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(54) **DUAL BLIND CURTAIN PROVIDED WITH DIVIDED TAPE BELT**

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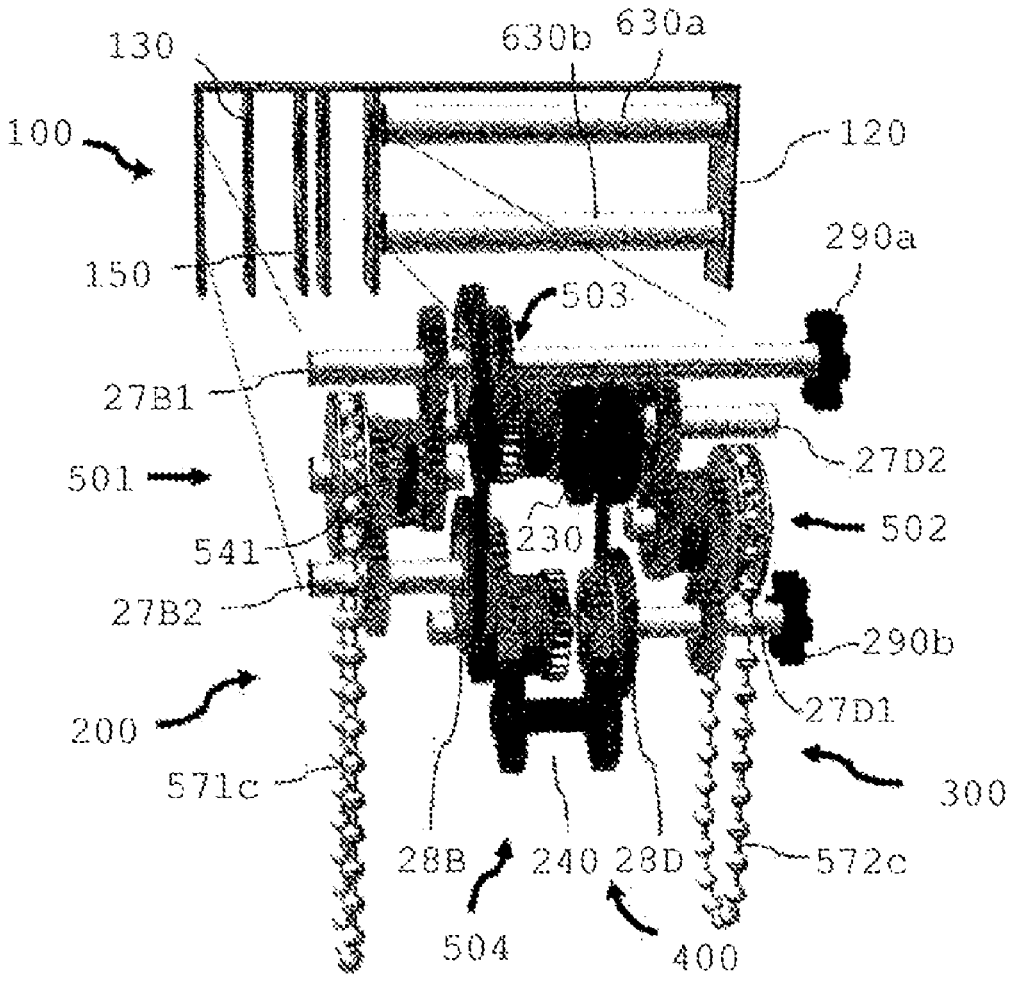
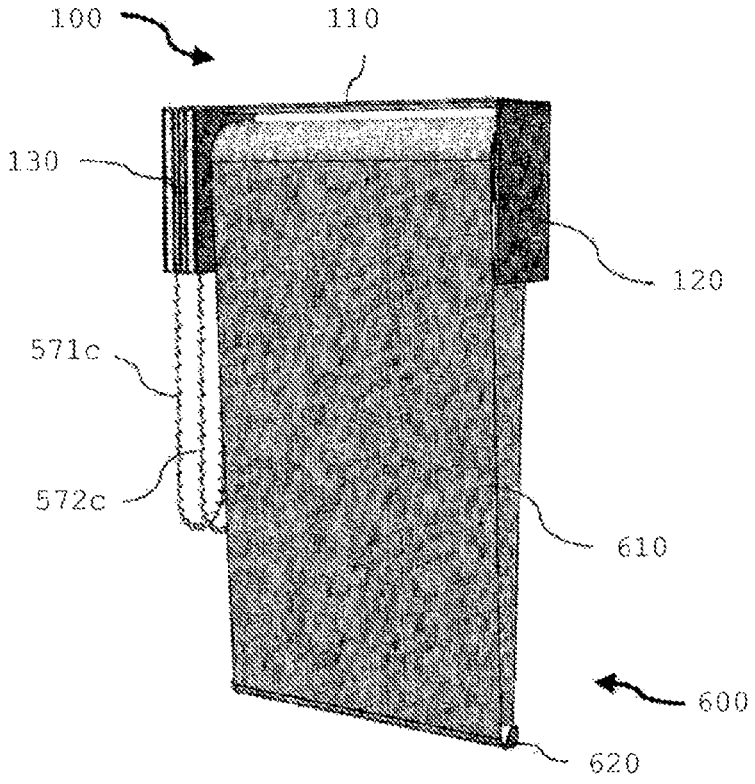
(57) **ABSTRACT**

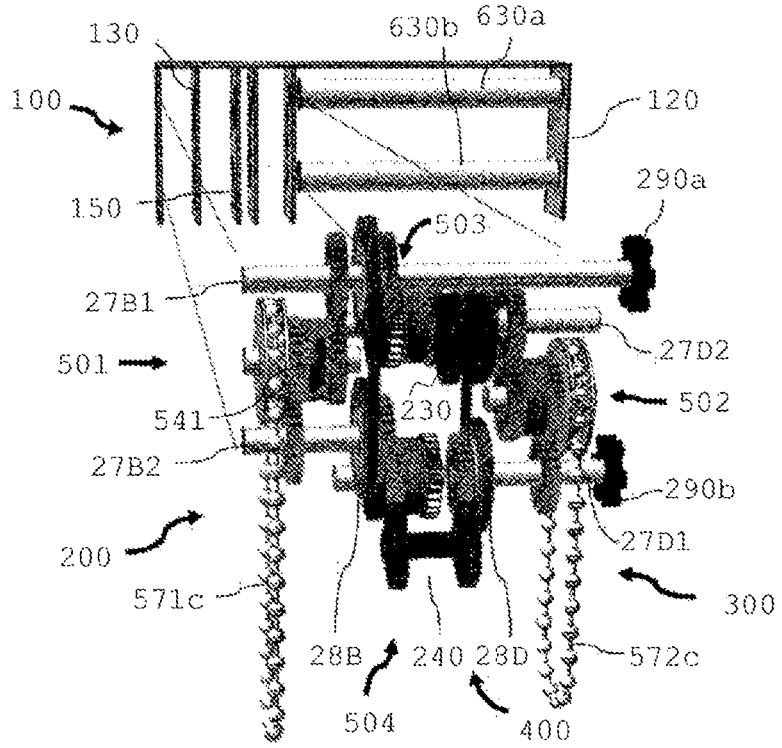
The present invention relates to a dual blind curtain installed at an indoor window, a wall surface, or the like and is used for purposes such as view blocking, light adjustment, heating, and indoor interior decoration effects, wherein: both opposite ends of a curtain sheet composed by connecting multiple patterns to each other are wound in roll shapes around two rotation shafts, respectively, and then curtain circulative rotation and curtain longitudinal rotation are allowed to be freely adjusted by the two rotation shafts; and a difference between the amounts of winding and unwinding of the curtain sheet around and from the rotation shafts, respectively, the difference being caused due to a change in the amount of winding of the curtain sheet around a corresponding shaft, can be identically maintained. Accordingly, the curtain sheet wound around the rotation shafts can be freely changed in patterns that a consumer wants.



[FIG. 1]



[FIG. 2]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501
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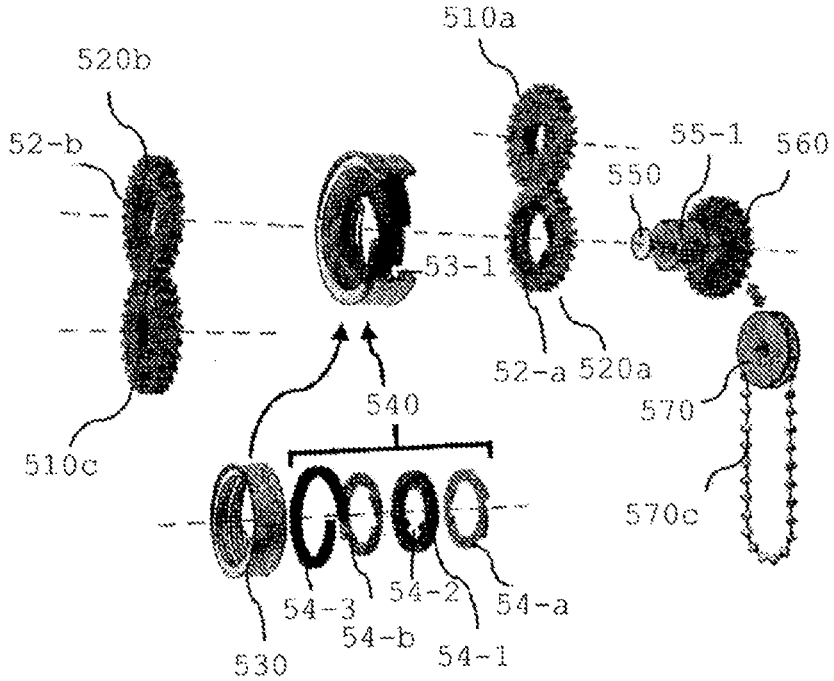
27D1
200
300
571c
28B
240
28D
572c
504
400

[FIG. 3]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52-b
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53-1
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510c
570c
54-3
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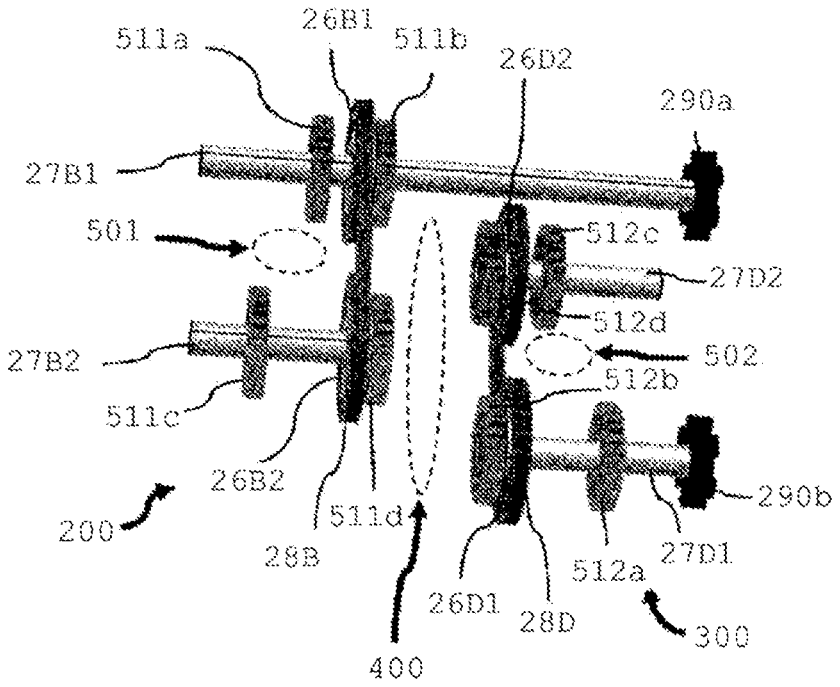
54-a
54-b
54-1
530

[FIG. 4]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27B1
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512d
27B2
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512b
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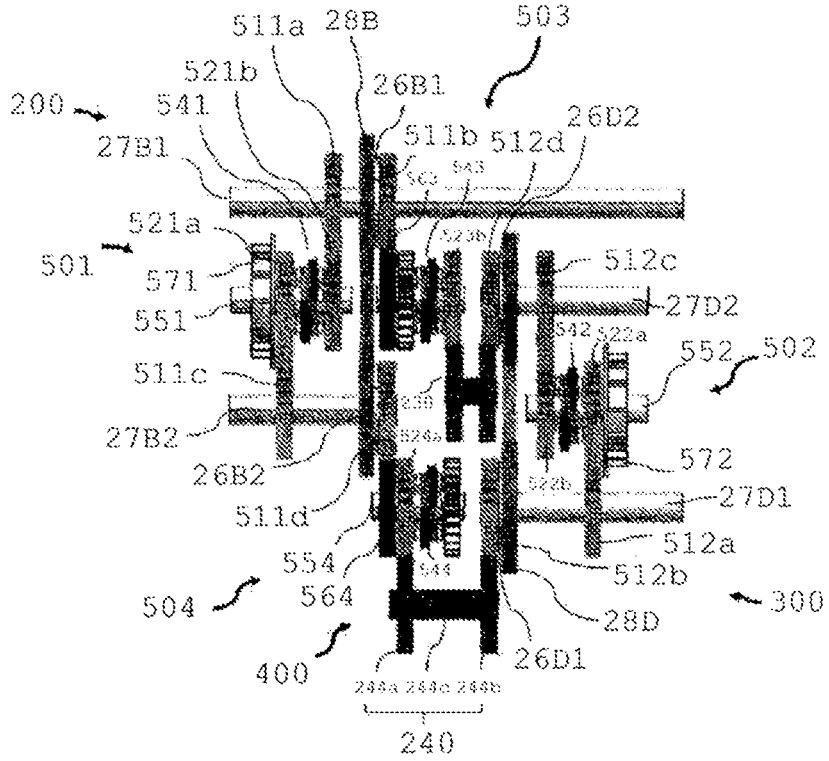
290b
200
511d
27D1
28B
512a
26D1
28D
300
400

[FIG. 5]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541
511b
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26D2
27B1
521a
501
512c
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551
552
502
511c
572
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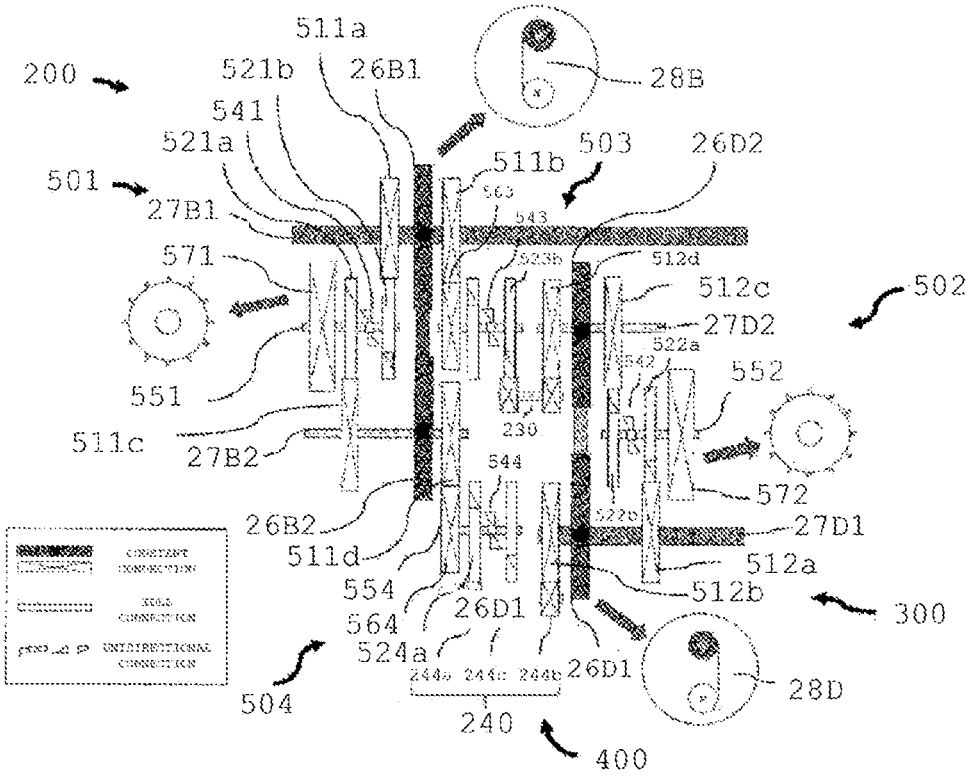
512a
554
512b
564
504
28D
300
400
26D1
240

[FIG. 6]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503
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27B1
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512d
512c
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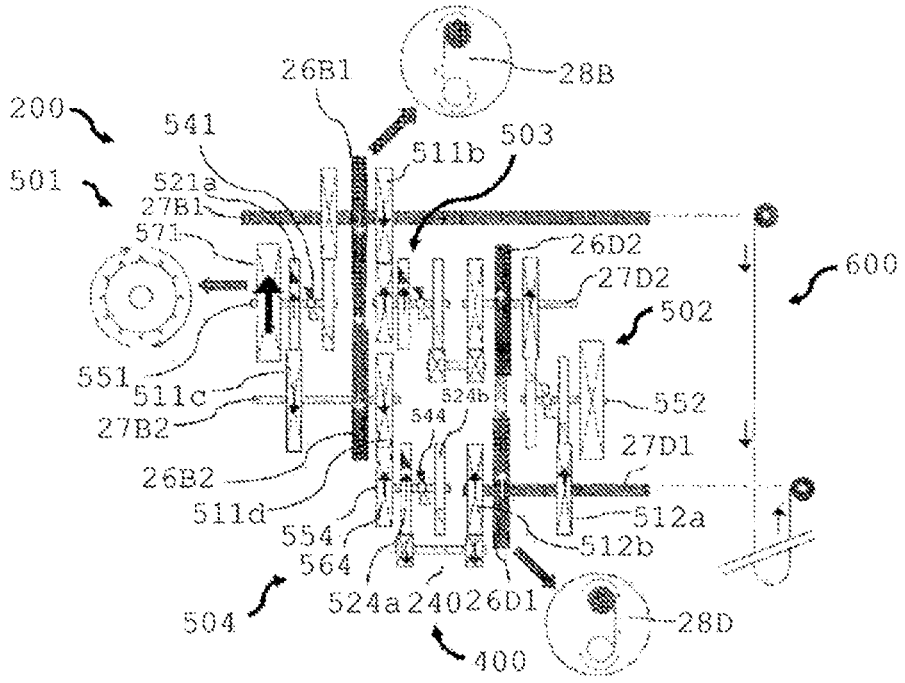
27D2
522a
552
551
511c
27B2
230
544
572
26B2
27D1
511d
512a
554
512b
564
26D1
300
524a
504
26D1
28D
240
400

[FIG. 7]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27B1
571
26D2
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27D2
502
551
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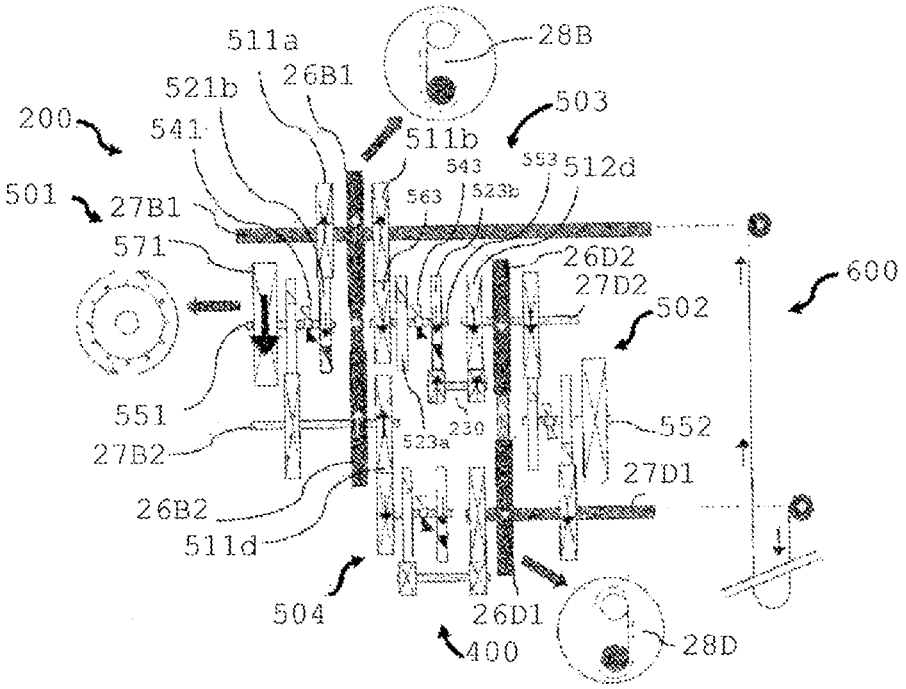
524b
544
27B2
27D1
26B2
511d
512a
554
512b
564
504
524a
240
26D1
28D
400

[FIG. 8]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543
553
512d
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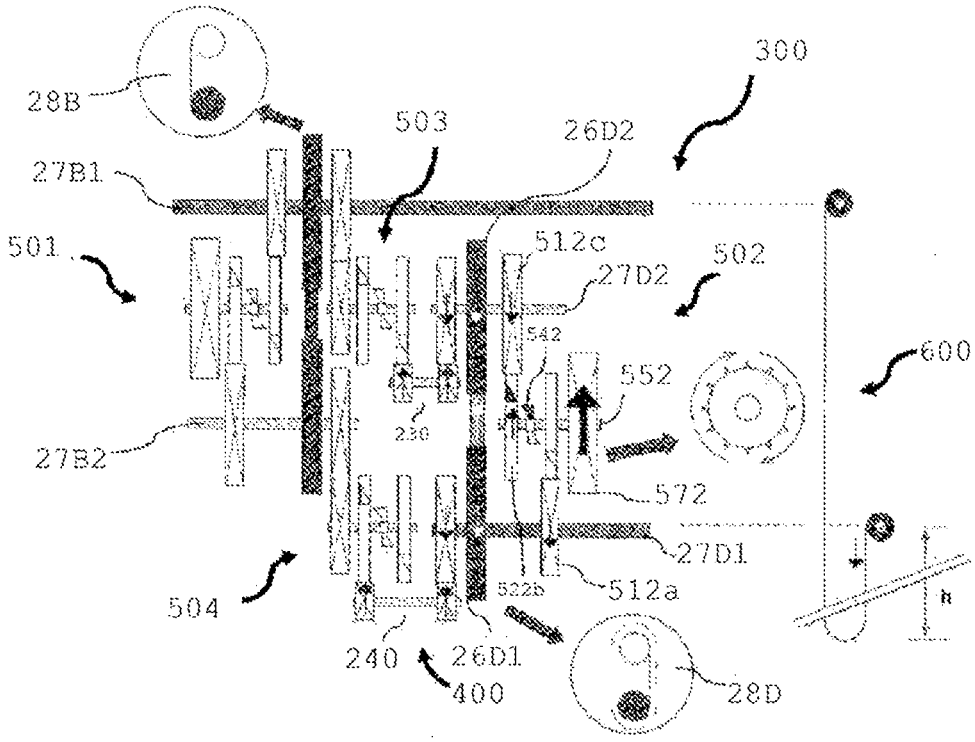
27D2
600
502
551
552
27B2
523a
230
27D1
26B2
511d
504
26D1
28D
400

[FIG. 9]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26D2
27B1
501
512c
27D2
502
542
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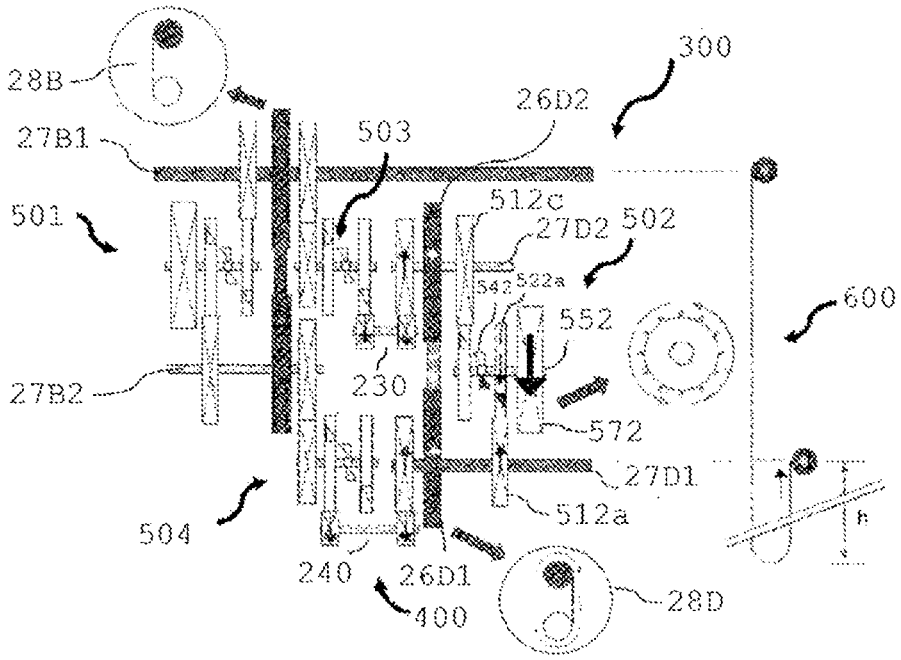
27B2
572
27D1
504
522b
512a
240
26D1
400
28D
h

[FIG. 10]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26D2
27B1
503
512c
501
27D2
502
542
502a
552
600
230
27B2
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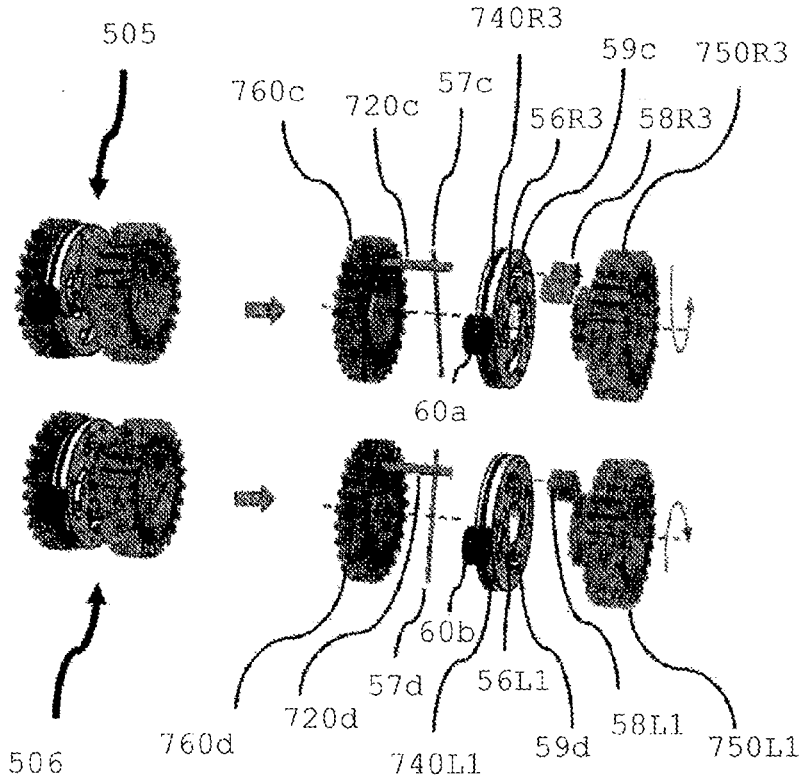
512a
504
240
26D1
28D
400

[FIG. 11]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740R3
59c
750R3
760c
720c
57c
56R3
58R3
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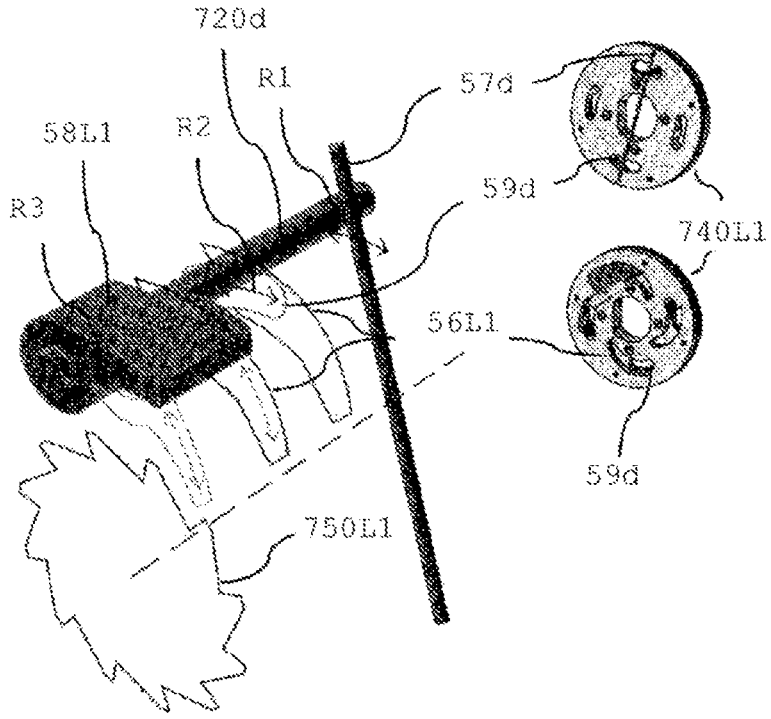
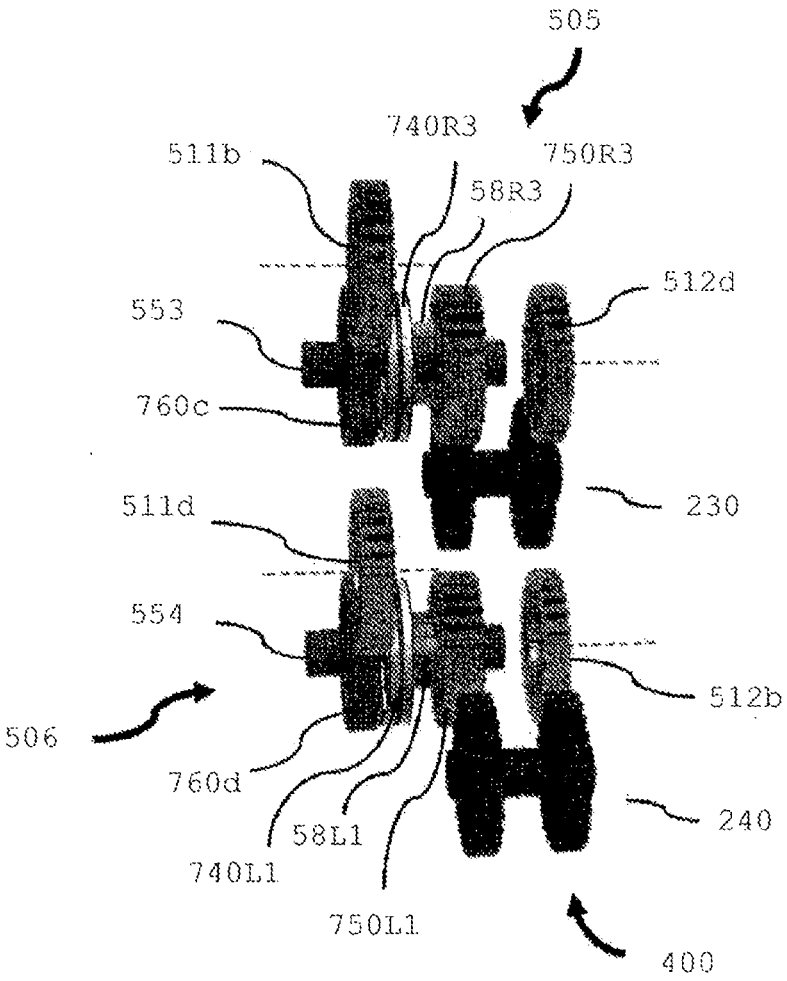
60b
57d
56L1
720d
58L1
760d
59d
750L1
740L1
506

[FIG. 12]



[FIG. 13]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750R3
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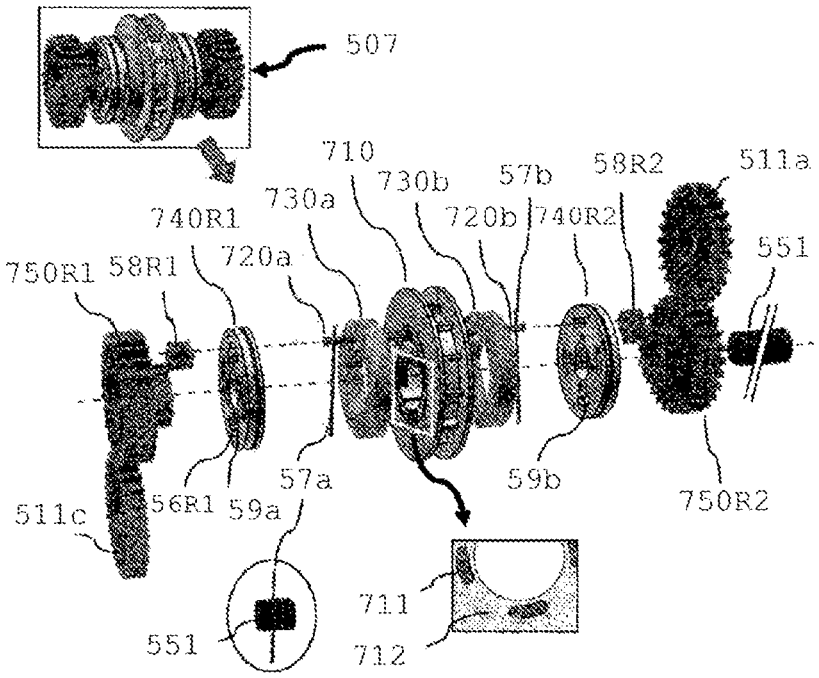
511d
554
512b
506
760d
240
58L1
740L1
750L1
400

[FIG. 14]


507
710
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511a
740R1
730a
720b
740R2
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750R1
58R1
720a
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59b
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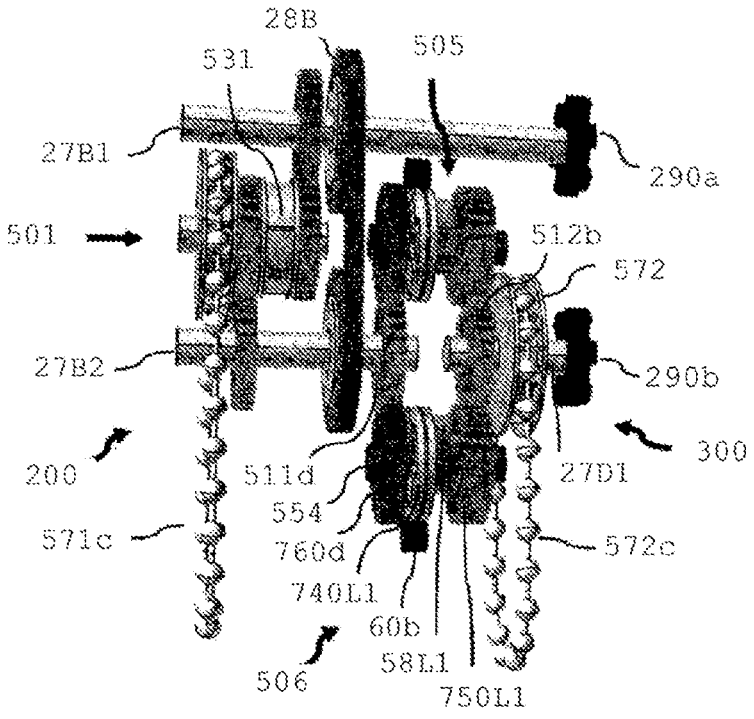
511c
56R1
59a
711
551
712

[FIG. 15]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290a
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512b
572
27B2
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511d
27D1
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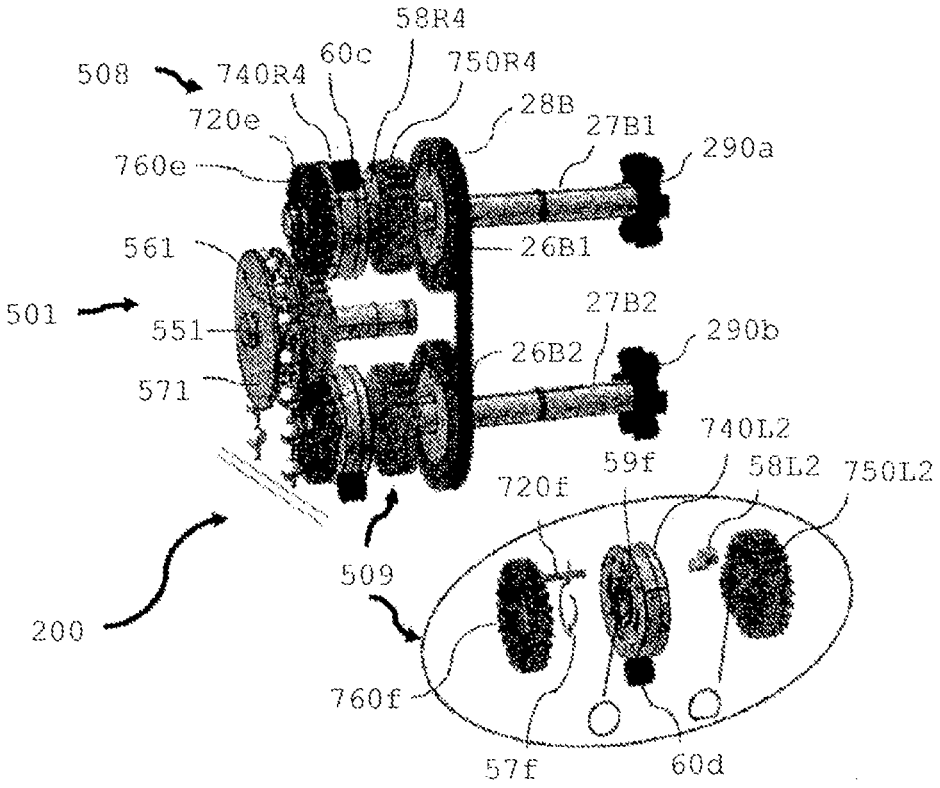
554
571c
760d
572c
740L1
60b
506
58L1
750L1

[FIG. 16]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26B2
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740L2
59f
58L2
750L2
720f
509
200
760f
57f
60d

DUAL BLIND CURTAIN PROVIDED WITH DIVIDED TAPE BELT

TECHNICAL FIELD

[0001] The present invention relates to a blind curtain, which is installed at an indoor window, an indoor wall surface, or the like and is used to obtain effects such as view blocking, light adjustment, heating, and indoor interior decoration. Specifically, the present invention relates to a dual blind curtain, in which when an operation of unfolding or winding up a curtain sheet is performed by winding opposite ends of a curtain sheet, which is formed by connecting a plurality of patterns to each other, around two winding shafts in a roll shape, respectively, and then rotating the two winding shafts, the two winding shafts can be freely adjusted to rotate facing each other or rotating along each other, thereby freely adjusting the curtain sheet wound around the winding shaft into a pattern that a consumer wants.

BACKGROUND ART

[0002] In general, blind curtains are used to block outdoor sunlight, protect indoor privacy, and improve indoor aesthetics, and the blind curtains are used in various forms, such as vertical blinds, horizontal blinds, overlapping-type blinds, or roll-type blinds.

[0003] Among them, in detail, the roll-type blind has a typical configuration in which a screen is wound on an outer periphery surface, a winding pipe provided at an end thereof with a handle is installed on an upper portion of an opening, the winding pipe is provided therein with a clutch and a rotation shaft, and an operative string is installed on one end of the winding pipe to interwork with the rotation shaft.

[0004] In the roll-type blind having the configuration as described above, the screen is wound or unfolded by selective pulling of the operative string, which is wound around the rotation shaft and divided into two strings to fall down.

[0005] The existing blind curtain satisfies the needs of consumers to some extent in terms of practicality, but does not satisfy the needs of various consumers in terms of interior decoration, which is another purpose of curtains. This is because in general the blind curtain uses a curtain sheet including one pattern, so that unless the entire blind curtain is replaced, there is no choice but to see the same curtain sheet all the time, resulting in a considerable problem in trying to obtain various effects required by customers, such as when trying to change interior decoration.

[0006] In order to satisfy the various needs for the interior decoration of customers as described above, the present inventor has solved the problem by installing a dual winding roll having two rotation shafts on a curtain model manufactured by connecting two or more curtain sheets in a height direction. Therefore, a device, in which two winding rolls are connected to each other to operate a plurality of curtain sheets, needs to have a unit that identically maintains an amount of the curtain unwinding from one shaft and an amount of the curtain winding around the other shaft due to a difference between the amounts of the curtains wound around the shafts, respectively. In this regard, there is a specialty that a rotation ratio at the center of the shaft is not always the same even when the amount of rotation at an outer diameter of the shaft wound with the curtain is the same. Such a specialty is special because high and low rotation ratios are present together, other than the rotation ratio of any one shaft is always high or low, and thus it is difficult to smoothly transmit power in any cases. This phenomenon has been solved using a curtain device, which is provided with a tape belt-type power transmission device and a rotation direction selection unit, devised by the present inventor.

[0007] In addition, two important functions of a configuration of a double blind of the present invention may be divided into a curtain circulation device and a curtain vertical opening/closing device, in which the curtain circulation device solves the problem through the tape belt and the rotation direction selection unit, and the curtain vertical opening/closing device solves the problem through a clutch and a push button assembly for operating the clutch, and solves the problem through a scissor clutch and a ball chain guide device instead of the push button assembly, such that it allows consumers to smoothly operate the dual blind by using a sense of use that is the same as that of existing blind products in the market.

[0008] However, in the previous invention, since the tape belt is driven through a sprocket provided in the scissor clutch by forming a hole in the tape belt in order to vertically open and close the curtain, it is not easy to secure durability of the tape belt, and as the tape belt is installed through a free fall to a height that is the same as an opening/closing height of the curtain, separate measures are thus required to secure safety related to the prevention of the tape belt from winding around a child's neck, so there is a need to provide a device for solving the above problem.

DISCLOSURE

Technical Problem

[0009] Therefore, the present invention has been devised to solve the problems, and an object of the present invention is to provide a dual blind curtain that may maximize practicality and durability by radically innovating functional limitations of existing technologies, also eliminating configurations of a rotary gear part, the push button assembly, the scissor clutch, and a lifting device part in the previous invention to construct simplicity and secure the stability by removing exposure of the tape belt, and facilitating the curtain circulation operation and the curtain vertical opening/closing operation by still satisfying a condition in which the dual blind curtain is operated by selectively connecting a rotation force to two shafts according to a rotation direction of a center sprocket, regardless of a difference occurred in a unique rotation amount required by each shaft of the connected rotations of two shafts as necessary and sufficient components.

Technical Solution

[0010] There is provided a dual blind curtain including: a body frame (**100**) including an upper plate (**110**) that forms an upper surface, side plates (**120**) that are provided on both sides of a bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**), and one or more partition plates (**130**) that are provided inside the bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**);

[0011] a B driving part (**200**) comprising: a first rotation direction selection part (**501**) including a center shaft (**551**)

that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center sprocket (**571**) that is installed on one side of the center shaft (**551**) and provided with a ball chain (**571***c*) to generate a rotation force, a second idle gear clutch (**521***b*) that is installed to idle around the center shaft (**551**), provided with a unidirectional connection port (**52-***b*) in one side thereof, and engaged with a main pulley primary gear (**511***a*), a first idle gear clutch (**521***a*) that idles on the center shaft (**551**), is provided with a unidirectional connection port (**52-***a*) in one side thereof to face the second idle gear clutch (**521***b*) at a predetermined interval, and engages with an auxiliary pulley primary gear (**511***c*), a center nut (**531**) that is positioned between the first and second idle gear clutches (**521***a* and **521***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), and a center bolt (**541**) that rotates in an inner portion of the center nut (**531**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54-***a* and **54-***b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**);

[0012] a main pulley shaft (**27**B**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley (**26**B**1**) that is constantly coupled to the main pulley shaft (**27**B**1**) and rotates; the main pulley primary gear (**511***a*) that engages with the second idle gear clutch (**521***b*) of the first rotation direction selection part (**501**) to transmit the rotation force to the main pulley (**26**B**1**); a main pulley secondary gear (**511***b*) that engages with a center gear (**563**) of a connection driving part (**400**) and transmits the rotation force transmitted to the main pulley (**26**B**1**); a winding rod connection port (**290***a*) that connects the rotation force generated in the main pulley (**26**B**1**) to a winding rod (**630***a*);

[0013] an auxiliary pulley (**26**B**2**) that is constantly coupled to an auxiliary pulley shaft (**27**B**2**) and rotates, in which the auxiliary pulley (**26**B**2**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); an auxiliary pulley primary gear (**511***c*) that engages with the first idle gear clutch (**521***a*) of the first rotation direction selection part (**501**) to transmit the rotation force to the auxiliary pulley (**26**B**2**); an auxiliary pulley secondary gear (**511***d*) that engages with a center gear (**564**) of the connection driving part (**400**) and transmits the rotation force transmitted to the auxiliary pulley (**26**B**2**); and

[0014] a tape belt B (**28**B) that connects the rotation force of the main pulley (**26**B**1**) or the rotation force of the auxiliary pulley (**26**B**2**) to each other and is connected to central axes of the main pulley (**26**B**1**) and the auxiliary pulley (**26**B**2**) to be wound in a stacked manner;

[0015] a D driving part (**300**) comprising: a second rotation direction selection part (**502**) including a center shaft (**552**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center sprocket (**572**) that is installed on one side of the center shaft (**552**) and provided with a ball chain (**572***c*) to generate a rotation force, a first idle gear clutch (**522***a*) that is installed to idle around the center shaft (**552**), provided with a unidirectional connection port (**52-***a*) in one side thereof, and engaged with a main pulley primary gear (**512***a*), a second idle gear clutch (**522***b*) that idles on the center shaft (**552**), is provided with a unidirectional connection port (**52-***b*) in one side thereof to face the first idle gear clutch (**522***a*) at a predetermined interval, and engages with an auxiliary pulley primary gear (**512***c*), a center nut (**532**) that is positioned between the first and second idle gear clutches (**522***a*, **522***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), and a center bolt (**542**) that rotates in an inner portion of the center nut (**532**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54-***a* and **54-***b*) on both sides of the bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**);

[0016] a main pulley shaft (**27**D**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley (**26**D**1**) that is constantly coupled to the main pulley shaft (**27**D**1**) and rotates; a main pulley primary gear (**512***a*) that engages with the first idle gear clutch (**522***a*) of the second rotation direction selection part (**502**) to transmit the rotation force to the main pulley (**26**D**1**); a main pulley secondary gear (**512***b*) that engages with a connection gear set (**240**) of the connection driving part (**400**) and transmits the rotation force transmitted to the main pulley (**26**D**1**); a winding rod connection port (**290***b*) that connects the rotation force generated in the main pulley (**27**D**1**) to a winding rod (**630***b*);

[0017] an auxiliary pulley (**26**D**2**) that is constantly coupled to an auxiliary pulley shaft (**27**D**2**) and rotates, in which the auxiliary pulley (**26**D**2**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); an auxiliary pulley primary gear (**512***c*) that engages with the second idle gear clutch (**522***b*) of the second rotation direction selection part (**502**) to transmit the rotation force to the auxiliary pulley (**26**D**2**); an auxiliary pulley secondary gear (**512***d*) that engages with a connection gear set (**230**) of the connection driving part (**400**) and transmits the rotation force transmitted to the auxiliary pulley (**26**D**2**); and

[0018] a tape belt D (**28**D) that connects the rotation force of the main pulley (**26**D**1**) or the rotation force of the auxiliary pulley (**26**D**2**) to each other and is connected to central axes of the main pulley (**26**D**1**) and the auxiliary pulley (**26**D**2**) to be wound in a stacked manner;

[0019] the connection driving part (**400**) comprising: a third rotation direction selection part (**503**) including a center shaft (**553**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an

outer circumferential surface thereof with a locking protrusion (**55-1**), a center gear (**563**) that is installed on one side of the center shaft (**553**) and engaged with the main pulley secondary gear (**511***b*) of the B driving part (**200**) to receive the rotation force, a first idle gear clutch (**523***a*) that is installed to idle around the center shaft (**553**) and provided with a unidirectional connection port (**52**-*a*) in one side thereof, a second idle gear clutch (**523***b*) that idles on the center shaft (**553**) and is provided with a unidirectional connection port (**52**-*b*) in one side thereof to face the first idle gear clutch (**523***a*) at a predetermined interval, a center nut (**533**) that is positioned between the first and second idle gear clutches (**523***a* and **523***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), a center bolt (**543**) that rotates in an inner portion of the center nut (**533**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54**-*a* and **54**-*b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**), and the connection gear set (**230**) that engages with the auxiliary pulley secondary gear (**512***d*) and transmits the rotation force of the second idle gear clutch (**523***b*) to the auxiliary pulley secondary gear (**512***d*); and

[0020] the fourth rotation direction selection part (**504**) including a center shaft (**554**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center gear (**564**) that is installed on one side of the center shaft (**554**) and engaged with the auxiliary pulley secondary gear (**511***d*) of the B driving part (**200**) to receive the rotation force, a first idle gear clutch (**524***a*) that is installed to idle around the center shaft (**554**) and provided with a unidirectional connection port (**52**-*a*) in one side thereof, a second idle gear clutch (**524***b*) that idles on the center shaft (**553**) and is provided with a unidirectional connection port (**52**-*b*) in one side thereof to face the first idle gear clutch (**524***a*) at a predetermined interval, a center nut (**533**) that is positioned between the first and second idle gear clutches (**524***a* and **524***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), a center bolt (**543**) that rotates in an inner portion of the center nut (**533**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54**-*a* and **54**-*b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**), and the connection gear set (**240**) that engages with the main pulley secondary gear (**512***b*) and transmits the rotation force of the first idle gear clutch (**524***a*) to the main pulley secondary gear (**512***b*); and

[0021] a curtain part (**600**) installed to idle on the side plate (**120**) and the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and including a curtain sheet (**610**) that is made by connecting one or more various pattern sheets and has one end wound around the winding rod (**630***a*) connected by the winding rod connection ports (**290***a* and **290***b*) and an opposite end wound around another winding rod (**630***b*) by the rotation force of the main pulley shafts (**27**B**1** and **27**D**1**), and a weight bar (**620**) that is formed on a center of the curtain sheet (**610**) wound around opposite ends thereof.

Advantageous Effects

[0022] When the double blind curtain according to the present invention is provided, effects of creating various types of curtain can be expected using one product of the present invention, such as creating various interior atmospheres of a curtain sheet on which various photos are printed, selecting a curtain sheet having different patterns and a configuration of mesh sheet-stripe sheet-mesh sheet-translucent sheet-mesh sheet-blackout sheet according to an amount of sunlight and selected for the purpose of covering upper and lower ranges of view and day and night, and using a curtain sheet on which a message is recorded for learning and publicity through an automatic circulation operation.

[0023] Further, a danger of winding around the child's neck can be removed by constructing simplicity of the configuration and removing the exposure of the tape belt, and simultaneously, a function of facilitating the curtain circulation operation and the curtain vertical opening/closing operation by satisfying a condition in which the dual blind curtain is operated by selectively connecting a rotation force to two shafts according to the rotation direction of the center sprocket, regardless of a difference occurred in a unique rotation amount required by each shaft of the connected rotations of two shafts as necessary and sufficient components of the device.

DESCRIPTION OF DRAWINGS

[0024] FIG. **1** is an external perspective view.

[0025] FIG. **2** is a perspective view representing a body frame and main driving parts.

[0026] FIG. **3** is an exploded perspective view of a rotation direction selection part.

[0027] FIG. **4** is a perspective view representing a connection between a tape belt and a shaft.

[0028] FIG. **5** is a perspective view of driving parts which are arranged in a plane.

[0029] FIG. **6** is a schematic view of FIG. **5**.

[0030] FIG. **7** is a schematic view of a process for transmitting a rotation force of curtain sheet downward circulation drive.

[0031] FIG. **8** is a schematic view of a process for transmitting a rotation force of curtain sheet upward circulation drive.

[0032] FIG. **9** is a schematic view of a process for transmitting a rotation force of curtain sheet pulling-down drive.

[0033] FIG. **10** is a schematic view of a process for transmitting a rotation force of curtain sheet pulling-up drive.

[0034] FIG. **11** is a view of a rotation direction configuration of a unidirectional complex clutch.

[0035] FIG. **12** is an explanatory view of a principle of operations of both surfaces of a pawl guide disc and a unidirectional complex clutch L.

[0036] FIG. **13** is a perspective view of a connection driving part including a unidirectional complex clutch.

[0037] FIG. **14** is a perspective view for explaining a bidirectional complex clutch.

[0038] FIG. **15** is a view in which unidirectional complex clutches R and L are simultaneously connected to one shaft of the connection driving part.

[0039] FIG. **16** is a view in which the unidirectional complex clutches R and L are applied to the rotation direction selection part.

BEST MODE

[0040] Hereinafter, preferred embodiments will be described in detail with respect to the accompanying drawings.

[0041] As shown in the accompanying drawings (see FIGS. **1** and **2**) of the present invention, a dual blind curtain is provided with two winding rods **630***a* and **630***b* inside a body frame **100** and formed with a curtain sheet **610** wound around the winding rods **630***a* and **630***b*, a B driving part **200**, a D driving part **300**, and a connection driving part **400** for changing a rotation direction of the winding rods **630***a* and **630***b*.

[0042] The body frame **100** includes an upper plate **110** that forms an upper surface, side plates **120** that are provided on both sides of a bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**, and one or more partition plates **130** that are provided inside the bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**.

[0043] As shown in the accompanying drawing (see FIG. **2**), a curtain part **600** provided inside the body frame **100** directly connects rotation forces of a main pulley shaft **27**B**1** of the B driving part **200** and a main pulley shaft **27**D**1** of the D driving part **300** to the winding rods **630***a* and **630***b* by winding rod connection ports **290***a* and **290***b* provided on an end thereof, or transmits the rotation forces by a method of adding another tool for transmitting power such as a chain, such that the curtain sheet **610** is vertically or circulatively wound.

[0044] The B driving part **200**, the D driving part **300**, and the connection driving part **400** are driven by a rotation force of center sprockets **571** and **572** rotated by ball chains **571***c* and **572***c*, which are manually operated by a user, the same operation principle applies to the B driving part **200** including a first rotation direction selection part **501** and a tape belt B **28**B, the D driving part **300** including a second rotation direction selection part **502** and a tape belt D **28**D, and the connection driving part **400** including a third rotation direction selection part **503** and a fourth rotation direction selection part **504**, and the B driving part **200**, the D driving part **300**, and the connection driving part **400** will be collectively referred to as a rotation direction selection part **500** and first described with reference to the drawing (see FIG. **3**).

[0045] As shown in the drawing (see FIG. **3**), a center shaft **550** is installed to rotate in a shaft hole **150** formed in the partition plate **130**, and a center gear **560** installed on one end of the center shaft **550** engages with another power to apply the rotation force to the center shaft **550** and may be installed in conjunction with the center sprocket **570** applied with the rotation force by the ball chain **570***c*. Meanwhile, the rotation force of the center gear **560** is transmitted to a center bolt **540** together with a locking protrusion **55-1** of the center shaft **550**, in which the center gear **560** is installed to idle on opposite ends of the locking protrusion **55-1** and positioned between first and second idle gear clutches **520***a* and **520***b* provided with unidirectional connection ports **52**-*a* and **52**-*b* on one side thereof facing each other, and in a rotation method of the center bolt **540**, a bolt thread **54-3** provided on an outer circumferential surface of a bolt body **54-1** is rotated forward and rearward along a guide direction of a nut groove **53-1** provided in an inner surface of the center nut **530** by the rotation force of the locking protrusion **55-1** fitted into a sliding hole **54-2** formed at the center of the bolt body **54-1**, and thus the bolt body **54-1** rotates while sliding forward and rearward the locking protrusion **55-1**. In this case, unidirectional connection ports **54**-*a* and **54**-*b* are provided on both sides of the bolt body **54-1** and connected to one of the corresponding unidirectional connection ports **52**-*a* and **52**-*b* provided in first and second idle gear clutches **520***a* and **520***b*. After one of the first and second idle gear clutches **520***a* and **520***b* are connected to the center bolt **540** slid in any one direction of the locking protrusion **55-1**, the center bolt **540** does not move further in a direction in which it moves inside the center nut **530**, and a body of the center nut **530** is installed to idle in the shaft hole **150** of the partition plate **130** to maintain the connected rotation direction until the rotation direction of the center shaft **550** is switched to an opposite direction by the center gear **560**. A rotation force generated in the first and second idle gear clutches **520***a* and **520***b* is transmitted to the main pulley primary gear **510***a* and an auxiliary pulley primary gear **510***c* through a connection path.

[0046] With the same configuration and principle as described in the rotation direction selection part **500**, the first rotation direction selection part **501**, the second rotation direction selection part **502**, the third rotation direction selection part **503**, and the fourth rotation direction selection part **504** selectively transmit the rotation force and the rotation direction of the center sprockets **571** and **572** or the center gears **563** and **564** to the first idle gear clutches **521***a*, **522***a*, **523***a*, and **524***a* and second idle gear clutches **521***b*, **522***b*, **523***b*, and **524***b*.

[0047] The following describes a connection between the tape belt B **28**B, the tape belt D **28**D, and a shaft with respect to the drawing (see FIG. **4**).

[0048] As described in the configuration collectively referred to as the rotation direction selection part **500**, when the rotation direction of the center sprocket **571** is determined by the ball chain **571***c* pulled in any one direction of the first rotation direction selection part **501**, the rotation force is selectively transmitted to either a main pulley primary gear **511***a* provided on the main pulley shaft **27**B**1** engaged with the first idle gear clutch **521***a* or an auxiliary pulley primary gear **511***c* provided on the auxiliary pulley shaft **27**B**2** engaged with the second idle gear clutch **521***b* accordingly, and in this case, a main pulley **26**B**1** rotating together with the main pulley primary gear **511***a* and an auxiliary pulley **26**B**2** rotating together with the auxiliary pulley primary gear **511***c* are connected to each other by the tape belt B **28**B. In such a configuration, the tape belt B **28**B is stacked on and wound around the main pulley **26**B**1** and the auxiliary pulley **26**B**2** according to the rotation of the shaft, like the curtain sheet **610** wound around the winding rods **630***a* and **630***b*, thereby serving to transmit the rotation force of one of the two shafts **27**B**1** and **27**B**2** to each other.

[0049] Meanwhile, when the rotation direction of a center sprocket **572** is determined by a ball chain **572***c* pulled in any one direction from the second rotation direction selection part **502** in the same manner as the configuration and connection structure of the first rotation direction selection part **501** and the tape belt B **28**B, the rotation force is

selectively transmitted to either the main pulley primary gear **512***a* provided on the main pulley shaft **27**D**1** engaged with the first idle gear clutch **522***a* or the auxiliary pulley primary gear **512***c* provided on the auxiliary pulley shaft **27**D**2** engaged with the second idle gear clutch **522***b* accordingly, and in this case, a main pulley **26**D**1** rotating together with the main pulley primary gear **512***a* and an auxiliary pulley **26**D**2** rotating together with the auxiliary pulley primary gear **512***c* are connected to each other by the tape belt D **28**D. In such a configuration, the tape belt D **28**D is stacked on and wound around the main pulley **26**D**1** and the auxiliary pulley **26**D**2** according to the rotation of the shaft, like the curtain sheet **610** wound around the winding rods **630***a* and **630***b*, thereby serving to transmit the rotation force of one of the two shafts **27**D**1** and **27**D**2** to each other.

[0050] Describing the tape belts **28**B and **28**D in more detail, the tape belt B **28**B and the tape belt D **28**D have the same shape and function, but a winding direction of the curtain sheet **610** according to a direction of the shaft rotation may be manipulated in an intended circulation direction or a vertical direction when an installation direction is formed to have a winding shape symmetrical by 180 degrees, and the number of winding rotations of the tape belts **28**B and **28**D wound around the main pulleys **26**B**1** and **26**D**1**, respectively needs to have the number of winding rotations the same as that of the curtain sheet **610** wound around the winding rods **630***a* and **630***b* corresponding to each other. In addition, as shown in the drawing (see FIG. **6**), the main pulley **26**B**1** of the B driving part **200** and the auxiliary pulley **26**D**2** of the D driving part **300** are connected to each other, and the main pulley **26**D**1** of the D driving part **300** and the auxiliary pulley **26**B**2** of the B driving part **200** are respectively connected. In this case, outer diameters of the tape belts **28**B and **28**D wound around the corresponding pulleys **26**B**1**:**26**D**2** and **26**D**1**:**26**B**2** are to be maintained at the same size.

[0051] This is because the rotation force is selectively connected to two shafts according to the rotation direction of the center sprocket regardless of a difference occurred in a unique rotation amount required by each shaft in the rotation of connected two shafts, which is the main principle of the present invention, thus satisfying a condition of manipulating the operation of the curtain as intended.

[0052] The following is a description that the B driving part **200** and the D driving part **300** are connected to the rotation force by the connection driving part **400**. First, as described in the rotation direction selection part **500** and as shown in the drawing (see FIG. **6**), in the third rotation direction selection part **503**, a center gear **563** that replaces the center sprocket **570** is provided on one end of a center shaft **553** installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and is connected with the rotation force through a connection gear set **230** installed to engage with the second idle gear clutch **523***b* in order to engage and receive the rotation force of the main pulley secondary gear **511***b* of the B driving part **200** to engage and transmit the rotation force to the auxiliary pulley secondary gear **512***d* of the D driving part **300**.

[0053] Next, as described in the rotation direction selection part **500** and as shown in the drawing (see FIG. **6**), in the fourth rotation direction selection part **504**, a center gear **564** that replaces the center sprocket **570** is provided on one end of a center shaft **554** installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and is connected with the rotation force through a connection gear set **240** installed to engage with the first idle gear clutch **524***a* in order to engage and receive the rotation force of the auxiliary pulley secondary gear **511***d* of the B driving part **200** to engage and transmit the rotation force to the main pulley secondary gear **512***b* of the D driving part **300**.

[0054] In this case, the configuration of the connection gear sets **230** and **240** serves to transmit the same rotation ratio and rotation direction as a component that is provided with a pair of connection gears **233***a*:**233***b*, and **244***a*:**244***b* on opposite ends of connection gear shafts **233***c* and **244***c* installed to rotate on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and engages with each gear.

[0055] Hereinabove, in order to explain an operation procedure for forming each configuration provided in the present invention, FIG. **5** is a schematic view of a driving part in a planar arranged manner and FIG. **6** is a schematic view of FIG. **5**, in which FIG. **6** is a basic view for systematically understanding a connection state of each configuration and a process for transmitting the rotation direction and the driving force of each step by indicating the rotation direction with an arrow. However, this is a drawing clarified to faithfully represent only the process for transmitting the rotation direction by omitting the center nuts **531**, **532**, **533**, and **534** of the rotation direction selection parts, respectively.

[0056] FIG. **7** is a schematic view of a process for transmitting a rotation force of curtain sheet downward circulation device, FIG. **8** is a schematic view of a process for transmitting a rotation force of curtain sheet upward circulation drive, FIG. **9** is a schematic view of a process for transmitting a rotation force of curtain sheet pulling-down drive, and FIG. **10** is a schematic view of a process for transmitting a rotation force of curtain sheet pulling-up drive.

[0057] Each operation method may be easily understood by applying the configuration and operation principle described in the rotation direction selection part **500** along a direction of the arrow, the general power transmission theory, the rotation principle of the engaging gear, the rotation principle of the rotation shaft connected by the belt, and the like, and thus additional description thereof will be omitted.

[0058] Meanwhile, after the entire curtain sheet **610** moves to the winding rod **630***b* through the downward circulation drive (see FIG. **7**) and wounded, when pulling-up drive (see FIG. **10**) of the curtain sheet **610** is performed, the tape belt D **28**D requires a length that allows the auxiliary pulley **26**D**2** to additionally rotate. In this regard, the auxiliary pulley **26**D**2** of the D driving part **300** has a diameter smaller than that of the main pulley **26**D**1**. In the drawing (see FIG. **10**), in case of the pulling-up drive of the curtain sheet **610**, the number of rotations of the curtain sheet **610** wound around the winding rod **630***b* is obtained, in which the curtain sheet **610** has a length the same as an installation height h of the blind that is wound around the winding rod **630***b* corresponding to the main pulley **26**D**1**, and the curtain sheet **610** needs to be small as a diameter formed by a

winding length of the tape belt **28**D having the number of rotations the same as that of the curtain sheet **610**. The length of the tape belt **28**D that is additionally wound around the auxiliary pulley **26**D**2** having the diameter as described above is secured, such that the pulling-up drive of the curtain sheet **610** is possible in all cases.

MODE FOR INVENTION

[0059] The following describes another embodiment of the present invention, and

[0060] relates to a unidirectional complex clutch including a latch wheel which is a tool for transmitting unidirectional power and a pawl guide groove which guides a movement of the pawl.

[0061] As shown in the accompanying drawings (see FIGS. **11**, **12**, and **13**),

[0062] the connection driving part **400** includes: a unidirectional complex clutch R **505** including the center shaft **553** that is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** formed with the plurality of shaft holes **150**, a latch wheel idle gear **750**R**3** that is provided with a latch wheel, which idles on one end of the center shaft **553** to transmit clockwise rotation to one side thereof, and transmits the rotation force to the auxiliary pulley secondary gear **512***d* through the engaged connection gear set **230**, a pawl shaft fixing idle gear **760***c* that is positioned on an opposite end of the center shaft **553** to receive the rotation force from the main pulley secondary gear **511***b*, and installed with one or more pawl shafts **720***c* each having a pawl **58**R**3** disposed thereon, in which the pawl **58**R**3** engages with the latch wheel idle gear **750**R**3** in a predetermined condition, a pawl shaft guide hole **59***c* that is positioned between the latch wheel idle gear **750**R**3** and the pawl shaft fixing idle gear **760***c* and through which the pawl shaft **720***c* passes, a pawl guide idle disc **740**R**3** that is provided with one or more pawl guide grooves **56**R**3** which make contact with the pawl shaft guide hole **59***c* to control a movement of the pawl **58**R**3** in a predetermined condition, a return bar spring **57***c* that is fixed to the center shaft **553** to return the pawl shaft **720***c* to a predetermined position of the pawl shaft guide hole **59***c* when there is no rotation load on the pawl shaft **720***c*, and a deceleration roller **60***a* that generates a load based on a rotation speed of the pawl guide idle disc **740**R**3**; and

[0063] a unidirectional complex clutch L **506** including the center shaft **554** that is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** formed with the plurality of shaft holes **150**, a latch wheel idle gear **750**L**1** that is provided with a latch wheel, which idles on one end of the center shaft **554** to transmit counterclockwise rotation to one side thereof, and transmits the rotation force to the main pulley secondary gear **512***b* through the engaged connection gear set **240**, a pawl shaft fixing idle gear **760***d* that is positioned on an opposite end of the center shaft **554** to receive the rotation force from the auxiliary pulley secondary gear **511***d*, and installed with one or more pawl shafts **720***d* each having a pawl **58**L**1** disposed thereon, in which the pawl **58**L**1** engages with the latch wheel idle gear **750**L**1** in a predetermined condition, a pawl shaft guide hole **59***d* that is positioned between the latch wheel idle gear **750**L**1** and the pawl shaft fixing idle gear **760***d* and through which the pawl shaft **720***d* passes, a pawl guide idle disc **740**L**1** that is provided with one or more pawl guide grooves **56**L**1** which make contact with the pawl shaft guide hole **59***d* to control a movement of the pawl **58**L**1** in a predetermined condition, a return bar spring **57***d* that is fixed to the center shaft **554** to return the pawl shaft **720***d* to a predetermined position of the pawl shaft guide hole **59***d* when there is no rotation load on the pawl shaft **720***d*, and a deceleration roller **60***b* that generates a load based on a rotation speed of the pawl guide idle disc **740**L**1**.

[0064] Since the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** are designed to operate in opposite directions to each other, a type of the configuration and the principle of the operation are the same, a description of the unidirectional complex clutch L **506** will be described first through the accompanying drawing (see FIG. **12**). In the B driving part **200**, a counterclockwise rotation force engaging with the auxiliary pulley secondary gear **511***d* and transmitted to the pawl shaft fixing idle gear **760***d* is connected to the latch wheel provided in the latch wheel idle gear **750**L**1** through the pawl shaft **720***d* and the pawl **58**L**1**, the pawl shaft **720***d* rotating while the pawl shaft fixing idle gear **760***d* rotates moves by a predetermined distance inside the pawl shaft guide hole **59***d* provided in the pawl guide disc **740**L**1**. In this case, the return bar spring **57***d* installed adjacent to the pawl shaft guide hole **59***d* is compressed and moved using a difference in the rotation speed between the pawl guide disc **740**L**1** and the pawl shaft fixing idle gear **760***d*, which are rotationally decelerated by the deceleration roller **60***b* that generates the load based on the rotation speed of the pawl guide disc **740**L**1**, and when the rotation load of the pawl shaft fixing idle gear **760***d* disappears, a pawl shaft **720***d* is returned to its original position by a force of the return spring **57***d*. Through movements R**1**, R**2**, and R**3** of the pawl shaft **720***d*, the pawl **58**L**1** disposed at the end of the pawl shaft **720***d* is moved. Meanwhile, one end of the pawl **58**L**1** is inserted into the pawl guide groove **56**L**1** in contact with the pawl shaft guide hole **59***d* provided in the pawl guide disc **740**L**1**, and guided in a direction of the pawl guide groove **56**L**1** designed toward the latch wheel provided on the latch wheel idle gear **750**L**1** according to the movements R**1**, R**2**, and R**3** of the pawl shaft **720***d*. As a result, the pawl **58**L**1** is coupled to or separated from the latch wheel provided on the latch wheel idle gear **750**L**1** according to the movements R**1**, R**2**, and R**3** of the pawl shaft **720***d*, and returned to its original position. Since the same operation process applies to the unidirectional complex clutch R **505** in which only the rotation direction is changed, an additional description thereof will be omitted.

[0065] Meanwhile, the rotation force generated in the D driving part **300** reaches the auxiliary pulley secondary gear **512***d* or the auxiliary pulley secondary gear **512***b* and is transmitted to the latch wheel idle gears **750**R**3** and **750**L**1** engaged by the connection gear sets **230** and **240**. However, currently, there is no rotation load on the pawl shaft fixing idle gears **760***c* and **760***d*, and thus the pawl shaft **720***d* is returned to its original position due to the force of the return rod spring **57***d*, so that the positions of the pawls **58**R**3** and **58**L**1** are in a state in which the rotation force is cut off. Therefore, the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** operate in the same manner as the third rotation direction selection part **503** and the fourth rotation direction selection part **504**.

[0066] The following describes another embodiment of the present invention, and

[0067] relates to a bidirectional complex clutch **507** that performs the same operation as the rotation direction selection part **500** having a unidirectional complex clutch in both directions.

[0068] As shown in the accompanying drawing (see FIG. **14**),

[0069] a bidirectional complex clutch **507** includes: an idle sprocket **710** that has one or more pawl shaft control grooves **711** and **712** provided on both sides thereof at a position spaced apart from each other in an opposite direction around the center shaft **551** installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**; latch wheel idle gears **750**R**1** and **750**R**2** that are installed to face a same component based on the idle sprocket **710**, are first provided with latch wheels which idle on opposite ends of the center shaft **551** to transmit unidirectional rotation to one side thereof, and transmits a generated rotation force to another shaft; pawl shaft fixing idle gears **730***a* and **730***b* that are positioned on both sides of the idle sprocket **710**, and installed with one or more pawl shafts **720***a* and **720***b* having pawls **58**R**1** and **58**R**2** disposed thereon, in which the pawls **58**R**1** and **58**R**2** are inserted into the pawl shaft control grooves **711** and **712** to receive the rotation force and engage with the latch wheel idle gears **750**R**1** and **750**R**2** in a predetermined condition; pawl shaft guide holes **59***a* and **59***b* that are positioned between the latch wheel idle gears **750**R**1** and **750**R**2** and the pawl shaft fixing idle gears **730***a* and **730***b*, respectively and through which the pawl shafts **720***a* and **720***b* pass; pawl guide idle discs **740**R**1** and **740**R**2** that are provided with one or more pawl guide grooves **56**R**1** and **56**R**2** which make contact with the pawl shaft guide holes **59***a* and **59***b* to control a movement of the pawls **58**R**1** and **58**R**2** in a predetermined condition; and return bar springs **57***a* and **57***b* that are installed on the center shaft **551** to return the pawl shafts **720***a* and **720***b* to a predetermined position of the pawl shaft guide holes **59***a* and **59***b* when there is no rotation load on the pawl shafts **720***a* and **720***b*.

[0070] Although the bidirectional complex clutch **507** has the same principle (see FIG. **12**) as the unidirectional complex clutches **505** and **506** according to the rotation direction, there is a slight difference in the configuration in which the bidirectional complex clutch **507** operates on the same shaft, whereas the unidirectional complex clutches **505** and **506** operate on each shaft, and this will be described through the accompanying drawing (see FIG. **14**).

[0071] In the idle sprocket **710** located at the center of the bidirectional complex clutch **507**, one or more pawl shaft control grooves **711** and **712** are provided in both sides thereof at a position spaced apart from each other in an opposite direction by a distance between the movements R**1**, R**2**, and R**3** of the pawl shaft. This means that when the idle sprocket **710** rotates in one direction (first, on the assumption of counterclockwise rotation: affect on the configuration on the left side in the accompanying drawing (see FIG. **14**)), one end of the pawl shaft **720***a* in the corresponding direction fixed to the pawl shaft idle disc **730***a* is inserted into the pawl shaft control groove **711**, and thus the idle sprocket **710** and the pawl shaft idle disc **730***a* are activated together, and in this case, while the pawl **58**R**1** arranged on the other end of the pawl shaft **720***a* moves to a point where it is coupled to the latch wheel of the latch wheel idle gear **750**R**1**, the opposite pawl shaft **720***b* does not need to be activated. That is, the opposite pawl shaft **720***b* is not activated by arranging a gap corresponding to a movement distance of the pawl shaft **720***a* in the opposite pawl shaft control groove **712**. This is because when the opposite pawl shaft **720***b* is activated before the pawl **58**R**1** arranged on the end of the pawl shaft **720***a* reaches the point where it is coupled to the latch wheel of the latch wheel idle gear **750**R**1**, the pawl guide disc **740**R**2** fixed to the center shaft **551** rotates, thereby preventing first intended engagement of the pawl **58**R**1** and the latch wheel of the latch wheel idle gear **750**R**1**. Since the same operation process is applied even when it is assumed that the idle sprocket **710** rotates in a clockwise rotation (affect on the configuration on the right side of the accompanying drawing (see FIG. **14**)), a description thereof will be omitted.

[0072] As described above, the idle sprocket **710** rotates in a state in which after the rotation force of the idle sprocket **710** located at the center of the bidirectional complex clutch **507** is connected to one of the latch wheel idle gears **750**R**1** and **750**R**2**, the rotation force on the other side is not coupled, such that the desired purpose of the operation is achieved and the same operation as the rotation direction selection parts **501** and **502** is performed.

[0073] The following describes another embodiment of the present invention, and

[0074] the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** are simultaneously connected to the auxiliary pulley secondary gear **511***d* of the B driving part **200** and the main pulley secondary gear **512***b* of the D driving part **300**, such that it is possible to shorten a power transmission path and simplify the configuration.

[0075] As shown in the accompanying drawing (see FIG. **15**),

[0076] there is provided a dual blind curtain including: a body frame **100** including an upper plate **110** that forms an upper surface, side plates **120** that are provided on both sides of a bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**, and one or more partition plates **130** that are provided inside the bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**;

[0077] a B driving part **200** comprising: a first rotation direction selection part **501** including a center shaft **551** that is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and provided on an outer circumferential surface thereof with a locking protrusion **55-1**, a center sprocket **571** that is installed on one side of the center shaft **551** and provided with a ball chain **571***c* to generate a rotation force, a second idle gear clutch **521***b* that is installed to idle around the center shaft **551**, provided with a unidirectional connection port **52-***b* in one side thereof, and engaged with a main pulley primary gear **511***a*, a first idle gear clutch **521***a* that idles on the center shaft **551**, is provided with a unidirectional connection port **52-***a* in one side thereof to face the second idle gear clutch **521***b* at a predetermined interval, and engages with an auxiliary pulley primary gear **511***c*, a center nut **531** that is positioned between the first and second idle gear clutches **521***a* and **521***b*, provided on an inner surface of a body thereof with a nut groove **53-1**, and installed so that the body idles at the shaft hole **150** of the partition plate

130, and a center bolt **541** that rotates in an inner portion of the center nut **531**, has a sliding hole **54-2** formed in a center of a body thereof to move left and right of the locking protrusion **55-1**, is provided with unidirectional connection ports **54-***a* and **54-***b* on both sides of a bolt body **54-1**, respectively, and is provided with a bolt thread **54-3** on an outer circumferential surface of the bolt body **54-1**;

[0078] a main pulley shaft **27**B**1** that is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of holes **150**; a main pulley **26**B**1** that is constantly coupled to the main pulley shaft **27**B**1** and rotates; the main pulley primary gear **511***a* that engages with the second idle gear clutch **521***b* of the first rotation direction selection part **501** to transmit the rotation force to the main pulley **26**B**1**; a winding rod connection port **290***a* that connects the rotation force generated in the main pulley **26**B**1** to a winding rod **630***a;*

[0079] an auxiliary pulley **26**B**2** that is constantly coupled to an auxiliary pulley shaft **27**B**2** and rotates, in which the auxiliary pulley **26**B**2** is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**; an auxiliary pulley primary gear **511***c* that engages with the first idle gear clutch **521***a* of the first rotation direction selection part **501** to transmit the rotation force to the auxiliary pulley **26**B**2**; an auxiliary pulley secondary gear **511***d* that engages with pawl shaft fixing idle gears **760***c*, **760***d* of a connection driving part **400** and transmits the rotation force transmitted to the auxiliary pulley **26**B**2**; and

[0080] a tape belt B **28**B that connects the rotation force of the main pulley **26**B**1** or the rotation force of the auxiliary pulley **26**B**2** to each other and is connected to central axes of the main pulley **26**B**1** and the auxiliary pulley **26**B**2** to be wound in a stacked manner;

[0081] a D driving part **300** including: a main pulley shaft **27**D**1** that is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of holes **150**; a main pulley secondary gear **512***b* that is installed on one end of the main pulley shaft **27**D**1** and connected to latch wheel idle gears **750**R**3** and **750**L**1** of the connection driving part **400**; a center sprocket **572** that is installed on the main pulley shaft **27**D**1** and provided with a ball chain **572***c* to generate the rotation force; and a winding rod connection port **290***b* connects the rotation force generated in the main pulley shaft **27**D**1** to a winding rod **630***b;*

[0082] the connection driving part **400** comprising: a unidirectional complex clutch R **505** including a latch wheel idle gear **750**R**3** that is provided with a latch wheel, which idles on one end of a center shaft **553** to transmit clockwise rotation to one side thereof, and engaged with the main pulley secondary gear **512***b* to transmit the rotation force, in which the center shaft **553** is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, a pawl shaft fixing idle gear **760***c* that is positioned on an opposite end of the center shaft **553** to receive the rotation force from the auxiliary pulley secondary gear **511***d*, and installed with one or more pawl shafts **720***c* each having a pawl **58**R**3** disposed thereon, in which the pawl **58**R**3** engages with the latch wheel idle gear **750**R**3** in a predetermined condition, a pawl shaft guide hole **59***c* that is positioned between the latch wheel idle gear **750**R**3** and the pawl shaft fixing idle gear **760***c* and through which the pawl shaft **720***c* passes, a pawl guide idle disc **740**R**3** that is provided with one or more pawl guide grooves **56**R**3** which make contact with the pawl shaft guide hole **59***c* to control a movement of the pawl **58**R**3** in a predetermined condition, a return bar spring **57***c* that is fixed to the center shaft **553** to return the pawl shaft **720***c* to a predetermined position of the pawl shaft guide hole **59***c* when there is no rotation load on the pawl shaft **720***c*, and a deceleration roller **60***a* that generates a load based on a rotation speed of the pawl guide idle disc **740**R**3**; and

[0083] the unidirectional complex clutch L **506** including a latch wheel idle gear **750**L**1** that is provided with a latch wheel, which idles on one end of a center shaft **554** to transmit counterclockwise rotation to one side thereof, and engaged with the main pulley secondary gear **512***b* to transmit the rotation force, in which the center shaft **554** is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, a pawl shaft fixing idle gear **760***d* that is positioned on an opposite end of the center shaft **554** to receive the rotation force from the auxiliary pulley secondary gear **511***d*, and installed with one or more pawl shafts **720***d* each having a pawl **58**L**1** disposed thereon, in which the pawl **58**L**1** engages with the latch wheel idle gear **750**L**1** in a predetermined condition, a pawl shaft guide hole **59***d* that is positioned between the latch wheel idle gear **750**L**1** and the pawl shaft fixing idle gear **760***d* and through which the pawl shaft **720***d* passes, a pawl guide idle disc **740**L**1** that is provided with one or more pawl guide grooves **56**L**1** which make contact with the pawl shaft guide hole **59***d* to control a movement of the pawl **58**L**1** in a predetermined condition, a return bar spring **57***d* that is fixed to the center shaft **554** to return the pawl shaft **720***d* to a predetermined position of the pawl shaft guide hole **59***d* when there is no rotation load on the pawl shaft **720***d*, and a deceleration roller **60***b* that generates a load based on a rotation speed of the pawl guide idle disc **740**L**1**; and

[0084] a curtain part **600** installed to idle on the side plate **120** and the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and including a curtain sheet **610** that is made by connecting one or more various pattern sheets and has one end wound around the winding rod **630***a* connected by the winding rod connection ports **290***a* and **290***b* and an opposite end wound around another winding rod **630***b* by the rotation force of the main pulley shafts **27**B**1** and **27**D**1**, and a weight bar **620** that is formed on a center of the curtain sheet **610** wound around opposite ends thereof.

[0085] In the above configuration, as briefly described above, the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** are simultaneously connected to the auxiliary pulley secondary gear **511***d* of the B driving part **200** and the main pulley secondary gear **512***b* of the D driving part **300**, and in more detail, the pawl shaft fixing idle gears **760***c* and **760***d* idle on individual shafts and simultaneously connected to the auxiliary pulley secondary gear **511***d* of the B driving part **200**. Thus, it generates a rotation force that is always in the same direction without interfering with mutual rotation, and the rotation force is transmitted to the main pulley secondary gear **512***b* by using

only one of couplings (the pawl **58**R**3** and the latch wheel idle gear **750**R**3** or the pawl **58**L**1** and the latch wheel idle gear **750**L**1**) formed by the pawl guide discs **740**R**3** and **740**L**1** provided on the individual center shafts **553** and **554**.

[0086] Therefore, the initial rotation force generated by the center sprocket **571** reaches the winding rod connection ports **290***a* and **290***b* of the both shafts **27**B**1** and **27**B**2** through the tape belt B **28**B and is transmitted to the winding rods **630***a* and **630***b*, such that the rotation force enables circulation drive of the curtain part **600**. Meanwhile, the rotation of the center sprocket **572** provided on the main pulley shaft **27**D**1** enables vertical drive of the curtain part **600**, in this case, the rotation force of the center sprocket **572** is not transmitted to the latch wheel idle gears **750**R**3** and **750**L**1** because there is no load on the B driving part **200**.

[0087] Through such a configuration, it is possible to shorten a power transmission path and simplify the configuration.

[0088] The following describes another embodiment of the present invention, and

[0089] in the present invention (see FIG. **15**), applying the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** to the connection driving part **400** is changed for the purpose of replacing the first rotation direction selection part **501** of the B driving part **200**, and the D driving part **300** in charge of vertical drive of the curtain is eliminated to have only a function for a circular motion of the dual blind curtain.

[0090] As shown in the accompanying drawing (see FIG. **16**),

[0091] there is provided a dual blind curtain including: a body frame **100** including an upper plate **110** that forms an upper surface, side plates **120** that are provided on both sides of a bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**, and one or more partition plates **130** that are provided inside the bottom surface of the upper plate **110** and formed with a plurality of shaft holes **150**;

[0092] a B driving part **200** comprising: a first rotation direction selection part **501** including a unidirectional complex clutch L**2** **509** which includes a center shaft **551** that is installed to idle on the partition plate **130**, which is provided inside the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, a center sprocket **571** that is installed on one side of the center shaft **551** and provided with a ball chain **571***c* to generate a rotation force, a center gear **561** that transmits the rotation force of the center sprocket **571** to pawl shaft fixing idle gears **760***e* and **760***f* of the unidirectional complex clutch R**2** **508** and the unidirectional complex clutch L**2** **509**, respectively,

[0093] a latch wheel fixing rotation gear **750**L**2** that is provided with a latch wheel, which is installed on one end of the auxiliary pulley shaft **27**B**2** to transmit counterclockwise rotation to one side thereof, in which the auxiliary pulley shaft **27**B**2** is installed to idle on the partition plate **130**, which is provided on the inner side of the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, a pawl shaft fixing idle gear **760***f* that is positioned on an opposite end of the auxiliary pulley shaft **27**B**2** to receive the rotation force from the center gear **561**, and installed with one or more pawl shafts **720***f* each having a pawl **58**L**2** disposed thereon, in which the pawl **58**L**2** engages with the latch wheel fixing rotation gear **750**L**2** in a predetermined condition, a pawl shaft guide hole **59***f* that is positioned between the latch wheel fixing rotation gear **750**L**2** and the pawl shaft fixing idle gear **760***f* and through which the pawl shaft **720***f* passes, a pawl guide idle disc **740**L**2** that is provided with one or more pawl guide grooves **56**L**2** which make contact with the pawl shaft guide hole **59***f* to control a movement of the pawl **58**L**2** in a predetermined condition, a return circular spring **57***f* that is installed on one side of the pawl guide idle disc **740**L**2** to idle with respect to a shaft to return the pawl shaft **720***f* to a predetermined position of the pawl shaft guide hole **59***f* when there is no rotation load on the pawl shaft **720***f*, and a deceleration roller **60***d* that generates a load based on a rotation speed of the pawl guide idle disc **740**L**2**, and

[0094] the complex clutch R**2** **508** which includes a latch wheel fixing rotation gear **750**R**4** that is provided with a latch wheel provided on one end of the main pulley shaft **27**B**1** to transmit clockwise rotation to one side thereof, and transmits the rotation force to a shaft, in which the main pulley shaft **27**B**1** is installed to idle on the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** has and formed with the plurality of shaft holes **150**, a pawl shaft fixing idle gear **760***e* that is positioned on an opposite end of the main pulley shaft **27**B**1** to receive the rotation force from the center gear **561**, and installed with one or more pawl shafts **720***e* each having a pawl **58**R**4** disposed thereon, in which the pawl **58**R**4** engages with the latch wheel fixing rotation gear **750**R**4** in a predetermined condition, a pawl shaft guide hole **59***e* that is positioned between the latch wheel fixing rotation gear **750**R**4** and the pawl shaft fixing idle gear **760***e* and through which the pawl shaft **720***e* passes, a pawl guide idle disc **740**R**4** that is provided with one or more pawl guide grooves **56**R**4** which make contact with the pawl shaft guide hole **59***e* to control a movement of the pawl **58**R**4** in a predetermined condition, a return circular spring **57***e* that is installed on one side of the pawl guide idle disc **740**R**4** to idle with respect to a shaft to return the pawl shaft **720***e* to a predetermined position of the pawl shaft guide hole **59***e* when there is no rotation load on the pawl shaft **720***e*, and a deceleration roller **60***c* that generates a load based on a rotation speed of the pawl guide idle disc **740**R**4**;

[0095] a main pulley **26**B**1** that is constantly coupled to the main pulley shaft **27**B**1** and rotates; a winding rod connection port **290***a* that connects the rotation force generated in the main pulley **26**B**1** to a winding rod **630***a;*

[0096] an auxiliary pulley **26**B**2** that is constantly coupled to the auxiliary pulley shaft **27**B**2** and rotates; a winding rod connection port **290***b* that connects the rotation force transmitted to the auxiliary pulley **26**B**2** to a winding rod **630***b*; and

[0097] a tape belt B **28**B that connects the rotation force of the main pulley **26**B**1** or the rotation force of the auxiliary pulley **26**B**2** to each other and is connected to central axes of the main pulley **26**B**1** and the auxiliary pulley **26**B**2** to be wound in a stacked manner; and

[0098] a curtain part **600** installed to idle on the side plate **120** and the partition plate **130**, which is provided inside the bottom surface of the upper plate **110** of the body frame **100** and formed with the plurality of shaft holes **150**, and including a curtain sheet **610** that is made by connecting one or more various pattern sheets and has one end wound around the winding rod **630***a* connected by the winding rod connection ports **290***a* and **290***b* and an opposite end wound around another winding rod **630***b* by the rotation force of the

main pulley shafts **27**B**1** and **27**D**1**, and a weight bar **620** that is formed on a center of the curtain sheet **610** wound around opposite ends thereof.

[0099] Describing the invention in detail, a configuration in which the unidirectional complex clutch R **505** and the unidirectional complex clutch L **506** are applied to the connection driving part **400** is changed for the purpose of replacing the first rotation direction selection part **501** and diverted to the unidirectional complex clutch R**2 508** and the unidirectional complex clutch L**2 509**. Since there is no change in the basic configuration and principle of the operation as compared with the above-described drawings (see FIG. **11**: a view of a rotation direction configuration of the unidirectional complex clutch, FIG. **12**: an operation principle of both sides of the pawl guide disc and unidirectional complex clutch L), a detailed description thereof will be omitted.

[0100] However, as a part that serves to transmit the rotation force generated by organic coupling of the component is changed, a method for coupling the shaft of the part is changed to fixing or idling, and a change in a shape of the return spring accordingly will be described in more detail.

[0101] That is, the pawl guide discs **740**R**1**, **740**R**2**, **740**R**3**, and **740**L**1** are fixedly installed to the connection driving part **400** to transmit the rotation force to the shafts, and accordingly, the return bar springs **57***a*, **57***b*, **57***d*, and **57***d* are also fixedly installed to the shafts.

[0102] In contrast, when applied to the first rotation direction selection part **501**, the pawl guide discs **740**R**1**, **740**R**2**, **740**R**3**, and **740**L**1** are installed to idle on the shafts to have shapes of the pawl guide idle discs **740**R**4** and **740**L**2**, and accordingly, as the springs, the return circular springs **57***e* and **57***f* are installed to idle with respect to the shaft rotation. In addition, as parts for transmitting the rotation force to the shaft, the latch wheel idle gears **750**R**1**, **750**R**2**, **750**R**3**, and **750**L**1**, which are installed to idle on the connection driving part **400**, are installed by changing a shape of the shaft coupling into the latch wheel fixing rotation gears **750**R**4** and **750**L**2** which are fixedly installed on the shafts.

[0103] As will be additionally described herein, even if the pawl guide discs **740**R**1**, **740**R**2**, **740**R**3**, and **740**L**1** are not fixedly installed to transmit the rotation force on the shafts and installed to idle when applied to the connection driving part **400**, they are only responsible for a power connection function, so that it is cleared that there is no problem in the basic operation principle of the invention.

INDUSTRIAL APPLICABILITY

[0104] The present invention is a technology that is applicable to a roll blind for a curtain, as well as to transmission of power when the rotation ratio of the two shafts is not constant, but changed, selective transmission of rotation power from one shaft to the two shafts, and connection of power of the two shafts having different rotation directions from each other such that only unidirectional power transmission is possible from one shaft to the other shaft.

1. A dual blind curtain comprising:

a body frame (**100**) including an upper plate (**110**) that forms an upper surface, side plates (**120**) that are provided on both sides of a bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**), and one or more partition plates (**130**) that are provided inside the bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**);

a B driving part (**200**) comprising: a first rotation direction selection part (**501**) including a center shaft (**551**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center sprocket (**571**) that is installed on one side of the center shaft (**551**) and provided with a ball chain (**571***c*) to generate a rotation force, a second idle gear clutch (**521***b*) that is installed to idle around the center shaft (**551**), provided with a unidirectional connection port (**52**-*b*) in one side thereof, and engaged with a main pulley primary gear (**511***a*), a first idle gear clutch (**521***a*) that idles on the center shaft (**551**), is provided with a unidirectional connection port (**52**-*a*) in one side thereof to face the second idle gear clutch (**521***b*) at a predetermined interval, and engages with an auxiliary pulley primary gear (**511***c*), a center nut (**531**) that is positioned between the first and second idle gear clutches (**521***a* and **521***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), and a center bolt (**541**) that rotates in an inner portion of the center nut (**531**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54**-*a* and **54**-*b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**); a main pulley shaft (**27**B**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley (**26**B**1**) that is constantly coupled to the main pulley shaft (**27**B**1**) and rotates; the main pulley primary gear (**511***a*) that engages with the second idle gear clutch (**521***b*) of the first rotation direction selection part (**501**) to transmit the rotation force to the main pulley (**26**B**1**); a main pulley secondary gear (**511***b*) that engages with a center gear (**563**) of a connection driving part (**400**) and transmits the rotation force transmitted to the main pulley (**26**B**1**); a winding rod connection port (**290***a*) that connects the rotation force generated in the main pulley (**26**B**1**) to a winding rod (**630***a*); an auxiliary pulley (**26**B**2**) that is constantly coupled to an auxiliary pulley shaft (**27**B**2**) and rotates, in which the auxiliary pulley (**26**B**2**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); an auxiliary pulley primary gear (**511***c*) that engages with the first idle gear clutch (**521***a*) of the first rotation direction selection part (**501**) to transmit the rotation force to the auxiliary pulley (**26**B**2**); an auxiliary pulley secondary gear (**511***d*) that engages with a center gear (**564**) of the connection driving part (**400**) and transmits the rotation force transmitted to the auxiliary pulley (**26**B**2**); and a tape belt B (**28**B) that connects the rotation force of the main pulley (**26**B**1**) or the rotation force of the auxiliary pulley (**26**B**2**) to each other and is connected to central axes of the main pulley (**26**B**1**)

and the auxiliary pulley (**26**B**2**) to be wound in a stacked manner; a D driving part (**300**) comprising: a second rotation direction selection part (**502**) including a center shaft (**552**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center sprocket (**572**) that is installed on one side of the center shaft (**552**) and provided with a ball chain (**572***c*) to generate a rotation force, a first idle gear clutch (**522***a*) that is installed to idle around the center shaft (**552**), provided with a unidirectional connection port (**52-***a*) in one side thereof, and engaged with a main pulley primary gear (**512***a*), a second idle gear clutch (**522***b*) that idles on the center shaft (**552**), is provided with a unidirectional connection port (**52-***b*) in one side thereof to face the first idle gear clutch (**522***a*) at a predetermined interval, and engages with an auxiliary pulley primary gear (**512***c*), a center nut (**532**) that is positioned between the first and second idle gear clutches (**522***a*, **522***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), and a center bolt (**542**) that rotates in an inner portion of the center nut (**532**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54-***a* and **54-***b*) on both sides of the bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**); a main pulley shaft (**27**D**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley (**26**D**1**) that is constantly coupled to the main pulley shaft (**27**D**1**) and rotates; a main pulley primary gear (**512***a*) that engages with the first idle gear clutch (**522***a*) of the second rotation direction selection part (**502**) to transmit the rotation force to the main pulley (**26**D**1**); a main pulley secondary gear (**512***b*) that engages with a connection gear set (**240**) of the connection driving part (**400**) and transmits the rotation force transmitted to the main pulley (**26**D**1**); a winding rod connection port (**290***b*) that connects the rotation force generated in the main pulley (**27**D**1**) to a winding rod (**630***b*); an auxiliary pulley (**26**D**2**) that is constantly coupled to an auxiliary pulley shaft (**27**D**2**) and rotates, in which the auxiliary pulley (**26**D**2**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); an auxiliary pulley primary gear (**512***c*) that engages with the second idle gear clutch (**522***b*) of the second rotation direction selection part (**502**) to transmit the rotation force to the auxiliary pulley (**26**D**2**); an auxiliary pulley secondary gear (**512***d*) that engages with a connection gear set (**230**) of the connection driving part (**400**) and transmits the rotation force transmitted to the auxiliary pulley (**26**D**2**); and a tape belt D (**28**D) that connects the rotation force of the main pulley (**26**D**1**) or the rotation force of the auxiliary pulley (**26**D**2**) to each other and is connected to central axes of the main pulley (**26**D**1**) and the auxiliary pulley (**26**D**2**) to be wound in a stacked manner; the connection driving part (**400**) comprising: a third rotation direction selection part (**503**) including a center shaft (**553**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center gear (**563**) that is installed on one side of the center shaft (**553**) and engaged with the main pulley secondary gear (**511***b*) of the B driving part (**200**) to receive the rotation force, a first idle gear clutch (**523***a*) that is installed to idle around the center shaft (**553**) and provided with a unidirectional connection port (**52-***a*) in one side thereof, a second idle gear clutch (**523***b*) that idles on the center shaft (**553**) and is provided with a unidirectional connection port (**52-***b*) in one side thereof to face the first idle gear clutch (**523***a*) at a predetermined interval, a center nut (**533**) that is positioned between the first and second idle gear clutches (**523***a* and **523***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), a center bolt (**543**) that rotates in an inner portion of the center nut (**533**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54-***a* and **54-***b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**), and the connection gear set (**230**) that engages with the auxiliary pulley secondary gear (**512***d*) and transmits the rotation force of the second idle gear clutch (**523***b*) to the auxiliary pulley secondary gear (**512***d*); and the fourth rotation direction selection part (**504**) including a center shaft (**554**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center gear (**564**) that is installed on one side of the center shaft (**554**) and engaged with the auxiliary pulley secondary gear (**511***d*) of the B driving part (**200**) to receive the rotation force, a first idle gear clutch (**524***a*) that is installed to idle around the center shaft (**554**) and provided with a unidirectional connection port (**52-***a*) in one side thereof, a second idle gear clutch (**524***b*) that idles on the center shaft (**553**) and is provided with a unidirectional connection port (**52-***b*) in one side thereof to face the first idle gear clutch (**524***a*) at a predetermined interval, a center nut (**533**) that is positioned between the first and second idle gear clutches (**524***a* and **524***b*), provided on an inner surface of a body thereof with a nut groove (**53-1**), and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), a center bolt (**543**) that rotates in an inner portion of the center nut (**533**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is

provided with unidirectional connection ports (**54**-*a* and **54**-*b*) on both sides of a bolt body (**54**-**1**), respectively, and is provided with a bolt thread (**54**-**3**) on an outer circumferential surface of the bolt body (**54**-**1**), and the connection gear set (**240**) that engages with the main pulley secondary gear (**512***b*) and transmits the rotation force of the first idle gear clutch (**524***a*) to the main pulley secondary gear (**512***b*); and a curtain part (**600**) installed to idle on the side plate (**120**) and the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and including a curtain sheet (**610**) that is made by connecting one or more various pattern sheets and has one end wound around the winding rod (**630***a*) connected by the winding rod connection ports (**290***a* and **290***b*) and an opposite end wound around another winding rod (**630***b*) by the rotation force of the main pulley shafts (**27**B**1** and **27**D**1**), and a weight bar (**620**) that is formed on a center of the curtain sheet (**610**) wound around opposite ends thereof.

2. The dual blind curtain of claim **1**, wherein the connection driving part (**400**) comprises: a unidirectional complex clutch R (**505**) including the center shaft (**553**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) formed with the plurality of shaft holes (**150**), a latch wheel idle gear (**750**R**3**) that is provided with a latch wheel, which idles on one end of the center shaft (**553**) to transmit clockwise rotation to one side thereof, and transmits the rotation force to the auxiliary pulley secondary gear (**512***d*) through the engaged connection gear set (**230**), a pawl shaft fixing idle gear (**760***c*) that is positioned on an opposite end of the center shaft (**553**) to receive the rotation force from the main pulley secondary gear (**511***b*), and installed with one or more pawl shafts (**720***c*) each having a pawl (**58**R**3**) disposed thereon, in which the pawl (**58**R**3**) engages with the latch wheel idle gear (**750**R**3**) in a predetermined condition, a pawl shaft guide hole (**59***c*) that is positioned between the latch wheel idle gear (**750**R**3**) and the pawl shaft fixing idle gear (**760***c*) and through which the pawl shaft (**720***c*) passes, a pawl guide idle disc (**740**R**3**) that is provided with one or more pawl guide grooves (**56**R**3**) which make contact with the pawl shaft guide hole (**59***c*) to control a movement of the pawl (**58**R**3**) in a predetermined condition, a return bar spring (**57***c*) that is fixed to the center shaft (**553**) to return the pawl shaft (**720***c*) to a predetermined position of the pawl shaft guide hole (**59***c*) when there is no rotation load on the pawl shaft (**720***c*), and a deceleration roller (**60***a*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**R**3**); and a unidirectional complex clutch L (**506**) including the center shaft (**554**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) formed with the plurality of shaft holes (**150**), a latch wheel idle gear (**750**L**1**) that is provided with a latch wheel, which idles on one end of the center shaft (**554**) to transmit counterclockwise rotation to one side thereof, and transmits the rotation force to the main pulley secondary gear (**512***b*) through the engaged connection gear set (**240**), a pawl shaft fixing idle gear (**760***d*) that is positioned on an opposite end of the center shaft (**554**) to receive the rotation force from the auxiliary pulley secondary gear (**511***d*), and installed with one or more pawl shafts (**720***d*) each having a pawl (**58**L**1**) disposed thereon, in which the pawl (**58**L**1**) engages with the latch wheel idle gear (**750**L**1**) in a predetermined condition, a pawl shaft guide hole (**59***d*) that is positioned between the latch wheel idle gear (**750**L**1**) and the pawl shaft fixing idle gear (**760***d*) and through which the pawl shaft (**720***d*) passes, a pawl guide idle disc (**740**L**1**) that is provided with one or more pawl guide grooves (**56**L**1**) which make contact with the pawl shaft guide hole (**59***d*) to control a movement of the pawl (**58**L**1**) in a predetermined condition, a return bar spring (**57***d*) that is fixed to the center shaft (**554**) to return the pawl shaft (**720***d*) to a predetermined position of the pawl shaft guide hole (**59***d*) when there is no rotation load on the pawl shaft (**720***d*), and a deceleration roller (**60***b*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**L**1**).

3. The dual blind curtain of claim **1** or **2**, wherein each of the first and second rotation direction selection parts (**501** and **502**) is provided with a bidirectional complex clutch (**507**) including: an idle sprocket (**710**) that has one or more pawl shaft control grooves (**711** and **712**) provided on both sides thereof at a position spaced apart from each other in an opposite direction around the center shaft (**551**) installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); latch wheel idle gears (**750**R**1** and **750**R**2**) that are installed to face a same component based on the idle sprocket (**710**), are first provided with latch wheels which idle on opposite ends of the center shaft (**551**) to transmit unidirectional rotation to one side thereof, and transmits a generated rotation force to another shaft; pawl shaft fixing idle gears (**730***a* and **730***b*) that are positioned on both sides of the idle sprocket (**710**), and installed with one or more pawl shafts (**720***a* and **720***b*) having pawls (**58**R**1** and **58**R**2**) disposed thereon, in which the pawls (**58**R**1** and **58**R**2**) are inserted into the pawl shaft control grooves (**711** and **712**) to receive the rotation force and engage with the latch wheel idle gears (**750**R**1** and **750**R**2**) in a predetermined condition; pawl shaft guide holes (**59***a* and **59***b*) that are positioned between the latch wheel idle gears (**750**R**1** and **750**R**2**) and the pawl shaft fixing idle gears (**730***a* and **730***b*), respectively and through which the pawl shafts (**720***a* and **720***b*) pass; pawl guide idle discs (**740**R**1** and **740**R**2**) that are provided with one or more pawl guide grooves (**56**R**1** and **56**R**2**) which make contact with the pawl shaft guide holes (**59***a* and **59***b*) to control a movement of the pawls (**58**R**1** and **58**R**2**) in a predetermined condition; and return bar springs (**57***a* and **57***b*) that are installed on the center shaft (**551**) to return the pawl shafts (**720***a* and **720***b*) to a predetermined position of the pawl shaft guide holes (**59***a* and **59***b*) when there is no rotation load on the pawl shafts (**720***a* and **720***b*).

4. A dual blind curtain comprising:

a body frame (**100**) including an upper plate (**110**) that forms an upper surface, side plates (**120**) that are provided on both sides of a bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**), and one or more partition plates (**130**) that are provided inside the bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**);

a B driving part (**200**) comprising: a first rotation direction selection part (**501**) including a center shaft (**551**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate

(**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and provided on an outer circumferential surface thereof with a locking protrusion (**55-1**), a center sprocket (**571**) that is installed on one side of the center shaft (**551**) and provided with a ball chain (**571***c*) to generate a rotation force, a second idle gear clutch (**521***b*) that is installed to idle around the center shaft (**551**), provided with a unidirectional connection port (**52-***b*) in one side thereof, and engaged with a main pulley primary gear (**511***a*), a first idle gear clutch (**521***a*) that idles on the center shaft (**551**), is provided with a unidirectional connection port (**52-***a*) in one side thereof to face the second idle gear clutch (**521***b*) at a predetermined interval, and engages with an auxiliary pulley primary gear (**511***c*), a center nut (**531**) that is positioned between the first and second idle gear clutches (**521***a* and **521***b*), provided with a nut groove (**53-1**) in an inner surface of a body thereof, and installed so that the body idles at the shaft hole (**150**) of the partition plate (**130**), and a center bolt (**541**) that rotates in an inner portion of the center nut (**531**), has a sliding hole (**54-2**) formed in a center of a body thereof to move left and right of the locking protrusion (**55-1**), is provided with unidirectional connection ports (**54-***a* and **54-***b*) on both sides of a bolt body (**54-1**), respectively, and is provided with a bolt thread (**54-3**) on an outer circumferential surface of the bolt body (**54-1**); a main pulley shaft (**27**B**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley (**26**B**1**) that is constantly coupled to the main pulley shaft (**27**B**1**) and rotates; the main pulley primary gear (**511***a*) that engages with the second idle gear clutch (**521***b*) of the first rotation direction selection part (**501**) to transmit the rotation force to the main pulley (**26**B**1**); a winding rod connection port (**290***a*) that connects the rotation force generated in the main pulley (**26**B**1**) to a winding rod (**630***a*); an auxiliary pulley (**26**B**2**) that is constantly coupled to an auxiliary pulley shaft (**27**B**2**) and rotates, in which the auxiliary pulley (**26**B**2**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**); an auxiliary pulley primary gear (**511***c*) that engages with the first idle gear clutch (**521***a*) of the first rotation direction selection part (**501**) to transmit the rotation force to the auxiliary pulley (**26**B**2**); an auxiliary pulley secondary gear (**511***d*) that engages with pawl shaft fixing idle gears (**760***c*, **760***d*) of a connection driving part (**400**) and transmits the rotation force transmitted to the auxiliary pulley (**26**B**2**); and the tape belt B (**28**B) that connects the rotation force of the main pulley (**26**B**1**) or the rotation force of the auxiliary pulley (**26**B**2**) to each other and is connected to central axes of the main pulley (**26**B**1**) and the auxiliary pulley (**26**B**2**) to be wound in a stacked manner; a D driving part (**300**) including: a main pulley shaft (**27**D**1**) that is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of holes (**150**); a main pulley secondary gear (**512***b*) that is installed on one end of the main pulley shaft (**27**D**1**) and connected to latch wheel idle gears (**750**R**3** and **750**L**1**) of the connection driving part (**400**); a center sprocket (**572**) that is installed on the main pulley shaft (**27**D**1**) and provided with a ball chain (**572***c*) to generate the rotation force; and a winding rod connection port (**290***b*) connects the rotation force generated in the main pulley shaft (**27**D**1**) to a winding rod (**630***b*); the connection driving part (**400**) comprising: a unidirectional complex clutch R (**505**) including a latch wheel idle gear (**750**R**3**) that is provided with a latch wheel, which idles on one end of a center shaft (**553**) to transmit clockwise rotation to one side thereof, and engaged with the main pulley secondary gear (**512***b*) to transmit the rotation force, in which the center shaft (**553**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), a pawl shaft fixing idle gear (**760***c*) that is positioned on an opposite end of the center shaft (**553**) to receive the rotation force from the auxiliary pulley secondary gear (**511***d*), and installed with one or more pawl shafts (**720***c*) each having a pawl (**58**R**3**) disposed thereon, in which the pawl (**58**R**3**) engages with the latch wheel idle gear (**750**R**3**) in a predetermined condition, a pawl shaft guide hole (**59***c*) that is positioned between the latch wheel idle gear (**750**R**3**) and the pawl shaft fixing idle gear (**760***c*) and through which the pawl shaft (**720***c*) passes, a pawl guide idle disc (**740**R**3**) that is provided with one or more pawl guide grooves (**56**R**3**) which make contact with the pawl shaft guide hole (**59***c*) to control a movement of the pawl (**58**R**3**) in a predetermined condition, a return bar spring (**57***c*) that is fixed to the center shaft (**553**) to return the pawl shaft (**720***c*) to a predetermined position of the pawl shaft guide hole (**59***c*) when there is no rotation load on the pawl shaft (**720***c*), and a deceleration roller (**60***a*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**R**3**); and the unidirectional complex clutch L (**506**) including a latch wheel idle gear (**750**L**1**) that is provided with a latch wheel, which idles on one end of a center shaft (**554**) to transmit counterclockwise rotation to one side thereof, and engaged with the main pulley secondary gear (**512***b*) to transmit the rotation force, in which the central shaft (**554**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), a pawl shaft fixing idle gear (**760***d*) that is positioned on an opposite end of the center shaft (**554**) to receive the rotation force from the auxiliary pulley secondary gear (**511***d*), and installed with one or more pawl shafts (**720***d*) each having a pawl (**58**L**1**) disposed thereon, in which the pawl (**58**L**1**) engages with the latch wheel idle gear (**750**L**1**) in a predetermined condition, a pawl shaft guide hole (**59***d*) that is positioned between the latch wheel idle gear (**750**L**1**) and the pawl shaft fixing idle gear (**760***d*) and through which the pawl shaft (**720***d*) passes, a pawl guide idle disc (**740**L**1**) that is provided with one or more pawl guide grooves (**56**L**1**) which make contact with the pawl shaft guide hole (**59***d*) to control a movement of the pawl (**58**L**1**) in a predetermined condition, a return bar spring (**57***d*) that

is fixed to the center shaft (**554**) to return the pawl shaft (**720***d*) to a predetermined position of the pawl shaft guide hole (**59***d*) when there is no rotation load on the pawl shaft (**720***d*), and a deceleration roller (**60***b*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**L**1**); and a curtain part (**600**) installed to idle on the side plate (**120**) and the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and including a curtain sheet (**610**) that is made by connecting one or more various pattern sheets and has one end wound around the winding rod (**630***a*) connected by the winding rod connection ports (**290***a* and **290***b*) and an opposite end wound around another winding rod (**630***b*) by the rotation force of the main pulley shaft (**27**B**1** and **27**D**1**), and a weight bar (**620**) that is formed on a center of the curtain sheet (**610**) wound around opposite ends thereof.

5. A dual blind curtain comprising:

a body frame (**100**) including an upper plate (**110**) that forms an upper surface, side plates (**120**) that are provided on both sides of a bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**), and one or more partition plates (**130**) that are provided inside the bottom surface of the upper plate (**110**) and formed with a plurality of shaft holes (**150**); a B driving part (**200**) comprising: a first rotation direction selection part (**501**) including a unidirectional complex clutch L**2** (**509**) which includes a center shaft (**551**) that is installed to idle on the partition plate (**130**), which is provided inside the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), a center sprocket (**571**) that is installed on one side of the center shaft (**551**) and provided with a ball chain (**571***c*) to generate a rotation force, a center gear (**561**) that transmits the rotation force of the center sprocket (**571**) to pawl shaft fixing idle gears (**760***e* and **760***f*) of the unidirectional complex clutch R**2** (**508**) and the unidirectional complex clutch L**2** (**509**), respectively, a latch wheel fixing rotation gear (**750**L**2**) that is provided with a latch wheel, which is installed on one end of the auxiliary pulley shaft (**27**B**2**) to transmit counterclockwise rotation to one side thereof, in which the auxiliary pulley shaft (**27**B**2**) is installed to idle on the partition plate (**130**), which is provided on the inner side of the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), a pawl shaft fixing idle gear (**760***f*) that is positioned on an opposite end of the auxiliary pulley shaft (**27**B**2**) to receive the rotation force from the center gear (**561**), and installed with one or more pawl shafts (**720***f*) each having a pawl (**58**L**2**) disposed thereon, in which the pawl (**58**L**2**) engages with the latch wheel fixing rotation gear (**750**L**2**) in a predetermined condition, a pawl shaft guide hole (**59***f*) that is positioned between the latch wheel fixing rotation gear (**750**L**2**) and the pawl shaft fixing idle gear (**760***f*) and through which the pawl shaft (**720***f*) passes, a pawl guide idle disc (**740**L**2**) that is provided with one or more pawl guide grooves (**56**L**2**) which make contact with the pawl shaft guide hole (**59***f*) to control a movement of the pawl (**58**L**2**) in a predetermined condition, a return circular spring (**57***f*) that is installed on one side of the pawl guide idle disc (**740**L**2**) to idle with respect to a shaft to return the pawl shaft (**720***f*) to a predetermined position of the pawl shaft guide hole (**59***f*) when there is no rotation load on the pawl shaft (**720***f*), and a deceleration roller (**60***d*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**L**2**), and the complex clutch R**2** (**508**) which includes a latch wheel fixing rotation gear (**750**R**4**) that is provided with a latch wheel provided on one end of the main pulley shaft (**27**B**1**) to transmit clockwise rotation to one side thereof, and transmits the rotation force to a shaft, in which the main pulley shaft (**27**B**1**) is installed to idle on the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) has and formed with the plurality of shaft holes (**150**), a pawl shaft fixing idle gear (**760***e*) that is positioned on an opposite end of the main pulley shaft (**27**B**1**) to receive the rotation force from the center gear (**561**), and installed with one or more pawl shafts (**720***e*) each having a pawl (**58**R**4**) disposed thereon, in which the pawl (**58**R**4**) engages with the latch wheel fixing rotation gear (**750**R**4**) in a predetermined condition, a pawl shaft guide hole (**59***e*) that is positioned between the latch wheel fixing rotation gear (**750**R**4**) and the pawl shaft fixing idle gear (**760***e*) and through which the pawl shaft (**720***e*) passes, a pawl guide idle disc (**740**R**4**) that is provided with one or more pawl guide grooves (**56**R**4**) which make contact with the pawl shaft guide hole (**59***e*) to control a movement of the pawl (**58**R**4**) in a predetermined condition, a return circular spring (**57***e*) that is installed on one side of the pawl guide idle disc (**740**R**4**) to idle with respect to a shaft to return the pawl shaft (**720***e*) to a predetermined position of the pawl shaft guide hole (**59***e*) when there is no rotation load on the pawl shaft (**720***e*), and a deceleration roller (**60***c*) that generates a load based on a rotation speed of the pawl guide idle disc (**740**R**4**); a main pulley (**26**B**1**) that is constantly coupled to the main pulley shaft (**27**B**1**) and rotates; a winding rod connection port (**290***a*) that connects the rotation force generated in the main pulley (**26**B**1**) to a winding rod (**630***a*); an auxiliary pulley (**26**B**2**) that is constantly coupled to the auxiliary pulley shaft (**27**B**2**) and rotates; a winding rod connection port (**290***b*) that connects the rotation force transmitted to the auxiliary pulley (**26**B**2**) to a winding rod (**630***b*); and the tape belt B (**28**B) that connects the rotation force of the main pulley (**26**B**1**) or the rotation force of the auxiliary pulley (**26**B**2**) to each other and is connected to central axes of the main pulley (**26**B**1**) and the auxiliary pulley (**26**B**2**) to be wound in a stacked manner; and a curtain part (**600**) installed to idle on the side plate (**120**) and the partition plate (**130**), which is provided inside the bottom surface of the upper plate (**110**) of the body frame (**100**) and formed with the plurality of shaft holes (**150**), and including a curtain sheet (**610**) that is made by connecting one or more various pattern sheets and has one end wound around the winding rod (**630***a*) connected by the winding rod connection ports (**290***a* and **290***b*) and an opposite end wound around another

winding rod (**630***b*) by the rotation force of the main pulley shaft (**27**B**1** and **27**D**1**), and a weight bar (**620**) that is formed on a center of the curtain sheet (**610**) wound around opposite ends thereof.

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