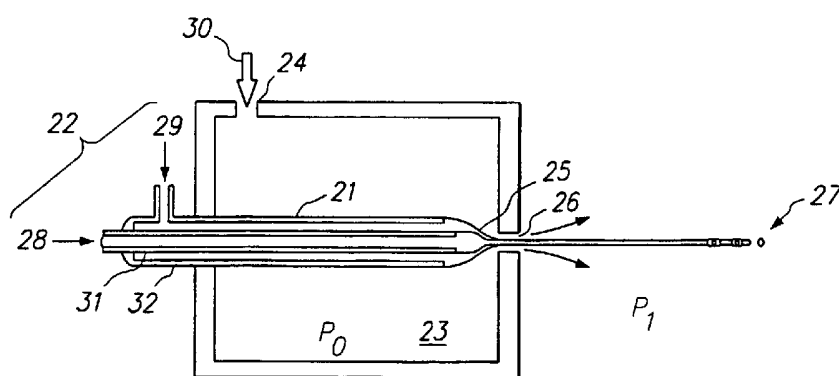


FIG. 2

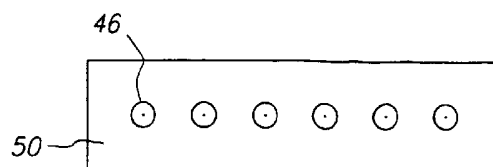


FIG. 3B

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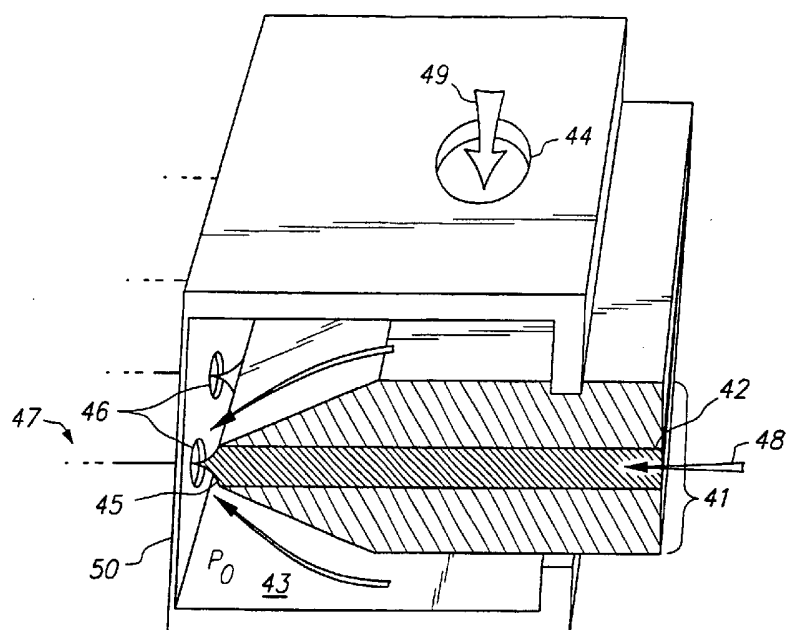


FIG. 3A

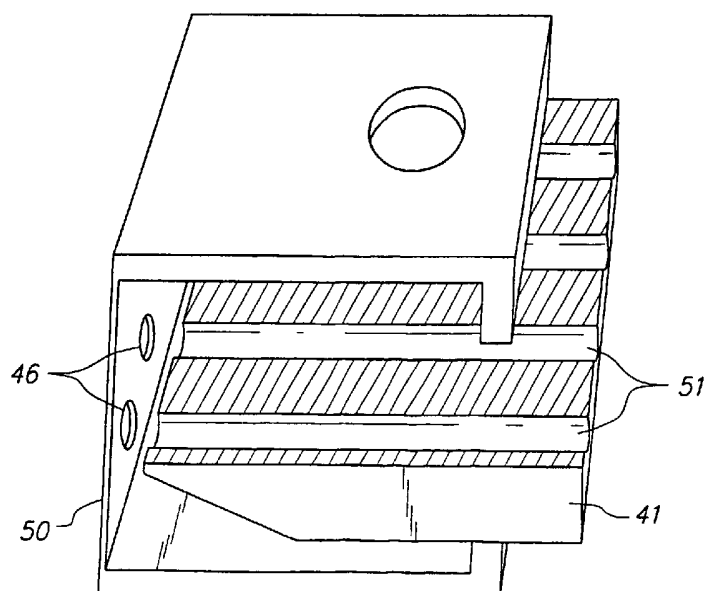


FIG. 3C

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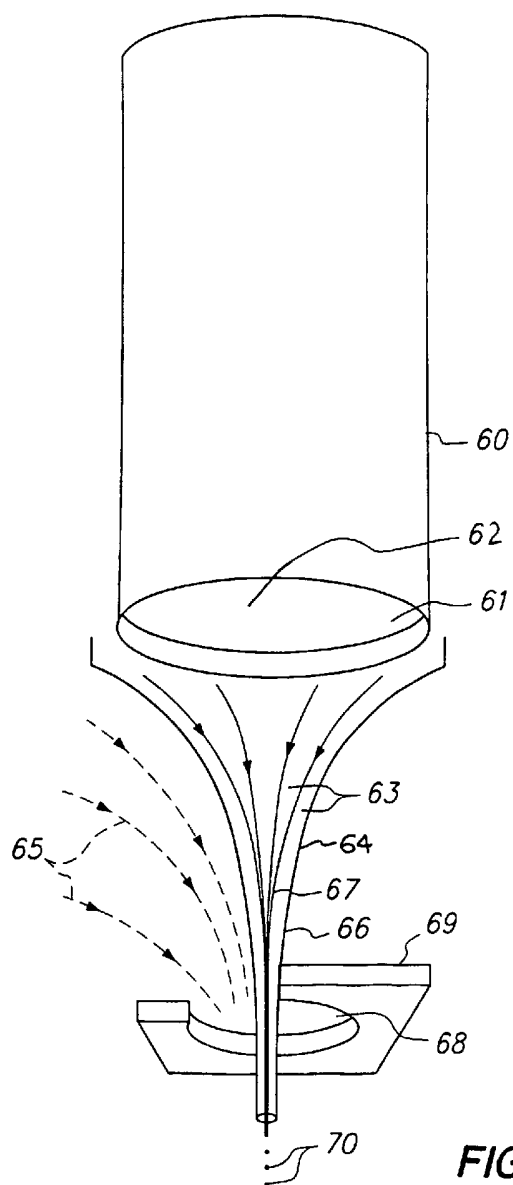
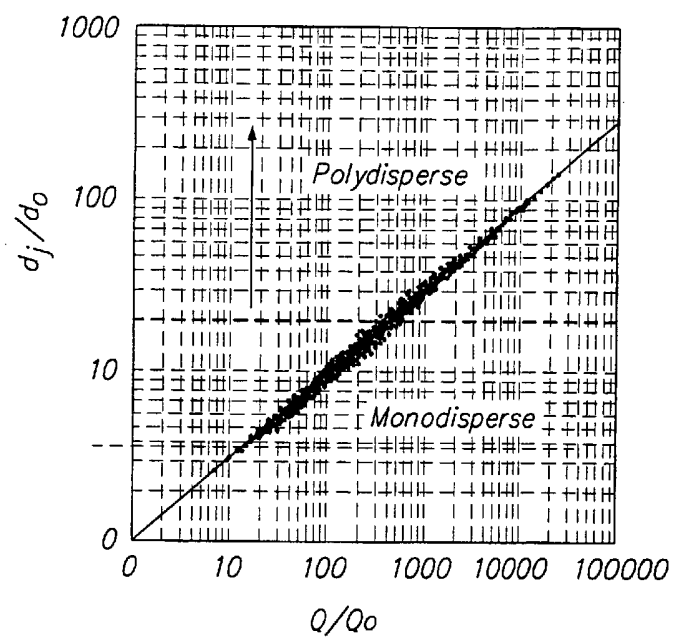


FIG. 4

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**FIG. 5**

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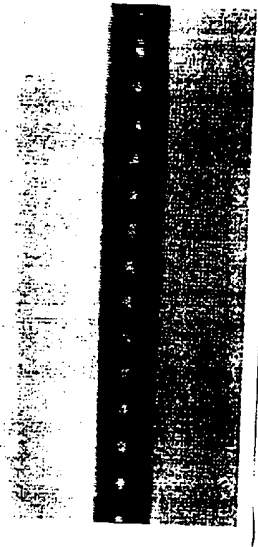


Fig. 6

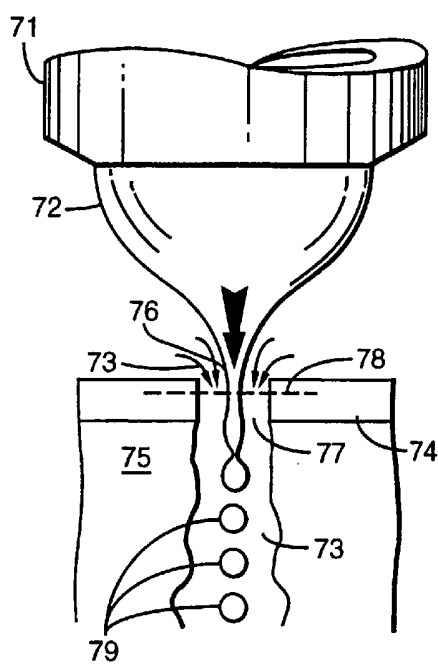


FIG. 7

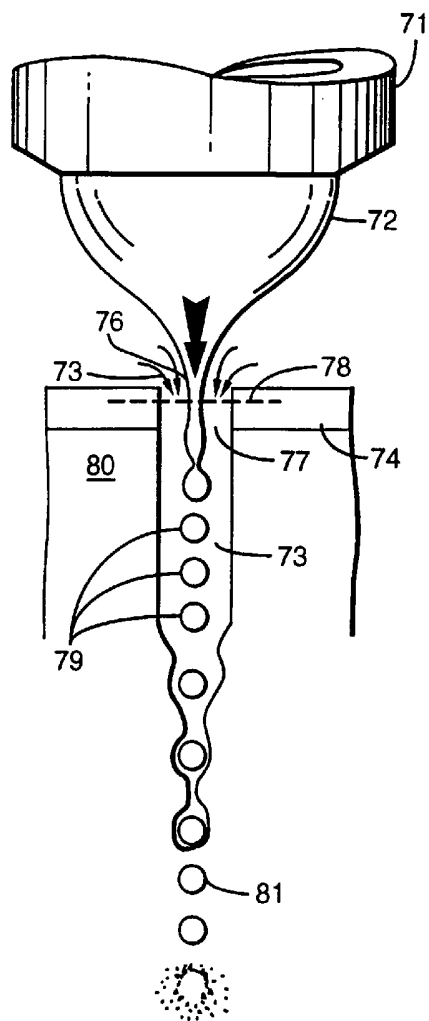


FIG. 8

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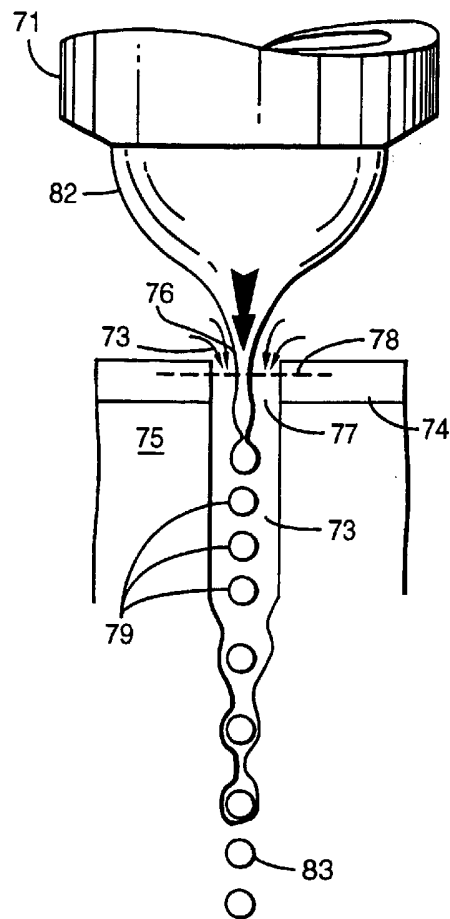


FIG. 9