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(54) **STEERING CONTROL DEVICE, STEERING CONTROL METHOD, AND COMPUTER PROGRAM PRODUCT**

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(57) **ABSTRACT**

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A steering control device includes a steering control unit configured to control an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator, and a turning control unit configured to control an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator. The turning control unit is configured to perform a gradual reduction control to gradually decrease the turning output torque during a linked control in which the steering control unit controls the actual steering angle in conjunction with the actual turning angle in response to a stop instruction. The turning control unit and the steering control unit are configured to cut energization to the turning actuator and the steering actuator after the gradual reduction control.

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(63) Continuation of application No. PCT/JP2021/033316, filed on Sep. 10, 2021.

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Oct. 23, 2020 (JP) ..... 2020-178238

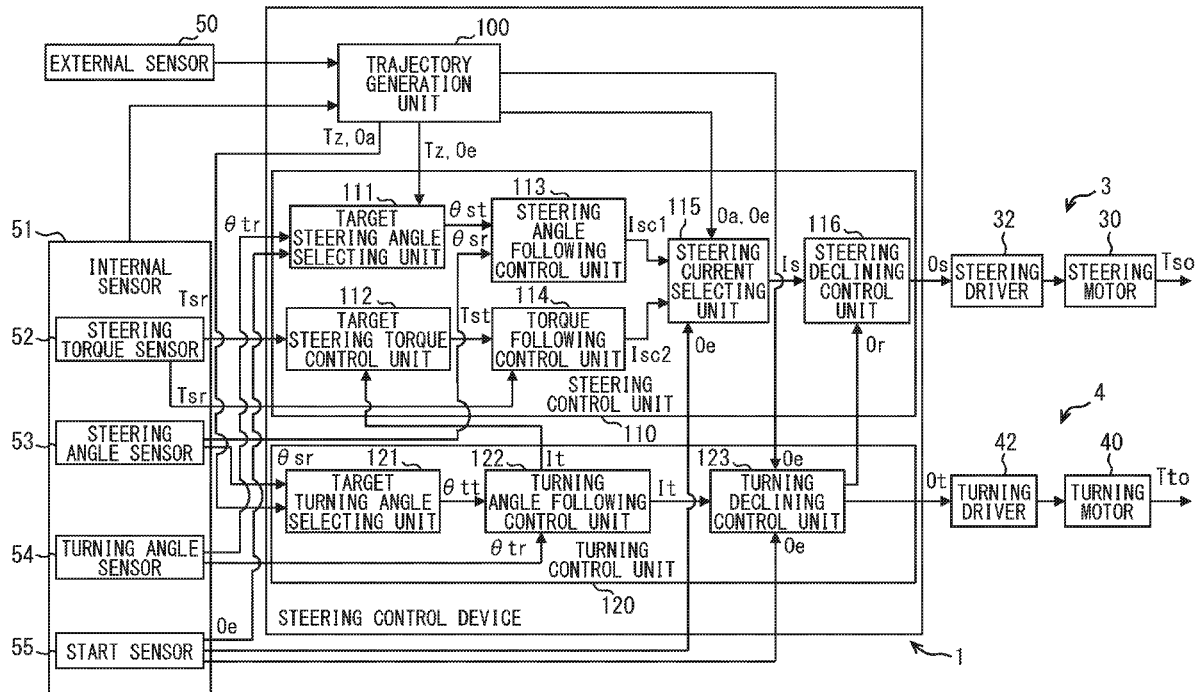


FIG. 1

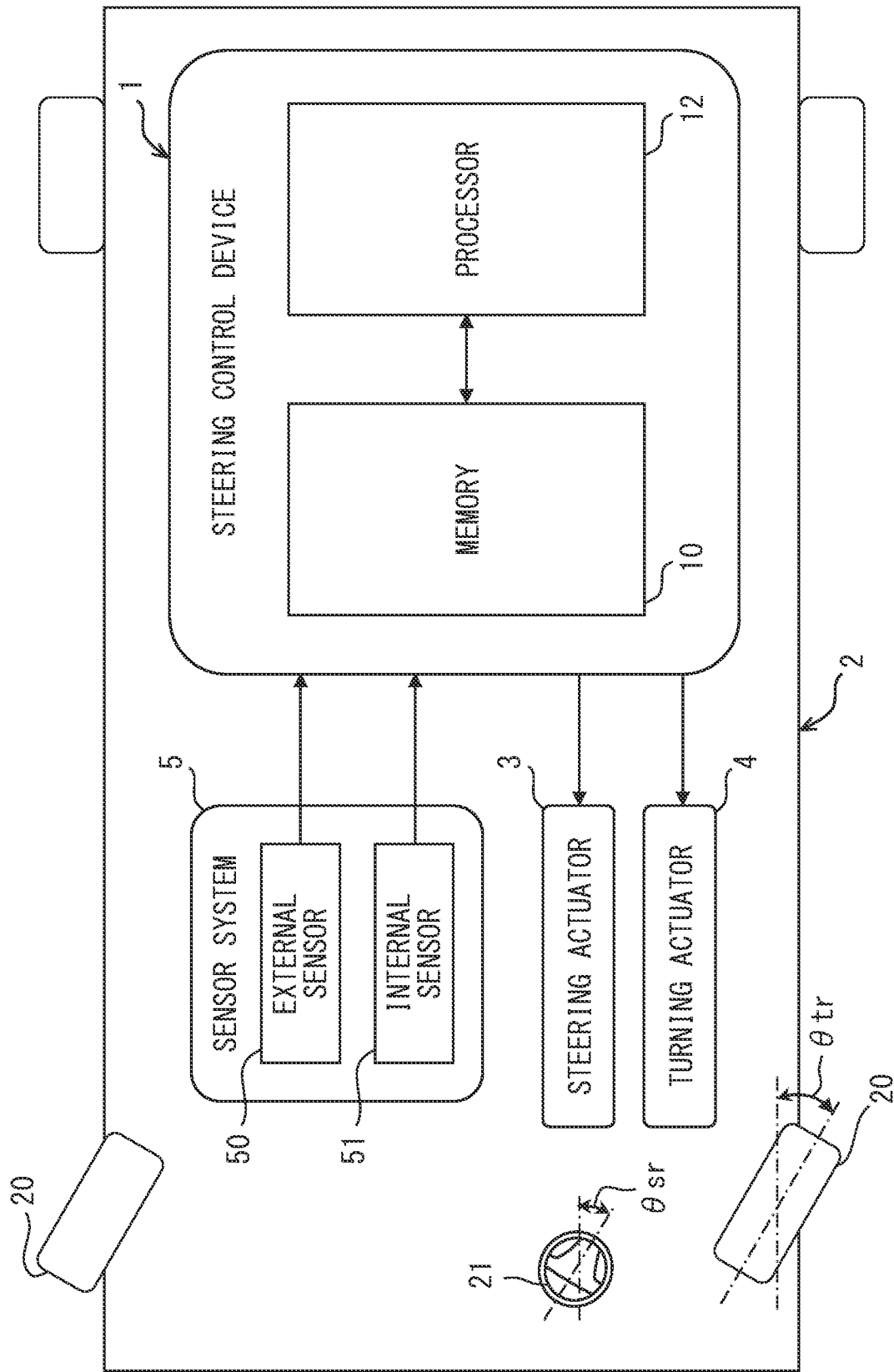


FIG. 2

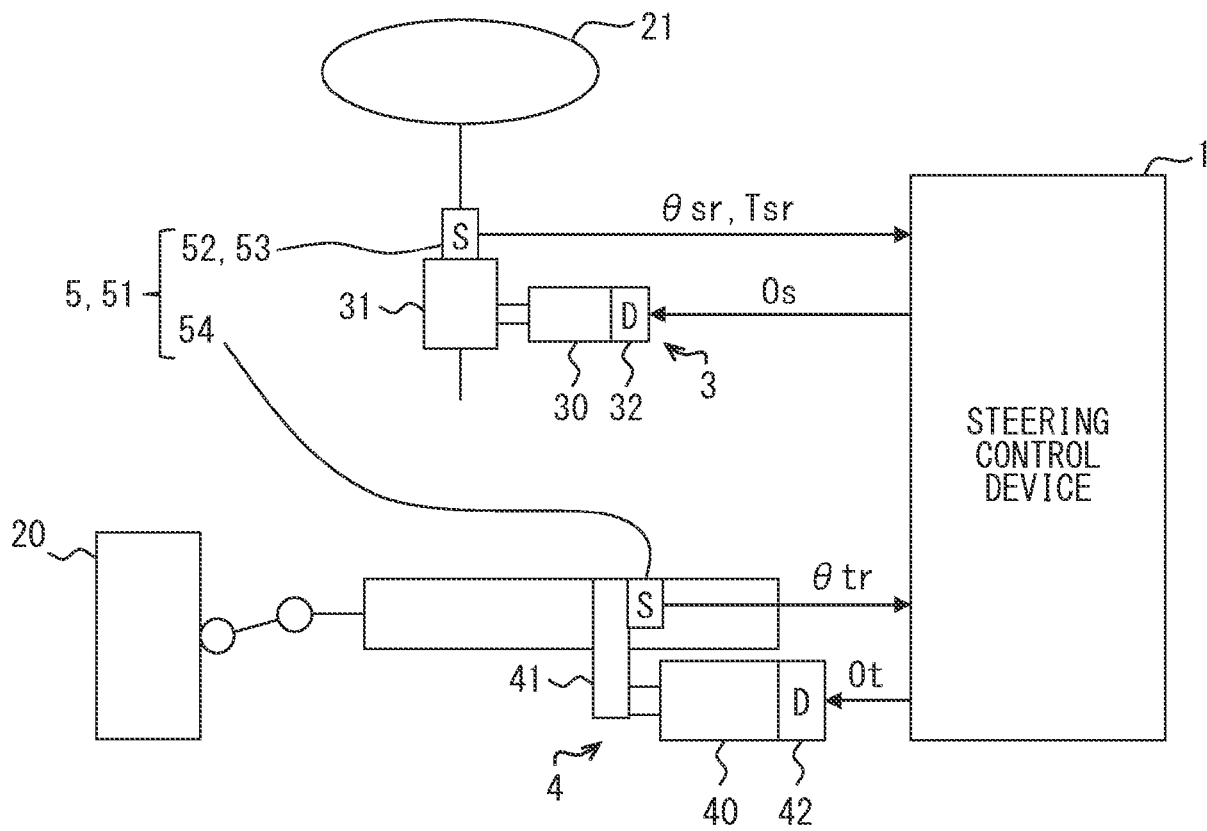


FIG. 3

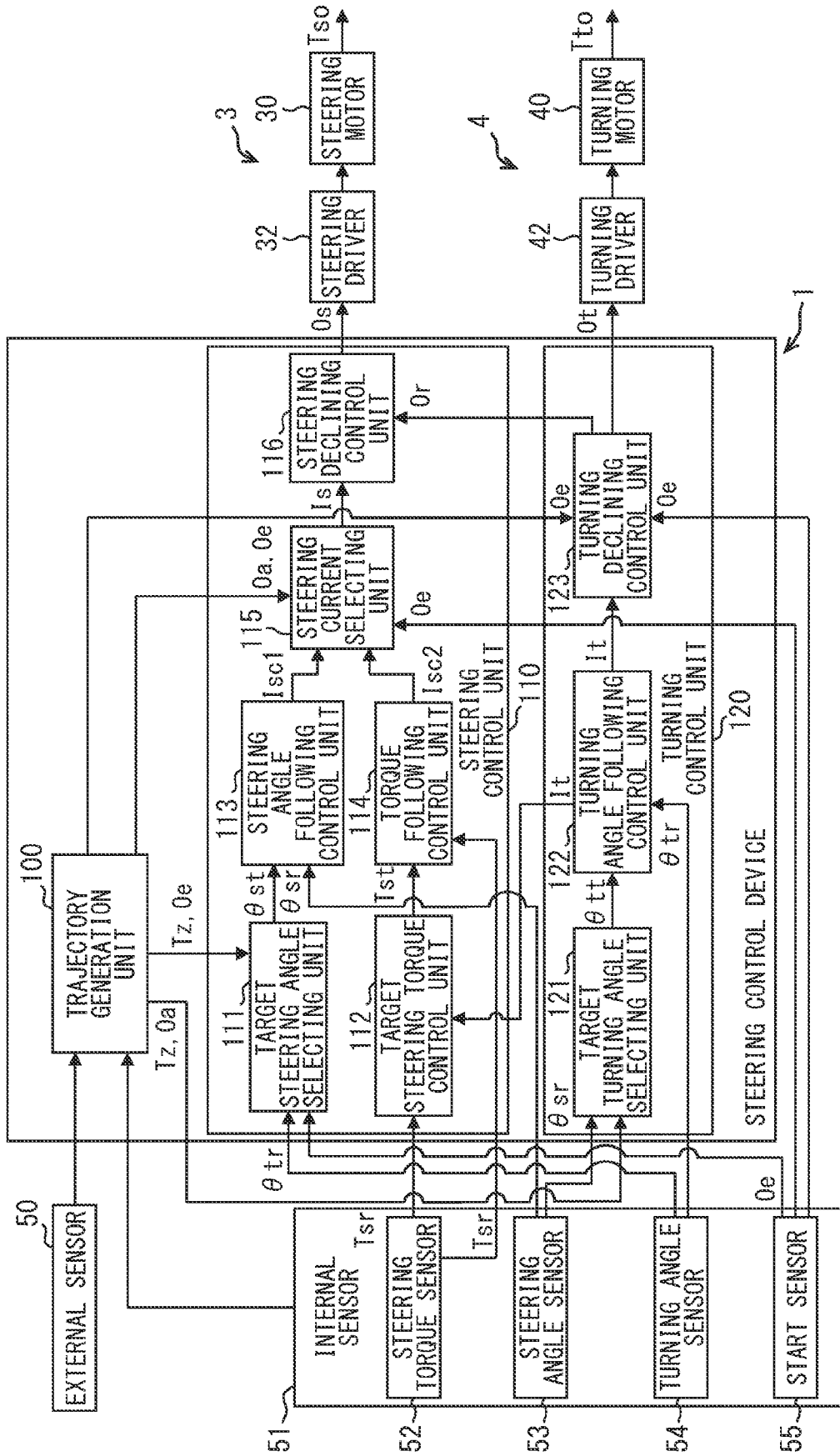


FIG. 4

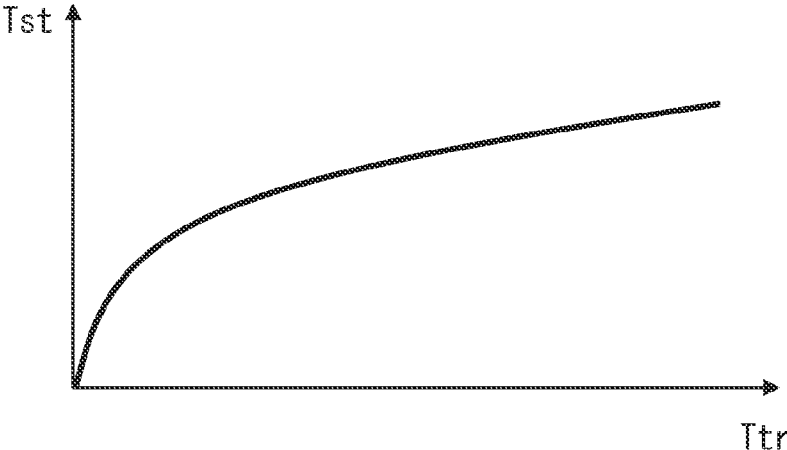
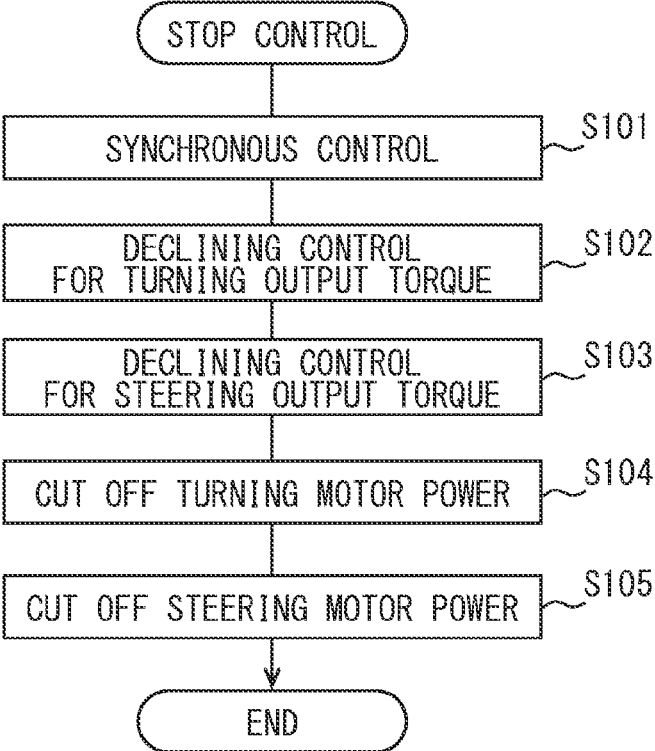
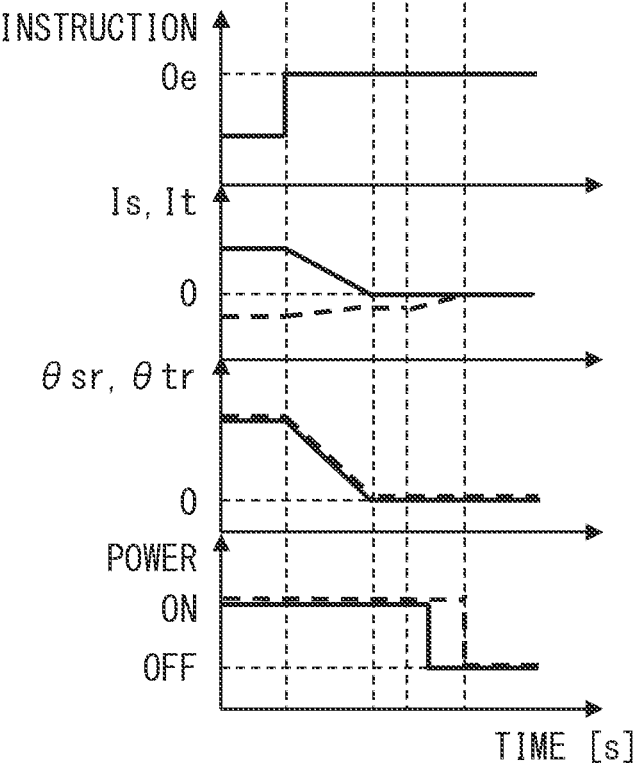


FIG. 5



**FIG. 6**



**FIG. 7**

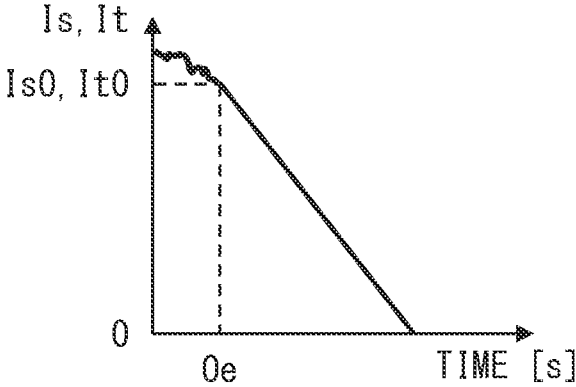


FIG. 8

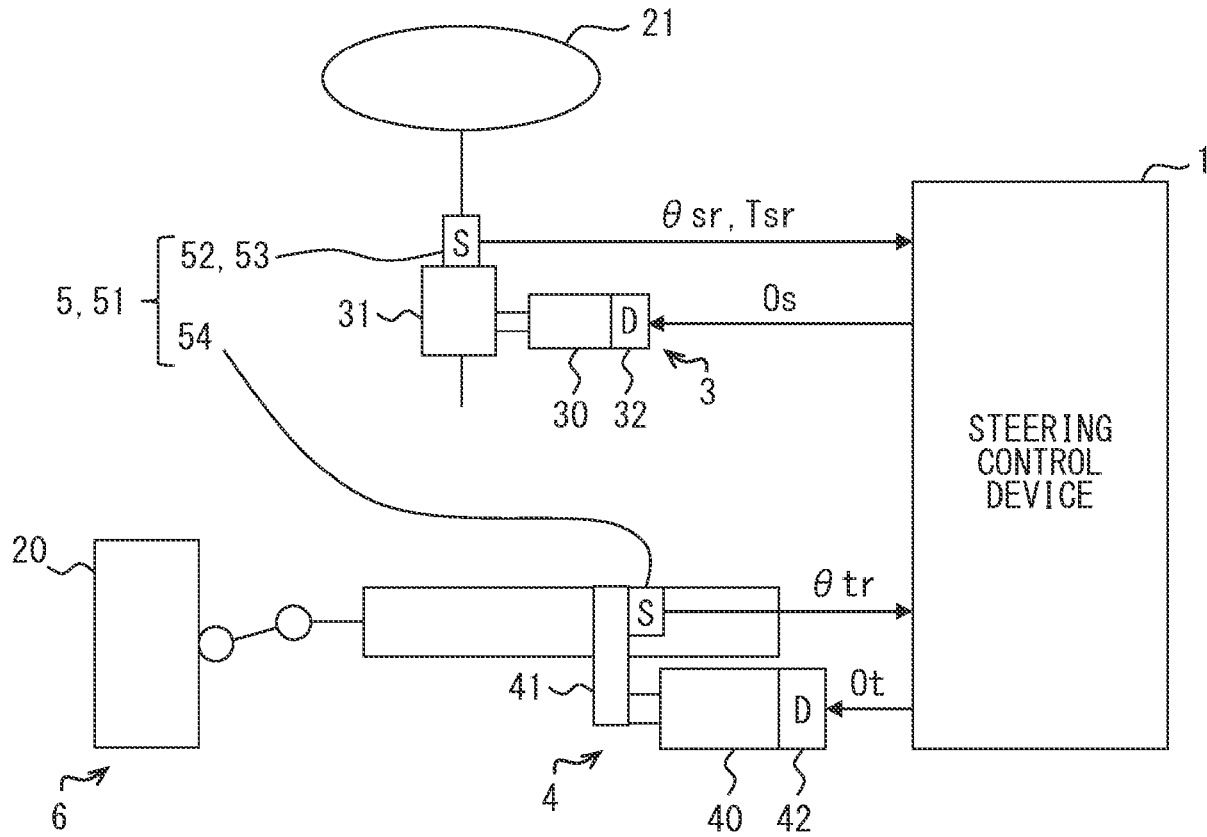
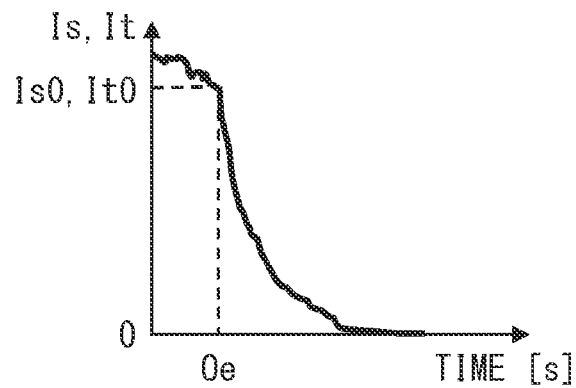


FIG. 9



## STEERING CONTROL DEVICE, STEERING CONTROL METHOD, AND COMPUTER PROGRAM PRODUCT

### CROSS REFERENCE TO RELATED APPLICATION

**[0001]** The present application is a continuation application of International Patent Application No. PCT/JP2021/033316 filed on Sep. 10, 2021, which designated the U.S. and claims the benefit of priority from Japanese Patent Application No. 2020-178238 filed on Oct. 23, 2020. The entire disclosures of all of the above applications are incorporated herein by reference.

### TECHNICAL FIELD

**[0002]** The present disclosure relates to steering control technology in vehicles.

### BACKGROUND

**[0003]** In recent years, techniques for electrically controlling the steering angle of a steering member of a vehicle and the turning angle of a turning tire have become widespread.

### SUMMARY

**[0004]** A first aspect of the present disclosure is a steering control device of a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator. The steering control device includes: a steering control unit configured to control an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and a turning control unit configured to control an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator. The turning control unit is configured to perform a gradual reduction control to gradually decrease the turning output torque during a linked control in which the steering control unit controls the actual steering angle in conjunction with the actual turning angle in response to a stop instruction of the vehicle. The turning control unit is configured to cut an energization to the turning actuator after the gradual reduction control of the turning output torque performed by the turning control unit. The steering control unit is configured to cut an energization to the steering actuator after the gradual reduction control of the turning output torque performed by the turning control unit.

**[0005]** A second aspect of the present disclosure is a steering control method for a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator. The steering control method includes: controlling an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and controlling an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator. In the controlling the actual turning angle, a gradual reduction control is performed to gradually decrease the turning output torque during a linked control in which the actual steering angle is controlled in conjunction with the actual turning angle in response to a stop instruction of the vehicle. In the controlling the actual turning angle, an energization to the

turning actuator is cut after the gradual reduction control of the turning output torque. In the controlling the actual steering angle, an energization to the steering actuator is cut after the gradual reduction control of the turning output torque.

**[0006]** A third aspect of the present disclosure is a computer program product for a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator. The computer program product is stored on at least one non-transitory computer readable medium and includes instructions configured to, when executed by at least one processor, cause the at least one processor to: control an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and control an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator. In the controlling the actual turning angle, a gradual reduction control is performed to gradually decrease the turning output torque during a linked control in which the actual steering angle is controlled in conjunction with the actual turning angle in response to a stop instruction of the vehicle. In the controlling the actual turning angle, an energization to the turning actuator is cut after the gradual reduction control of the turning output torque. In the controlling the actual steering angle, an energization to the steering actuator is cut after the gradual reduction control of the turning output torque.

### BRIEF DESCRIPTION OF THE DRAWINGS

**[0007]** FIG. 1 is a schematic diagram showing a vehicle equipped with a steering control device according to a first embodiment.

**[0008]** FIG. 2 is a schematic diagram showing the overall configuration of the steering control device according to the first embodiment.

**[0009]** FIG. 3 is a block diagram showing a detail configuration of the steering control device according to the first embodiment.

**[0010]** FIG. 4 is a graph for explaining the steering control unit according to the first embodiment.

**[0011]** FIG. 5 is a flowchart illustrating the steering control method according to the first embodiment.

**[0012]** FIG. 6 is a graph showing a stop control mode according to the first embodiment.

**[0013]** FIG. 7 is a graph showing a gradual reduction control according to the first embodiment.

**[0014]** FIG. 8 is a schematic diagram showing a vehicle equipped with a steering control device according to a second embodiment.

**[0015]** FIG. 9 is a graph showing a gradual reduction control according to the second embodiment.

### EMBODIMENTS

#### Comparative Example

**[0016]** In a comparative example of the present disclosure, the turning angle of the turning tire is controlled by the turning torque output to the turning tire via the electric actuator. In the comparative example, the lock device limits the rotation of the turning tire in a locking state. The lock device is configured to be in the locking state before the power is off in response to the stop instruction of the vehicle, and release



the locking state in response to the start instruction of the vehicle.

**[0017]** However, in the comparative example, if the turning tire is twisted when the energization to the electric actuator is cut by turning off the power in response to the stop instruction of the vehicle, the restoring force in the turning tire caused by the twist continues to act on the locking device. Therefore, there is a concern that the components from the turning tires to the electric actuator and the locking device may shorten their lives.

**[0018]** In contrast, if the locking device does not work in response to the stop instruction of the vehicle, the service life can be secured. However, in this case, the torque output to the turning tire via the electric actuator suddenly disappears in response to the cutoff of power to the electric actuator. As a result, the turning tire is rapidly restored to eliminate the twist, and accordingly the turning tire and chassis of the vehicle supporting the turning tire vibrate. Further, when the locking device is not working, the steering member to which the vibration of the turning tire propagates also vibrate.

**[0019]** Hereinafter, embodiments will be described with reference to the drawings. In the following description, the same reference symbols are assigned to corresponding components in each embodiment in order to avoid repetitive descriptions. When only a part of the configuration is described in the respective embodiments, the configuration of the other embodiments described before may be applied to other parts of the configuration. Further, not only the combinations of the configurations explicitly shown in the description of the respective embodiments, but also the configurations of the plurality of embodiments can be partially combined together even if the configurations are not explicitly shown if there is no problem in the combination in particular.

#### First Embodiment

**[0020]** As shown in FIG. 1, a steering control device 1 according to a first embodiment is mounted in a vehicle 2. The vehicle 2 is capable of executing a constant or temporary autonomous driving in an automated driving mode. Here, the automated driving mode may be achieved with an autonomous operation control, such as conditional driving automation, advanced driving automation, or full driving automation, where the system in operation performs all driving tasks. The automated driving mode may be achieved with an advanced driving assistance control, such as driving assistance or partial driving automation, where the occupant performs some or all driving tasks. The automated driving mode may be achieved by combining or switching between the autonomous driving control and the advanced driver assistance control.

**[0021]** At least one pair of turning tires 20 among the wheels of the vehicle 2 can be turned under the control of the steering control device 1. A steering wheel 21, which is a steering member of the vehicle 2, can be held by an occupant in the vehicle compartment. A steering actuator 3, a turning actuator 4, and a sensor system 5 are mounted on the vehicle 2 together with the steering control device 1. As shown in FIGS. 2 and 3, the steering actuator 3 includes a steering motor 30, a steering speed reducer 31 and a steering driver 32. The steering actuator 3 is mechanically associated with the steering wheel 21 of the vehicle 2.

**[0022]** The steering actuator 3 controls a current applied to an electric steering motor 30 by a steering driver 32 according to an output instruction  $O_s$  from the steering control device 1. The steering actuator 3 generates a steering torque by the steering motor 30 to which a current is applied. The steering actuator 3 amplifies the steering torque generated by the steering motor 30 using a steering speed reducer 31 such as a planetary gear and then outputs the amplified steering torque. The output steering torque is transmitted from the steering actuator 3 to the steering wheel 21 as a reaction force, so that the actual steering angle  $\theta_{sr}$  (see FIG. 1) of the steering wheel 21 can be changed.

**[0023]** Here, the actual steering angle  $\theta_{sr}$  and its target steering angle  $\theta_{st}$  are given a positive (i.e., plus) value on the right side and a negative (i.e., minus) value on the left side with respect to the front-rear direction of the vehicle 2, respectively. Similarly, the output instruction  $O_s$  to the steering actuator 3 is given a positive or negative value.

**[0024]** The turning actuator 4 includes a turning motor 40, a turning speed reducer 41 and a turning driver 42. The turning actuator 4 is mechanically linked to the turning tires 20 of the vehicle 2. The turning actuator 4 constitutes a steer-by-wire system that mechanically disconnects and electrically cooperates with the steering wheel 21 and the steering actuator 3.

**[0025]** The turning actuator 4 controls a current applied to an electric turning motor 40 by a turning driver 42 according to an output instruction  $O_t$  from the steering control device 1. The turning actuator 4 generates a turning torque by a turning motor 40 to which a current is applied. The turning actuator 4 amplifies the turning torque generated by the turning motor 40 by a turning speed reducer 41 such as a rack gear, and then outputs the amplified turning torque. The output turning torque is transmitted from the turning actuator 4 to the turning tires 20 as driving force, thereby changing the actual turning angle  $\theta_{tr}$  (see FIG. 1) of the turning tires 20.

**[0026]** Here, the actual turning angle  $\theta_{tr}$  and its target turning angle  $\theta_{tt}$  are given a positive (i.e., plus) value on the right side and a negative (i.e., minus) value on the left side with respect to the front-rear direction of the vehicle 2, respectively. Similarly, the output instruction  $O_t$  to the turning actuator 4 is given a positive or negative value.

**[0027]** As shown in FIGS. 1 to 3, the sensor system 5 includes an external sensor 50 and an internal sensor 51. The external sensor 50 is configured to acquire information about the outside of the vehicle 2, which is the surrounding environment of the vehicle 2. The external sensor 50 may acquire the external information by detecting an object existing in the outside of the vehicle 2. The external sensor 50 of the detection type is at least one of a camera, a LIDAR (Light Detection and Ranging/Laser Imaging Detection and Ranging), a radar, sonar, and the like, for example. The external sensor 50 may acquire the external information by receiving a signal from an artificial satellite of a GNSS (Global Navigation Satellite System) present in the outside of the vehicle 2 or a signal from a roadside device of ITS (Intelligent Transport Systems). The external sensor 50 of the signal reception type is at least one of, for example, a GNSS receiver, a telematics receiver, and the like.

**[0028]** The internal sensor 51 is configured to acquire information about the inside of the vehicle 2, which is the internal environment of the vehicle 2. The internal sensor 51 may acquire the internal information by detecting a specific

motion physical quantity in the inside of the vehicle 3. The physical quantity detection type internal sensor 51 is, for example, a steering torque sensor 52, a steering angle sensor 53, a turning angle sensor 54, a speed sensor, an acceleration sensor, an inertia sensor, a yaw rate sensor, an actuation sensor 55, and the like, and includes at least sensors 52, 53, 54, 55. The steering torque sensor 52 acquires an actual steering torque  $T_{sr}$  applied to the steering wheel 21. Here, the steering angle sensor 53 acquires the actual steering angle  $\theta_{sr}$  of the steering wheel 21. The turning angle sensor 54 acquires the actual turning angle  $\theta_{tr}$  of the turning tires 20. The actuation sensor 55 outputs an actuation instruction of the vehicle 2 in response to switching on an actuation switch by an occupant of the vehicle 2. The actuation sensor 55 outputs a stop instruction  $O_e$  of the vehicle 2 in response to switching off the actuation switch by an occupant of the vehicle 2. The actuation sensor 55 outputs an idle reduction instruction and a restart instruction when the vehicle 2 is in the idle reduction.

[0029] The steering control device 1 is connected with the steering actuator 3, the turning actuator 4 and the sensor system 5 through at least one of LAN (Local Area Network), a wire harness, an internal bus, or the like. The steering control device 1 includes at least one dedicated computer. The dedicated computer that constitutes the steering control device 1 may be a driving control ECU (Electronic Control Unit) that implements driving control including an automatic operation mode. The dedicated computer that constitutes the steering control device 1 may be at least one of a steering ECU that controls the steering actuator 3 and a turning ECU that controls the turning actuator 4. The dedicated computer that constitutes the steering control device 1 may be a locator ECU that estimates state quantities of the vehicle 2 including the position of the vehicle 2. The dedicated computer of the steering control device 1 may be a navigation ECU configured to navigate the driver of the vehicle 2. The dedicated computer that constitutes the steering control device 1 may be an HCU (i.e., HMI (i.e., Human Machine Interface) Control Unit) that controls information presentation of the vehicle 2.

[0030] As shown in FIG. 1, the steering control device 1 includes at least one memory 10 and at least one processor 12 by including such a dedicated computer. The memory 10 is at least one type of non-transitory tangible storage medium, such as a semiconductor memory, a magnetic storage medium, and an optical storage medium, for non-transitory storage of computer readable programs and data. The processor 12 includes, as a core, at least one type of, for example, a CPU (Central Processing Unit), a GPU (Graphics Processing Unit), an RISC (Reduced Instruction Set Computer) CPU, and the like.

[0031] The processor 12 executes multiple instructions included in a steering control program stored in the memory 10. Accordingly, the steering control device 1 provides a plurality of functional units, which are functional blocks for interlocking steering and turning in the vehicle 2 by a control, as shown in FIG. 3. As described above, in the steering control device 1, the processor 12 executes a plurality of instructions according to the steering control program stored in the memory 10 so that multiple functional units are established in order to control the steering and turning of the vehicle 2 in conjunction with each other. These multiple functional units include a trajectory generation unit

100, a steering control unit 110, and a turning control unit 120.

[0032] The trajectory generation unit 100 generates the planned trajectory  $T_z$  based on input information from the sensor system 5 or physical information based thereon. The planned trajectory  $T_z$  indicates a traveling route that defines the time-series change of the state quantity (self-state quantity) of the vehicle 2. The self-state quantity is a physical quantity including at least the position of the vehicle 2. Such a self-state quantity may include, for example, at least one of traveling speed, acceleration, yaw angle, and the like in addition to the position.

[0033] The trajectory generation unit 100 is configured to generate control instructions for requests to the steering control unit 110 and the turning control unit 120. Specifically, the trajectory generation unit 100 is configured to generate an automated driving instruction  $O_a$  for performing the automated driving mode as the driving control of the vehicle 2. The trajectory generation unit 100 is configured to generate the stop instruction  $O_e$  for performing the stop control mode as the driving control of the vehicle 2 in which the vehicle 2 is stopped in the automated driving mode. The stop instruction  $O_e$  in the automated driving mode is at least one of a complete stop instruction of the vehicle 2, an idle reduction instruction of the vehicle 2, and a power stop instruction of the actuators 3, 4. The trajectory generation unit 100 may output the stop instruction  $O_e$  immediately before a scheduled time determined by the trajectory generation unit 100 for stopping the vehicle 2, on the scheduled time, or immediately after the scheduled time. In contrast, when the driving control of the vehicle 2 is a manual driving mode, the stop instruction  $O_e$  for performing the stop control mode is output from the actuation sensor 55. The stop instruction  $O_e$  in the manual driving mode is at least one of a complete stop instruction of the vehicle 2 according to the switching off operation of the actuation switch, and an idle reduction instruction of the vehicle 2.

[0034] The steering control unit 110 is configured to selectively perform a control of a steering output torque  $T_{so}$  output from the steering actuator 3, a control of the actual steering angle  $\theta_{sr}$  through the steering actuator 3, and a control of the actual steering torque through the steering actuator 3. For this purpose, the steering control unit 110 includes, as sub-functional units, a target steering angle selecting unit 111, a target steering torque control unit 112, a steering angle following control unit 113, a torque following control unit 114, a steering current selecting unit 115, and a steering declining control unit 116.

[0035] The stop instruction  $O_e$  and the planned trajectory  $T_z$  are input to the target steering angle selecting unit 111 from the trajectory generation unit 100. The stop instruction  $O_e$  is input to the target steering angle selecting unit 111 from the actuation sensor 55. The actual turning angle  $\theta_{tr}$  is input to the target steering angle selecting unit 111 from the turning angle sensor 54. The target steering angle selecting unit 111 is configured to select, based on these inputs, the target steering angle  $\theta_{st}$  that is the control target of the actual steering angle  $\theta_{sr}$ .

[0036] Specifically, when the stop instruction  $O_e$  is not input from the trajectory generation unit 100 and the actuation sensor 55, the target steering angle selecting unit 111 selects the target steering angle  $\theta_{st}$  acquired to follow the planned trajectory  $T_z$ . In contrast, when the stop instruction  $O_e$  is input from the trajectory generation unit 100 or the

actuation sensor 55, the target steering angle selecting unit 111 performs a linked control in which the actual steering angle  $\theta_{sr}$  changes in conjunction with the actual turning angle  $\theta_{tr}$ . In the linked control, the target steering angle selecting unit 111 selects, as the target steering angle  $\theta_{st}$ , a converted angle that is calculated by multiplying the actual turning angle  $\theta_{tr}$  by a predetermined conversion ratio.

[0037] The target steering angle  $\theta_{st}$  is input to the steering angle following control unit 113 from the target steering angle selecting unit 111. The steering angle following control unit 113 executes steering angle following control such as PID control to make the actual steering angle  $\theta_{sr}$  follow the target steering angle  $\theta_{st}$ . As a result of the steering angle following control, the steering angle following control unit 113 determines, as a first steering current candidate value  $I_{sc1}$ , a candidate value of a steering current applied to the steering motor 30.

[0038] The actual steering torque  $T_{sr}$  is input from the steering torque sensor 52 to the target steering torque control unit 112, and a turning current value  $I_t$  is input from a turning angle following control unit 122 of the turning control unit 120 to the target steering torque control unit 112. The target steering torque control unit 112 is configured to estimate an actual turning torque  $T_{tr}$  applied on the turning tire 20 based on the actual steering torque  $T_{sr}$  and a value calculated by multiplying the turning current value  $I_t$  by a predetermined torque conversion coefficient. The target steering torque control unit 112 is configured to determine a target steering torque  $T_{st}$  corresponding to the estimated actual turning torque  $T_{tr}$  based on a map shown in FIG. 4, for example.

[0039] As shown in FIG. 3, the target steering torque  $T_{st}$  is input from the target steering torque control unit 112 to the torque following control unit 114. The actual steering torque  $T_{sr}$  is input from the steering torque sensor 52 to the torque following control unit 114. The torque following control unit 114 executes torque following control such as PID control to make the actual steering torque  $T_{sr}$  follow the target steering torque  $T_{st}$ . As a result of the torque following control, the torque following control unit 114 determines, as a second steering current candidate value  $I_{sc2}$ , a candidate value of a steering current applied to the steering motor 30.

[0040] The stop instruction  $O_e$  and the automated driving instruction  $O_a$  are input from the trajectory generation unit 100 to the steering current selecting unit 115. The stop instruction  $O_e$  is input to the steering current selecting unit 115 from the actuation sensor 55. The first steering current candidate value  $I_{sc1}$  is input from the steering angle following control unit 113 to the steering current selecting unit 115. The second steering current candidate value  $I_{sc2}$  is input from the torque following control unit 114. The steering current selecting unit 115 is configured to select, as a steering current value  $I_s$ , one of the first steering current candidate value  $I_{sc1}$  and the second steering current candidate value  $I_{sc2}$  depending of the mode.

[0041] Specifically, the steering current selecting unit 115 selects the first steering current candidate value  $I_{sc1}$  as the steering current value  $I_s$  when the automated driving instruction  $O_a$  is input from the trajectory generation unit 100. Further, the steering current selecting unit 115 selects the first steering current candidate value  $I_{sc1}$  as the steering current value  $I_s$  when the stop instruction  $O_e$  is input from the trajectory generation unit 100 or the actuation sensor 55. Accordingly, in the automated driving mode and the stop

control mode of the vehicle 2, the first steering current candidate value  $I_{sc1}$  is selected as the steering current value  $I_s$ .

[0042] In contrast, the steering current selecting unit 115 selects the second steering current candidate value  $I_{sc2}$  as the steering current value  $I_s$  when on one of the automated driving instruction  $O_a$  and the stop instruction  $O_e$  is input. Accordingly, in the manual driving mode of the vehicle 2, the second steering current candidate value  $I_{sc2}$  is selected as the steering current value  $I_s$ .

[0043] The steering current value  $I_s$  is input from the steering current selecting unit 115 to the steering declining control unit 116. The steering declining instruction  $O_r$  is input from the turning declining control unit 123 of the turning control unit 120 to the steering declining control unit 116. When the steering declining instruction  $O_r$  is not input, the steering declining control unit 116 generates the output instruction  $O_s$  to instruct the steering actuator 3 to apply the steering current value  $I_s$ . As a result, when the steering current value  $I_s$  is the first steering current candidate value  $I_{sc1}$ , the actual steering angle  $\theta_{sr}$  of the steering wheel 21 is controlled to be an angle corresponding to the output instruction  $O_s$  through the steering actuator 3 that works based on the generated output instruction  $O_s$ . In contrast, when the steering current value  $I_s$  is the second steering current candidate value  $I_{sc2}$ , the actual steering torque  $T_{sr}$  of the steering wheel 21 is controlled to be an angle corresponding to the output instruction  $O_s$  through the steering actuator 3 that works based on the generated output instruction  $O_s$ .

[0044] Further, when the steering declining instruction  $O_r$  is input, the steering declining control unit 116 generates the output instruction  $O_s$  for instructing the steering actuator 3 to apply the current value adjusted by the gradual reduction control. As a result, the steering actuator 3 that works depending on the generated output instruction  $O_s$  controls the steering output torque  $T_{so}$  for the steering wheel 21.

[0045] As shown in FIG. 3, the turning control unit 120 is configured to selectively perform the control of the turning output torque  $T_{to}$  output from the turning actuator 4 and the control of the actual turning angle  $\theta_{tr}$  through the turning actuator 4. The turning control unit 120 includes a target turning angle selecting unit 121, a turning angle following control unit 122, and a turning declining control unit 123.

[0046] The automated driving instruction  $O_a$  and the planned trajectory  $T_z$  are input to the target turning angle selecting unit 121 from the trajectory generation unit 100. The actual steering angle  $\theta_{sr}$  is input to the target turning angle selecting unit 121 from the steering angle sensor 53. The target turning angle selecting unit 121 is configured to select, based on these inputs, the target turning angle  $\theta_{tt}$  that is the control target for the actual turning angle  $\theta_{tr}$ .

[0047] Specifically, when the automated driving instruction  $O_a$  is input from the trajectory generation unit 100, the target turning angle selecting unit 121 selects the target turning angle  $\theta_{tt}$  acquired to follow the planned trajectory  $T_z$ . In contrast, when the automated driving instruction  $O_a$  is not input from the trajectory generation unit 100, the target turning angle selecting unit 121 performs the linked control in which the actual turning angle  $\theta_{tr}$  changes in conjunction with the actual steering angle  $\theta_{sr}$ . In the linked control, the target turning angle selecting unit 121 selects, as the target turning angle  $\theta_{tt}$ , a converted angle that is calculated by multiplying the actual steering angle  $\theta_{sr}$  by a predetermined conversion ratio.

**[0048]** The target turning angle  $\theta_{tt}$  is input to the turning angle following control unit **122** from the target turning angle selecting unit **121**. The turning angle following control unit **122** executes turning angle following control such as PID control to make the actual turning angle  $\theta_{tr}$  follow the target turning angle  $\theta_{tt}$ . As a result of the steering angle following control, the turning angle following control unit **122** determines the turning current value  $I_t$  applied to the turning motor **40**.

**[0049]** The turning current value  $I_t$  is input from the turning angle following control unit **122** to the turning declining control unit **123**. The stop instruction  $O_e$  is input from the trajectory generation unit **100** to the turning declining control unit **123**. The stop instruction is input from the actuation sensor **55** to the turning declining control unit **123**. When the stop instruction  $O_e$  is not input, the turning declining control unit **123** generates the output instruction  $O_t$  to instruct the turning actuator **4** to apply the turning current value  $I_t$ . As a result, the actual turning angle  $\theta_{tr}$  of the turning tire **20** is controlled to be an angle corresponding to the output instruction  $O_t$  by the turning actuator **4** works depending on the generated output instruction  $O_t$ .

**[0050]** In contrast, when the stop instruction  $O_e$  is input, the turning declining control unit **123** generates the output instruction  $O_t$  for instructing the turning actuator **4** to apply the current value adjusted by the gradual reduction control. As a result, the turning actuator **4** that works depending on the generated output instruction  $O_t$  controls the turning output torque  $T_{to}$  for the turning tire **20**. The steering output torque  $T_{tr}$  substantially matches the actual turning torque  $T_{tr}$  of the turning tire **20**.

**[0051]** The flow of the method performed in the stop control mode in the steering control method for controlling to link the steering and turning of the vehicle **2** by the trajectory generation unit **100**, the steering control unit **110**, and the turning control unit **120** will be described with reference to FIGS. 5-7. This flow is started in response to the stop instruction  $O_e$  from one of the trajectory generation unit **100** and the actuation sensor **55**. Further, in FIG. 5, "S" means steps of the process executed by instructions included in the steering control program. In FIG. 6, except for the graph of the stop instruction  $O_e$ , the change over time of the value related to turning is illustrated by a solid line, and the change over time of the value related to steering is illustrated by a dotted line.

**[0052]** In **S101** of FIG. 5, the target steering angle selecting unit **111** selects, in response to the stop instruction  $O_e$ , the target steering angle  $\theta_{st}$  such that the actual steering angle  $\theta_{sr}$  changes together with the actual turning angle  $\theta_{tr}$ . As a result, the selected target steering angle  $\theta_{st}$  is converted by the steering angle following control unit **113** into the first steering current candidate value  $I_{sc1}$ , and the converted value  $I_{sc1}$  is input from the steering current selecting unit **115** to the steering declining control unit **116** as the steering current value  $I_s$  in response to the stop instruction  $O_e$ . Therefore, the output instruction  $O_s$  instructing the steering current value  $I_s$  is output, and accordingly the actual steering angle  $\theta_{sr}$  is controlled in conjunction with the actual turning angle  $\theta_{tr}$  as shown in FIG. 6. The linked control stating in **S101** continues until **S103** starts. Accordingly, **S102** is performed during the linked control.

**[0053]** As shown in FIG. 5, in **S102**, the turning declining control unit **123** gradually decreases the turning current value  $I_t$  as shown in FIGS. 6, 7. The turning declining con-

trol unit **123** stores the turning current value  $I_t$ , as an initial value  $I_{t0}$  which is to be gradually decreased, at the time when the stop instruction  $O_e$  is input. The turning declining control unit **123** substantially continuously decreases the turning current value  $I_t$  from the initial value  $I_{t0}$  such that the rate of decrease of the turning current value  $I_t$  per unit time is constant. The turning declining control unit **123** outputs, to the turning actuator **4**, the output instruction  $O_t$  that instructs the decrease of the turning current value  $I_t$ . According to this, the turning output torque  $T_{to}$  output from the turning actuator **4** to the turning tire **20** is controlled according to the output instruction  $O_t$  to gradually decrease such that the rate of decrease per unit time is constant. As a result of the above, when the turning current value  $I_t$  reaches 0, the turning declining control unit **123** terminates the gradual decrease of the turning current value  $I_t$ , and then outputs the steering declining instruction  $O_r$  to the steering declining control unit **116**. The gradual decrease of the turning current value  $I_t$  may be terminated when the turning current value  $I_t$  reaches a predetermined value close to 0.

**[0054]** As shown in FIG. 5, in **S103**, the steering declining control unit **116** performs the gradual decrease of the steering current value  $I_s$  shown in FIGS. 6, 7 in response to the steering declining instruction  $O_r$  after the gradual reduction control of the turning current value  $I_t$  by the turning declining control unit **123** ends. The steering declining control unit **116** stores, as an initial value  $I_{s0}$  of the steering current value  $I_s$  which is to be gradually decreased, the steering current value  $I_s$  at the time when the steering declining instruction  $O_r$  is input. The steering declining control unit **116** substantially continuously decreases the steering current value  $I_s$  from the initial value  $I_{s0}$  such that the rate of decrease of the steering current value  $I_s$  per unit time is constant. The steering declining control unit **116** outputs, to the steering actuator **3**, the output instruction  $O_s$  that instructs the decrease of the steering current value  $I_s$ . According to this, the steering output torque  $T_{so}$  output from the steering actuator **3** to the steering wheel **21** is controlled according to the output instruction  $O_s$  to gradually decrease such that the rate of decrease per unit time is constant. As a result of the above, the steering declining control unit **116** terminates the gradual decrease of the steering current value  $I_s$  when the steering current value  $I_s$  reaches 0. The gradual decrease of the steering current value  $I_s$  may be terminated when the steering current value  $I_s$  reaches a predetermined value close to 0.

**[0055]** As shown in FIG. 5, after the gradual reduction control of the steering current value  $I_s$  by the **116** ends, the turning declining control unit **123** cuts the energization from the turning driver **42** to the turning motor **40** in **S104**. Further, after the gradual reduction control of the steering current value  $I_s$  by the **116** ends, the steering declining control unit **116** cuts the energization from the steering driver **32** to the steering motor **30** in **S105**. The timings of **S104** in which the energization to the steering motor **30** is cut and **S105** in which the energization to the turning motor **40** is cut may offset (in FIGS. 5, 6, **S104** is performed first), and may be substantially the same. In the present embodiment shown in FIG. 6, the cut of the energization of the steering motor **30** and the cut of the energization of the turning motor **40** are realized by shutting down (turning off, in FIG. 6) power by the corresponding driver **32**, **42**. However, when the applied

current is decreased to 0 in S102, S103, the cut of the energization is realized when the current reaches 0.

[0056] In the present embodiment, S101, S103, S105 correspond to a steering control step, and S102, S104 correspond to a turning control step.

#### Operation Effect

[0057] The operation and effects of the first embodiment described above will be described below.

[0058] According to the first embodiment, during the linked control in which the actual steering angle  $\theta_{sr}$  is controlled in conjunction with the actual turning angle  $\theta_{tr}$  in response to the stop instruction  $O_e$  of the vehicle 2, the turning output torque  $T_{to}$  output from the turning actuator 4 is controlled to gradually decrease, and then the energization to the turning actuator 4 and the steering actuator 3 is cut. According to this, the twist of the turning tire 20 is gradually eliminated according to the gradual reduction control of the turning output torque  $T_{to}$  before the energization to the turning actuator 4 and the steering actuator 3 is cut, and accordingly the turning tire 20 is limited from sudden restoration. Accordingly, the vibration of the turning tire 20 and the vehicle chassis supporting the turning tire 20 due to the restoration of the turning tire 20 can be suppressed in response to the stop instruction  $O_e$ . Further, since the gradual reduction control of the turning tire  $T_{to}$  is performed during the linked control of the actual steering angle  $\theta_{sr}$  and the actual turning angle  $\theta_{tr}$ , the vibration of the steering wheel 21 which moves in conjunction with the turning tire 20 can be suppressed in response to the stop instruction  $O_e$ .

[0059] In the first embodiment, the turning output torque  $T_{to}$  can be controlled to gradually decrease by the gradual decrease of the current to the turning actuator 4. Accordingly, the vibration of the vehicle 2 due to the sudden restoration of the turning tire 20 can be suppressed in response to the stop instruction of the vehicle 2.

[0060] In the first embodiment, the current to the turning actuator 4 is decreased to 0, and thus the turning output torque  $T_{to}$  is also decreased to 0. Accordingly, the gradual elimination of the twist of the turning tire 20 can be continued until the turning output torque  $T_{to}$  becomes zero, thereby the vibration can be further suppressed in response to the stop instruction of the vehicle.

[0061] In the first embodiment, the energization of the turning actuator 4 and the steering actuator 3 is cut after the steering output torque  $T_{so}$  from the steering actuator 3 is controlled to be gradually decreased. According to this, even when the steering output torque  $T_{so}$  remains after the linked control of the actual steering angle  $\theta_{sr}$  and the actual turning angle  $\theta_{tr}$ , the movement and vibration of the steering wheel 21 due to the remaining torque  $T_{so}$  can be suppressed by the gradual change of the torque according to the gradual decrease. Therefore, it is possible to suppress the steering wheel 21 from hindering the effect of suppressing the vibration in response to the stop command  $O_e$  of the vehicle 2.

[0062] In the first embodiment, the steering output torque  $T_{so}$  can be controlled to gradually decrease by the gradual decrease of the current to the steering actuator 3. Therefore, it is possible to suppress the steering wheel 21 from hindering the effect of suppressing the vibration in response to the stop command  $O_e$  of the vehicle 2.

[0063] In the first embodiment, the current to the steering actuator 3 is decreased to 0, and thus the steering output

torque  $T_{so}$  is also decreased to 0. Accordingly, the steering output torque  $T_{so}$  can be gradually decreased to be zero, and thus it is possible to suppress the vibration in response to the stop instruction  $O_e$  of the vehicle 2.

[0064] In the first embodiment, the energization of the turning actuator 4 and the steering actuator 3 is cut by shutting down the power after the gradual reduction control of the turning output torque  $T_{to}$  and the steering output torque  $T_{so}$ . According to this, the vibration of the turning tire 20 and the vehicle chassis due to the restoration of the turning tire 20 can be suppressed by the power shutdown in response to the stop instruction  $O_e$ .

#### Second Embodiment

[0065] A second embodiment shown in FIGS. 8, 9 is a modification of the first embodiment. In the second embodiment, the configurations of the turning declining control unit 123 and the steering declining control unit 116 are different from the first embodiment.

[0066] The turning declining control unit 123 of the second embodiment includes a low-pass filter whose cutoff frequency is lower than the resonance frequency of a turning system 6 shown in FIG. 8. The turning system 6 includes at least the turning tire 20 of the vehicle 2. The turning system 6 may include the turning actuator 4. The turning system 6 may include actuation members such as a tire rod located between the turning actuator 4 and the turning tire 20. The turning declining control unit 123 is configured to gradually decrease the turning current value  $I_t$  by the low-pass filter. Accordingly, in addition to the turning current value  $I_t$ , the decrease rate per unit time of the turning current value  $I_t$  is also gradually reduced as shown in FIG. 9.

[0067] The steering declining control unit 116 of the second embodiment includes the low-pass filter having a predetermined cutoff frequency. The steering declining control unit 116 is configured to gradually decrease the steering current value  $I_s$  by the low-pass filter. Accordingly, in addition to the steering current value  $I_s$ , the decrease rate per unit time of the steering current value  $I_s$  is also gradually reduced as shown in FIG. 9.

#### Operation Effect

[0068] The operation and effect of the second embodiment described above will be described below.

[0069] In the second embodiment, the current applied to the turning actuator 4 is gradually decreased by using the low-pass filter whose cutoff frequency is lower than the resonance frequency of the turning system 6 including the turning tire 20 of the vehicle 2. According to this, the vibration of the turning system 6 and the vehicle chassis due to the restoration of the turning tire 20 can be suppressed in response to the stop instruction  $O_e$ .

#### Other Embodiments

[0070] While some embodiments of the present disclosure have been described above, the present disclosure should not be interpreted to be limited to these embodiments, and can be applied to various other embodiments and combinations thereof without departing from the scope of the subject matter of the present disclosure.

[0071] The turning actuator 4 of the modified example may constitute a power steering system that is mechanically

linked to the steering wheel **21** and the steering actuator **3** and is controllable independently of the actuator **3**.

**[0072]** The dedicated computer of the steering control device **1** in a modification example may be at least one outside center computer communicating with the vehicle **2**. The dedicated computer of the steering control device **1** of the modification example may include at least one of a digital circuit and an analog circuit as a processor. In particular, the digital circuit is at least one type of, for example, an ASIC (Application Specific Integrated Circuit), a FPGA (Field Programmable Gate Array), an SOC (System on a Chip), a PGA (Programmable Gate Array), a CPLD (Complex Programmable Logic Device), and the like. Such a digital circuit may include a memory in which a program is stored.

**[0073]** In a modification example of the first and second embodiments, the turning current value  $I_t$  may be intermittently decreased instead of continuously being decreased. In a modification example of the first and second embodiments, the steering current value  $I_s$  may be intermittently decreased instead of continuously being decreased.

**[0074]** In a modification example of the second embodiment, the turning current value  $I_t$  may be gradually decreased such that the decrease rate gradually increases, instead of gradually decreasing such that decrease rate per unit time gradually decreases. In a modification example of the second embodiment, the steering current value  $I_s$  may be gradually decreased such that the decrease rate gradually increases, instead of gradually decreasing such that decrease rate per unit time gradually decreases.

**[0075]** In a modification, the first embodiment and the second embodiment may be combined. Specifically, only one of the turning declining control unit **123** and the steering declining control unit **116** may decrease the current by the low-pass filter as in the second embodiment.

What is claimed is:

1. A steering control device of a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator, the steering control device comprising:

a steering control unit configured to control an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and  
 a turning control unit configured to control an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator, wherein  
 the turning control unit is configured to perform a gradual reduction control to gradually decrease the turning output torque during a linked control in which the steering control unit controls the actual steering angle in conjunction with the actual turning angle in response to a stop instruction of the vehicle,  
 the turning control unit is configured to cut an energization to the turning actuator after the gradual reduction control of the turning output torque performed by the turning control unit, and  
 the steering control unit is configured to cut an energization to the steering actuator after the gradual reduction control of the turning output torque performed by the turning control unit.

2. The steering control device according to claim 1, wherein the turning control unit is configured to perform the gradual reduction control of the turning output torque by gradually decreasing a current applied to the turning actuator.

3. The steering control device according to claim 2, wherein the turning control unit is configured to gradually decrease the current applied to the turning actuator to zero.

4. The steering control device according to claim 2, wherein the turning control unit is configured to gradually decrease the current applied to the turning actuator by a low-pass filter whose cutoff frequency is lower than a resonance frequency of a turning system that includes the turning tire of the vehicle.

5. The steering control device according to claim 1, wherein the turning control unit is configured to cut the energization to the turning actuator after a gradual reduction control of the steering output torque performed by the steering control unit to gradually decrease the steering output torque, and

the steering control unit is configured to cut the energization to the steering actuator after the gradual reduction control of the steering output torque performed by the steering control unit.

6. The steering control device according to claim 5, wherein the steering control unit is configured to perform the gradual reduction control of the steering output torque by gradually decreasing a current applied to the steering actuator.

7. The steering control device according to claim 6, wherein the steering control unit is configured to gradually decrease the current applied to the steering actuator to zero.

8. The steering control device according to claim 1, wherein the turning control unit is configured to cut the energization to the turning actuator by shutting off power, and the steering control unit is configured to cut the energization to the steering actuator by shutting off power.

9. A steering control method for a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator, the steering control method comprising:

controlling an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and

controlling an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator, wherein

in the controlling the actual turning angle, a gradual reduction control is performed to gradually decrease the turning output torque during a linked control in which the actual steering angle is controlled in conjunction with the actual turning angle in response to a stop instruction of the vehicle,

in the controlling the actual turning angle, an energization to the turning actuator is cut after the gradual reduction control of the turning output torque, and

in the controlling the actual steering angle, an energization to the steering actuator is cut after the gradual reduction control of the turning output torque.

10. The steering control method according to claim 9, wherein

in the controlling the actual turning angle, the gradual reduction control of the turning output torque is performed by gradually decreasing a current applied to the turning actuator.

11. The steering control method according to claim 10, wherein

in the controlling the actual turning angle, the current applied to the turning actuator is gradually decreased to zero.

**12.** The steering control method according to claim **10**, wherein

in the controlling the actual turning angle, the current applied to the turning actuator is gradually decreased by a low-pass filter whose cutoff frequency is lower than a resonance frequency of a turning system that includes the turning tire of the vehicle.

**13.** The steering control method according to claim **9**, wherein

in the controlling the actual turning angle, the energization to the turning actuator is cut after a gradual reduction control of the steering output torque to gradually decrease the steering output torque, and

in the controlling the actual steering angle, the energization to the steering actuator is cut after the gradual reduction control of the steering output torque.

**14.** The steering control method according to claim **13**, wherein

in the controlling the actual steering angle, the gradual reduction control of the steering output torque is performed by gradually decreasing a current applied to the steering actuator.

**15.** The steering control method according to claim **14**, wherein

in the controlling the actual steering angle, the current applied to the steering actuator is gradually decreased to zero.

**16.** The steering control method according to claim **9**, wherein

in the controlling the actual turning angle, the energization to the turning actuator is cut by shutting off power, and

in the controlling the actual steering angle, the energization to the steering actuator is cut by shutting off power.

**17.** A computer program product for a vehicle for controlling and linking a movement of a steering member by a steering actuator with a movement of a turning tire by a turning actuator, the computer program product being stored on at least one non-transitory computer readable medium and comprising instructions configured to, when executed by at least one processor, cause the at least one processor to:

control an actual steering angle of the steering member by controlling a steering output torque output from the steering actuator; and

control an actual turning angle of the turning tire by controlling a turning output torque output from the turning actuator, wherein

in the controlling the actual turning angle, a gradual reduction control is performed to gradually decrease the turning output torque during a linked control in which the actual steering angle is controlled in conjunction with the actual turning angle in response to a stop instruction of the vehicle,

in the controlling the actual turning angle, an energization to the turning actuator is cut after the gradual reduction control of the turning output torque, and

in the controlling the actual steering angle, an energization to the steering actuator is cut after the gradual reduction control of the turning output torque.

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