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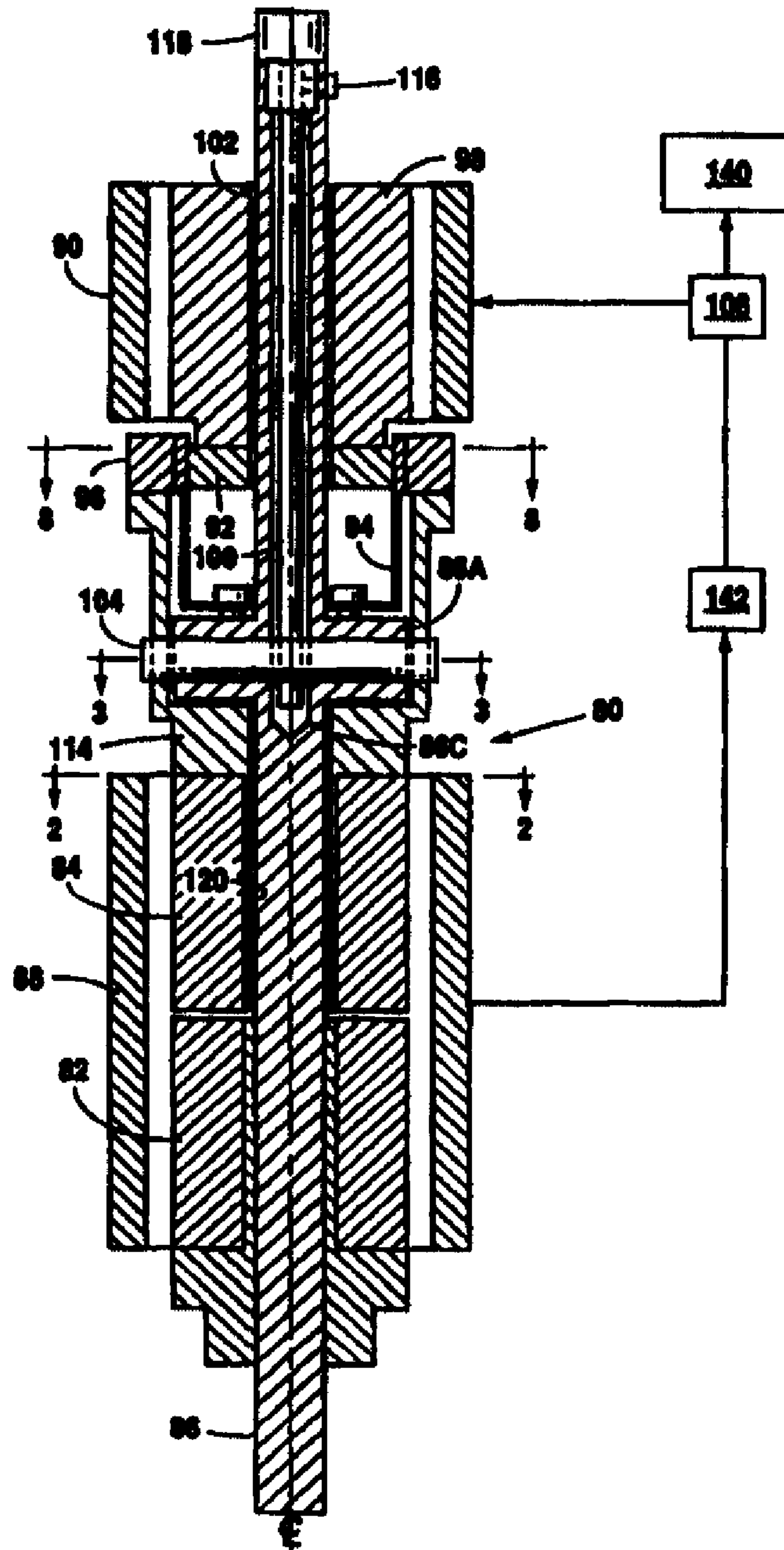
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(54) **GENERATEUR ROTATIF D'ENERGIE A SORTIE VARIABLE**

(54) **VARIABLE OUTPUT ROTARY POWER GENERATOR**





(57) La présente invention concerne un appareil de fonds de puits destiné à générer rapidement et à réguler l'énergie électrique à sortie variable par la modification de l'alignement d'une paire d'aimants permanents (84, 82), disposés axialement l'un près de l'autre et tournant à l'intérieur d'un induit (88) qui possède des enroulements conducteurs d'électricité. Chacun des aimants permanents (84, 82) possède plusieurs segments d'aimants permanents dont la magnétisation alterne de manière circulaire. Un des aimants permanents (82) est fixé à l'arbre d'entraînement (86), l'autre aimant permanent (84) étant monté mobile sur l'arbre d'entraînement (82) pour assurer l'alignement ou le désalignement, selon les besoins, des magnétisations des segments magnétiques respectifs sur la paire d'aimants permanents (84, 82). Lorsque les magnétisations sont complètement alignées, un maximum de puissance électrique est généré dans les enroulements de l'induit principal (88). Inversement, lorsque les magnétisations sont complètement désalignées, aucune énergie électrique n'est générée. Ainsi peut-on réguler la puissance de sortie en modifiant l'alignement de la paire d'aimants permanents (84, 82) au moyen d'un générateur de couple de traînée qui crée un couple de traînée, transmis à l'aimant permanent mobile (84) par un convertisseur de couple.

(57) The present invention is directed to a downhole apparatus for quickly generating and regulating variable output electric power by varying the alignment of a pair of axially adjacent permanent magnets (84, 82) rotating within an armature (88) having electrically conductive windings. Each of the permanent magnets (84, 82) comprises a plurality of permanent magnetic segments having circumferentially alternating magnetizations. One of the permanent magnets (82) is fixed to the drive shaft (86) and the other permanent magnet (84) is movably mounted on the drive shaft (82) to enable the alignment or misalignment, as desired, of the magnetizations of the respective magnetic segments on the pair of permanent magnets (84, 82). When the magnetizations are completely aligned, the maximum electrical power is generated in the windings of the main armature (88); conversely, when the magnetizations are completely misaligned, zero electrical power is generated. Thus, the output power is regulated by varying the alignment of the pair of permanent magnets (84, 82), which is accomplished with a drag torque generator which creates a drag torque that is transmitted to the movable permanent magnet (84) by a torque converter.

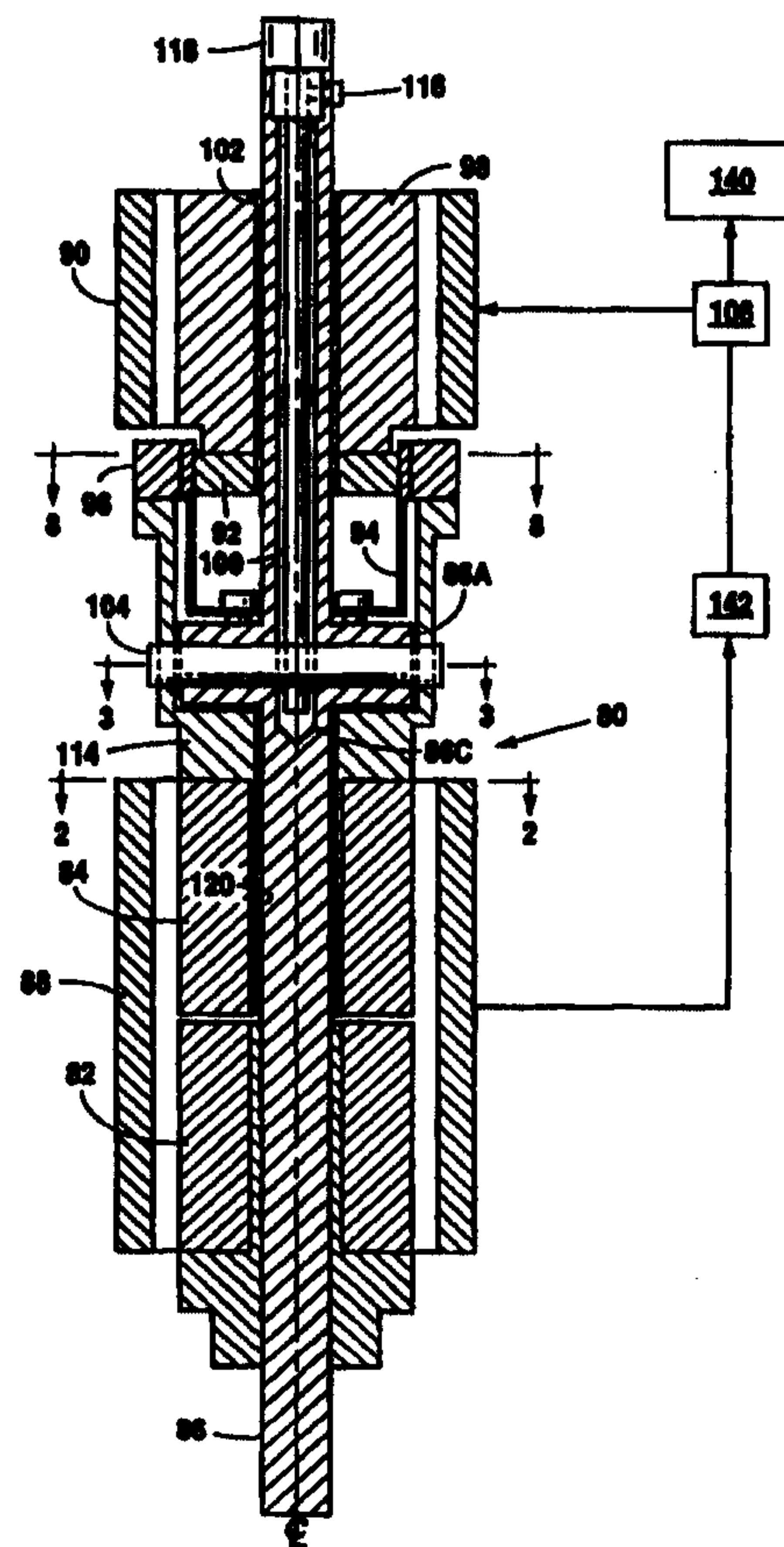
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(54) Title: VARIABLE OUTPUT ROTARY POWER GENERATOR**(57) Abstract**

The present invention is directed to a downhole apparatus for quickly generating and regulating variable output electric power by varying the alignment of a pair of axially adjacent permanent magnets (84, 82) rotating within an armature (88) having electrically conductive windings. Each of the permanent magnets (84, 82) comprises a plurality of permanent magnetic segments having circumferentially alternating magnetizations. One of the permanent magnets (82) is fixed to the drive shaft (86) and the other permanent magnet (84) is movably mounted on the drive shaft (82) to enable the alignment or misalignment, as desired, of the magnetizations of the respective magnetic segments on the pair of permanent magnets (84, 82). When the magnetizations are completely aligned, the maximum electrical power is generated in the windings of the main armature (88); conversely, when the magnetizations are completely misaligned, zero electrical power is generated. Thus, the output power is regulated by varying the alignment of the pair of permanent magnets (84, 82), which is accomplished with a drag torque generator which creates a drag torque that is transmitted to the movable permanent magnet (84) by a torque converter.



5 VARIABLE OUTPUT ROTARY POWER GENERATOR

This application claims priority from U.S. provisional application Ser. No. 60/071,611 filed January 16, 1998.

BACKGROUND OF THE INVENTION**10 1. Field of the Invention**

This invention relates generally to an apparatus for generating electric power downhole within an earth borehole. More specifically, this invention relates to a downhole apparatus for generating variable electric output by varying the alignment of permanent magnets
15 rotating within an armature having electrically conductive windings.

2. Description of the Related Art

In the field of petroleum well drilling and logging, recent advancements in drilling and logging technology have produced tools that require increasingly higher levels of electric energy downhole.
20 Moreover, for many modern drilling and logging systems, the electric energy requirements vary over a wide dynamic range of system operating conditions. Thus, during certain operating conditions, such systems require reduced electric energy, and the dissipation of any excess electric energy must not result in destructive effects within the
25 generator and/or the associated regulating electronics. Additionally, for a generator driven by a rotating shaft from a conventional mud-powered turbine, the rotational speed of the input shaft ("input RPM") often varies over a wide range, which, for a conventional permanent magnet generator, presents significant difficulties with respect to the
30 dissipation of excess energy because such dissipation must occur in a downhole environment that typically involves elevated ambient temperatures.

Before the present invention, existing downhole electric generators were not directed to meeting these variable energy

5 requirements. For example, U.S. Pat. No. 3,970,877, issued to Russell
et al. on July 20, 1976, discloses a method for generating downhole
electric energy using a means responsive to turbulence in the drilling
mud flow to convert vibratory motion into an electrical output.
However, the method of the '877 patent is directed to low-power
10 generation rather than high-power generation. Another drawback of
the method of the '877 patent is that it requires electronic devices to
rectify and smooth the electrical output, which is initially in the form of
relatively high-voltage pulses.

United States Pat. No. 4,396,071, issued to Stephens on August
15 2, 1983, discloses an apparatus for regulating the electrical output
produced by a conventional mud-powered turbine by means of a by-pass
valve to control the amount of mud flow passing through the turbine.
Although the '071 apparatus is directed to providing relatively constant
electrical output to meet the electrical demands of a downhole
20 measurement while drilling system, the '071 apparatus attempts to
accomplish that goal indirectly by controlling the input RPM to the
electrical generator rather than directly controlling the electrical output
of the generator regardless of the input RPM, which would be more
desirable. Additionally, the by-pass valve of the '071 apparatus would
25 suffer from the destructive effects of erosion that are frequently
encountered in the use of typical drilling fluids (muds). Similarly, U.S.
Pat. No. 4,491,738, issued to Kamp on January 1, 1985, discloses a
machine for generating electrical energy by controlling drilling fluid
dynamics, preferably in response to fluid pressure changes created by a
30 conventional mud-pulse telemetry system, to move a reciprocating
anchor comprising a plurality of magnets inside a stator, and U.S. Pat.
No. 4,515,225, issued to Dailey on May 7, 1985, discloses an apparatus
in which a fluid separate from the drilling mud is used to activate an

5 electrical generator. Again, however, the '738 and '225 machines are not directed to meeting the aforementioned variable electrical requirements.

It would, therefore, be a significant advancement in the art to provide an improved downhole apparatus for generating variable
10 electrical output over a wide range of input RPM and downhole system electrical requirements.

SUMMARY OF THE INVENTION

Accordingly, this invention is directed to a downhole apparatus for generating and regulating variable electric output for a downhole
15 drilling and/or logging system (generally referred to herein as a "system" or "downhole system"). The present invention accomplishes that objective by providing a pair of axially adjacent permanent magnets, each of which comprises a plurality of permanent magnetic segments having circumferentially alternating magnetizations, on a
20 drive shaft which is connected to a rotating shaft (such as a conventional mud-powered turbine) and thereby rotated within a fixed main armature having windings for carrying the generated electrical output. One of the permanent magnets is fixed to the drive shaft, and the other permanent magnet is movably mounted on the drive shaft to
25 enable the alignment or misalignment, as desired, of the magnetizations of the respective magnetic segments on the pair of permanent magnets. When the magnetizations are completely aligned, the maximum electrical output is generated in the windings of the main armature; conversely, when the magnetizations are completely
30 misaligned, zero electrical output is generated. The present invention thus avoids the aforementioned problem concerning dissipation of excess energy by not generating excess energy. Depending on the electrical requirements of the downhole system, the electrical output of

5 this invention may be tailored to meet current, voltage, or power specifications, as desired. A typical requirement is to provide relatively constant voltage regardless of the input RPM of the rotating drive shaft.

To accomplish this electric energy regulation, the present invention includes a drag torque generator which creates a drag torque that is transmitted to the movable permanent magnet by a torque converter (reduction gearing). The drag torque generator preferably comprises an electrically conductive rotor rotatably mounted to the drive shaft, which rotates inside a fixed drag armature having windings for carrying an electrical control current. A controller, which is responsive to the generated output, controls the electrical control current in the windings of the drag armature. The electrical control current produces a first magnetic field, which induces an eddy current in the electrically conductive rotor. In turn, the induced eddy current produces a second magnetic field that opposes the first magnetic field thereby creating a drag torque on the rotor. The rotor is connected to the torque converter, which converts the drag torque into a control torque to properly position the movable magnet and thereby regulate the amount of electrical output generated in the windings of the main armature.

BRIEF DESCRIPTION OF THE DRAWINGS

This invention may best be understood by reference to the following drawings:

Fig. 1 is a schematic axial cross-sectional view of an electric generator in accordance with the present invention.

Fig. 2 is a schematic cross-sectional view taken in direction 2-2 of Fig. 1 showing a movable magnet of the electric generator of Fig. 1.

5 Fig. 3 is a schematic cross-sectional view taken in direction 3-3 of Fig. 1 showing a stop pin and cooperating structure of the electric generator of Fig. 1.

Fig. 4 is a schematic elevational view taken in direction 4-4 of Fig. 3 showing a stop pin and cooperating structure of the electric
10 generator of Fig. 1.

Figs. 5A and 5B are schematic elevational views taken in direction 5-5 of Fig. 2 showing two different relative positions of fixed and movable magnets of the electric generator of Fig. 1.

Fig. 6 is a schematic axial cross-sectional view of an alternative
15 embodiment of an electric generator in accordance with the present invention.

Fig. 7 is a schematic cross-sectional view taken in direction 7-7 of Fig. 6 showing a biasing mechanism of the electric generator of Fig. 6.

Fig. 8 is a schematic cross-sectional view taken in direction 8-8 of
20 Figs. 1 and 6 showing a harmonic drive mechanism of the electric generators of Figs. 1 and 6.

Fig. 9 is a schematic axial cross-sectional view of an alternative embodiment of an electric generator in accordance with the present invention.

25 Fig. 10 is a schematic cross-sectional view taken in direction 10-10 of Fig. 9 showing an alternative drag element of the electric generator of Fig. 9.

Fig. 11 is a schematic exploded perspective view of a drag element and drag armature of the electric generator of Fig. 9.

30 DETAILED DESCRIPTION OF A PREFERRED EMBODIMENT

Figure 1 illustrates an electric generator 80 in accordance with the present invention for supplying electrical energy to a downhole system 140. Electric generator 80 is driven by a drive shaft 86 that is

5 preferably connected to a conventional mud-powered turbine (not
shown) and supported by bearings (not shown). Electric generator 80
comprises permanent magnets 82 and 84, which are preferably of equal
length and magnetic strength and which rotate inside a fixed main
armature 88 to generate downhole electric energy. Because such
10 electric energy is needed over a wide range of rotation speeds of drive
shaft 86 (*i.e.*, the mud-powered turbine) and electrical demands of
system 140, the electrical output must be controlled. The present
invention controls the electrical output by providing a regulator for
varying the relative rotational position of movable magnet 84 with
15 respect to fixed magnet 82. Specifically, fixed magnet 82 is fixedly
attached to drive shaft 86, but movable magnet 84 is mounted to a
carriage 114 that is mounted to drive shaft 86 with a bearing 120 such
that carriage 114 may rotate with respect to drive shaft 86. The degree
of relative rotation between carriage 114 and drive shaft 86 is
20 preferably limited by a stop pin 104 as discussed below.

As will be readily apparent to persons skilled in the art, the
present invention may be used to generate AC or DC electrical energy.
If this invention is used to generate DC electrical energy, a rectifier 142
is provided as shown in Fig. 1 to rectify the output from main armature
25 88 before it is fed into controller 106 and on to system 140. Similar
arrangements are shown in Figs. 6 and 9 for alternative embodiments
of this invention.

As shown in Figs. 2, 5A, and 5B (in which main armature 88,
carriage 114, bearing 120, and drive shaft 86 are not shown for clarity),
30 magnets 82 and 84 comprise a plurality of longitudinal permanent
magnet segments, which are preferably bonded to carriage 114. The
magnetization of the magnet segments alternates circumferentially
from north pointing radially outward to north pointing radially inward.

5 When magnets 82 and 84 are completely aligned as shown in Fig. 5A, the maximum electrical output will be generated. Conversely, when magnets 82 and 84 are completely misaligned as shown in Fig. 5B, zero electrical output will be generated. For the preferred embodiment shown, this range of movement is 45° (angle θ in Fig. 3). Thus, the
10 requisite amount of electrical output is achieved by positioning magnets 82 and 84 between these two extremes. As shown in Figs. 1, 3, and 4, a preferred embodiment limits this range of motion to the appropriate degree by means of a stop pin 104 that rotates within a transverse cavity in the form of a pair of symmetric sectors 86B within an enlarged
15 portion 86A of drive shaft 86. A biasing element 100, preferably of hexagonal cross-section, is installed through an axial cavity 86C in one end of drive shaft 86 and into a matching, preferably hexagonal, shaped hole in stop pin 104. Biasing element 100 serves to bias carriage 114 and movable magnet 84 in the maximum-output position with stop pin
20 104 against one extreme of sectors 86B. This biasing effect is accomplished by applying a torsional preload on biasing element 100 in the direction of the rotation of drive shaft 86 and securing biasing element 100 in the preloaded position with a set screw 116 contained in an end fitting 118. Stop pin 104 protrudes through a hole in carriage
25 114 and thereby rotates with carriage 114 when carriage 114 is rotated by a drag torque, as discussed below. After carriage 114 has been rotated from its initial position with respect to drive shaft 86 by means of a drag torque as discussed below, stop pin 104 serves to return carriage 114 to its initial position by means of biasing element 100.

30 Persons skilled in the art will recognize that the hexagonal shape of biasing element 100 and the corresponding hole in stop pin 104 are simply a convenient means of fastening biasing element 100 to stop pin 104 using a segment of a conventional hex key (Allen wrench). In

5 general, the shape need not be hexagonal so long as another means of
fastening biasing element 100 to stop pin 104 is provided.
Furthermore, the means of biasing carriage 114 and movable magnet
84 toward a certain position could take a variety of other forms, such as
a coil spring. Moreover, the biasing mechanism could be located outside
10 rather than inside drive shaft 86 if, for instance, electrical wires need to
be routed through the inside of drive shaft 86. Also, although a
preferred embodiment comprises a biasing mechanism, a biasing
mechanism is not absolutely necessary for all applications and could be
eliminated, if desired.

15 Persons skilled in the art will also recognize that the
configuration of magnets 82 and 84 and the relative rotation limiting
device for varying the amount of electrical output generated by an
embodiment of this invention may take a variety of other forms. For
example, the relative rotation may be limited to less than that which
20 would be required to achieve complete misalignment of magnets 82 and
84 such that the maximum allowable rotation produces a certain
fraction of maximum output instead of zero output. Alternatively,
magnets 82 and 84 may be made of unequal axial length such that
rotation into the completely misaligned position produces a certain
25 fraction of maximum output instead of zero output. Additionally, the
number of magnet segments comprising magnets 82 and 84 may be
varied such that a rotation angle other than 45° is required to achieve
complete misalignment. As shown in Fig. 2, the cross-sections of
magnets 82 and 84 preferably have a circular outer shape and a
30 polygonal inner shape. A circular outer shape is preferable for
providing an optimal magnetic field to cooperate with main
armature 88, and a polygonal inner shape is preferable for ease of
manufacture and to help prevent the magnet segments from de-bonding

5 from carriage 114 due to torsional loads. However, the outer and inner
shapes of magnets 82 and 84 may comprise other suitable shapes, as
will be readily recognized by persons skilled in the art. Because
magnets 82 and 84 may be of polygonal cross-section or circular cross-
section, the term "circumferential" as used herein to describe magnets
10 82 and 84 should be understood to mean the peripheral dimension of
those elements, whether flat or curved. Also, although the magnet
segments comprising magnets 82 and 84 are preferably of equal
circumferential dimensions, they may be of unequal circumferential
dimensions, if desired.

15 Further, the regulator for varying the position of movable
magnet 84 with respect to fixed magnet 82 may take a variety of forms.
Referring to Fig. 1, a preferred regulator comprises a drag element 98
mounted to a bearing 102 on drive shaft 86. Drag element 98, which
rotates inside a fixed drag armature 90, is preferably made of copper
20 and serves as a path for developing an eddy current. It should be
understood that copper is referred to as a preferred material for certain
elements of this invention, but any suitable conductive material could
be used in place of copper for such elements. In typical downhole
operations, fluctuations in parameters such as input RPM, electrical
25 demands of system 140, and ambient temperature tend to cause
fluctuations in the electrical output from main armature 88. Therefore,
a preferred regulator includes a controller 106 which contains suitable
electronics for monitoring the electrical output from main armature 88
and making appropriate adjustments to the input to drag armature 90,
30 as discussed below, to modify the electrical output from main armature
88 and thereby meet the electrical requirements of system 140.
Specifically, controller 106 generates an appropriate electrical control
current in the windings of drag armature 90, which sets up a first

5 magnetic field. The rotation of drag element 98 within the first
magnetic field creates an eddy current in drag element 98, which is a
function of (1) the magnetic field created by drag armature 90, (2) the
speed of rotation of drag element 98, (3) the conductivity of drag
element 98, and (4) the axial length of drag element 98. In turn, the
10 eddy current in drag element 98 produces a second magnetic field that
opposes the first magnetic field, which creates a drag torque on drag
element 98. Thus, drag element 98 (rotor) and drag armature 90
(stator) function as a drag torque generator. The drag torque causes
drag element 98 to rotate relative to drive shaft 86 in the direction
15 opposite that of the drive shaft rotation. Because drag element 98 is
connected to movable magnet 84 through a torque converter as
discussed below, the drag torque rotates movable magnet 84 relative to
fixed magnet 82 by an appropriate amount according to the applied
electrical control current. The relative movement of movable magnet 84
20 with respect to fixed magnet 82 modifies the electrical output from
main armature 88. Thus, as controller 106 senses deviations in the
output from main armature 88, controller 106 makes appropriate
modifications to the electrical control current in drag armature 90 to
cause appropriate modifications to the output from main armature 88
25 and thereby meet the electrical requirements of system 140.

Persons reasonably skilled in the art will recognize that the
required drag torque may be generated by a variety of other rotor/stator
configurations, such as: (1) a copper drag element rotating inside
permanent magnets housed in a fixed armature; (2) permanent
30 magnets rotating inside a fixed copper cylinder; (3) a copper drag
element rotating inside a motor-driven, rotatable drag armature
comprising a series of alternately magnetized permanent magnet
segments, similar to magnets 82 and 84 as shown in Fig. 2, which can

5 be rotated in either direction to advance or retard the drag element, as
appropriate; or (4) a drag element, comprising a series of alternately
magnetized permanent magnet segments similar to magnets 82 and 84
as shown in Fig. 2, rotating within a stator comprising windings which
can be energized to control the speed and direction of a rotating
10 magnetic field and thus advance or retard the drag element, as
appropriate. The foregoing options (1) and (2) would not include a
controller 106 and therefore would not be responsive to the output from
main armature 88; rather, those two open-loop options would be
responsive only to changes in drive shaft speed and would simply limit
15 the output from main armature 88. By contrast, the latter two options
(3) and (4) would provide an additional advantage of helping to reduce
the time that the apparatus takes to return to the initial, maximum-
output position by enabling the application of a "reverse" drag torque
(*i.e.*, a torque in the same direction as the rotation of drive shaft 86) to
20 drag element 98, thereby assisting biasing element 100 in moving
carriage 114 and movable magnet 84 back to their initial position. If
desired, option (1) or (2) could be used in conjunction with the other
drag torque generator configurations described herein to provide both a
rudimentary limit to the output and a more sophisticated output control
25 mechanism. The rudimentary limit provided by option (1) or (2) in such
a hybrid configuration may be desirable, for example, to prevent an
electrical overload in the event of failure of the electronics in controller
106. Of course, the drag torque could also be supplied by a mechanical
brake.

30 In the above-described configurations for drag element 98 and
drag armature 90, the drag torque is supplied by the interaction of
opposing magnetic fields formed around the circumference of drag
element 98 (referred to hereafter as "circumferential configurations").

5 However, it should also be understood that the drag torque could be
supplied by the interaction of opposing magnetic fields formed in a
plane normal to the longitudinal axis of drive shaft 86 (referred to
hereafter as "axial configurations"). Referring to Fig. 9, in an axial
10 configuration, a disc-like drag element 198 would replace drag element
98 from a circumferential configuration, and a disc-like drag armature
190 would replace drag armature 90. Thus, the opposing magnetic
fields would be formed in region 200 between drag element 198 and
drag armature 190. Appropriate selections for drag element 198 and
15 drag armature 190 would therefore create axial configuration
counterparts to any of the above-described circumferential
configurations. Specifically, axial configurations could comprise: (1) a
copper drag element rotating adjacent to a fixed drag armature having
windings for carrying electrical current to induce an eddy current in the
copper drag element; (2) a copper drag element rotating adjacent to a
20 fixed drag armature comprising permanent magnets for inducing an
eddy current in the copper drag element; (3) a drag element comprising
permanent magnets rotating adjacent to a fixed copper drag armature
for carrying an eddy current induced by the rotating permanent
magnets of the drag element; (4) a copper drag element rotating
25 adjacent to a motor-driven, rotatable drag armature comprising a series
of alternately magnetized permanent magnets, similar to those shown
in Fig. 10, for inducing an eddy current in the drag element and which
can be rotated in either direction to advance or retard the drag element,
as appropriate; or (5) a drag element, comprising a series of alternately
30 magnetized permanent magnets as shown in Fig. 10, rotating adjacent
to a fixed drag armature comprising windings which can be energized
to control the speed and direction of a rotating magnetic field to interact
with the magnetic field of the magnets on the drag element and thus

5 advance or retard the drag element, as appropriate. As an example,
Figs. 9, 10, and 11 illustrate the cooperation of drag element 198 and
drag armature 190 for the above-described axial configuration (5). In
Fig. 11, drive shaft 86 is not shown for clarity. Also, for the sake of
clarity, Figs. 10 and 11 do not show a backing plate that is preferably of
10 high magnetic permeability and is preferably bonded to the magnets of
drag element 198 on the face of the magnets that is not adjacent to drag
armature 190 (*i.e.*, the face of the magnets that is visible in Figs. 10 and
11). Such a backing plate, which is preferably machined as an integral
part of the polygonal core of drag element 198, serves to more
15 adequately complete the magnetic circuit of the magnets of drag
element 198 and also serves as an extra means of attachment to
prevent the magnets from flying off the core due to centrifugal forces of
rotation.

To achieve the desired movement of carriage 114 with as small a
20 drag torque as possible, the drag torque generated on drag element 98
is preferably multiplied using a torque converter as it is transmitted to
carriage 114. In a preferred embodiment, the torque converter
comprises a harmonic drive mechanism such as those sold by Harmonic
Drive Technologies, Inc. and HD Systems, Inc. Alternatively, the
25 torque converter could comprise other known gear mechanisms, such as
a planetary gear mechanism. Although it may be possible to eliminate
the torque converter in certain embodiments of this invention, the
absence of a torque converter would increase the input torque
requirements to unacceptable levels in most instances.

30 As shown in Figs. 1 and 8, a preferred harmonic drive
mechanism comprises a wave generator 92, a flexspline 94, and a
circular spline 96. Drag element 98 is fixedly connected to wave
generator 92, and circular spline 96 is fixedly connected to carriage 114

5 which comprises movable magnet 84. Circular spline 96 is relatively stiff and has internal teeth to engage flexspline 94. Flexspline 94, which is of slightly smaller diameter than circular spline 96 and has fewer teeth (usually two fewer) than circular spline 96, is relatively flexible and has external teeth to engage circular spline 96. Wave
10 generator 92 comprises an elliptical, thin raced ball bearing that fits inside flexspline 94 and causes flexspline 94 to engage circular spline 96 at each end of the major axis of the ellipse. Wave generator 92, flexspline 94, and circular spline 96 cooperate such that each revolution of wave generator 92 causes circular spline 96 to rotate by only two
15 teeth, for example. Thus, the drag torque on drag element 98 is multiplied as transmitted to carriage 114 and movable magnet 84 as a control torque. A tradeoff for achieving this torque multiplication is that the harmonic drive mechanism increases the response time of the apparatus. However, if desired, the use of a motor-driven, rotatable
20 drag armature as mentioned above would help to decrease the response time.

Because power generator 80 comprises a brushless, noncontact apparatus, it has an additional advantage of being capable of operating while immersed in oil. Thus, if oil is needed for pressure balancing due
25 to high downhole pressures, this generator can safely operate in an oil-filled compartment.

Persons skilled in the art will recognize that other advantageous configurations are possible to vary the amount of electrical output generated by an embodiment of this invention. For example, referring
30 to Fig. 6, circular spline 96 could be fixedly attached to a spline mounting 130, which is fixedly attached to drive shaft 86. In this alternative configuration, flexible spline 94 is rotatably mounted on drive shaft 86 with a bearing 122. Flexible spline 94 is fixedly

5 connected to carriage 114 with guide pins 124, which extend through
clearance guide slots 130A in spline mounting 130, as shown in Fig. 7.
The biasing mechanism in this alternative configuration preferably
comprises springs 126 fastened to guide pins 124 and bias pins 128,
which protrude from spline mounting 130. Drag element 98 and wave
10 generator 92 are configured in the same manner as in Fig. 1. However,
in the configuration of Fig. 6, the direction of the harmonic drive output
is reversed from that of Fig. 1 such that the control torque is in the
direction of rotation of drive shaft 86. Springs 126 and bias pins 128
are therefore configured to bias movable magnet 84 in the direction
15 opposite the rotation of drive shaft 86. Thus, through conversion into
an opposite control torque, the drag torque on drag element 98 operates
to decrease the generated electrical output by misaligning magnets 82
and 84 in the opposite direction as that described above for the
configuration of Fig. 1. Additionally, the reactive torque produced by
20 the generated electrical output would help to decrease the response
time required to return to the initial, maximum-output position.

Yet another advantageous configuration may be to fix the initial
relationship of magnets 82 and 84 in a certain degree of misalignment
such that the default electrical output is somewhat less than the
25 maximum possible output. Indeed, it may be beneficial in certain
applications to have an initial relationship of complete misalignment of
magnets 82 and 84 such that the initial electrical output is zero. By
selecting the proper arrangement of the harmonic drive output
direction and the direction of the biasing torque, the drag torque could
30 be made to increase or decrease the electrical output, as desired.
However, in a preferred embodiment of this invention, controller 106 is
powered by a portion of the output from main armature 88. Therefore,
an initial relationship of complete misalignment of magnets 82 and 84

5 which produces zero initial output generally would not be desirable unless an alternate power source is provided for controller 106.

Still another advantageous configuration may be to have a threaded cooperation of movable magnet 84 on drive shaft 86 such that the drag torque created on drag element 98 translates movable magnet
10 84 axially and thereby changes the electrical output by changing the percentage of movable magnet 84 that is encompassed by main armature 88. Such a threaded configuration would also vary the electrical output by changing the separation distance between fixed magnet 82 and movable magnet 84.

15 Because the present invention is intended to be able to operate at elevated downhole temperatures, the various magnets referred to herein preferably comprise samarium-cobalt (Sm-Co) magnets. Although some other types of magnets, such as neodymium-iron-boron (Nd-Fe-B) magnets, generally provide better magnetic flux at lower
20 temperatures, Sm-Co magnets maintain better energy density at temperatures above about 150°C. However, any suitable type of magnets may be used, if desired.

Although the foregoing specific details describe a preferred embodiment of this invention, persons reasonably skilled in the art of
25 electric power generation for petroleum well drilling and logging will recognize that various changes may be made in the details of the apparatus of this invention without departing from the spirit and scope of the invention as defined in the appended claims. Therefore, it should be understood that this invention is not to be limited to the specific
30 details shown and described herein.

5

CLAIMS

I claim:

1. An apparatus for generating variable downhole electrical energy from a rotating shaft, comprising:

a drive shaft having a longitudinal drive shaft axis;

10 a first permanent magnet fixedly attached to said drive shaft and comprising a first plurality of longitudinal permanent magnetic segments having magnetizations circumferentially alternating between north pointing inward toward said drive shaft axis and north pointing outward away from said drive shaft axis;

15 a second permanent magnet movably mounted on said drive shaft axially adjacent to said first permanent magnet and comprising a second plurality of longitudinal permanent magnetic segments having magnetizations circumferentially alternating between north pointing inward toward said drive shaft axis and north pointing outward away
20 from said drive shaft axis;

a main armature surrounding at least a portion of each of said first and second permanent magnets and having windings for generating electrical output due to the rotation of said first and second permanent magnets within said main armature; and

25 a regulator operably connected to said second permanent magnet for regulating said electrical output by varying the position of said second permanent magnet with respect to said first permanent magnet.

2. The apparatus of claim 1 wherein said first plurality and said second plurality are equal.

30 3. The apparatus of claim 2 wherein said first plurality and said second plurality are equal to eight.

4. The apparatus of claim 1 wherein said first and second permanent magnets have a circular outer cross-sectional shape.

5 5. The apparatus of claim 1 wherein said first and second permanent magnets have a polygonal outer cross-sectional shape.

6. The apparatus of claim 5 wherein said outer cross-sectional shape is octagonal.

7. The apparatus of claim 1 wherein said first and second
10 permanent magnets have a circular inner cross-sectional shape.

8. The apparatus of claim 1 wherein said first and second permanent magnets have a polygonal inner cross-sectional shape.

9. The apparatus of claim 8 wherein said inner cross-sectional shape is octagonal.

15 10. The apparatus of claim 1 wherein the circumferential dimension of each of said first plurality of longitudinal permanent magnetic segments is substantially the same.

11. The apparatus of claim 1 wherein the circumferential dimension of each of said second plurality of longitudinal permanent
20 magnetic segments is substantially the same.

12. The apparatus of claim 1 wherein the circumferential dimension of each of said first plurality of longitudinal permanent magnetic segments and said second plurality of longitudinal permanent magnetic segments is substantially the same.

25 13. The apparatus of claim 1 wherein the axial dimension of each of said first plurality of longitudinal permanent magnetic segments and said second plurality of longitudinal permanent magnetic segments is substantially the same.

14. The apparatus of claim 1 further comprising a biasing
30 mechanism for biasing said second permanent magnet toward a predetermined position with respect to said first permanent magnet.

15. The apparatus of claim 1 wherein said regulator may vary the rotational alignment of said second permanent magnet with respect

5 to said first permanent magnet between a first relative position for which said electrical output is a minimum value and a second relative position for which said electrical output is a maximum value.

16. The apparatus of claim 1 wherein said regulator may vary the axial separation of said second permanent magnet with respect to
10 said first permanent magnet between a first relative position for which said electrical output is a minimum value and a second relative position for which said electrical output is a maximum value.

17. The apparatus of claim 1 wherein said regulator may vary the rotational alignment and the axial separation of said second
15 permanent magnet with respect to said first permanent magnet between a first relative position for which said electrical output is a minimum value and a second relative position for which said electrical output is a maximum value.

18. The apparatus of any one of claims 15, 16, and 17 wherein
20 said minimum value is zero.

19. The apparatus of any one of claims 15, 16, and 17 wherein said minimum value is greater than zero.

20. An apparatus for generating variable downhole electric energy from a rotating shaft for a downhole system, comprising:

25 a drive shaft having a longitudinal drive shaft axis, said drive shaft having a first end and a second end along said drive shaft axis;

a first permanent magnet fixedly attached to said drive shaft and comprising a first plurality of longitudinal permanent magnetic segments having magnetizations circumferentially alternating between
30 north pointing inward toward said drive shaft axis and north pointing outward away from said drive shaft axis;

a carriage rotatably mounted on said drive shaft axially adjacent to said first permanent magnet and comprising a second permanent

5 magnet comprising a second plurality of longitudinal permanent magnetic segments having magnetizations circumferentially alternating between north pointing inward toward said drive shaft axis and north pointing outward away from said drive shaft axis;

a main armature surrounding at least a portion of each of said
10 first and second permanent magnets and having windings for generating electrical output due to the rotation of said first and second permanent magnets within said main armature; and

a regulator operably connected to said carriage and comprising a drag torque generator for generating a drag torque on said carriage in
15 the direction opposite the rotation of said drive shaft thereby regulating said electrical output by varying the rotational alignment of said second permanent magnet with respect to said first permanent magnet.

21. The apparatus of claim 20 wherein said drag torque generator comprises:

20 an electrically conductive drag element rotatably mounted to said drive shaft and operably connected to said carriage; and

a fixed drag armature surrounding at least a portion of said drag element and having permanent magnets for generating a first magnetic field;

25 wherein said first magnetic field induces an eddy current in said drag element which in turn creates a second magnetic field opposed to said first magnetic field thereby generating said drag torque.

22. The apparatus of claim 20 wherein said drag torque generator comprises:

30 a drag element rotatably mounted to said drive shaft and operably connected to said carriage, said drag element comprising a third plurality of longitudinal permanent magnetic segments having magnetizations circumferentially alternating between north pointing

5 inward toward said drive shaft axis and north pointing outward away from said drive shaft axis for creating a first magnetic field; and

a fixed, electrically conductive drag armature surrounding at least a portion of said drag element;

10 wherein said first magnetic field induces an eddy current in said drag armature which in turn creates a second magnetic field opposed to said first magnetic field thereby generating said drag torque.

23. The apparatus of claim 20 wherein:

15 said regulator further comprises an electronic controller operably connected to said main armature, said drag torque generator, and said downhole system for monitoring said electrical output, comparing said electrical output to the electrical requirements of said downhole system, and sending control signals to said drag torque generator to modify said electrical output to match the electrical requirements of said downhole system; and

20 said drag torque generator comprises (a) an electrically conductive drag element rotatably mounted to said drive shaft and operably connected to said carriage and (b) a fixed drag armature surrounding at least a portion of said drag element and having windings for carrying an electrical control current in response to said control signals from said controller;

25 wherein said electrical control current creates a first magnetic field which induces an eddy current in said drag element which in turn creates a second magnetic field opposed to said first magnetic field thereby generating said drag torque.

30 24. The apparatus of claim 20 wherein:

said regulator further comprises an electronic controller operably connected to said main armature, said drag torque generator, and said downhole system for monitoring said electrical output, comparing said

5 electrical output to the electrical requirements of said downhole system,
and sending control signals to said drag torque generator to modify said
electrical output to match the electrical requirements of said downhole
system; and

said drag torque generator comprises (a) an electrically
10 conductive drag element rotatably mounted to said drive shaft and
operably connected to said carriage and (b) a rotatable drag armature
surrounding at least a portion of said drag element and comprising a
third plurality of longitudinal permanent magnetic segments having
15 magnetizations circumferentially alternating between north pointing
inward toward said drive shaft axis and north pointing outward away
from said drive shaft axis for creating a first magnetic field, said drag
armature being rotationally responsive to said control signals from said
controller;

wherein said first magnetic field induces an eddy current in said
20 drag element which in turn creates a second magnetic field opposed to
said first magnetic field thereby generating said drag torque.

25. The apparatus of claim 24 wherein said drag armature is
capable of rotation in such a manner as to create a reverse drag torque.

26. The apparatus of claim 20 wherein:

25 said regulator further comprises an electronic controller operably
connected to said main armature, said drag torque generator, and said
downhole system for monitoring said electrical output, comparing said
electrical output to the electrical requirements of said downhole system,
and sending control signals to said drag torque generator to modify said
30 electrical output to match the electrical requirements of said downhole
system; and

said drag torque generator comprises (a) a drag element
rotatably mounted to said drive shaft and operably connected to said

5 carriage, said drag element comprising a third plurality of longitudinal
permanent magnetic segments having magnetizations circumferentially
alternating between north pointing inward toward said drive shaft axis
and north pointing outward away from said drive shaft axis for creating
a first magnetic field and (b) a fixed drag armature surrounding at least
10 a portion of said drag element and having windings for carrying an
electrical control current in response to said control signals from said
controller;

wherein said electrical control current creates a second, rotating
magnetic field opposed to said first magnetic field thereby generating
15 said drag torque.

27. The apparatus of claim 26 wherein said second, rotating
magnetic field is capable of rotation in such a manner as to create a
reverse drag torque.

28. The apparatus of claim 20 wherein said drag torque
20 generator comprises:

an electrically conductive drag element rotatably mounted to said
drive shaft and operably connected to said carriage; and

a fixed drag armature axially adjacent to said drag element and
having permanent magnets for generating a first magnetic field;

25 wherein said first magnetic field induces an eddy current in said
drag element which in turn creates a second magnetic field opposed to
said first magnetic field thereby generating said drag torque.

29. The apparatus of claim 20 wherein said drag torque
generator comprises:

30 a drag element rotatably mounted to said drive shaft and
operably connected to said carriage, said drag element comprising a
third plurality of permanent magnetic segments having magnetizations
circumferentially alternating between north pointing toward said first

5 end of said drive shaft and north pointing toward said second end of
said drive shaft for creating a first magnetic field; and

a fixed, electrically conductive drag armature axially adjacent to
said drag element;

10 wherein said first magnetic field induces an eddy current in said
drag armature which in turn creates a second magnetic field opposed to
said first magnetic field thereby generating said drag torque.

30. The apparatus of claim 20 wherein:

15 said regulator further comprises an electronic controller operably
connected to said main armature, said drag torque generator, and said
downhole system for monitoring said electrical output, comparing said
electrical output to the electrical requirements of said downhole system,
and sending control signals to said drag torque generator to modify said
electrical output to match the electrical requirements of said downhole
system; and

20 said drag torque generator comprises (a) an electrically
conductive drag element rotatably mounted to said drive shaft and
operably connected to said carriage and (b) a fixed drag armature
axially adjacent to said drag element and having windings for carrying
an electrical control current in response to said control signals from
25 said controller;

wherein said electrical control current creates a first magnetic
field which induces an eddy current in said drag element which in turn
creates a second magnetic field opposed to said first magnetic field
thereby generating said drag torque.

30 31. The apparatus of claim 20 wherein:

said regulator further comprises an electronic controller operably
connected to said main armature, said drag torque generator, and said
downhole system for monitoring said electrical output, comparing said

5 electrical output to the electrical requirements of said downhole system,
and sending control signals to said drag torque generator to modify said
electrical output to match the electrical requirements of said downhole
system; and

said drag torque generator comprises (a) an electrically
10 conductive drag element rotatably mounted to said drive shaft and
operably connected to said carriage and (b) a rotatable drag armature
axially adjacent to said drag element and comprising a third plurality
of permanent magnetic segments having magnetizations
15 circumferentially alternating between north pointing toward said first
end of said drive shaft and north pointing toward said second end of
said drive shaft for creating a first magnetic field, said drag armature
being rotationally responsive to said control signals from said
controller;

wherein said first magnetic field induces an eddy current in said
20 drag element which in turn creates a second magnetic field opposed to
said first magnetic field thereby generating said drag torque.

32. The apparatus of claim 31 wherein said drag armature is
capable of rotation in such a manner as to create a reverse drag torque.

33. The apparatus of claim 20 wherein:

25 said regulator further comprises an electronic controller operably
connected to said main armature, said drag torque generator, and said
downhole system for monitoring said electrical output, comparing said
electrical output to the electrical requirements of said downhole system,
and sending control signals to said drag torque generator to modify said
30 electrical output to match the electrical requirements of said downhole
system; and

said drag torque generator comprises (a) a drag element
rotatably mounted to said drive shaft and operably connected to said

5 carriage, said drag element comprising a third plurality of permanent
magnetic segments having magnetizations circumferentially
alternating between north pointing toward said first end of said drive
shaft and north pointing toward said second end of said drive shaft for
creating a first magnetic field and (b) a fixed drag armature axially
10 adjacent to said drag element and having windings for carrying an
electrical control current in response to said control signals from said
controller;

wherein said electrical control current creates a second, rotating
magnetic field opposed to said first magnetic field thereby generating
15 said drag torque.

34. The apparatus of claim 33 wherein said second, rotating
magnetic field is capable of rotation in such a manner as to create a
reverse drag torque.

35. The apparatus of claim 20 wherein said regulator further
20 comprises a torque converter operably connected to said drag torque
generator and said carriage for converting said drag torque to a control
torque and transferring said control torque to said carriage.

36. The apparatus of claim 35 wherein said torque converter
comprises a harmonic drive mechanism.

25 37. The apparatus of claim 36 wherein said harmonic drive
mechanism comprises:

a circular spline fixedly mounted to said carriage and having a
plurality of internal teeth;

30 a flexspline fixedly mounted to said drive shaft and having a
plurality of external teeth for engaging said plurality of internal teeth
of said circular spline, with said plurality of external teeth being fewer
than said plurality of internal teeth; and

5 a wave generator fixedly mounted to said drag torque generator and having an elliptical, thin raced ball bearing operatively engaged inside said flexspline such that said external teeth of said flexspline engage said internal teeth of said circular spline at each end of the major axis of said elliptical bearing; and

10 wherein said wave generator, flexspline, and circular spline cooperate such that each revolution of said wave generator causes said circular spline and said carriage to rotate by only a fraction of a revolution.

15 38. The apparatus of claim 36 wherein said harmonic drive mechanism comprises:

a spline mounting fixedly mounted to said drive shaft and having at least one guide slot;

a circular spline fixedly mounted to said spline mounting and having a plurality of internal teeth;

20 a flexspline rotatably mounted to said drive shaft and having a plurality of external teeth for engaging said plurality of internal teeth of said circular spline, with said plurality of external teeth being fewer than said plurality of internal teeth;

25 a wave generator fixedly mounted to said drag torque generator and having an elliptical, thin raced ball bearing operatively engaged inside said flexspline such that said external teeth of said flexspline engage said internal teeth of said circular spline at each end of the major axis of said elliptical bearing; and

30 at least one guide pin fixedly attached to said flexspline and freely passing through said at least one guide slot in said spline mounting and fixedly attached to said carriage;

5 wherein said wave generator, flexspline, and circular spline cooperate such that each revolution of said wave generator causes said flexspline and said carriage to rotate by only a fraction of a revolution.

39. The apparatus of claim 38 wherein said spline mounting further comprises at least one bias pin and wherein said apparatus
10 further comprises a biasing mechanism comprising at least one spring respectively connecting said at least one bias pin to said at least one guide pin.

40. The apparatus of claim 35 wherein said torque converter comprises a planetary gear mechanism.

41. The apparatus of claim 20 wherein said drag torque
15 generator comprises a mechanical brake.

42. The apparatus of claim 20 wherein said regulator may vary the rotational alignment of said second permanent magnet with respect to said first permanent magnet between a first relative position
20 for which said electrical output is a minimum value and a second relative position for which said electrical output is a maximum value.

43. The apparatus of claim 42 wherein said minimum value is zero.

44. The apparatus of claim 42 wherein said minimum value is
25 greater than zero.

45. The apparatus of claim 42 further comprising a biasing mechanism for biasing said second permanent magnet toward said first relative position.

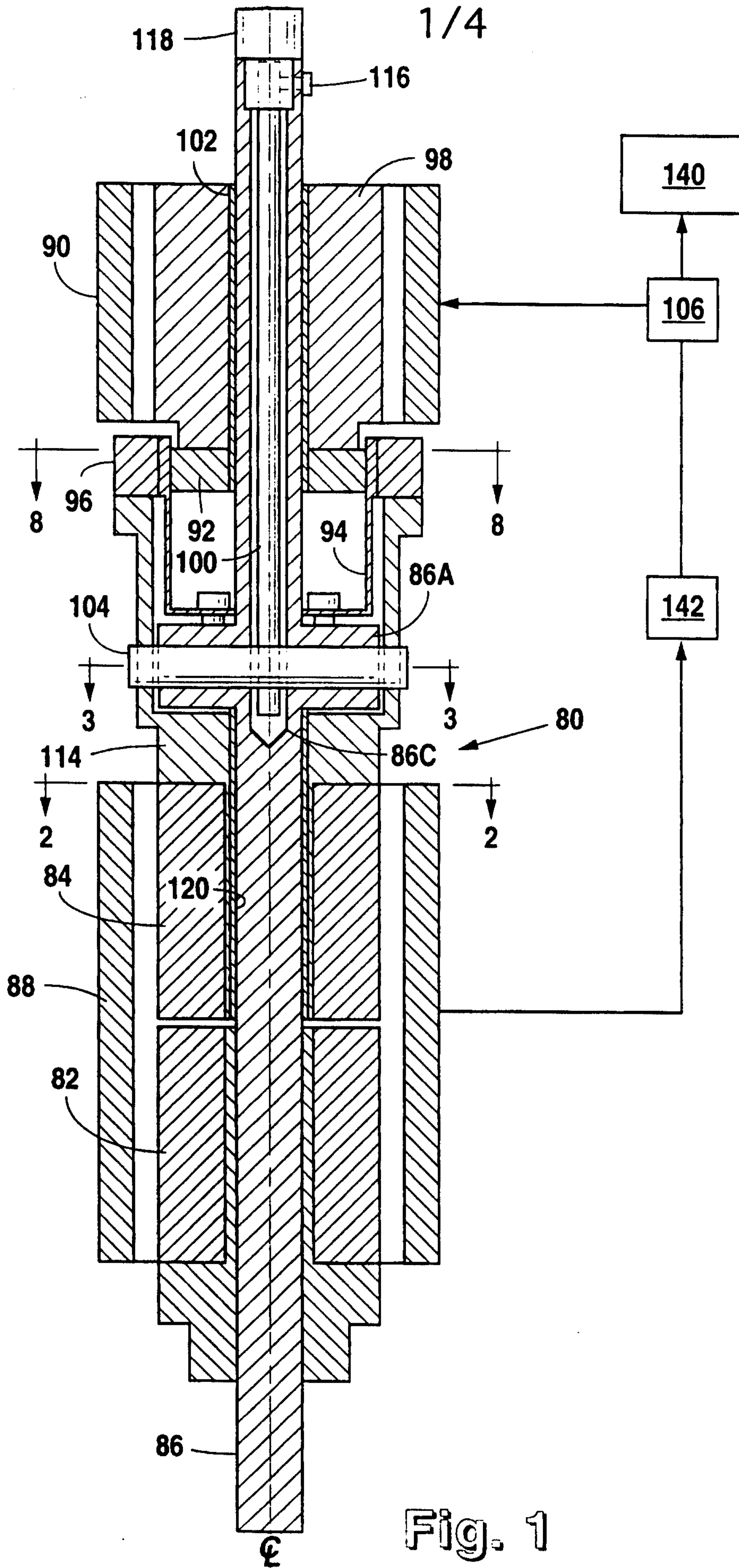
46. The apparatus of claim 42 further comprising a biasing
30 mechanism for biasing said second permanent magnet toward said second relative position.

47. The apparatus of any one of claims 45 and 46 wherein:

5 said drive shaft has an axial cavity and a transverse cavity intersecting said axial cavity; and

 said biasing mechanism comprises (a) a stop pin disposed within said transverse cavity and operably connected to said carriage, (b) a longitudinal biasing element disposed within said axial cavity and
10 operably connected to said stop pin, and (c) a means for applying a torsional preload to said biasing element such that said stop pin tends to engage a portion of the boundary of said transverse cavity.

48. The apparatus of any one of claims 45 and 46 wherein said biasing mechanism comprises a spring connecting said drive shaft to
15 said carriage.



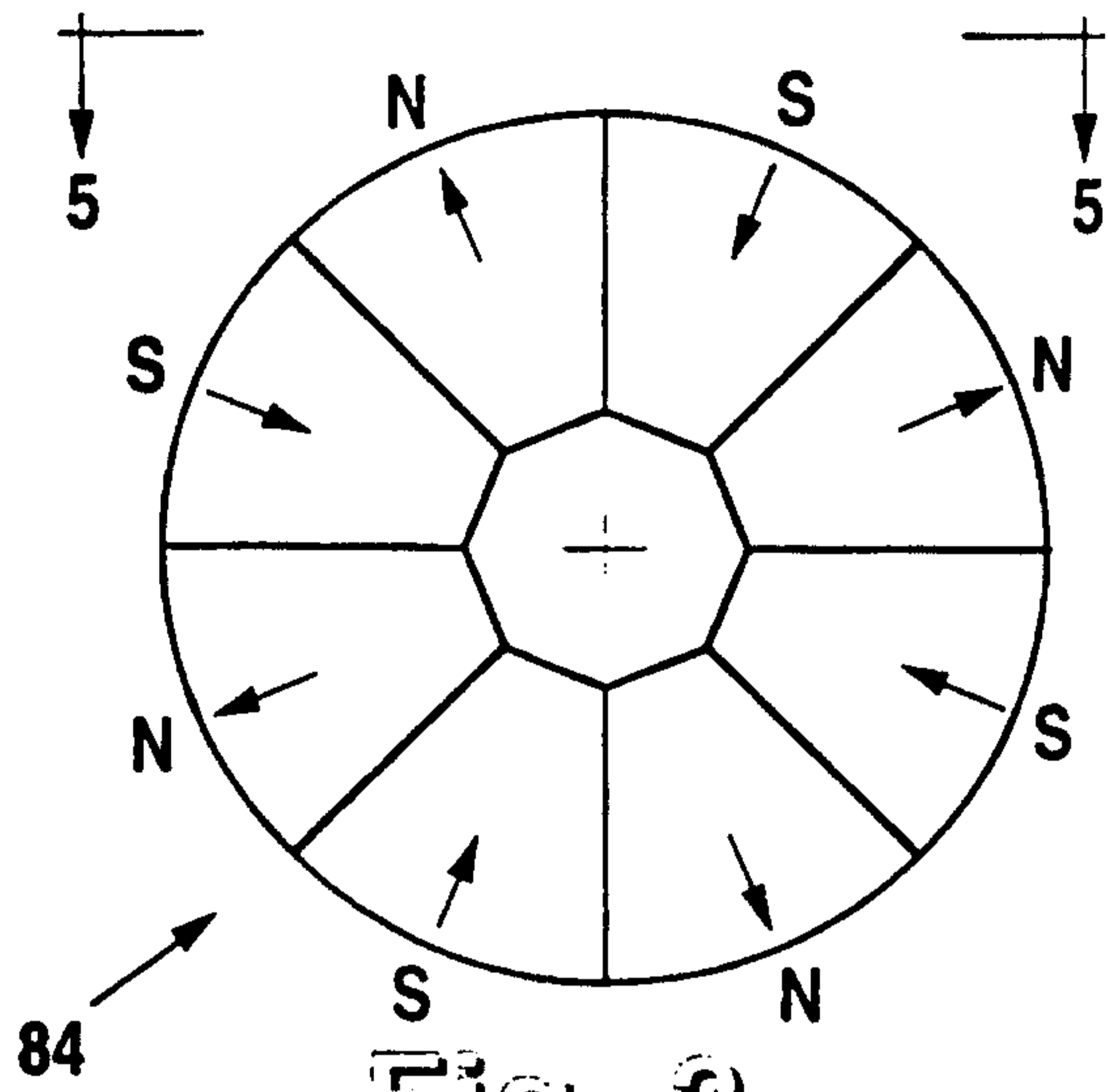


Fig. 2

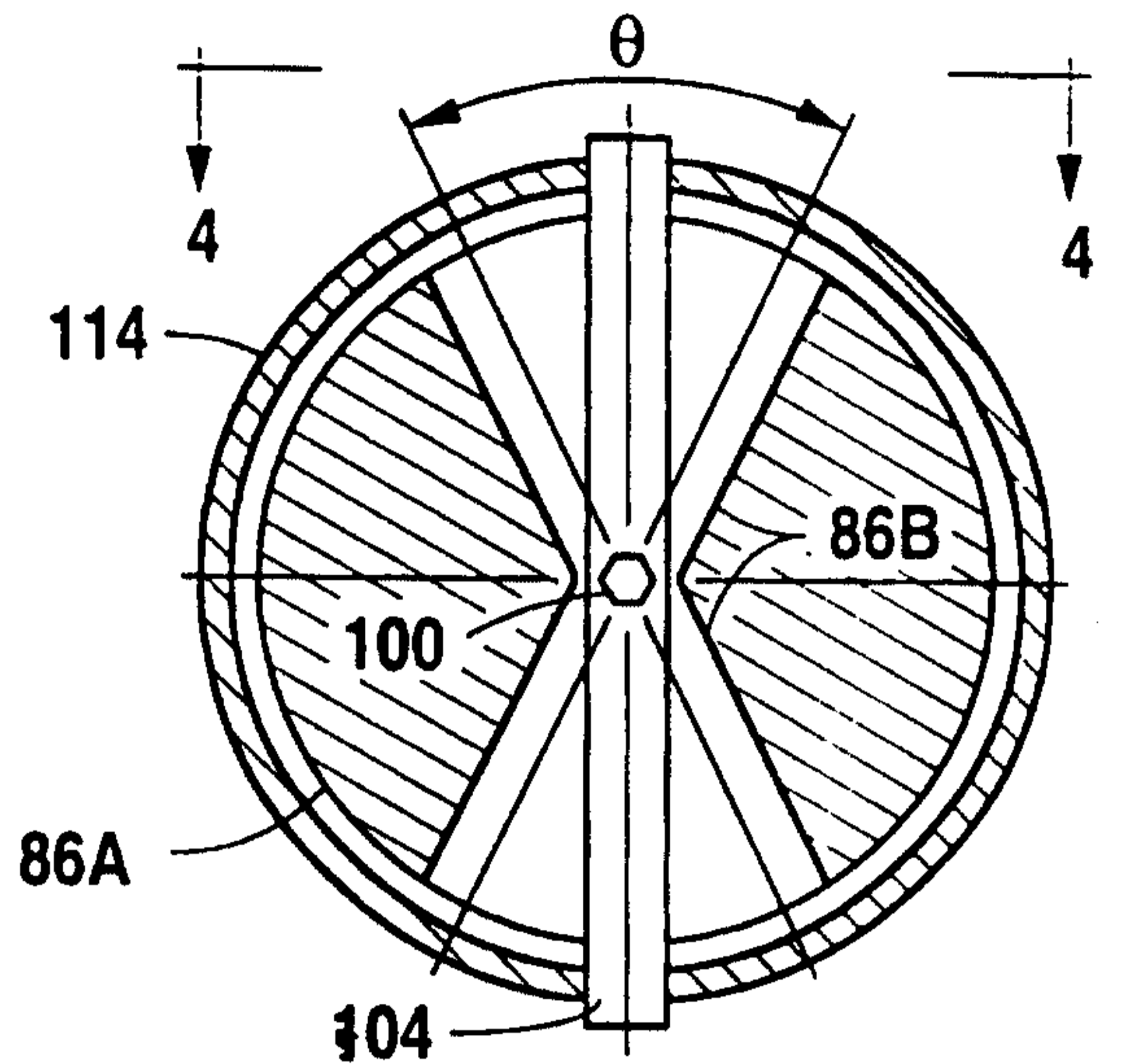


Fig. 3

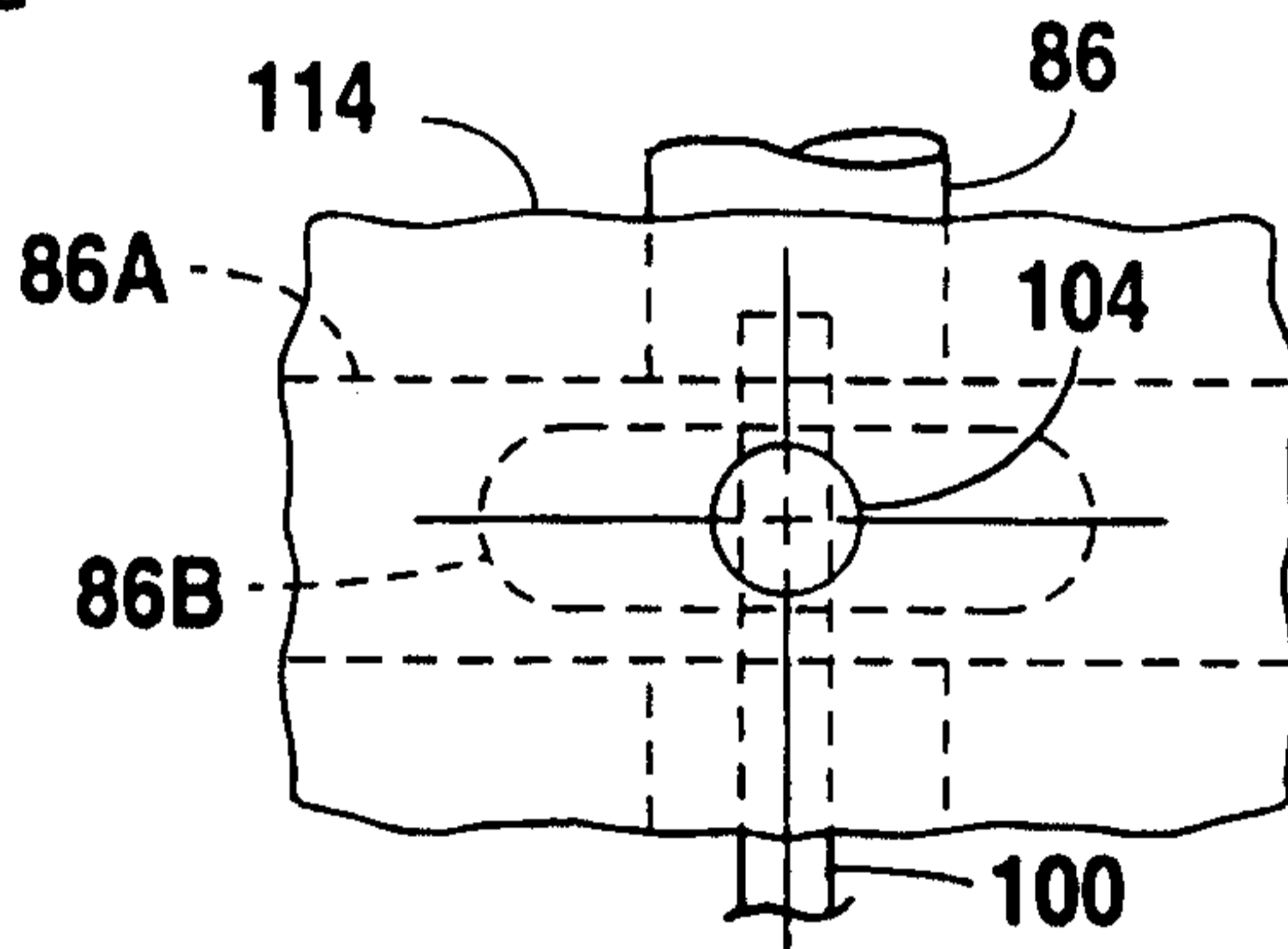


Fig. 4

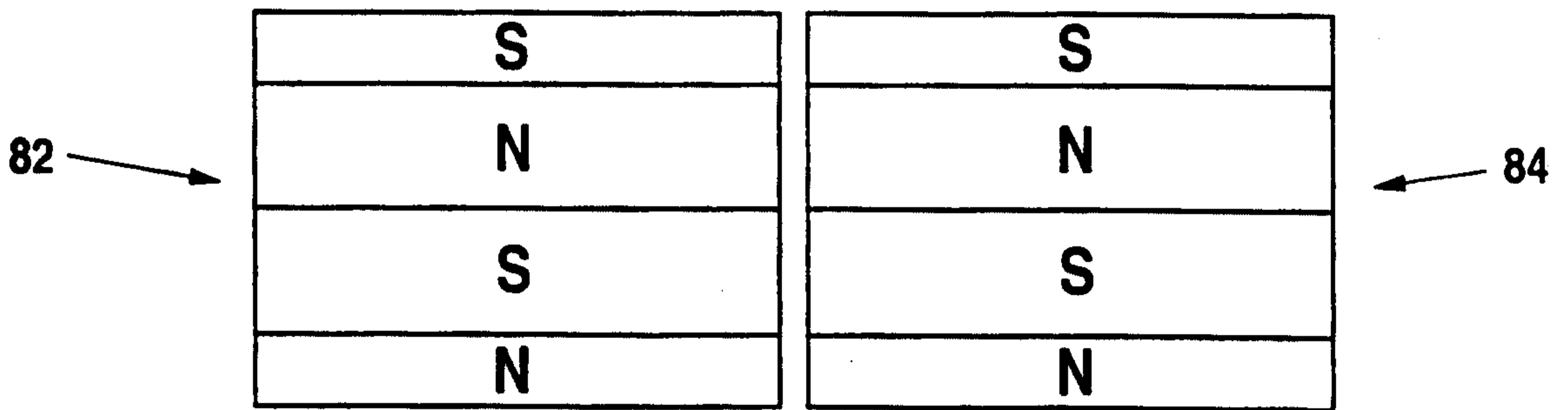


Fig. 5A

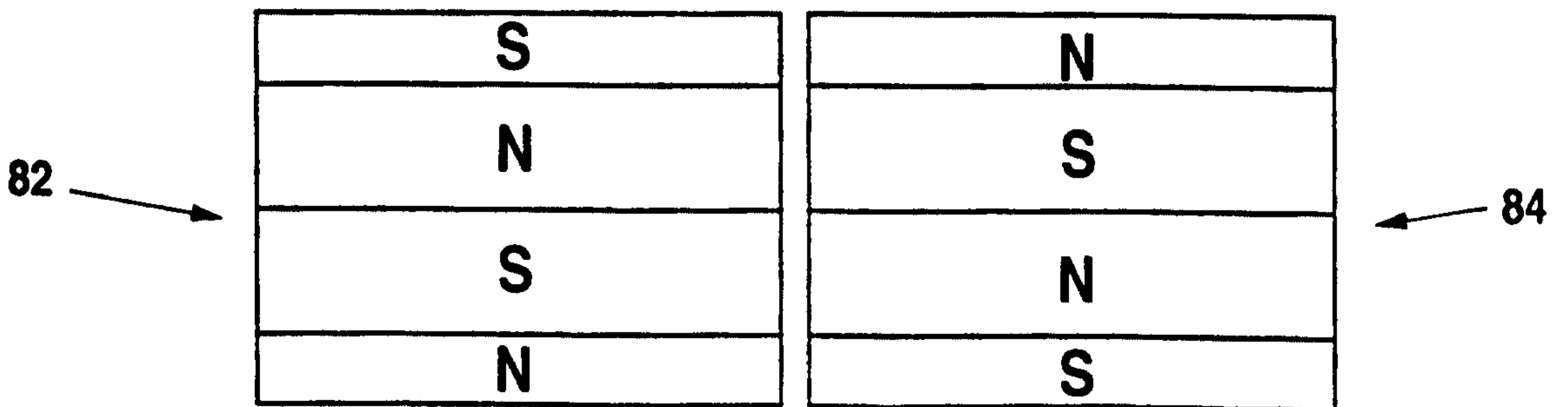


Fig. 5B

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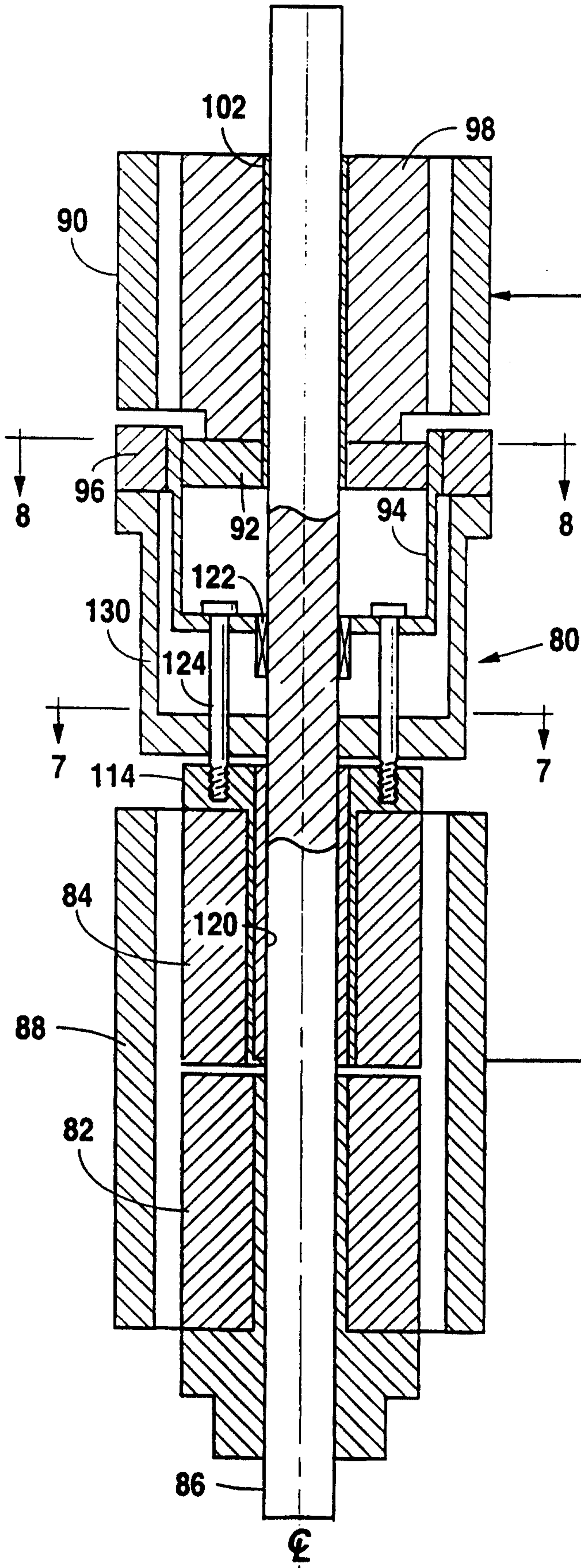


Fig. 6

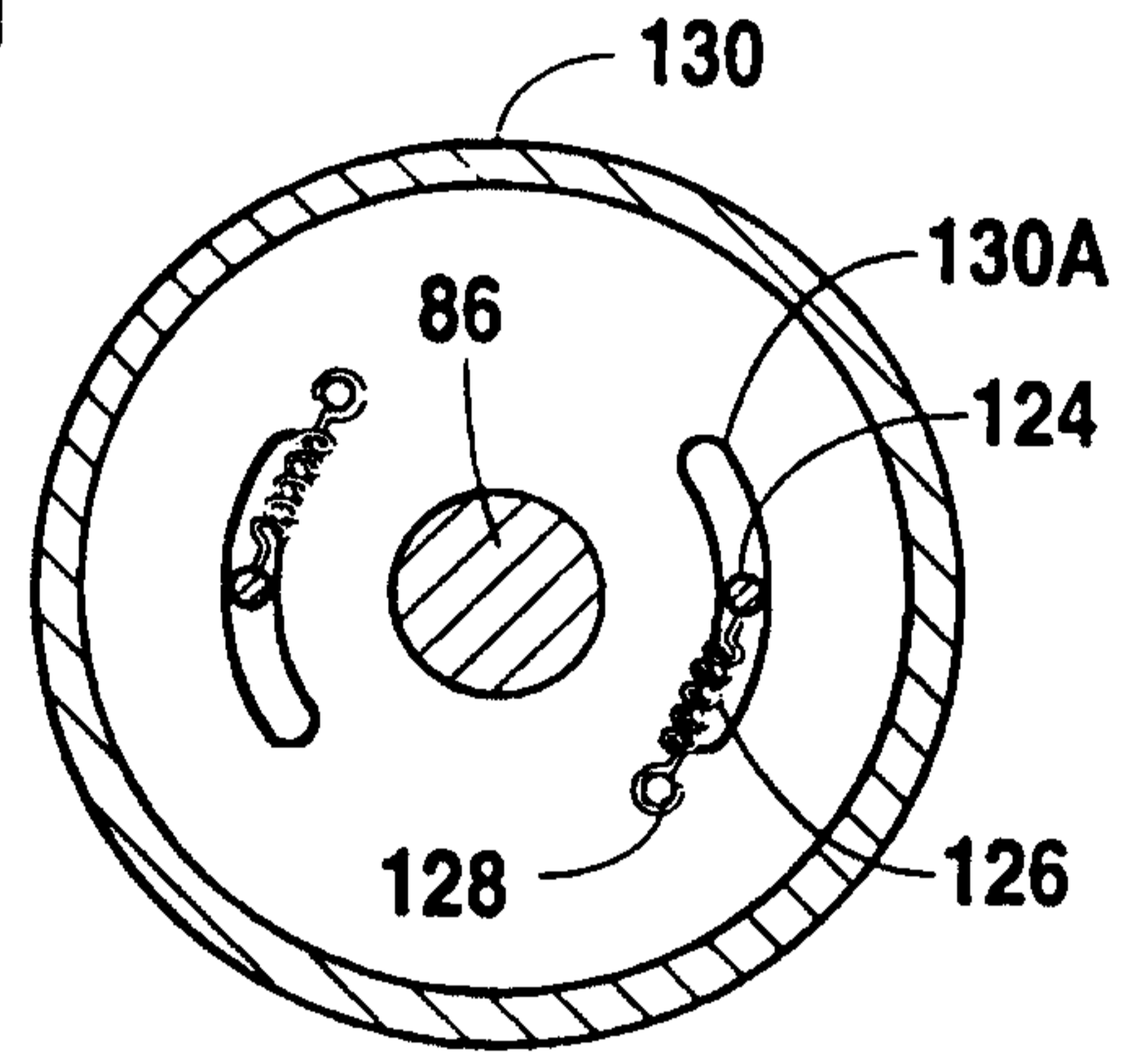
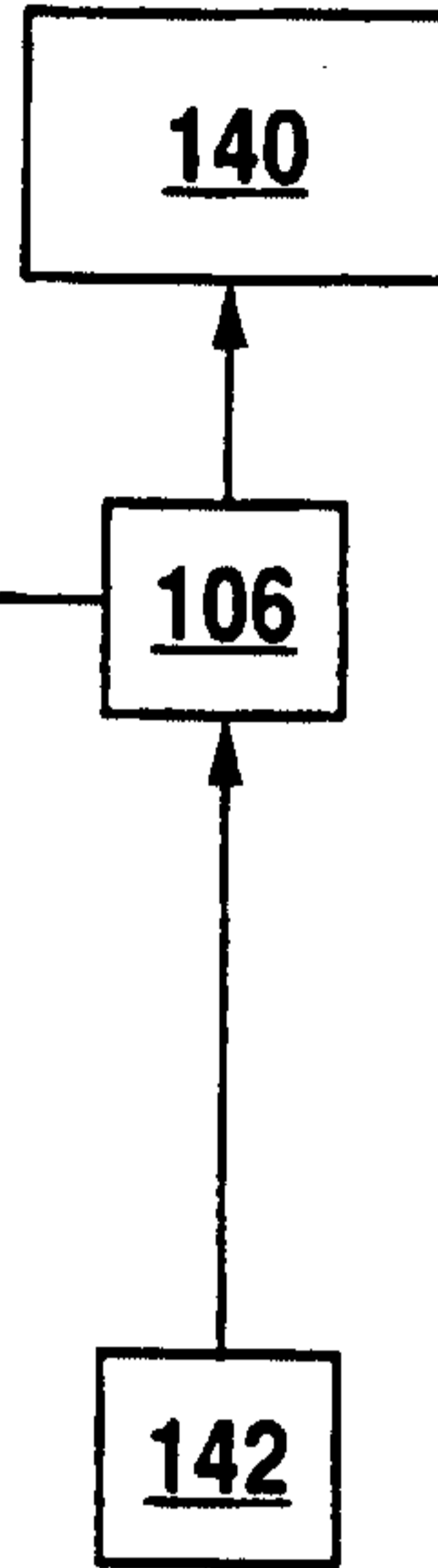


Fig. 7

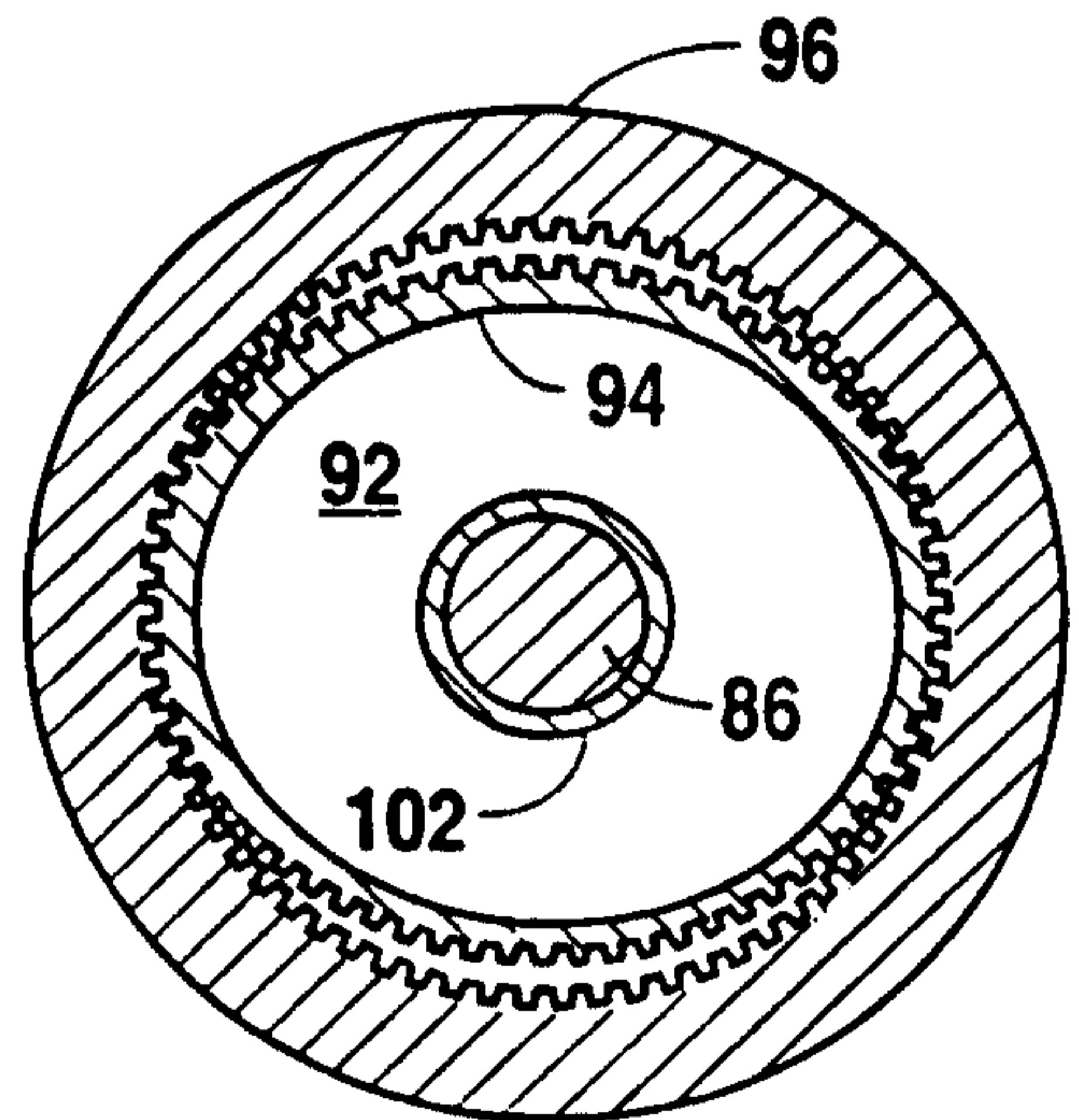


Fig. 8

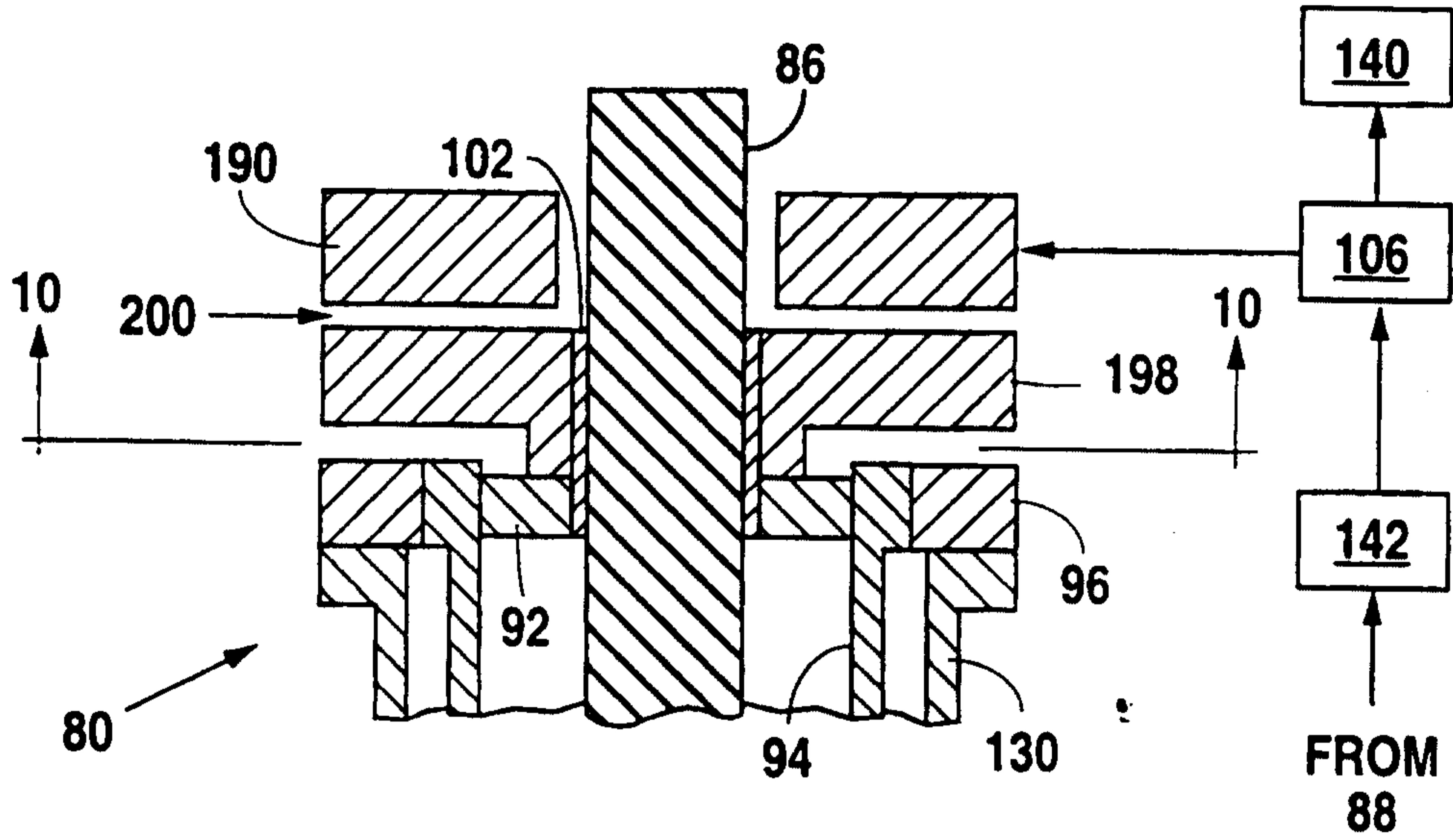


Fig. 9

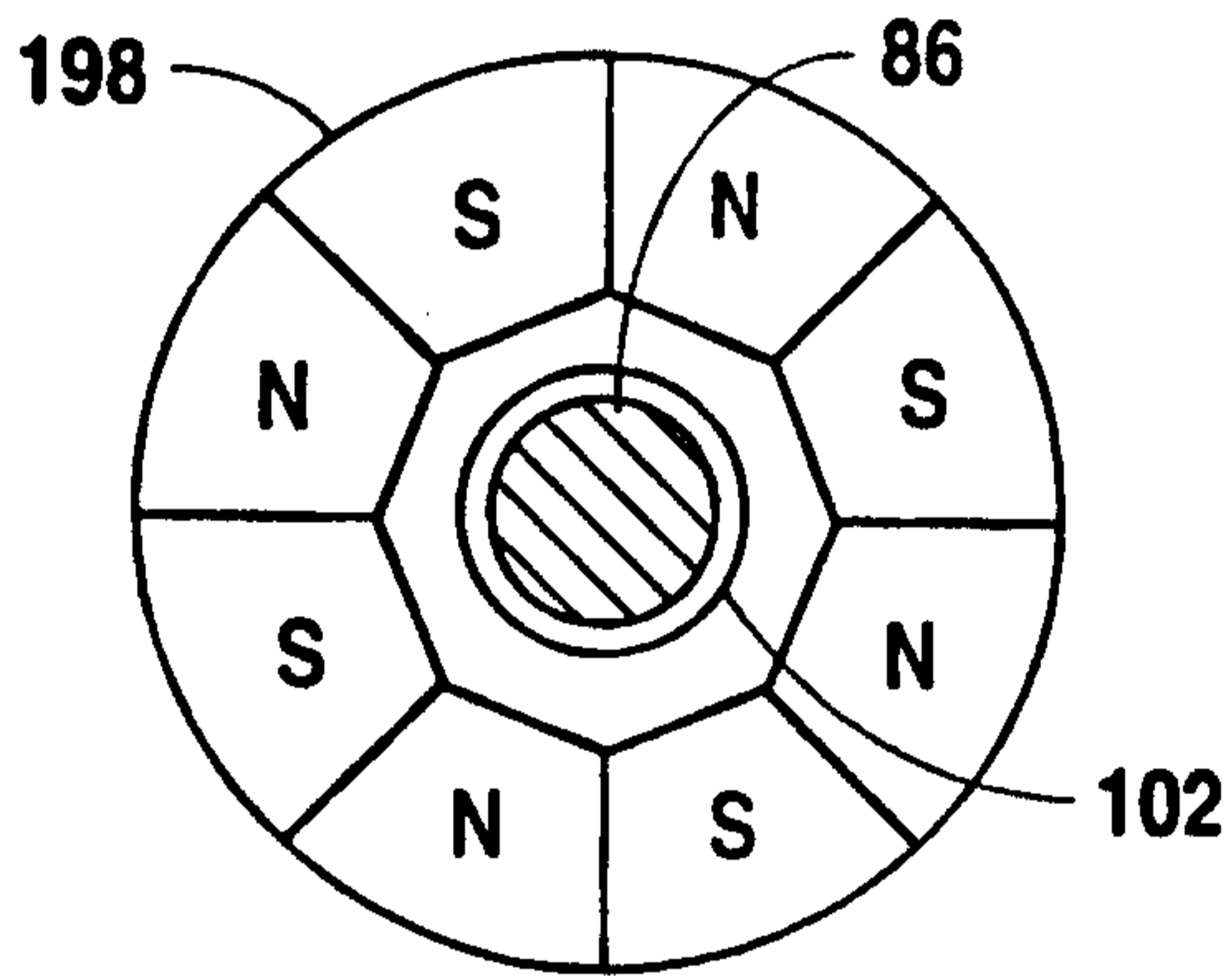


Fig. 10

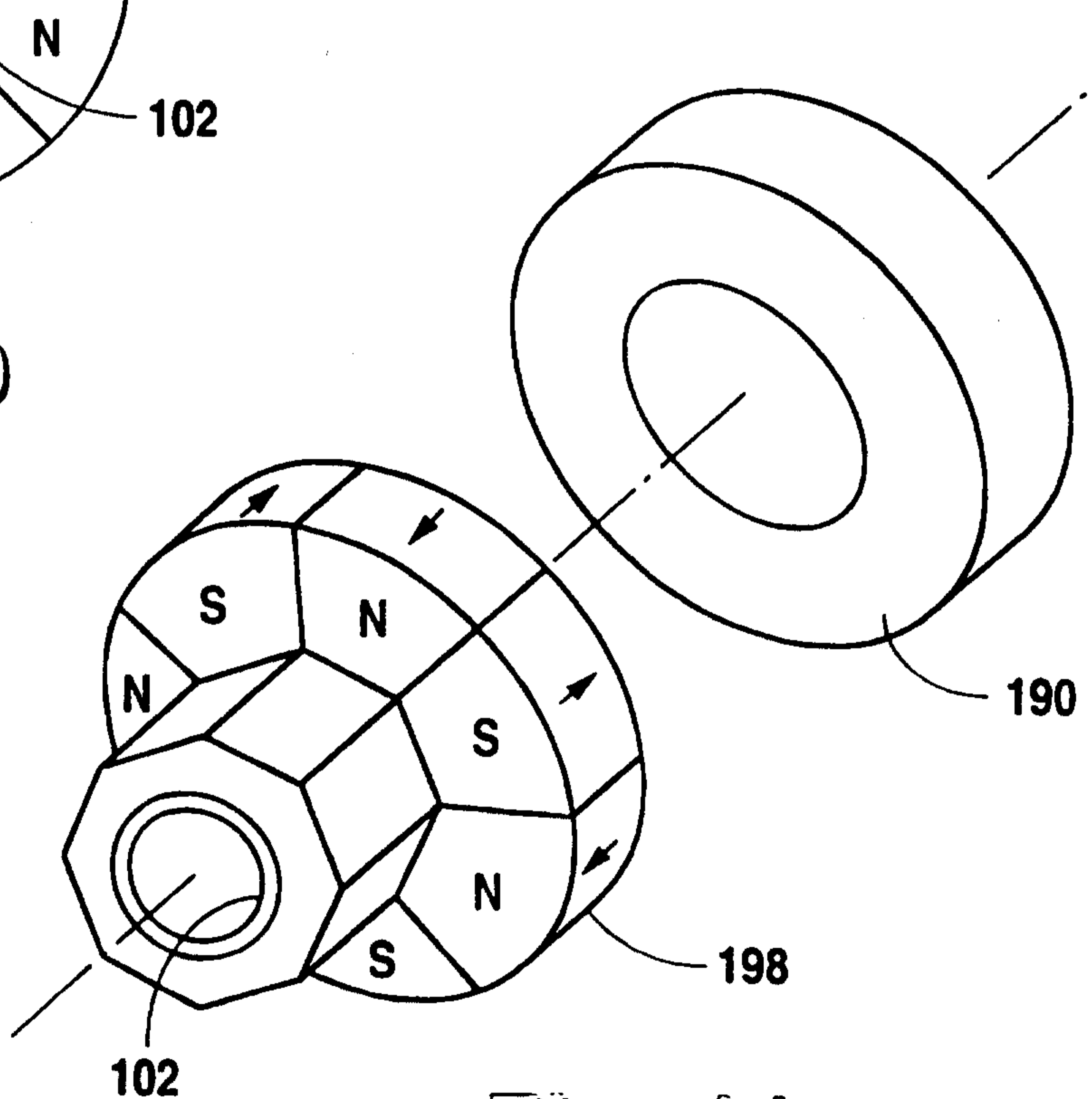


Fig. 11

