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(54) **HYDROKINETIC TORQUE COUPLING
DEVICE HAVING TURBINE-PISTON
LOCKUP CLUTCH, AND RELATED
METHODS**

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(57) **ABSTRACT**

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CPC **F16H 45/02** (2013.01); **F16H 2045/0205** (2013.01); **F16H 2045/0226** (2013.01); **F16H 2045/0263** (2013.01); **F16H 2045/0284** (2013.01)

A hydrokinetic torque coupling device features an impeller, a casing having a first engagement surface, a damper assembly, a turbine-piston including a drive component with a second engagement surface, and a clutch plate having clutch plate engagement surfaces interposed between and axially movable relative to at least one of the first and second clutch plate engagement surfaces. The turbine-piston is axially displaceable relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another to non-rotatably lock the casing relative to the input part of the damper assembly.

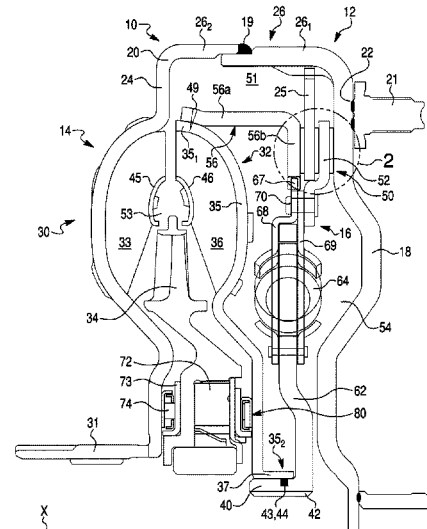
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CPC F16H 45/02; F16H 2045/0205; F16H 2045/0226; F16H 2045/0263; F16H 2045/0284; F16H 2045/0221
See application file for complete search history.

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19 Claims, 5 Drawing Sheets



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Fig. 1

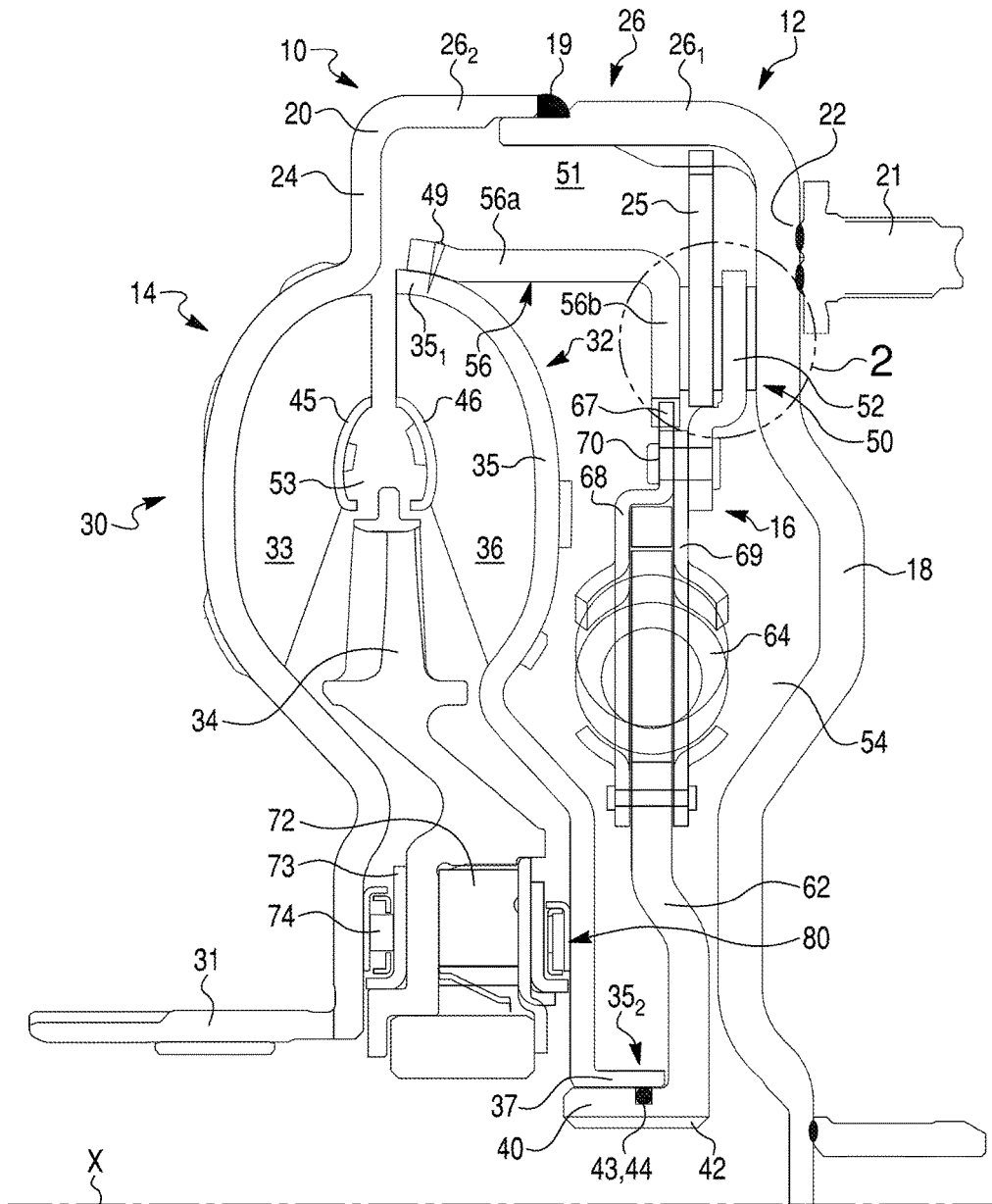


Fig. 2

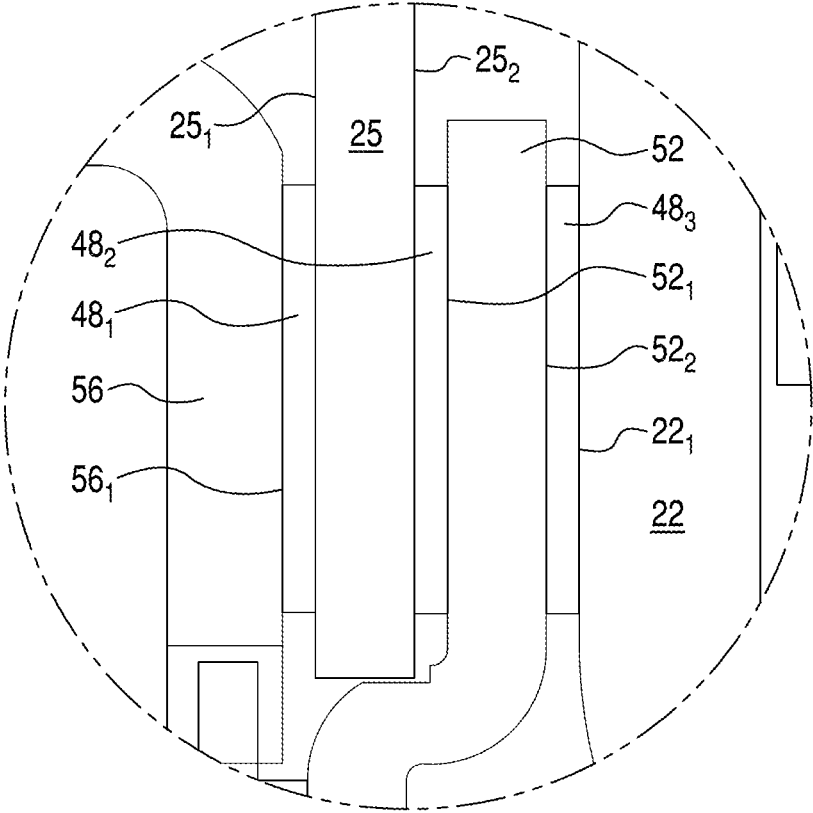


Fig. 3

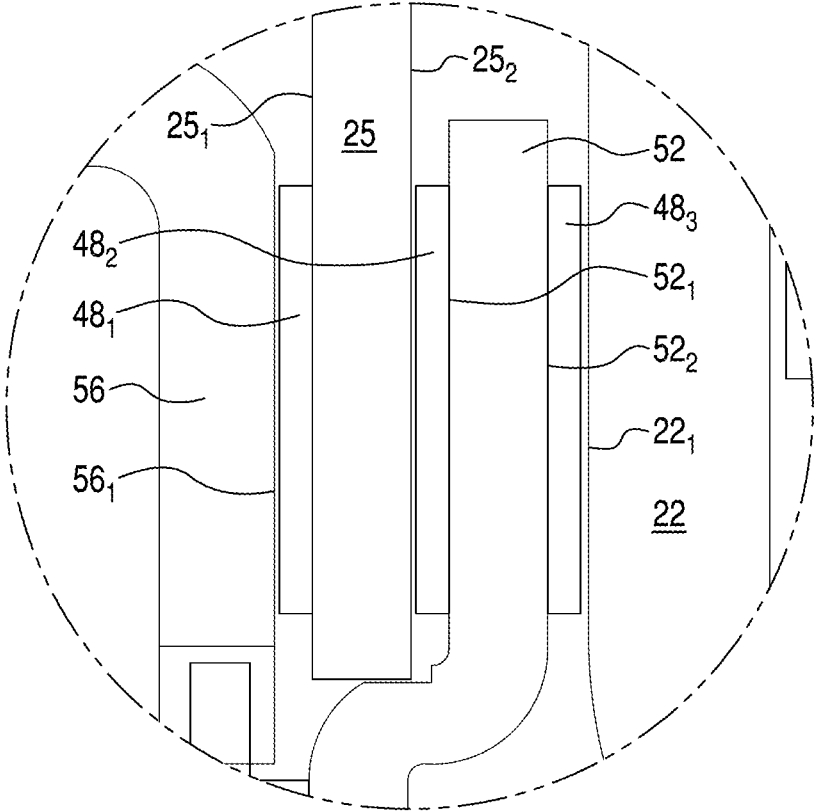


Fig. 4

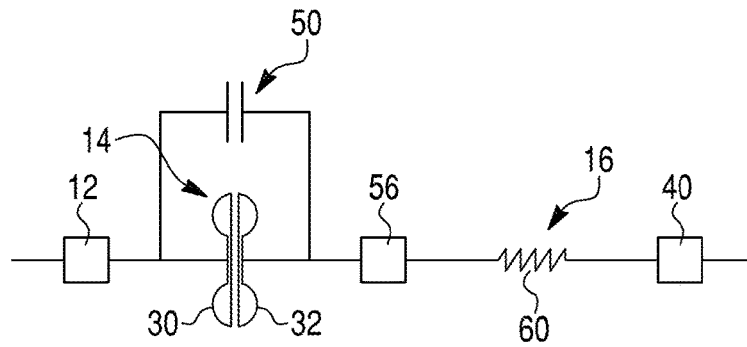


Fig. 5

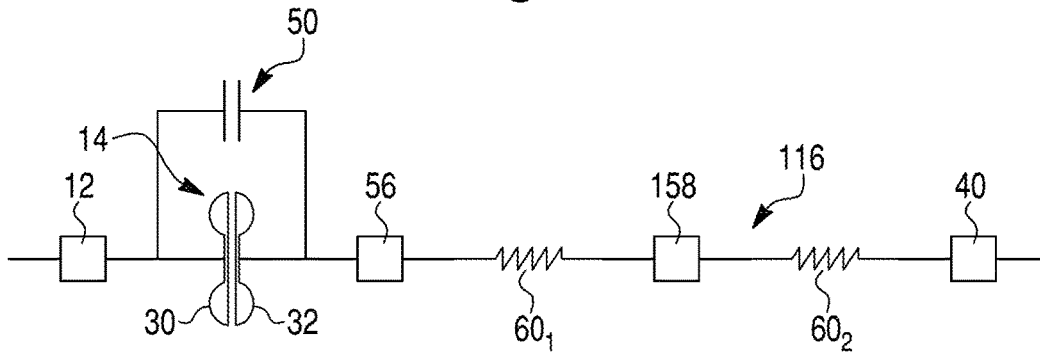


Fig. 6

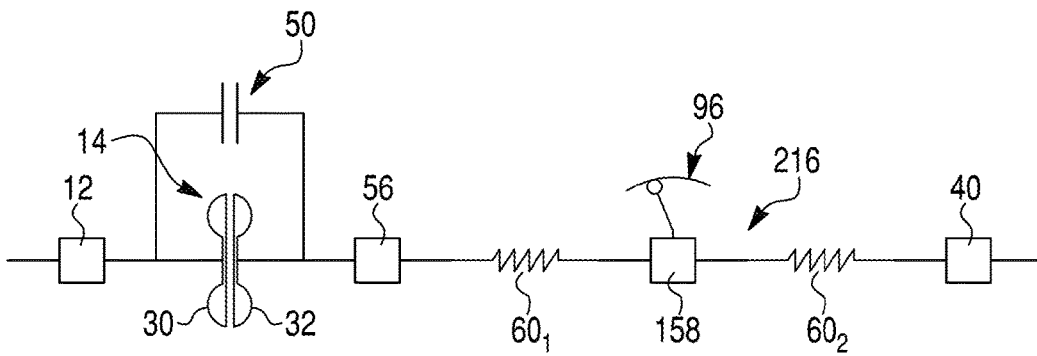
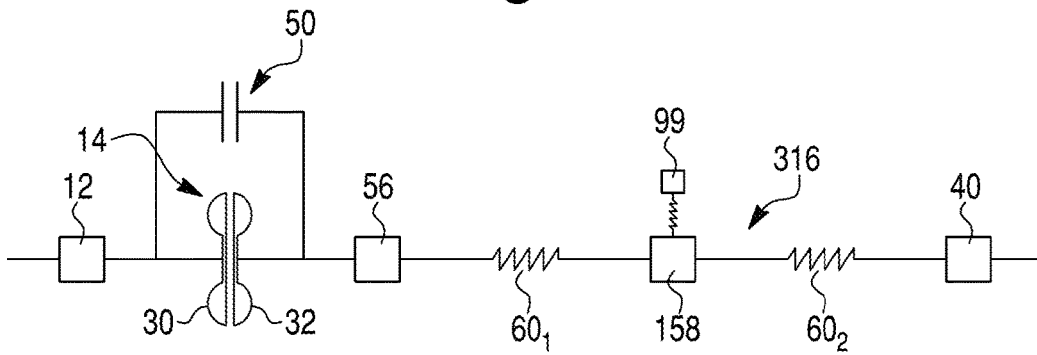


Fig. 7



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HYDROKINETIC TORQUE COUPLING DEVICE HAVING TURBINE-PISTON LOCKUP CLUTCH, AND RELATED METHODS

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention generally relates to hydrokinetic torque coupling devices, and more particularly to a hydrokinetic torque coupling device including a turbine-piston lockup clutch for mechanically coupling driving and driven shafts.

2. Description of the Related Art

Generally, vehicles with automatic transmissions are equipped with a hydrokinetic torque coupling device for fluidly coupling the driving shaft of an engine to a driven shaft of a transmission. Lockup clutches are known for mechanically coupling the driving and driven shafts under certain operating conditions. Lockup clutches and their operation are described in, for example, U.S. Pat. No. 8,276,723 and U.S. Pat. No. 7,191,879.

While hydrokinetic torque coupling devices with lockup clutches have proven to be useful for vehicular driveline applications and conditions, improvements that may enhance their performance and cost are possible.

As taught hereinbelow, such improvements may derive from, for example, reducing the spatial requirements of components of the hydrokinetic torque coupling device and/or consolidating functions of two or more components into a single component.

BRIEF SUMMARY OF THE INVENTION

According to a first aspect of the invention, there is provided a hydrokinetic torque coupling device for coupling together a driving shaft and a driven shaft. The hydrokinetic torque coupling device of this first aspect comprises an impeller including an impeller shell, a casing having a first engagement surface and including a casing shell connected to the impeller shell, an axially displaceable turbine-piston hydrodynamically drivable by the impeller, a damper assembly comprising an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub, and a clutch plate having opposite clutch plate engagement surfaces. The turbine-piston includes a turbine-piston shell and a drive component having a second engagement surface. The clutch plate is interposed between and axially movable relative to at least one of the first and second engagement surfaces. The turbine-piston is axially displaceable relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.

A second aspect of the invention provides a method of assembling a hydrokinetic torque coupling device. A torque converter is provided that comprises an impeller including an impeller shell, and an axially displaceable turbine-piston including a turbine-piston shell and a drive component. The torque converter is operatively connected to a casing shell, a damper assembly including an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub, and a clutch plate

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having opposite clutch plate engagement surfaces to assemble the hydrokinetic torque coupling device, which includes a casing including the casing shell. The clutch plate is interposed between and axially displaceable relative to at least one of a first engagement surface of the casing and a second engagement surface of the drive component. Turbine-piston is axially displaceable relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.

A third aspect of the invention provides a method of operating a hydrokinetic torque coupling device. The hydrokinetic torque coupling device comprises an impeller including an impeller shell, a casing having a first engagement surface and including a casing shell connected to the impeller shell, an axially displaceable turbine-piston hydrodynamically drivable by the impeller and including a turbine-piston shell and a drive component having a second engagement surface, a damper assembly including an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub, and a clutch plate having opposite clutch plate engagement surfaces interposed between and axially movable relative to at least one of the first and second engagement surfaces. The method comprises axially displacing the turbine-piston relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.

Other aspects of the invention, including apparatus, devices, systems, coupling devices, converters, processes, and the like which constitute part of the invention, will become more apparent upon reading the following detailed description of the exemplary embodiments.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWING(S)

The accompanying drawings are incorporated in and constitute a part of the specification. The drawings, together with the general description given above and the detailed description of the exemplary embodiments and methods given below, serve to explain the principles of the invention. The objects and advantages of the invention will become apparent from a study of the following specification when viewed in light of the accompanying drawings, in which like elements are given the same or analogous reference numerals and wherein:

FIG. 1 is a fragmented half-view in axial cross section of a hydrokinetic torque coupling device equipped with a turbine-piston in accordance with a first exemplary embodiment of the present invention;

FIG. 2 is an enlarged fragmentary view of circle 2 of FIG. 1 in a lockup mode;

FIG. 3 is the same enlarged fragmentary view of circle 2 of FIG. 1, but with the first exemplary embodiment shown in the non-lockup mode;

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FIG. 4 is a simplified diagram of a hydrodynamic torque coupling device including a turbine-piston with a single damper assembly;

FIG. 5 is a simplified diagram of another hydrodynamic torque coupling device including a turbine-piston with a dual or double damper assembly;

FIG. 6 is a simplified diagram of still another hydrodynamic torque coupling device including a turbine-piston with dual or double damper assemblies and a pendulum vibration absorber; and

FIG. 7 is a simplified diagram of a further hydrodynamic torque coupling device including a turbine-piston with dual or double damper assemblies and a vibration absorbing spring-mass system.

DETAILED DESCRIPTION OF EXEMPLARY EMBODIMENT(S) AND EMBODIED METHOD(S) OF THE INVENTION

Reference will now be made in detail to exemplary embodiments and methods of the invention as illustrated in the accompanying drawings, in which like reference characters designate like or corresponding parts throughout the drawings. It should be noted, however, that the invention in its broader aspects is not limited to the specific details, representative devices and methods, and illustrative examples shown and described in connection with the exemplary embodiments and methods.

This description of exemplary embodiments is intended to be read in connection with the accompanying drawings, which are to be considered part of the entire written description. In the description, relative terms such as "horizontal," "vertical," "up," "down," "upper," "lower," "right," "left," "top," and "bottom" as well as derivatives thereof (e.g., "horizontally," "downwardly," "upwardly," etc.) should be construed to refer to the orientation as then described or as shown in the drawing figure under discussion. These relative terms are for convenience of description and normally are not intended to require a particular orientation. Terms concerning attachments, coupling and the like, such as "connected" and "interconnected," refer to a relationship wherein structures are secured or attached to one another either directly or indirectly through intervening structures, as well as both movable or rigid attachments or relationships, unless expressly described otherwise. The term "operatively connected" is such an attachment, coupling or connection that allows the pertinent structures to operate as intended by virtue of that relationship. Additionally, the words "a" and "an" as used in the claims mean "at least one."

An exemplary embodiment of a hydrokinetic torque coupling device is generally represented in the accompanying drawings by reference numeral 10, as best shown in the fragmentary sectional view in FIG. 1. The hydrokinetic torque coupling device 10 is operable to fluidly or mechanically couple a driving shaft and a driven shaft of a motor vehicle, such as an automobile. In the typical case, the driving shaft is an output shaft of an internal combustion engine (not shown) of the motor vehicle and the driven shaft is connected to an automatic transmission of the motor vehicle.

The hydrokinetic torque coupling device 10 includes a sealed casing 12 filled with a fluid, such as oil or transmission fluid. The sealed casing 12, a hydrodynamic torque converter 14 and a torsional vibration damper (also referred to herein as a damper assembly) 16 are all rotatable about a rotational axis X. The drawings discussed herein show half-views, that is, the cross section of a portion or fragment

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of the hydrokinetic torque coupling device 10 above rotational axis X. Generally, the device 10 is symmetrical about the rotational axis X. Herein, the axial and radial orientations are considered with respect to the rotational axis X of the torque coupling device 10. The relative terms such as "axially," "radially," and "circumferentially" are with respect to orientations parallel to, perpendicular to, and circularly around the rotational axis X, respectively.

The sealed casing 12 according to the first exemplary embodiment as illustrated in FIG. 1 includes a first casing shell 18 and a second casing shell 20 fixedly and sealingly connected together, such as by weld 19 at their outer peripheries, and non-movable relative to one another yet rotatable about the axis X. The first shell 18 is interconnected to and non-movable relative to the driving shaft, more typically a flywheel (not shown) that is fixed to and non-rotatable relative to the driving shaft, so that the casing 12 turns at the same speed that the engine operates. Specifically, in the illustrated embodiment of FIG. 1 the casing 12 is rotatably driven by the internal combustion engine and is non-rotatably coupled to the flywheel thereof with studs 21, shown in FIG. 1. Each of the first and second casing shells 18, 20 may be made, for example, integrally by press-forming one-piece metal sheets.

The first casing shell 18 includes a first sidewall portion 22 extending substantially radially relative to the direction from the rotational axis X (i.e., in a plane that is generally transverse to the rotational axis X) and a cylindrical first outer wall portion 26₁ extending substantially axially from the first sidewall portion 22 toward the second casing shell 20. Similarly, the second casing shell 20 includes a second sidewall portion 24 extending substantially radially relative to the direction from the rotational axis X and a cylindrical second outer wall portion 26₂ extending substantially axially from the second sidewall portion 24 toward the first casing shell 18. The first sidewall portion 22 includes a first engagement surface 22₁, best shown in FIGS. 2 and 3. The first and second outer wall portions 26₁, 26₂ collectively establish an annular outer wall 26 substantially parallel with the rotation axis X. The weld 19 fixedly secures the outer wall portions 26₁ and 26₂ together. Although not shown, the casing 12 may include additional components, such as an annular cylindrical shell welded at opposite ends to and interconnecting the first and second outer wall portions 26₁, 26₂ together.

Extending radially inward from the annular outer wall 26, more particularly from the outer wall portion 26₁, is a clutch plate 25 having opposite clutch plate engagement surfaces 25₁ and 25₂. A spline connection may be used to connect the clutch plate 25 to the annular outer wall 26 so as to allow axial displacement of the clutch plate 25 relative to the casing 12, while securing the clutch plate 25 non-rotatably relative to the casing 12. As best shown in FIG. 3, a first frictional liner 48₁ is secured to the clutch plate engagement surface 25₁.

The torque converter 14 includes an impeller (sometimes referred to as the pump or impeller wheel) 30, a turbine-piston 32, and a stator (sometimes referred to as the reactor) 34 interposed axially between and operatively connecting the impeller 30 and the turbine-piston 32. The impeller 30, the turbine-piston 32, and the stator 34 are coaxially aligned with one another on the rotational axis X. The impeller 30, the turbine-piston 32, and the stator 34 collectively form a torus. The impeller 30 and the turbine-piston 32 may be fluidly (or hydrodynamically) coupled to one another as known in the art.

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The second casing shell **20** of the casing **12** also forms and serves as the impeller shell of the impeller **30**. Accordingly, the impeller shell **20** sometimes is referred to as part of the casing **12**. The impeller **30** further includes an impeller core ring **45**, and a plurality of impeller blades **33** fixedly attached, such as by brazing, to the impeller core ring **45** and the impeller shell **20**. The impeller **30**, including its shell **20**, the impeller core ring **45**, and the impeller blades **33**, is secured to the first casing shell **18** of the casing **12** so as to be non-rotatable relative to the casing **12** and hence inter-connected to the drive shaft (or flywheel) of the engine to rotate at the same speed as the engine output. The impeller **30** also includes an impeller hub **31** fixedly secured to the impeller shell **20**. The impeller hub **31** is arranged for engagement with a hydraulic pump of the transmission.

The turbine-piston **32** is a consolidation or incorporation of a turbine with a lockup clutch piston. The turbine component of the turbine-piston **32** includes a turbine-piston shell **35**, a turbine-piston core ring **46**, and a plurality of turbine-piston blades **36** fixedly attached, such as by brazing, to the turbine-piston core ring **46** and the turbine-piston shell **35**. The spinning of the impeller **30** causes transmission fluid in the torus to spin the turbine blades **36**, and hence the turbine-piston core ring **46** and the turbine-piston shell **35** in hydrodynamic transmission mode. The impeller shell **20** and the turbine-piston shell **35** collectively define a substantially toroidal inner chamber (or torus chamber) **53** therebetween.

The stator **34** is positioned between the impeller **30** and turbine-piston **32** to redirect fluid from the turbine-piston **32** back to the impeller **30** in an efficient manner. The stator **34** is typically mounted on a one-way clutch **72** to prevent the stator **34** from counter-rotation. A thrust bearing **74** is interposed between a side wall **73** of the stator **34** and the impeller shell **20** of the casing **12**.

The hydrokinetic torque coupling device **10** further includes an output hub (also referred to as a hub) **40** that is rotatable about the rotational axis X. The output hub **40** is operatively coupled to and coaxial with the driven shaft. For example, the output hub **40** may be provided with internal splines **42** for non-rotatably coupling the output hub **40** to the driven shaft, such as a transmission input shaft, provided with complementary external splines or grooves. Alternatively, a weld or other connection may be used to fix the output hub **40** to the driven shaft. A radially outer surface of the output hub **40** includes an annular slot **43** for receiving a sealing member, such as an O-ring **44**. A sealing member (not shown), mounted to a radially inner peripheral surface of the output hub **40**, creates a seal at the interface of a transmission input shaft (not shown) and the output hub **40**.

Extending axially at a radially inner peripheral end **35₂** of the turbine-piston shell **35** is a substantially cylindrical flange **37** that is proximate to the rotational axis. The substantially cylindrical flange **37** of the turbine-piston **32** is rotatable relative to the output hub **40**. The sealing member (e.g., O-ring) **44** creates a seal at the interface of the substantially cylindrical flange **37** and the output hub **40**. As discussed in further detail below, the turbine-piston **32** is axially movably relative to the output hub **40** along this interface.

A drive component **56** is affixed to the turbine-piston shell **35**, such as by an annular weld and/or fasteners **49** near a radially distal end **35₁** of the turbine-piston shell **35**. The drive component **56** includes an axially extending tubular portion **56a** and a clutch portion **56b** extending radially downward from the tubular portion **56a**. Although the portions **56a**, **56b** are embodied as integral with one another, e.g., made of a single or unitary component, the portions

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56a, **56b** may be separate components connected together. As best shown in FIGS. **2** and **3**, the clutch portion **56b** of the drive component **56** has an engagement surface **56₁** that faces and is parallel to engagement surface **25₁** of the clutch plate **25**.

The damper assembly **16** is housed in the casing **12** axially between the turbine-piston **32** and the first casing shell **18**, as shown in FIG. **1**. The damper assembly **16** includes first and second input disk parts **68**, **69** connected to one another by rivet **70**. The first input disk part **68** is operatively connected to the radially extending clutch portion **56b** of the drive component **56** at connection **67** so that the drive component **56** rotationally drives the damper assembly **16**. The connection **67** may be splined to allow the drive component **56** to be displaced axially relative to the damper assembly **16** by the turbine-piston **32** moving in and out of lockup mode, which is discussed in greater detail below.

The input disk parts **68**, **69** are interconnected to a driven (or output) member **62** through a plurality of circumferentially extending elastic damping members **64**. The driven member **62** has windows in which the elastic damping members **64** are set. The input disk parts **68**, **69** engage first ends of the damping members **64**, and the driven member **62** engages second ends of the second damping members **64**. The disk parts **68**, **69** are thus rotatable relative to the driven member **62**, with the damping members **64** absorbing torsional vibration due to their elasticity. The damping members **64** are configured as helical (or coil) springs having principal axes oriented substantially circumferentially. Other elastic members may be selected to replace or supplement the springs.

As shown in FIG. **1**, the output hub **40** and driven member **62** are integral with one another. Alternatively, the driven member **62** may be non-integrally connected to the output hub **40**. The non-rotatable connection between the driven member **62** and the output hub **40** may be formed by splines or welding.

The hydrokinetic torque coupling device **10** includes an additional clutch plate **52** extending radially between the clutch plate **25** and the first sidewall portion **22**. A radially inner end of the clutch plate **52** is connected to the damper assembly **16** by rivets **70**, which also connect disk parts **68**, **69**. The clutch plate **52** is axially movable relative to the casing **12**. This axial movement of the clutch plate **52** may be accomplished by several designs, such as providing sliding engagement between the damper assembly **16** and the output hub **40**, providing sliding engagement between the output hub **40** and the driven shaft along splines **42**, and/or configuring the damper assembly **16** to allow the damper retainer plates **68**, **69** to move axially relative to the output plate **62**. As best shown in FIGS. **2** and **3**, the additional clutch plate **52** has a clutch plate engagement surface **52₁** that faces and is parallel to the clutch plate engagement surface **25₂** of the clutch plate **25**, and another clutch plate engagement surface **52₂** that faces and is parallel to the (first) engagement surface **22₁** of the first sidewall portion **22**.

The drive component **56** and clutch plates **25** and **52** collectively provide a multi-face lockup clutch **50**. From left to right in the figures, the lockup clutch **50** includes engagement surface pairs **56₁** and **25₁**, **25₂** and **52₁**, and **52₂** and **22₁** that collectively establish a multi-interface lockup clutch **50**. The engagement surfaces of each pair are movable towards and away from one another, e.g., **56₁** and **25₁** are movable towards each other into lockup mode and away from each other out of lockup mode, etc. One or more of the engage-

ment surfaces 56_1 , 25_1 , 25_2 , 52_1 , 52_2 , and 22_1 may be provided with a frictional ring (or friction lining) 48_1 , 48_2 , or 48_3 (collectively referred to as 48). As best shown in FIG. 3, in the illustrated embodiment friction ring 48_1 is secured to the clutch plate engagement surface 25_1 and friction rings 48_2 and 48_3 are secured to the clutch plate engagement surfaces 52_1 , 52_2 , respectively. The friction rings 48 may be secured to the engagement surfaces, for example, by adhesive bonding and/or with fasteners. The friction rings 48 are made of a friction material for improved frictional performance. Friction rings 48 may be secured to any, all, or none of the engagement surfaces.

The turbine-piston 32 is axially displaceable to move the second engagement surface 56_1 axially towards and away from the first engagement surface 22_1 of the casing 12 for positioning the hydrokinetic torque coupling device 10 respectively into and out of a lockup mode in which the casing 12 is mechanically locked to the input part of the damper assembly 16 . Axial displacement of the turbine-piston 32 into the lockup mode frictionally interlocks the clutch plate engagement surfaces 52_2 and 25_1 (including the friction ring(s) 48_3 and 48_1 secured thereto) to the first and second engagement surfaces 22_1 and 56_1 , respectively, and frictionally interlocks the clutch plate engagement surfaces 25_2 and 52_1 to one another via friction rings 48_2 . Torque is transferred from the engine to the casing 12 , then by way of the frictionally interlocked engagement surfaces (including the frictional linings 48 thereof) to the damping assembly 16 and the output hub 40 . Notably, the friction rings 48_1 , 48_2 , and 48_3 may have circumferentially spaced grooves (not shown) extending generally radially so as to fluidly connect the radially outer chamber 51 and the damper chamber 54 with one another in the lockup mode for cooling friction surfaces of the lockup clutch 50 by the working fluid.

The lockup clutch 50 bypasses the hydrodynamic fluid coupling of the torque converter 14 and mechanically non-rotatably couples the driving and driven shafts. The provision of multiple lockup interfaces between engagement surfaces increases torque capacity of the device 10 .

Axial displacement of the turbine-piston 32 out of lockup mode releases the frictional interlocking of the engagement surfaces. In the non-lockup mode, normal operation of the torque converter 14 selectively fluidly (hydrodynamically) couples and decouples the impeller 30 to and from the turbine-piston 32 .

As discussed above, the turbine-piston 32 is axially movable toward and away from the impeller shell 20 between a lockup position and a non-lockup (open) position. Axial movement of the turbine-piston 32 is accomplished by changing the pressure differential between the opposite sides of the turbine-piston shell 35 . Referring to FIG. 1, a torus chamber 53 is to the left side of the turbine-piston shell 35 , a damper chamber 54 is to the other (right) side of the turbine-piston shell 35 , and a radially outer chamber 51 is located above (as shown in FIG. 1) or radially outside of the tubular portion $56a$ of the drive component 56 and the lockup clutch 52 . In the illustrated embodiment of FIG. 1, the torus chamber 53 and the radially outer chamber 51 remain in fluid communication with one another, irrespective of whether the lockup clutch is in or out of lockup mode. A pressure increase in the torus chamber 53 relative to the damper chamber 54 (or stated differently a pressure decrease in the damper chamber 54 relative to the torus chamber 53) shifts the turbine-piston 32 axially against the direction of torque transmission, i.e., towards the input side of the casing 12 , that is left to right in FIG. 1, into lockup mode. On the other hand, a pressure decrease in the torus chamber 53

relative to the damper chamber 54 (or stated differently a pressure increase in the damper chamber 54 relative to the torus chamber 53) shifts the turbine-piston 32 axially in the direction of torque transmission, i.e., towards the output side of the casing, that is right to left in FIG. 1, out of lockup mode. Pressure changes are created by control of the fluid, e.g., hydraulic fluid or oil, in the chambers 51 , 53 and 54 .

In operation, the lockup clutch 50 is generally activated after the hydrodynamic coupling of the driving and driven shafts, typically at relatively constant speeds, in order to avoid the loss of efficiency caused in particular by slip phenomena between the turbine-piston 32 and the impeller 30 . Because of the axial pressures acting on the turbine-piston 32 for movement between its lockup and non-lockup positions, the turbine-piston shell 35 may be made somewhat thicker than typical turbine shells that do not form or function as the lockup piston. Although not shown, a biasing member, such as a spring, may be included in the hydrokinetic torque coupling device 10 to axially urge the turbine-piston 32 into or out of lockup mode.

The turbine-piston 32 both forms the shell component of the turbine and the piston component of the lockup clutch 50 , as described above. By consolidating two components that are normally separate from one another into a single component, space is saved in the hydrokinetic torque coupling device 10 . This space-saving structure provides several design options. For example, the hydrokinetic torque coupling device 10 can be made smaller and lighter. Alternatively, the free space within the casing 12 can be used to add additional components, such as damping components. Additionally, and as discussed in greater detail below, the drive component 56 simplifies assembly of the hydrokinetic torque coupling device 10 .

An exemplary method for assembling the hydrokinetic torque coupling device 10 according to the embodiment of FIGS. 1 and 2 will now be explained. It should be understood that this exemplary method may be practiced in connection with the other embodiments described herein. This exemplary method is not the exclusive method for assembling the hydrokinetic torque coupling devices described herein. While the methods for assembling the hydrokinetic torque coupling device 10 may be practiced by sequentially performing the steps as set forth below, it should be understood that the methods may involve performing the steps in different sequences.

The impeller 30 , the turbine-piston 32 (without the drive component 56), the stator 34 , and the damper 16 may each be preassembled. The turbine-piston 32 includes, as noted above, the turbine-piston shell 35 and the turbine blades 36 attached to the turbine-piston shell 35 . The drive component 56 and the clutch plates 25 and 52 may be, for example, stamped from metal.

The impeller 30 , the stator 34 , and the turbine-piston 32 subassemblies are assembled together. The drive member 56 is secured, such as by welding at the weld 49 , to the turbine-piston shell 35 . The damper assembly 16 is added. The cylindrical flange 37 of the turbine-piston 32 is mounted to slidingly engage the output hub 40 (splined with the driven shaft at 42) with the seal 44 therebetween. The clutch portion $56b$ of the drive component 56 is engaged at the spline connection 67 with the damper assembly 16 as described above. The first casing shell 18 with the clutch plate 25 pre-installed is fitted so that the clutch plates 25 and 52 and the clutch portion $56b$ of the drive component 56 are aligned as shown in FIG. 1. The first casing shell 18 is non-moveably and sealingly secured, such as by welding at 19 , to the second casing shell 20 , as best shown in FIG. 1.

Various modifications, changes, and alterations may be practiced with the above-described embodiment. The features of the embodiments described herein may be practice with one another and are substitutable in numerous combinations.

The torque converters and hydrodynamic torque coupling devices described herein may incorporate different damper assemblies. For example, FIG. 4 shows a hydrodynamic torque coupling devices including the impeller 30 and the turbine-piston 32 for establishing the hydrodynamic transmission mode and the lockup clutch 50 for lockup mode transmission. The impeller 30/turbine-piston 32 combination and the lockup clutch 50 are arranged parallel to one another and serially between the casing 12 and the turbine-piston shell 35. The elastic damping members 60 of the damper assembly 16 and the output member 40 are arranged serially downstream of the turbine-piston shell 35 in FIG. 4. The diagram of FIG. 4 generally corresponds to the arrangement of the embodiment shown in FIGS. 1-3.

FIG. 5 shows an alternative damper assembly 116 similar to that of FIG. 4, but in which the damper assembly 116 is modified to include the elastic damping members 60 as a first set of circumferentially extending elastic damping members 60₁ and to further include a second set of circumferentially extending elastic damping members 60₂ positioned radially inward of the first set of elastic damping members 60₁. An intermediate member 158 is interposed serially between the first and second sets of elastic damping members 60₁ and 60₂ for absorbing torsional vibration. In turn, the second set of elastic damping members 60₂ is interposed serially between the intermediate member 158 and the output member 40. The intermediate member 158 is rotatable relative to the drive component 56 via the elasticity of damping members 60₁, and is rotatable relative to the output member 40 via the elasticity of damping members 60₂.

A damper assembly 216 shown in FIG. 6 is similar to that of FIG. 5, but further includes pendulum vibration absorbers 96 coupled to an intermediate member 158. Pendulum vibration absorbers 96 are well known in the art and described in, for example, U.S. patent application Ser. No. 14/305,128 filed Jun. 16, 2014, GB598811 to Stone, U.S. Pat. No. 6,026,940 to Sudau, and EP1744074 to Grahl. The pendulum vibration absorbers 96 may be coupled to the circumference of the intermediate member 158 and may be arranged on both sides of the intermediate member 158.

A damper assembly 316 shown in FIG. 7 is similar to that of FIG. 5, but further includes a spring mass system 99 coupled to an intermediate member 158. Spring-mass systems are well known in the art and described in, for example, WO 2004/018897 to Haller. The spring-mass system 99 may be coupled to the circumference of the intermediate member 158. The spring of the spring-mass system 99 may be a coil spring, such as a steel spring. The damper may be any linear or non-linear damper, including for example a viscous damper. The spring and mass may be embodied as two components or one integral component. The spring-mass system may have a linear or non-linear constant or variable stiffness, and a constant or variable mass.

The features of the above-described embodiments are substitutable in numerous combinations.

The foregoing description of the exemplary embodiment(s) of the present invention has been presented for the purpose of illustration in accordance with the provisions of the Patent Statutes. It is not intended to be exhaustive or to limit the invention to the precise forms disclosed. The embodiments disclosed hereinabove were

chosen in order to best illustrate the principles of the present invention and its practical application to thereby enable those of ordinary skill in the art to best utilize the invention in various embodiments and with various modifications as suited to the particular use contemplated, as long as the principles described herein are followed. This application is therefore intended to cover any variations, uses, or adaptations of the invention using its general principles. Further, this application is intended to cover such departures from the present disclosure as come within known or customary practice in the art to which this invention pertains. Thus, changes can be made in the above-described invention without departing from the intent and scope thereof. It is also intended that the scope of the present invention be defined by the claims appended thereto.

What is claimed is:

1. A hydrokinetic torque coupling device for coupling a driving shaft and a driven shaft together, comprising:
 - an impeller comprising an impeller shell;
 - a casing comprising a casing shell connected to the impeller shell, the casing having a first engagement surface;
 - an axially displaceable turbine-piston coaxially aligned with and hydrodynamically drivable by the impeller, the turbine-piston comprising a turbine-piston shell and a drive component having a second engagement surface;
 - a damper assembly comprising an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub; and
 - a clutch plate directly connected to the damper assembly and axially displaceable relative to the casing, the clutch plate having opposite clutch plate engagement surfaces interposed between and axially movable relative to at least one of the first and second engagement surfaces,
 wherein the turbine-piston is axially displaceable relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.
2. The hydrokinetic torque coupling device of claim 1, wherein the input part of the damper assembly is non-rotatably and axially movably connected to the drive component.
3. The hydrokinetic torque coupling device of claim 1, further comprising an additional clutch plate connected to and axially displaceable relative to the casing.
4. The hydrokinetic torque coupling device of claim 1, wherein:
 - the input part of the damper assembly is non-rotatably and axially movably connected to the drive component; and
 - the hydrokinetic torque coupling device further comprises an additional clutch plate connected to and axially displaceable relative to the casing.
5. The hydrokinetic torque coupling device of claim 4, wherein the additional clutch plate is interposed between the drive component and the clutch plate, and wherein the clutch plate is interposed between the additional clutch plate and the first engagement surface of the casing.
6. The hydrokinetic torque coupling device of claim 4, wherein the drive component, the clutch plate, and the

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additional clutch plate are rotatable relative to one another when the hydrokinetic torque coupling device is out of the lockup mode.

7. The hydrokinetic torque coupling device of claim 4, wherein one of the clutch plate engagement surfaces of the additional clutch plate comprises a first frictional lining, and wherein one of the clutch plate engagement surfaces of the clutch plate comprises a second frictional lining.

8. The hydrokinetic torque coupling device of claim 4, wherein one of the clutch plate engagement surfaces of the additional clutch plate comprises a first frictional lining, and wherein the opposite clutch plate engagement surfaces of the clutch plate comprise second and third frictional linings, respectively.

9. The hydrokinetic torque coupling device of claim 1, wherein the drive component of the turbine piston comprises an axially extending tubular portion fixed to the turbine-piston shell and a radially extending clutch portion having the second engagement surface.

10. The hydrokinetic torque coupling device of claim 9, wherein the turbine-piston comprises a plurality of turbine blades, and wherein the axially extending tubular portion of the drive component is radially outside of the turbine blades.

11. The hydrokinetic torque coupling device of claim 9, wherein the axially extending tubular portion and the radially extending clutch portion of the drive component are integral with one another.

12. The hydrokinetic torque coupling device of claim 1, wherein the first and second engagement surfaces and the clutch plate engagement surfaces extend radially at an angle of about 90 degrees relative to a rotational axis of the casing.

13. The hydrokinetic torque coupling device of claim 1, wherein the casing shell forms the first engagement surface of the casing.

14. The hydrokinetic torque coupling device of claim 13, wherein the damper assembly comprises an intermediate member, a first set of circumferentially extending elastic damping members drivingly coupling the input part to the intermediate member, the output part connected to and non-rotatable relative to the output hub, a second set of circumferentially extending elastic damping members drivingly coupling the intermediate member to the output part, and a centrifugal pendulum oscillator mounted to the intermediate member.

15. The hydrokinetic torque coupling device of claim 13, wherein the damper assembly comprises an intermediate member, a first set of circumferentially extending elastic damping members drivingly coupling the input part to the intermediate member, the output part connected to and non-rotatable relative to the output hub, a second set of circumferentially extending elastic damping members drivingly coupling the intermediate member to the output part, and a spring mass system coupled to the intermediate member.

16. The hydrokinetic torque coupling device of claim 1, wherein the turbine-piston is axially displaceable towards an input side of the hydrokinetic torque coupling device in order to frictionally couple the clutch plate engagement surfaces and the first and second engagement surfaces together and position the hydrokinetic torque coupling device in the lockup mode, and wherein the turbine-piston is

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axially movable towards an output side of the hydrokinetic torque coupling device so that the first and second engagement surfaces are not frictionally coupled and the hydrokinetic torque coupling device is not in the lockup mode.

17. The hydrokinetic torque coupling device of claim 1, further comprising a stator situated between the impeller and the turbine-piston.

18. A method of assembling a hydrokinetic torque coupling device for coupling a driving shaft and a driven shaft together, comprising:

providing a torque converter comprising an impeller including an impeller shell, and an axially displaceable turbine-piston including a turbine-piston shell and a drive component; and

operatively connecting the torque converter to a casing shell, a damper assembly comprising an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub, and a clutch plate having opposite clutch plate engagement surfaces to assemble the hydrokinetic torque coupling device comprising a casing including the casing shell, wherein the clutch plate is directly connected to the damper assembly and interposed between and axially movable relative to at least one of a first engagement surface of the casing and a second engagement surface of the drive component, wherein the turbine-piston is axially displaceable relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.

19. A method of operating a hydrokinetic torque coupling device comprising an impeller including an impeller shell, a casing having a first engagement surface and including a casing shell connected to the impeller shell, an axially displaceable turbine-piston coaxially aligned with and hydrodynamically drivable by the impeller and including a turbine-piston shell and a drive component having a second engagement surface, a damper assembly comprising an input part drivenly interconnected to the turbine-piston and an output part operatively connectable to an output hub, and a clutch plate directly connected to the damper assembly and having opposite clutch plate engagement surfaces interposed between and axially movable relative to at least one of the first and second engagement surfaces, said method comprising:

axially displacing the turbine-piston relative to the casing to move the second engagement surface axially towards and away from the first engagement surface for positioning the hydrokinetic torque coupling device into and out of a lockup mode in which the first and second engagement surfaces and the clutch plate engagement surfaces frictionally interlock with one another so that the casing is mechanically locked to and non-rotatable relative to the input part of the damper assembly.

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