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(54) **PUMP AND BLOCKING DEVICE**

(71) Applicant: **Watson Marlow GmbH**, Rommerskirchen (DE)

(72) Inventors: **Achim Staedele**, Unterheinriet (DE); **Eric Mayer**, Heilbronn (DE)

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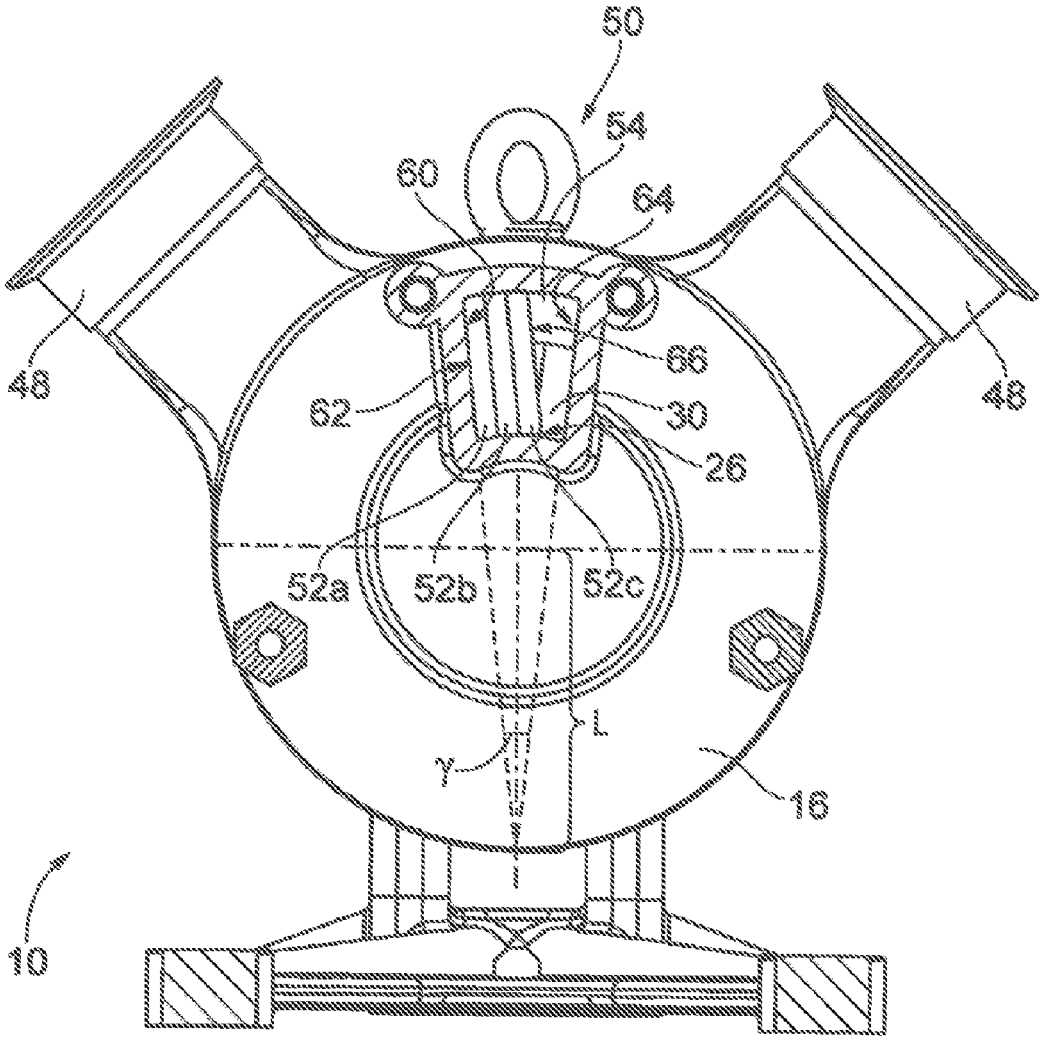
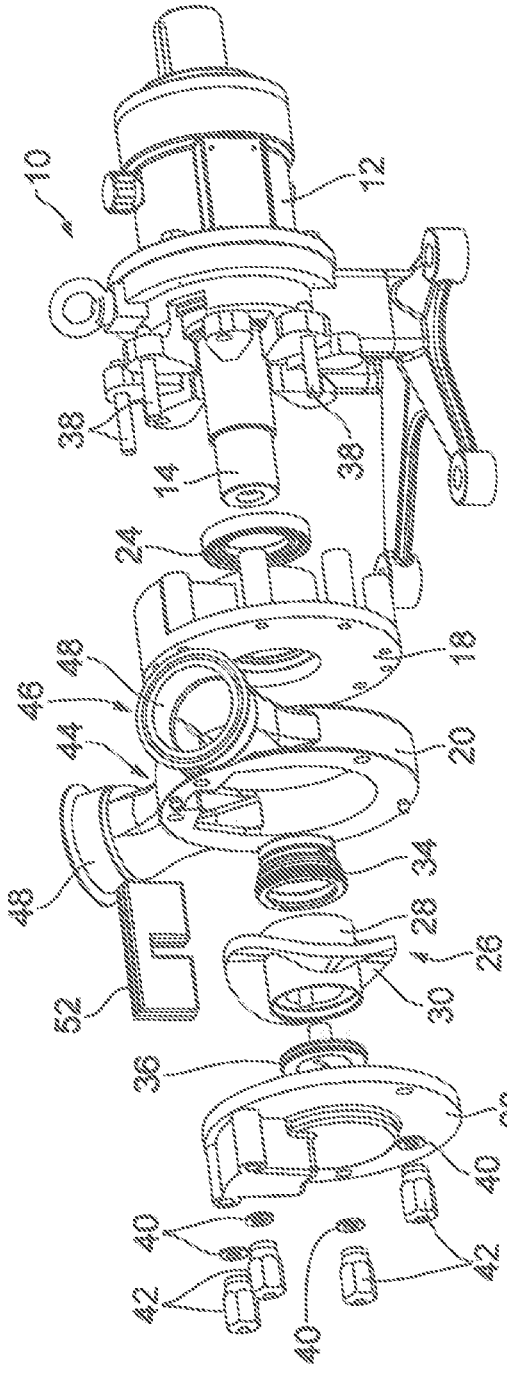
(57) **ABSTRACT**

A blocking device for a pump having a rotor that is rotatable about a rotation axis in a pump duct and comprises a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner, wherein the blocking device comprises a plurality of blocking elements which are configured to block the pump duct in the axial direction on both sides of the rotor collar, wherein each of the plurality of blocking elements has a slot with a U-shaped sealing profile for abutting against the rotor collar, a sealing face for abutting against the rotor hub, and two contacting faces for abutting against a seat of the pump duct and/or against a contacting face of another blocking element of the plurality of blocking elements, and to a pump having a corresponding blocking device.

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12
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38
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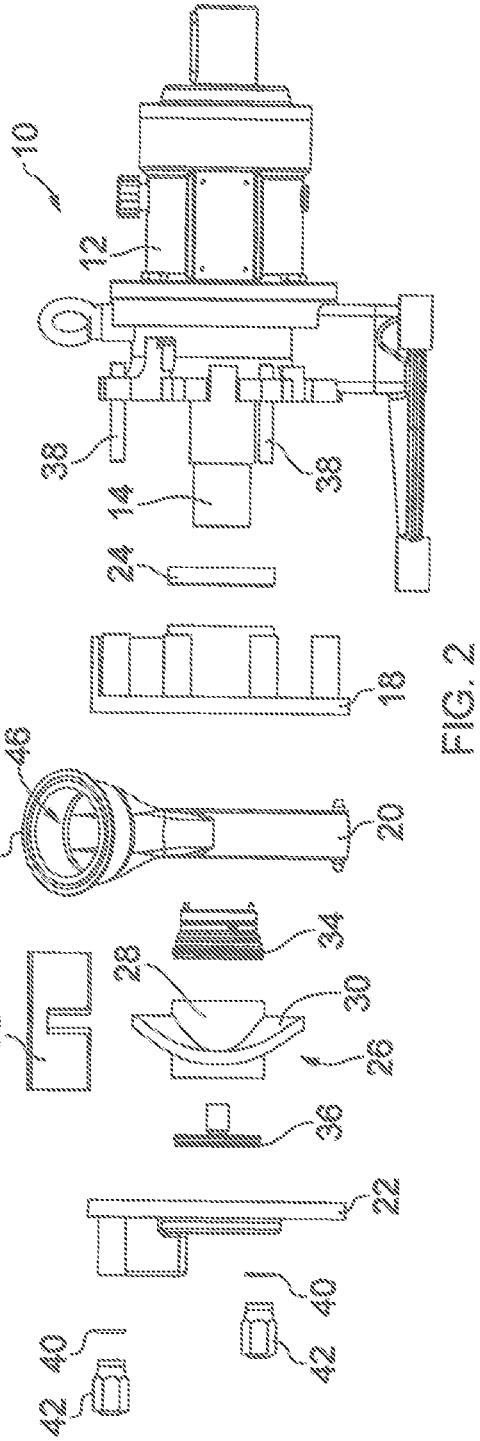
40

FIG. 1


10
12
38
38
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34
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36
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42

FIG. 2

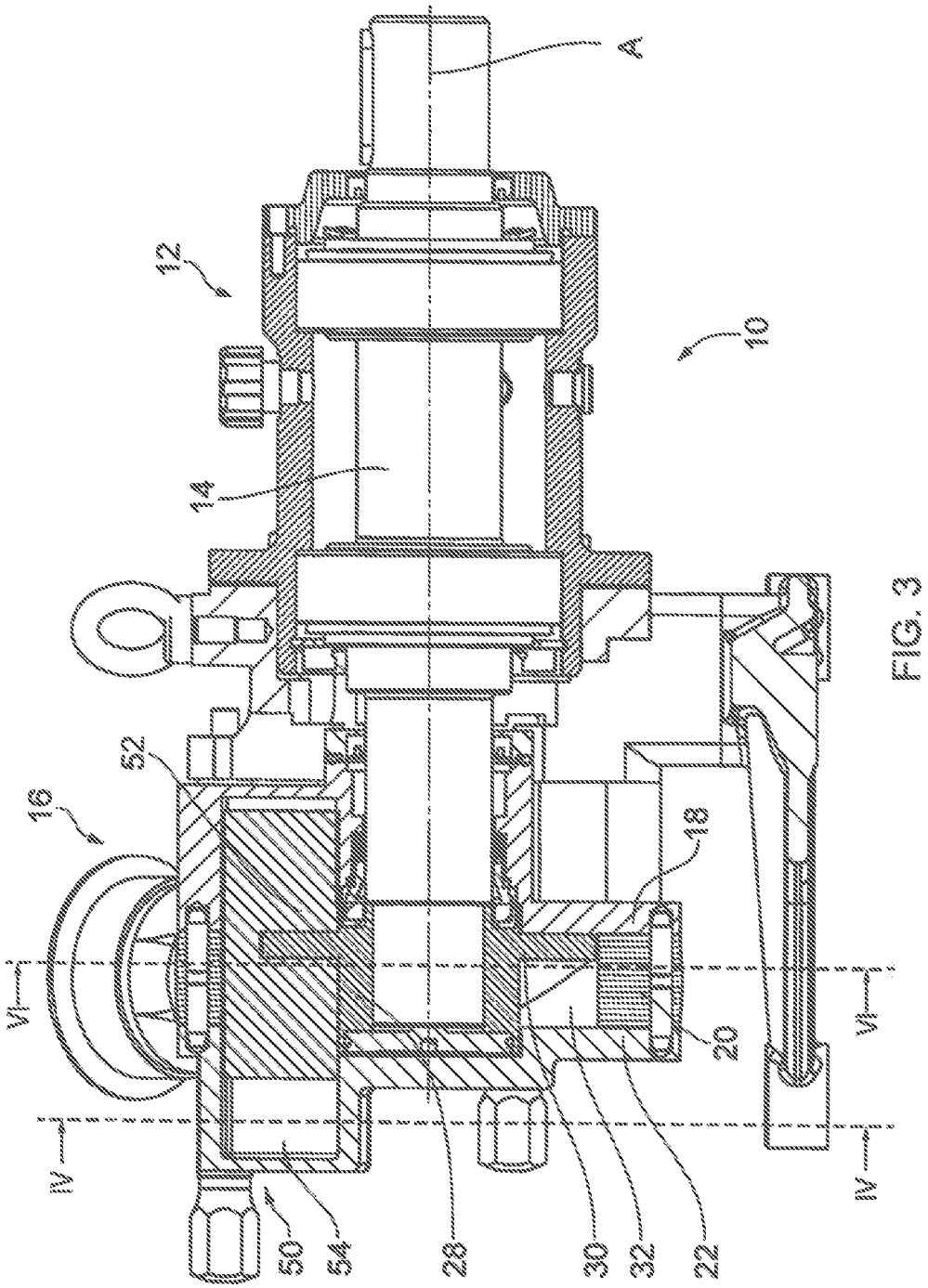

FIG. 3

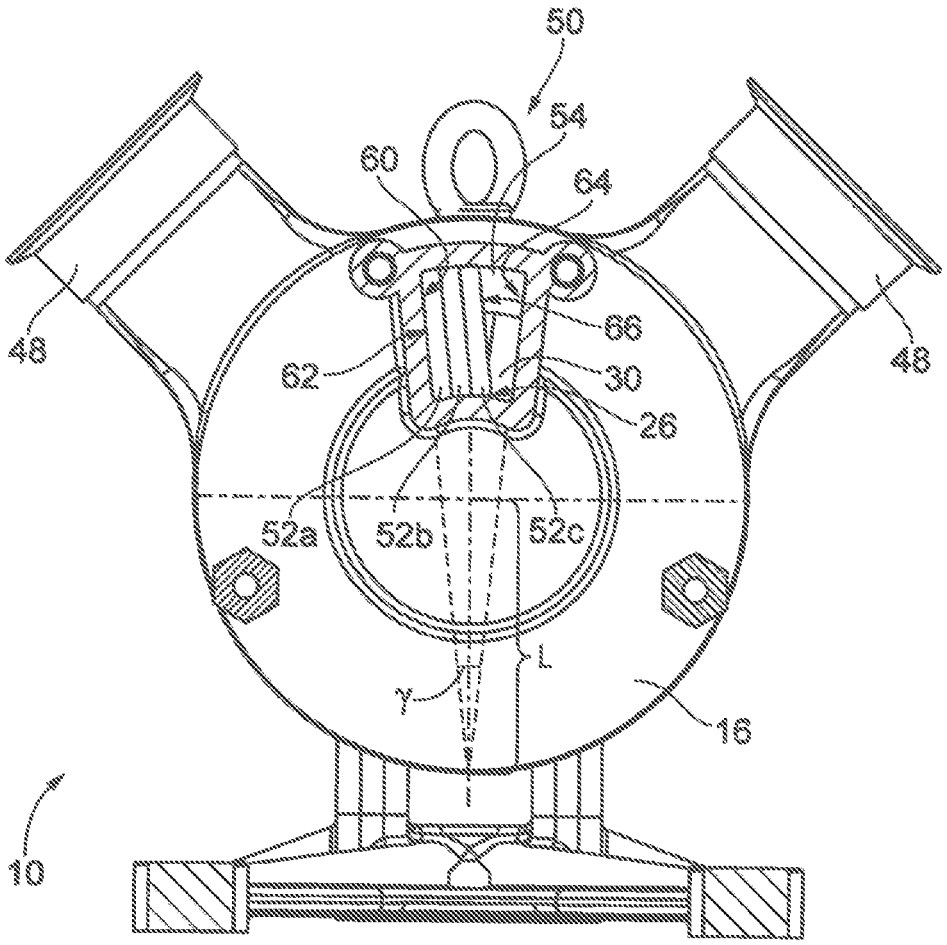

FIG. 4

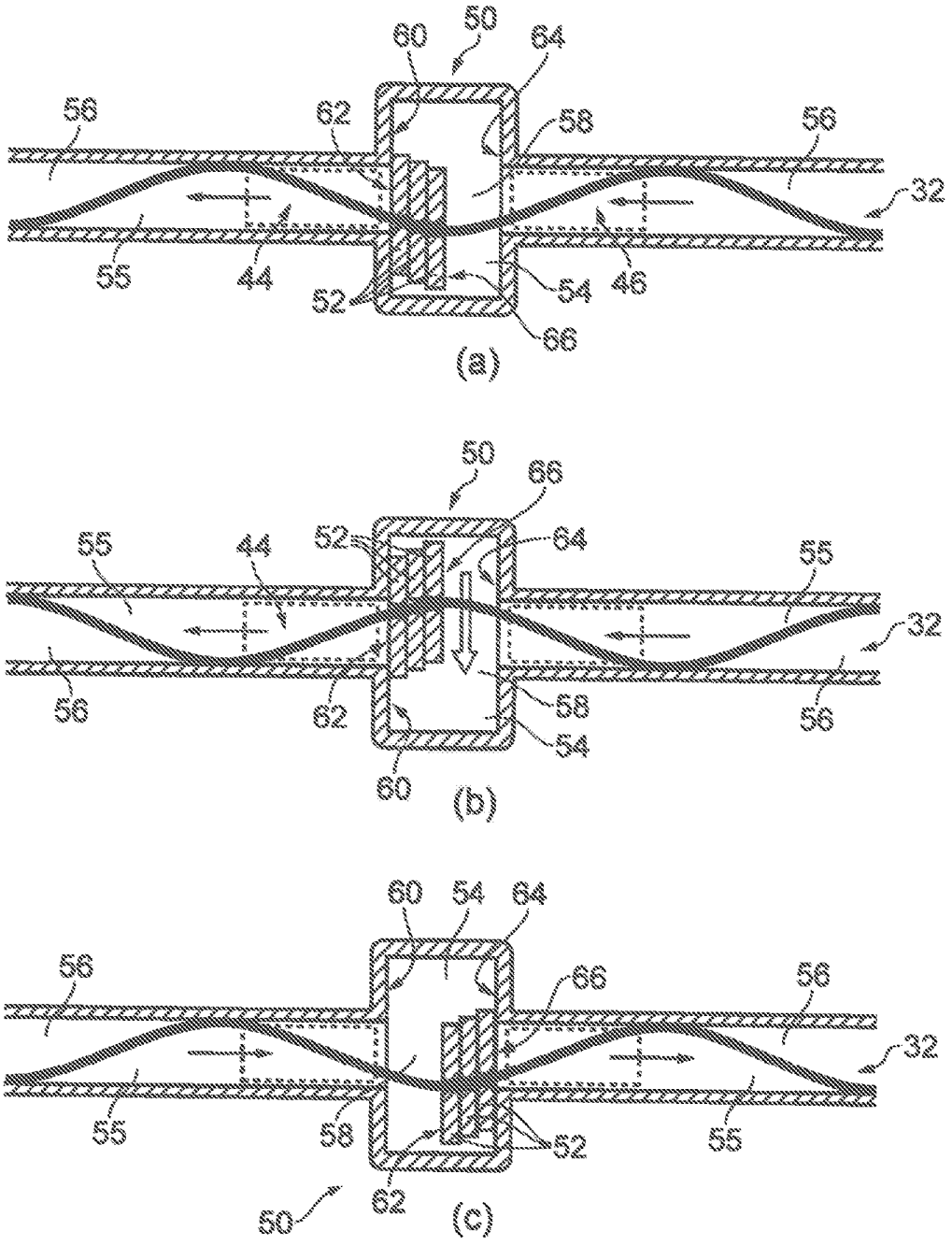
60
50
64
56
62
58
56
32
55
44
52
54
46
66
(a)
50
66
55
44
52
64
55
32
56
62
58
56
54
60
(b)
60
54
64
56
66
56
32
55
58
52
55
62
50
(c)

FIG. 5

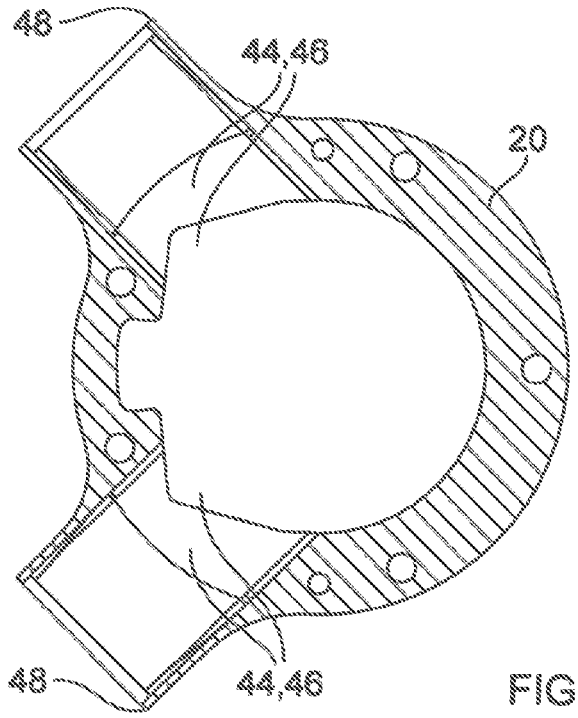

FIG. 6

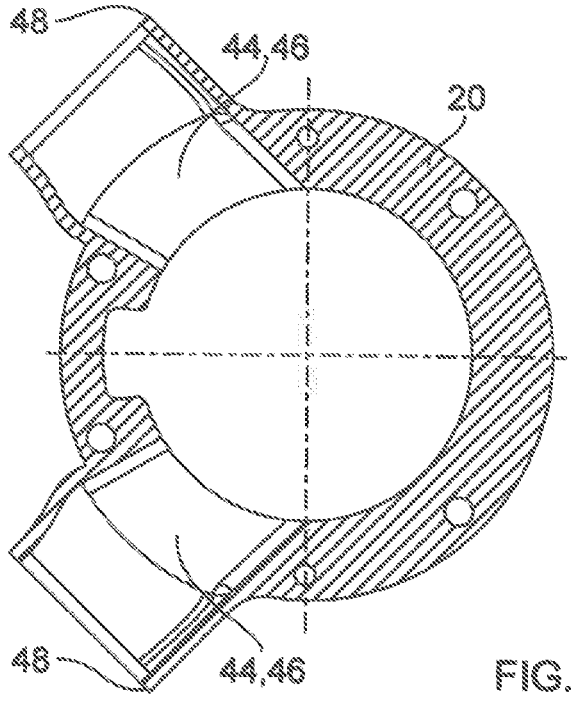

FIG. 7

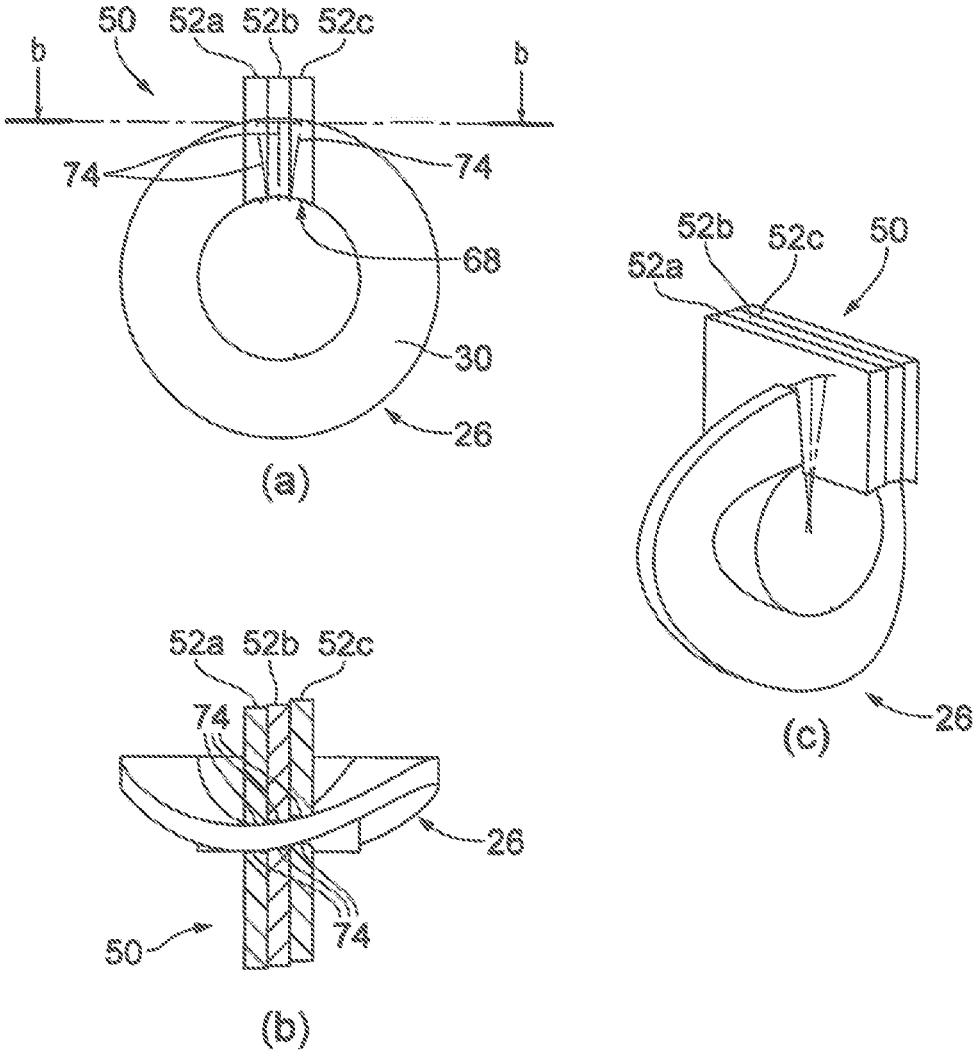
50
52a
52b
52c
b
b
74
74
68
30
26
(a)
52b
52c
50
52a
26
(c)
52a
52b
52c
74
26
50
74
(b)

FIG. 8

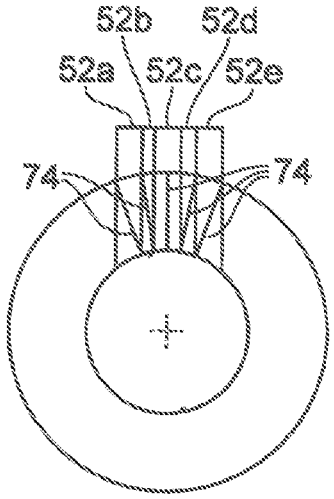

FIG. 9

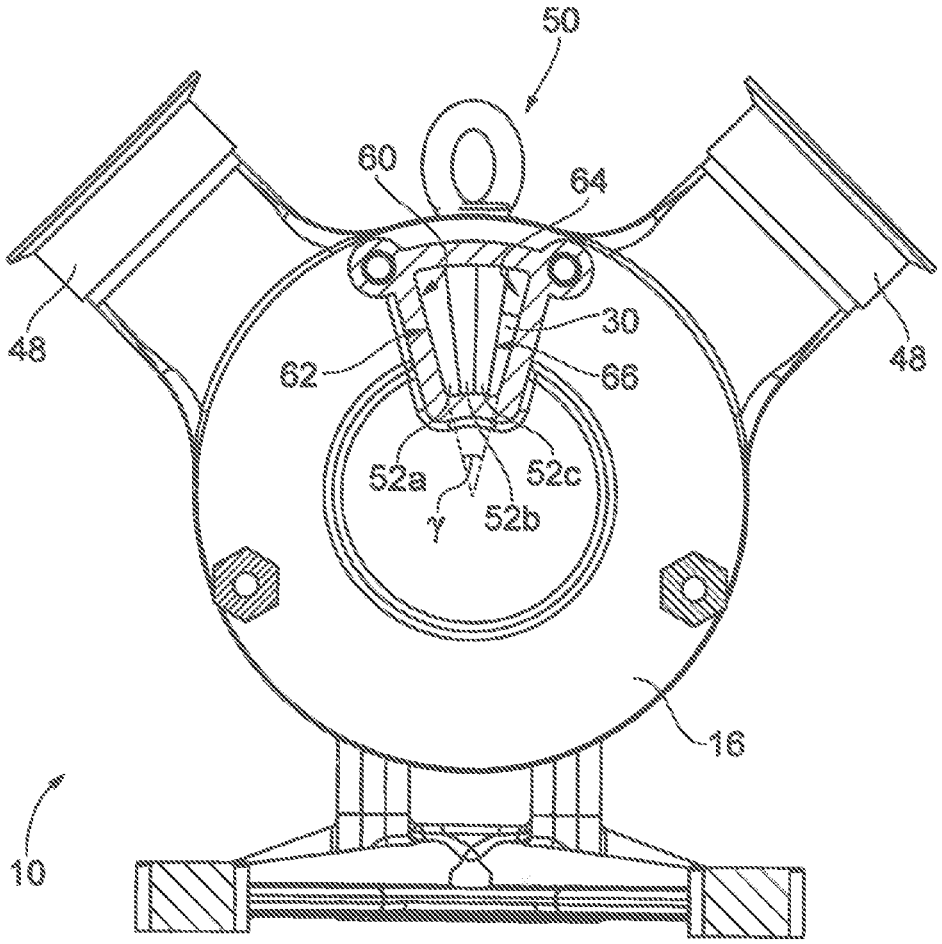

FIG. 10

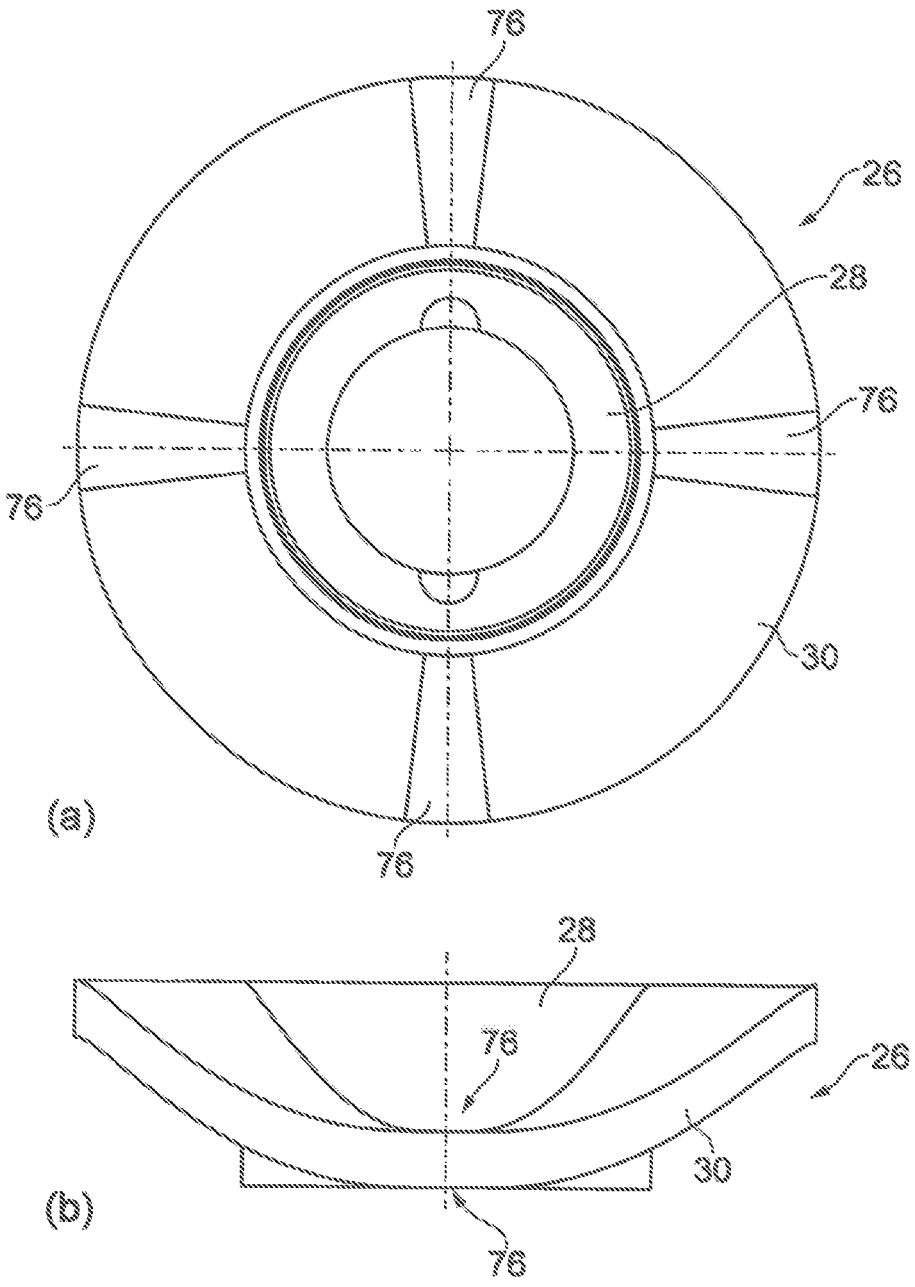

FIG. 11

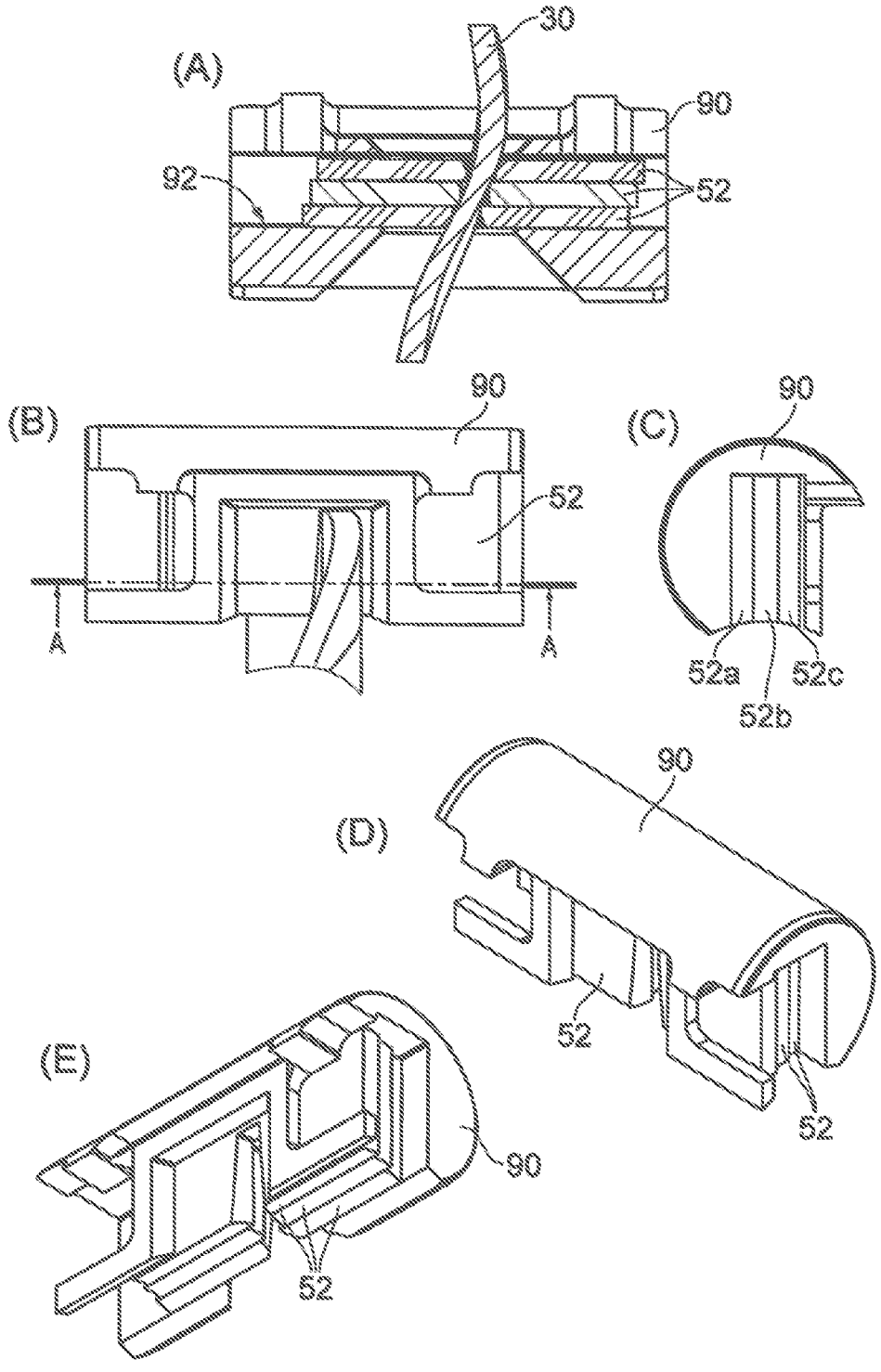

FIG. 12

PUMP AND BLOCKING DEVICE

TECHNICAL FIELD

[0001] The invention relates to a pump having a rotor that is rotatable about a rotation axis and comprises a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner.

BACKGROUND

[0002] Such pumps are known as sinusoidal pumps. In a pump duct, a fluid to be pumped is pumped from an inlet to an outlet by rotation of the rotor. A blocking device is provided which prevents the fluid to be pumped from being transported back from the outlet to the inlet. The blocking device has a blocking element which comprises the rotor collar and blocks the pump duct in the axial direction on both sides of the rotor collar. The blocking element is mounted in a guide which allows a one-dimensional movement in the axial direction in a manner corresponding to the rotor collar encircling the rotor hub in an undulating manner.

SUMMARY

[0003] A blocking device according to the invention is provided for a pump having a rotor that is rotatable about a rotation axis in a pump duct and comprises a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner. The blocking device comprises a plurality of blocking elements which are configured to block the pump duct in the axial direction on both sides of the rotor collar. Each of the plurality of blocking elements has a slot with a U-shaped sealing profile for abutting against the rotor collar, a sealing face for abutting against the rotor hub, and two contacting faces for abutting against a seat of the pump duct and/or against a contacting face of another blocking element of the plurality of blocking elements. The plurality of blocking elements form a plurality of sealing lines on the rotor, with the result that the sealing action of the blocking element is improved. Since the blocking elements can abut against one another via the contacting faces, complex mounting of the different blocking elements is not necessary.

[0004] Preferably, an odd number of blocking elements are provided. In this way, a middle blocking element can define a central axis or plane, relative to which the further blocking elements are arranged.

[0005] At least two blocking elements can each be formed in a uniform manner. As a result of the use of uniform blocking elements, the design of the blocking device is simplified and assembly errors can be prevented and the production costs can be reduced. Preferably, all of the blocking elements are formed in a uniform manner. It is also possible for example for two respective blocking elements to be formed in a uniform manner and to be mounted in different orientations in the blocking device, thereby allowing for example a symmetrical configuration of the blocking device.

[0006] According to a first preferred embodiment, the blocking elements each have parallel contacting faces. In this way, the blocking device can be configured in a compact manner. Furthermore, this allows for example employment of the blocking device according to the invention in known pump housings.

[0007] According to an alternative embodiment, the blocking elements can each have contacting faces which are arranged at an angle and are each parallel to the radial direction of the rotor. This simplifies the geometry of the blocking elements and of the sealing faces and profiles, since blocking elements abutting against one another are each arranged in the radial direction of the rotor.

[0008] Preferably, the blocking elements each have a sealing profile which comprises a sealing lip extending in the radial direction of the rotor. This allows a good sealing function of the respective blocking elements.

[0009] The invention furthermore relates to a pump having a rotor that is rotatable about a rotation axis and comprises a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner, a pump housing having a pump duct which connects a first inlet/outlet space to a second inlet/outlet space, and an above-described blocking device.

BRIEF DESCRIPTION OF THE DRAWINGS

[0010] Further features and advantages of the invention can be gathered from the following description and from the drawings, to which reference is made. In the drawings:

[0011] FIG. **1** shows a pump according to the invention in an exploded perspective view having a blocking device according to the invention;

[0012] FIG. **2** shows the pump from FIG. **1** in an exploded side view;

[0013] FIG. **3** shows a sectional view of the pump from FIG. **1** in the axial direction;

[0014] FIG. **4** shows a sectional view of the pump from FIG. **3** on the section plane IV-IV;

[0015] FIG. **5** shows schematic views of the pump duct of a pump according to the invention;

[0016] FIG. **6** shows a sectional view of the central housing component of the pump from FIG. **3** on the section plane VI-VI;

[0017] FIG. **7** shows a sectional view of the central housing component according to an alternative embodiment;

[0018] FIG. **8** shows detail views of a blocking element and rotor according to a first embodiment of the invention;

[0019] FIG. **9** shows a detail view of a blocking element and rotor according to a second embodiment of the invention;

[0020] FIG. **10** shows a sectional view of a pump having a blocking element according to a third embodiment of the invention;

[0021] FIG. **11** shows detail views of a rotor of the pump from FIG. **1**; and

[0022] FIG. **12** shows views of the blocking device according to the invention in a pump having a guide component.

DETAILED DESCRIPTION

[0023] FIGS. **1** and **2** each show a pump **10** in an exploded view. The pump **10** comprises a shaft mounting unit **12** which supports a shaft **14**. Attached to the shaft mounting unit **12** is a pump housing **16** having a first axial housing component **18**, a central annular housing component **20** and a second axial housing component **22**.

[0024] Provided between the first axial housing component **18** and the shaft mounting unit **12** is a sealing element **24**.

[0025] The shaft **14** projects into the pump housing **16** in a manner supported on one side. A rotor **26** comprises a rotor hub **28** and a rotor collar **30** that extends from the rotor hub **28** in the radial direction and encircles it in an undulating manner. The rotor **26** is fastened to the shaft **14** via a fastening bolt **36**. The one-sided support allows a simple configuration of the pump housing **16**, since it is in particular not necessary to support the shaft **14** in the second axial housing component **22**. Some other kind of support of the shaft **14**, for example a support on both sides, can also be provided.

[0026] In the following text, references to an axial direction relate to the rotation axis of the rotor **26** and references to a radial direction relate to a corresponding radial direction centered on the rotation axis. "Axially rearward" relates to the direction pointing towards the shaft mounting unit **12** and "axially forward" relates to the direction pointing towards the pump housing **16**. The first axial housing component **18** is thus the axially rear housing component and the second axial housing component **22** is thus the axially front housing component.

[0027] Provided between the rotor **26** and the first axial housing component **18** is a mechanical face seal **34**. Instead of the mechanical face seal, some other sealing element can also be provided.

[0028] The mounting of the shaft **14**, the sealing element **24** and the mechanical face seal **34** and the fastening of the rotor **26** to the shaft **14** can also be configured in some other manner.

[0029] In the embodiment shown, the pump housing **16** is held together via four bolts **38**, washers **40** and nuts **42**, wherein the bolts **38** each extend from the shaft mounting unit **12** through all three housing components **18**, **20**, **22**. However, some other fastening method can also be provided. For example, independent fastening of the housing components **18**, **20**, **22** to one another and of the pump housing **16** to the shaft mounting unit **12** can be provided or independent fastening of the second axial housing component **22** can be provided. This allows modular assembly and disassembly of the pump **10**. Alternative ways of fastening the housing components **18**, **20**, **22** can also be provided. For example, the housing component **18** can be fastened to the shaft mounting unit **12** and the housing components **20** and **22** can be fastened to the housing component **18** via grub screws in the housing component **18**.

[0030] The central annular housing component **20** has a first inlet/outlet space **44** and a second inlet/outlet space **46**, which are each formed with a connection element **48** for connection to a pipeline.

[0031] A blocking device **50** comprises a plurality of blocking elements **52** and is configured to block a pump duct in the axial direction on both sides of the rotor collar **30**. Each of the plurality of blocking elements **52** has a slot **72** with a U-shaped sealing profile **70** for abutting against the rotor collar **30**, a sealing face **68** for abutting against the rotor hub **28**, and two contacting faces **62**, **66** for abutting against a seat of the pump duct and/or against a contacting face of another blocking element **52** of the plurality of blocking elements **52**.

[0032] FIG. **3** shows the pump **10** in a sectional view on a section plane perpendicularly through the rotation axis A of the rotor **26** and the shaft **14**. The housing components **18**, **20** and **22** form a pump duct **32** together with the rotor hub **28**, said pump duct **32** extending annularly around the rotor hub **28**. The rotor collar **30** divides the pump duct **32** into various fluid chambers **55** by way of its undulating shape, wherein the radially outer end of the undulating rotor collar **30** adjoins the radial outer wall, formed by the annular housing component **18**, of the pump duct **32** in a sealing manner.

[0033] The blocking device **50** is arranged in an upper sector, in the embodiment shown, of the pump duct **32**. Each of the blocking elements **52** abuts in a sealing manner against the two axial side faces of the rotor collar **30** and against the rotor hub **28**. Upon rotation of the rotor **26**, the blocking elements **52** can each move independently of one another in the axial direction within a chamber **54** as per the undulating shape of the rotating rotor collar **30**.

[0034] The chamber **54** is formed by the pump housing **16** in the embodiment shown and has a seat **60** which forms the transition between the chamber **54** and the annular pump duct **31**.

[0035] FIG. **4** shows a section through the chamber **54** of the pump **10** on the section plane IV-IV in FIG. **3** in the case of an anticlockwise direction of rotation of the rotor **26**.

[0036] A first blocking element **52***a* abuts against the seat **60** of the chamber **54** with a first contacting face **62** in every axial position, and abuts against the rotor collar **30** with the U-shaped sealing profile and against the rotor hub **28** with the sealing face **68**. A second blocking element **52***b* abuts against the second contacting face of the first blocking element **52***a* with a first contacting face and forms a second sealing line on the rotor collar **30** and the rotor hub **28** with the U-shaped sealing profile and the sealing face **68**. A third blocking element **52***c* abuts against the second contacting face of the second blocking element **52***b* with a first contacting face and forms a third sealing line on the rotor collar **30** and the rotor hub **28** in an analogous manner. The plurality of blocking elements **52** therefore block the annular pump duct **31** and prevent the fluid to be pumped from being transported back through the annular pump duct **31**.

[0037] The second contacting face **66** of the third blocking element **52***c* is configured, in the embodiment shown, to abut against a second seat **64** in a second operating direction, in which the rotor rotates clockwise, and thus to block the annular pump duct. This is described further below in conjunction with FIG. **5**.

[0038] An exchange duct **58** is formed within the chamber **54** and allows fluid to flow in the axial direction between the axially front fluid chamber and the axially rear fluid chamber. In this way, compression of the fluid by a change in volume during an axial movement of the blocking element and the rotor collar is avoided.

[0039] It is also possible for the blocking elements **52** to form an exchange duct **58** which extends in the axial direction between an axially front fluid chamber **55** and an axially rear fluid chamber **55** on the opposite side of the rotor collar **30**. This can be formed for example as a groove in one or more blocking elements or between two contacting faces, abutting against one another, of two blocking elements **52**.

[0040] The chamber **54** has four inner walls. A radially internal wall of the chamber **54** is formed in the shape of a circular arc about the rotation axis of the rotor **26** axially on both sides of the rotor **26** and has the same radius as or a slightly smaller radius than the rotor hub **28** in order to ensure a good fit of the blocking elements **52** on the rotor hub **28**.

[0041] A radially external wall of the chamber **54** has a profile that is in the shape of a circular arc about the rotation axis of the rotor **26**. It is also possible for the radially external wall of the chamber **54** to be formed such that it is spaced apart from the blocking elements **52**, such that the fluid to be pumped on the pressure side can pass between the radially external wall of the chamber **54** and the blocking elements **52** and thus presses the blocking elements **52** against the rotor hub **26**.

[0042] In the circumferential direction, the chamber **54** is formed by two flat walls that are located in the circumferential direction and each surround the flow duct in a U-shaped manner and form a first and second seat **60**, **64** for the blocking elements **52** in the embodiment shown. In this way, the pump **10** can be operated on both sides.

[0043] In the embodiment shown, the blocking elements **52** are each formed with contacting faces **62**, **66** that extend in a parallel manner and are each spaced apart from one another by a thickness d of each particular blocking element **52**. The two flat walls that are located in the circumferential direction are formed in this embodiment such that the blocking element **52** can be displaced by an angle γ in the circumferential direction within the chamber **54** between the first and second seats **60**, **64**. In the embodiment shown, the angle γ is 10°. The angle γ can be in a range from 5° to 40°, wherein the angle is preferably in a range from 5° to 20°.

[0044] To this end, the two flat walls that are located in the circumferential direction are in the radial direction with respect to a center point which is shifted on a central axis of the pump by the distance L, wherein $L=(D/2)/\sin(\gamma/2)$ and D is the overall thickness of all the blocking elements **52** (in this case D=3d). In this way, the centerline of the middle blocking element **52***b* is in each case oriented in the radial direction with respect to the rotation axis A when the first or second blocking element **52***a* or **52***b* abuts respectively against the first or second seat **60**, **64** of the chamber **54** by way of its contacting faces **62** or **66**. The first and second seats are thus each formed in planes which are oriented at the angle γ to one another.

[0045] In order to compensate for a change in volume on account of the axial movement of the rotor collar **30** and of the blocking elements **52**, the exchange duct **58** is formed in the blocking device **50**. This allows a flow of fluid to be pumped between the axially front fluid chamber and the axially rear fluid chamber within the blocking device. Therefore, a compact configuration of the blocking device **50** is allowed, since the chamber **54** of the blocking device does not have to be connected to one of the inlet/outlet spaces **44**, **46**.

[0046] In the chamber **54**, the ratio of the area of the axial flow cross section of the exchange duct **58** to the axial projection area of the rotor collar **30** and of those parts of the blocking elements **52** that project beyond the rotor collar is preferably at least 0.2 and is for example in the range from 0.2 to 0.6. This allows sufficient volume compensation with a compact construction of the blocking device **50**.

[0047] Sub-figures (a) to (c) of FIG. **5** each show a schematic view of the pump duct **31** with the rotor **26** and the blocking device **50**.

[0048] In the embodiment shown, the pump duct is formed by the pump housing **16** itself, i.e. from the three housing components **18**, **20**, **22**. In this way, as can be seen in FIG. **5**, installation space can be saved on in the region of the pump duct. Furthermore, the assembly and disassembly and also cleaning of the pump **10** are simplified.

[0049] The inlet and the outlet of the fluid to be pumped takes place via radially external inlet/outlet spaces **44**, **46** which are shown by way of dashed lines. In the embodiment shown, the inlet/outlet spaces are formed in a symmetrical manner to one another, in order to allow bidirectional operation of the pump **10**.

[0050] The pump duct **32** is formed in an annular manner and extends with a constant cross section from the first radially external inlet/outlet space **44** to the second radially external inlet/outlet space **46**. The blocking device **50** is arranged in the annular pump duct **32** between the two inlet/outlet spaces **44**, **46** and prevents a backflow of the fluid to be pumped counter to the operating direction of the pump. In the region of the radially external inlet/outlet spaces **44**, **46**, fluid to be pumped can flow in the radial direction into the fluid chambers **55** formed by the rotor **26** and the pump housing. When the rotor **26** is rotated, the fluid chambers are moved further along the annular pump duct **32**, wherein one respective fluid chamber **56** closes and allows fluid transport in the pumping direction. On the outlet side of the pump **10**, the fluid chambers move into the region of the blocking device **50**, which blocks the pump duct **32**, with the result that the fluid to be pumped flows in the radial direction out of the fluid chambers and into the outlet-side radially external inlet/outlet space.

[0051] The pump **10** is therefore a positive displacement pump which transports a trapped fixed volume in the closed fluid chamber **56**.

[0052] The function of the blocking device **50** is explained in the following text. The blocking device **50** is arranged between the first inlet/outlet space **44** and the second inlet/outlet space **46** and comprises a plurality of blocking elements **52** which block the pump duct **31** in the axial direction on both sides of the rotor collar **30**. In the embodiment shown in FIG. **5**, three blocking elements **52** are provided.

[0053] The blocking device **50** is configured for bidirectional operation of the pump **10**. To this end, the blocking device **50** has a first seat **60** for the first blocking element **52***a* on the side of the first inlet/outlet space **44**, against which the first blocking element **52***a* abuts by way of a first contacting face **62** in a first operating direction for pumping from the first inlet/outlet space **44** to the second inlet/outlet space **46**, see FIG. **5**(*a*) and (*b*).

[0054] The blocking device also has a second seat **64** for the third blocking element **52***c* on the side of the second inlet/outlet space **46**, against which the third blocking element **52***c* abuts by way of a second contacting face **66** in a second operating direction for pumping from the second inlet/outlet space **46** to the first inlet/outlet space, see FIG. **5**(*c*).

[0055] The spacing between the first seat **60** and the second seat **64** in the circumferential direction is greater than the spacing between the first contacting face **62** and the second contacting face **66** in the circumferential direction.

[0056] When the operating direction of the bidirectional pump **10** is changed, all three blocking elements **52** move from the first seat **60** to the second seat **64** such that the third blocking element **52***c* abuts against the second seat **64** by way of its second contacting face **66**, the second blocking element **52***b* abuts against the first contacting face, facing the first inlet/outlet space **44**, of the third blocking element **52***c* via its second contacting face facing the second inlet/outlet

space **46**, and the first blocking element **52***a* abuts against the first contacting face, facing the first inlet/outlet space **44**, of the second blocking element **52***b* via its second contacting face facing the second inlet/outlet space **46**, and the respectively other contacting face **66**, **62** is spaced apart from the pump housing **16**. Furthermore, the resistance in the fluid to be pumped is reduced and thus the pressure force from the blocking elements to the rotor is reduced, with the result that the frictional forces and thus also the wear to the blocking elements **52** are reduced.

[0057] As can clearly be seen in FIG. **5**(*a*) and (*b*), the volume in chamber **54** changes when the rotor **26** is rotated (from right to left in the drawing) on account of the undulating shape of the rotor collar and the blocking elements **52** moving in the axial direction. Since the blocking device **50** is arranged between the two inlet/outlet spaces **44**, **46**, it is at least sometimes possible for an axial portion of the chamber **54** of the blocking device **50** not to be connected to the associated outlet space **44**, **46**.

[0058] In order to allow this change in volume to be compensated, the exchange duct **58** is formed between the axially front fluid chamber and the axially rear fluid chamber. A fluid flow is shown in the axial direction by the arrow in FIG. **5**(*b*).

[0059] FIG. **6** shows a sectional view through the central housing component **20** in accordance with the section plane VI-VI in FIG. **3**. The housing component **20** is arranged such that the blocking device **50** with the chamber **54** is arranged in a manner rotated by 90° compared with the embodiment shown in FIG. **3**, i.e. on the horizontal central axis of the annular pump duct **32**. Preferably, the pump **10** is formed such that the pump housing **16** can be attached to the shaft mounting unit **12** at different angles.

[0060] The inlet/outlet spaces **44**, **46** are formed radially externally on the annular pump duct **32**, wherein a first part of the inlet/outlet spaces **44**, **46** is formed over the entire axial height of the pump duct in that the central housing component **20** is spaced apart from the pump duct **32** in the radial direction in the region of the inlet/outlet spaces **44**, **46**. In the embodiment shown, the radial spacing of the housing component **20** narrows in the circumferential direction in the respective end region of the inlet/outlet spaces **44**, **46**, such that the first part of the inlet/outlet spaces **44**, **46** is approximately triangular in axial view. A second part of the inlet/outlet spaces **44**, **46** is formed in the housing component **20** and forms a transition to the connection elements **48**.

[0061] The inlet/outlet spaces **44**, **46** are formed in the left-hand upper quadrant and in the left-hand lower quadrant in the housing component **20** in the embodiment shown and each extend as far as the vertical central axis of the annular pump duct **32**. This allows the emptying of residues from the pump.

[0062] FIG. **7** shows a sectional view through the central housing component **20** as per the alternative embodiment. The embodiment differs from the embodiment shown in FIG. **6** in that the housing component **20** is not spaced apart from the pump duct **32** in the radial direction in the region of the inlet/outlet spaces **44**, **46**.

[0063] Sub-figure (a) of FIG. **8** shows an axial plan view of the rotor **26** and the blocking elements **52**. Sub-figure (b) shows a sectional view through the blocking elements **52** in accordance with the section plane b-b in sub-figure (a), and sub-figure (c) shows a perspective view of the components of sub-figure (a).

[0064] As can be seen in sub-figure **8** (a), the first and second contacting faces of the blocking elements **52***a*, **52***b* and **52***c* are each formed in a manner parallel to one another. The middle blocking element **52***b* is formed in a symmetrical manner to its central plane in particular with regard to the sealing face **68** abutting against the rotor hub. The first and third blocking elements **52***a* and **52***c* are each formed in a mirror-symmetrical manner to one another with regard to the central plane of the middle blocking element **52***b*. Since the blocking elements **52***a*, **52***b*, **52***c* are each also formed in a mirror-symmetrical manner to a respective central plane located parallel to the rotor plane, two uniform blocking elements can be used for the two outer blocking elements **52***a* and **52***b*, these being accordingly mounted on the rotor collar **30** in a manner rotated by 180° with respect to one another.

[0065] As can be seen in sub-figure **8** (b), the U-shaped sealing profiles **70** on the slot **72** of the blocking elements **52** are each configured as convex sealing lips **74**. In order to allow optimal contact of the blocking elements **52** against the rotor collar **30**, the sealing lips **74** each extend in the radial direction of the rotor. Thus, the sealing lip **74** of the middle blocking element **52***b* extends parallel to the lateral contacting faces, while the sealing lips **74** of the outer blocking elements **52***a* and **52***c* extend in a manner inclined at an angle to the lateral contacting faces.

[0066] The blocking elements **52** can furthermore also each have an inclined face which is directed at least partially in the axial direction and is configured to press the particular blocking element **52** against the rotor hub **28** in the event of an axial movement in the fluid to be pumped. For example, the surface remote from the rotor hub **28** can be formed in a roof-like manner. Alternatively, an inclined face can be formed in a groove in a blocking element **52** or in a groove between two blocking elements **52**.

[0067] FIG. **9** shows a second embodiment of a blocking device **50** having five blocking elements **52***a* to **52***e*. In an analogous manner to the preceding embodiment, the first and second contacting faces of each blocking element **52***a* to **52***e* extend parallel to one another. The sealing lip **74** of the middle blocking element **52***c* extends parallel to the contacting faces, while the sealing lips **74** of the outer blocking elements **52***a*, **52***b*, **52***d* and **52***e* each extend in an inclined manner with respect to the contacting faces. In an analogous manner to the preceding embodiment, the blocking elements **52***a* and **52***e* and the blocking elements **52***b* and **52***d* are each formed in a uniform manner. In contrast to the preceding embodiment, the different blocking elements **52***a* to **52***e* are formed with different thicknesses, i.e. the outer blocking elements have a greater spacing between the contacting faces than the inner blocking elements. In this way, installation space can be saved for the inner blocking elements, in the case of which the sealing lip extends parallel or at a small angle to the contacting faces, while a correspondingly greater thickness is required for the outer blocking elements **52***a* and **52***e* on account of the profile of the sealing lips **74** with a relatively large angle to the contacting faces.

[0068] FIG. **10** shows a third embodiment of a blocking device **50**, in which the blocking elements **52** are configured such that the first and second contacting faces **62**, **66** are arranged at an angle and each extend in the radial direction of the rotor **26**. In this way, all of the blocking elements **52** can be formed in an identical manner, with the result that the

production costs are reduced and the assembly of the pump and the replacement of blocking elements **52** are simplified.

[0069] In FIG. **10**, the two flat walls of the chamber **54** that are located in the circumferential direction are likewise arranged in the radial direction of the rotor **26**. The first and second seats are thus each formed in planes which are oriented at the angle γ to one another.

[0070] The pump housing and the rotor are otherwise formed in an analogous manner to the preceding embodiments.

[0071] Alternatively, it is also possible for the two walls that are located in the circumferential direction and the contacting faces **62**, **66** of the blocking elements **52** to each have generally cylindrical shapes, in particular curved shapes, that are coordinated with one another. Furthermore, the outer contacting faces **62**, **66**, in the circumferential direction, of the respectively outer blocking elements **52**, which are configured to abut against the first and second seats **60**, **64**, can have a different shape from the contacting faces by way of which the blocking elements **52** abut against one another, for example by way of a wedge shape or arcuate shape of the blocking element **52**.

[0072] The shapes of the two walls that are located in the circumferential direction and of the contacting faces **62**, **66** of the blocking elements **52** can be selected such that the blocking elements are pressed against the rotor hub **26** by the pressure difference when the pump is in operation.

[0073] Sub-figures (a) and (b) of FIG. **11** each show a view of the rotor **26**, wherein sub-figure (a) shows an axial plan view of the rotor **26** and sub-figure (b) shows a radial plan view of the rotor **26**.

[0074] The rotor collar **30** extends in the radial direction from the rotor hub **28** and encircles the rotor hub **28** in an undulating manner. In the embodiment shown, the rotor collar **30** is in the two axial extreme positions at two opposite points each. Thus, the rotor collar forms two fluid chambers on each of the two axial sides of the rotor collar.

[0075] In the embodiment shown, the rotor collar **30** extends in a flattened manner at the axial extreme positions **76**, with the result that the sealing is improved at the axial end faces of the pump duct **32**, which are formed by the two axial housing components **18** and **22**. This allows in particular an enlargement of a gap between the rotor collar **30** and the axial end faces of the pump duct **32**. This allows the pump to generate greater pressures with larger gap dimensions.

[0076] In the embodiment shown, the rotor **26** is produced from an anti-seizure alloy.

[0077] Preferably, a sealing face, in the form of a circumferential groove, for a mechanical seal is provided in the rotor hub **26**.

[0078] It is also possible for other rotor shapes to be used for the pump.

[0079] The pump housing can also be formed in some other manner. For example, the blocking device **50** can also be provided in a known pump housing which provides only one-sided pumping operating. The plurality of blocking elements **52** can in particular also be guided in a guide which allows only a unidirectional movement in the axial direction. In FIG. **12**, a blocking device **50** according to the invention is in a guiding component **90** which allows a linear movement of the blocking elements **52** in the axial direction and forms a seat **92** for one of the blocking elements **52**. The blocking elements are formed in an analogous manner to the embodiment shown in FIG. **4** and FIG. **8**. FIG. **12** (A) shows a sectional view through the blocking device **50**. FIG. **12** (B) shows a view of the high-pressure side of the pump. FIG. **12** (C) shows a view in the axial direction, wherein the high-pressure side of the pump is arranged on the right and the low-pressure side of the pump is arranged on the left. FIGS. **12** (D) and **12** (E) each show perspective views.

1.-**7**. (canceled)

8. Blocking device for a pump, said pump including a rotor that is rotatable about a rotation axis in a pump duct and including a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner, said blocking device comprising:

a plurality of blocking elements which are configured to block the pump duct in the axial direction on both sides of the rotor collar, wherein each of the plurality of blocking elements has a slot with a U-shaped sealing profile for abutting against the rotor collar, a sealing face for abutting against the rotor hub, and two contacting faces for abutting against a seat of the pump duct and/or against a contacting face of another blocking element of the plurality of blocking elements.

9. Blocking device according to claim **8**, wherein an odd number of blocking elements, are provided.

10. Blocking device according to claim **8**, wherein two blocking elements are each formed in a uniform manner.

11. Blocking device according to claim **8**, wherein the blocking elements each have parallel contacting faces.

12. Blocking device according to claim **8**, wherein the blocking elements, each have contacting faces which are arranged at an angle and are each parallel to the radial direction of the rotor.

13. Blocking device according to claim **8**, wherein the blocking each have a sealing profile which comprises a sealing lip extending in the radial direction of the rotor.

14. Pump having a rotor that is rotatable about a rotation axis and comprises a rotor hub and a rotor collar that extends from the rotor hub in the radial direction and encircles it in an undulating manner, a pump housing having a pump duct which connects a first inlet/outlet space to a second inlet/outlet space, and a blocking device according to claim **8**.

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